EGT2
ENGINEERING TRIPOS PART IIA

Monday 23 April 20182 to 3.40

## Module 3D4

## STRUCTURAL ANALYSIS AND STABILITY

Answer not more than three questions.
All questions carry the same number of marks.
The approximate percentage of marks allocated to each part of a question is indicated in the right margin.

Write your candidate number not your name on the cover sheet.

## STATIONERY REQUIREMENTS

Single-sided script paper
Graph paper

## SPECIAL REQUIREMENTS TO BE SUPPLIED FOR THIS EXAM

CUED approved calculator allowed
Attachment: Data Sheet for Question 2: Stiffness Matrices (1 page)
Engineering Data Book

## 10 minutes reading time is allowed for this paper at the start of the exam. <br> You may not start to read the questions printed on the subsequent pages of this question paper until instructed to do so.

## Version FC/3

1 (a) Figure 1(a) shows a thin-walled unsymmetric channel-like cross-section with constant thickness $t$.
(i) Compute the St. Venant torsion constant of the cross-section.
(ii) Compute directions of the principal axes and the principal second moments of area of the cross-section.
(b) Figure 1(b) shows a thin-walled channel-like cross section with a single symmetry axis and a constant thickness $t$. Compute the shear centre of the cross-section.
(c) Describe how you would compute the shear centre of the cross-section shown in Fig. 1(a). No numerical computations are necessary.


Fig. 1

## Version FC/3

2 (a) Figure 2(a) shows a circular arc cantilever with radius $r$ which lies in the $x-y$ plane. The flexural stiffness of the cantilever is $E I$ and its torsional stiffness is $G J=3 E I$. It is loaded at its free end by a force $F$ acting in the negative $z$-direction. Compute the tip displacement at the cantilever's free end.
(b) Figure 2(b) shows the side view of a continuous beam over three spans with uniform flexural rigidity $E I$. The beam is simply supported at its left end, clamped at its right end and continuous over the two mid-span supports. The centre span caries a uniformly distributed load $w$.
(i) Sketch the bending moment diagram over the entire beam length, marking its salient features. No numerical computations are necessary.
(ii) Compute the hogging moment over support B using the displacement method.

(a)

(b)

Fig. 2

## Version FC/3

3 Figure 3(a) shows a pin-ended strut with bending stiffness EI carrying a compressive load $P$. Consider the deformation of the strut shown in Fig. 3(b), where the strut takes up a shape $y=\psi(x)$.
(a) Show that the bending energy $U$ stored in the strut is given by

$$
U=\frac{1}{2} E I \int_{-L}^{L}\left(\frac{d^{2} \psi}{d x^{2}}\right)^{2} d x
$$

and that the work $V$ done by the load $P$ is given by

$$
V=\frac{1}{2} P \int_{-L}^{L}\left(\frac{d \psi}{d x}\right)^{2} d x
$$

(b) Estimate the buckling load of the strut using a mode shape

$$
\psi_{1}=a\left(1-\left(\frac{x}{L}\right)^{2}\right) .
$$

(c) The buckling load of the strut will be estimated using a mode shape

$$
\psi_{2}=a\left(1-\left(\frac{x}{L}\right)^{2}\right)+b\left(1-\left(\frac{x}{L}\right)^{4}\right) .
$$

The total potential energy maybe expressed as

$$
U-V=\frac{1}{2}\left[\begin{array}{ll}
a & b
\end{array}\right][\boldsymbol{K}]\left[\begin{array}{l}
a \\
b
\end{array}\right] .
$$

Determine the matrix $\boldsymbol{K}$ and without further calculation explain how this may be used to compute the buckling load.

(a)

(b)

Fig. 3

## Version FC/3

4 Figure 4 shows three frame structures that are all braced against out-of-plane displacements. Each member has flexural rigidity $E I$ for bending deformations in the plane of the frame.
(a) The central point A of the beam shown in Fig. 4(a) has a support that prevents displacement but not rotation. Show that the stiffness of the structure against rotation at A is $4 E I / L$.
(b) A vertical load $F$ is applied at A to the frame shown in Fig. 4(b). Using the $s \& c$ functions given in Table 1, where the stiffness factor $s$ and carry-over factor $c$ are given for a column with Euler buckling load $P_{E}$ carrying a compressive load $P$, estimate the critical load to cause buckling.
(c) A vertical load $F$ is applied at A to the frame shown in Fig. 4(c). Again using $s \& c$ functions, estimate the critical load to cause buckling, and describe the mode of bucking. [50\%]

| $P / P_{E}$ | $s$ | $c$ |
| :---: | :---: | :---: |
| 0.0 | 4.0000 | 0.5000 |
| 0.2 | 3.7297 | 0.5550 |
| 0.4 | 3.4439 | 0.6242 |
| 0.6 | 3.1403 | 0.7136 |
| 0.8 | 2.8159 | 0.8330 |
| 1.0 | 2.4674 | 1.0000 |
| 1.2 | 2.0901 | 1.2487 |
| 1.4 | 1.6782 | 1.6557 |


| $P / P_{E}$ | $s$ | $c$ |
| :---: | :---: | :---: |
| 1.6 | 1.2240 | 2.4348 |
| 1.8 | 0.7170 | 4.4969 |
| 2.0 | 0.1428 | 24.6841 |
| 2.2 | -0.5194 | -7.5107 |
| 2.4 | -1.3006 | -3.3703 |
| 2.6 | -2.2490 | -2.2312 |
| 2.8 | -3.4449 | -1.7081 |
| 3.0 | -5.0320 | -1.4157 |

Table 1

Version FC/3


Fig. 4

END OF PAPER

## Data Sheet for Question 2: Stiffness Matrices.

## Beam type I



$$
\left[\begin{array}{l}
S_{A} \\
M_{A} \\
S_{B} \\
M_{B}
\end{array}\right]=\left[\begin{array}{cccc}
\frac{12 E I}{L^{3}} & -\frac{6 E I}{L^{2}} & -\frac{12 E I}{L^{3}} & -\frac{6 E I}{L^{2}} \\
-\frac{6 E I}{L^{2}} & \frac{4 E I}{L} & \frac{6 E I}{L^{2}} & \frac{2 E I}{L} \\
-\frac{12 E I}{L^{3}} & \frac{6 E I}{L^{2}} & \frac{12 E I}{L^{3}} & \frac{6 E I}{L^{2}} \\
-\frac{6 E I}{L^{2}} & \frac{2 E I}{L} & \frac{6 E I}{L^{2}} & \frac{4 E I}{L}
\end{array}\right]\left[\begin{array}{l}
w_{A} \\
\phi_{A} \\
w_{B} \\
\phi_{B}
\end{array}\right]
$$

## Beam type II



$$
\left[\begin{array}{l}
S_{A} \\
M_{A} \\
S_{B} \\
M_{B}
\end{array}\right]=\left[\begin{array}{cccc}
\frac{3 E I}{L^{3}} & -\frac{3 E I}{L^{2}} & -\frac{3 E I}{L^{3}} & 0 \\
-\frac{3 E I}{L^{2}} & \frac{3 E I}{L} & \frac{3 E I}{L^{2}} & 0 \\
-\frac{3 E I}{L^{3}} & \frac{3 E I}{L^{2}} & \frac{3 E I}{L^{3}} & 0 \\
0 & 0 & 0 & 0
\end{array}\right]\left[\begin{array}{l}
w_{A} \\
\phi_{A} \\
w_{B} \\
\phi_{B}
\end{array}\right]
$$

