## Part IB Paper 4: Thermofluid Mechanics <br> THERMODYNAMICS

## Examples Paper 6

## Advanced Steam Power Plant

S1. In a combined gas turbine-steam turbine cycle the gas turbine has a heat input of 100 MW and a thermal efficiency of $35 \%$. It rejects all of its heat to a steam cycle which has a thermal efficiency of $25 \%$.

What is the power output and thermal efficiency of the combination?

Q1. The exhaust gas from the gas turbine of a combined cycle power plant enters the heat recovery steam generator (HRSG) at a temperature of $600^{\circ} \mathrm{C}$ and leaves at $140^{\circ} \mathrm{C}$. The gas may be assumed to have the same properties as air with a value of cp appropriate to a temperature midway between these limits. The steam in the steam generator is at a uniform pressure of 20 bar. It enters as a sub-cooled liquid at $100^{\circ} \mathrm{C}$ and leaves as superheated steam at a temperature of $400^{\circ} \mathrm{C}$. The flow in the steam generator is steady throughout.
(a) Draw a T-x diagram of the HRSG. Calculate the ratio of the mass flow rates of exhaust gas and steam.
(b) Use the result of (a) to calculate the temperature difference between the steam and the gas at the point where the water first reaches its boiling point (called the pinch point)
(c) Calculate the ratio of the rate of entropy increase of the steam within the steam generator to the rate of entropy decrease of the gas, and comment on the significance of its value.

## Refrigerators and Heat Pumps

Q2. Calculate the shaft work required to compress 1 kg of dry saturated $\mathrm{R}-134 \mathrm{~A}$ at 0.15 MPa in a steady flow compressor to a pressure of 1 MPa , for each of the following types of process:
(a) adiabatic and reversible;
(b) adiabatic and with an isentropic efficiency of $80 \%$.
(c) Neglect any changes in kinetic or potential energy. Also, sketch each process on a $p-h$ and a $T-s$ diagram.

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Q3. A vapour compression refrigerator uses $\mathrm{R}-134 \mathrm{~A}$ as the working fluid. The pressure in the condenser is 10 bar and in the evaporator it is 1.5 bar.
(a) What are the saturation temperatures in the condenser and the evaporator?

The refrigerator operates in steady flow, the refrigerant leaves the condenser as saturated liquid and is throttled adiabatically to the evaporator pressure. It leaves the evaporator as saturated vapour and is compressed adiabatically to the condenser pressure, leaving the compressor with a temperature 20 K above the saturation temperature in the condenser (i.e. with 20 K of superheat).
(b) Sketch the cycle on a $p-h$ and a $T-s$ diagram.
(c) Calculate the heat absorbed in the evaporator per unit mass of fluid.
(d) Calculate the heat rejected in the condenser per unit mass of fluid.
(e) Calculate the $\mathrm{COP}_{\mathrm{R}}$ of the plant.
(f) Compare the $\mathrm{COP}_{\mathrm{R}}$ with that for a Carnot cycle working between the temperature in the evaporator and that in the condenser and suggest reasons for the difference.
(g) If the refrigerator has to remove 250 Watts from the cold space, what must be the mass flow rate of refrigerant around the cycle?

Q4. A heat pump is required to operate on a vapour compression cycle with a temperature of $-20^{\circ} \mathrm{C}$ in the evaporator. The refrigerant is $\mathrm{R}-134 \mathrm{~A}$, which enters the compressor as saturated vapour and leaves at a temperature of $65^{\circ} \mathrm{C}$. On exit from the condenser the refrigerant is a saturated liquid at $45^{\circ} \mathrm{C}$. It then passes through a throttle valve back into the evaporator.
(a) Find the pressure in (i) the evaporator and (ii) the condenser.
(b) Calculate the $\mathrm{COP}_{\mathrm{P}}$ of the heat pump.
(c) The throttle valve is replaced by a turbine having an isentropic efficiency of $70 \%$. Calculate the work output of the turbine per unit mass of refrigerant.
(d) If this work is used to help drive the compressor what is the $\mathrm{COP}_{\mathrm{P}}$ of the modified plant?

## Engineering

## Gas Mixtures and Psychrometry

S1. A gas mixture consists of 3 kg of nitrogen and 5 kg of carbon dioxide at a pressure of 3 bar and a temperature of 300 K . Calculate:
(a) the mass fraction and mole fraction of each constituent
(b) the partial pressure of each constituent
(c) the equivalent molar mass and specific gas constant of the mixture
(d) the volume and density of the mixture.

Q5. A sample of moist air at an atmospheric pressure of 1.013 bar and a temperature of $35^{\circ} \mathrm{C}$ has absolute humidity $0.007 \mathrm{kgw} / \mathrm{kga}$. Calculate the partial pressure of the water vapour in the air, the relative humidity and the dewpoint temperature.

Q6. A dehumidifier, shown below, operates by passing $1 \mathrm{kgs}^{-1}$ of moist air over a cooling coil which removes water vapour by condensation. The inlet air is at an atmospheric pressure of 1.013 bar , a temperature of $30^{\circ} \mathrm{C}$ and has a relative humidity of $45 \% .30 \%$ of the mass flow of water vapour at inlet to the dehumidifier leaves as liquid water, $\dot{m}_{w}$. The outlet air is saturated (relative humidity $100 \%$ ). The liquid water, water vapour and air leave the dehumidifier at the same temperature, $T_{\text {exit }}$.
(a) Calculate $T_{\text {exit }}$.
(b) Calculate the rate of heat transfer in the cooler. (NB consider the enthalpy of air, water vapour and liquid water entering and leaving the device.)


## Combustion

Q7. Write down the chemical equation, complete with all coefficients, for the following reactions
(a) Combustion of propane $\mathrm{C}_{3} \mathrm{H}_{8}$ at an air-fuel ratio of 16:1
(b) Combustion of benzene $\mathrm{C}_{6} \mathrm{H}_{6}$ at an equivalence ratio of 1.05
(c) Combustion of ethanol $\mathrm{C}_{2} \mathrm{H}_{5} \mathrm{OH}$ at stoichiometric conditions

Q8. The combustion chamber of a space rocket operates in a steady flow. The fuel, saturated liquid hydrogen at 20 K , flows at a rate of $24 \mathrm{~kg} / \mathrm{s}$. The oxidant is saturated liquid oxygen at 88 K , which is supplied at a rate of $72 \mathrm{~kg} / \mathrm{s}$. At these temperatures, hydrogen and oxygen require $452 \mathrm{~kJ} / \mathrm{kg}$ and $214 \mathrm{~kJ} / \mathrm{kg}$ respectively of heat input to vaporise. Below 298 K , the specific heat capacities of gaseous hydrogen and oxygen may be assumed constant and equal to the perfect gas values given in the Thermofluids Tables. All of the oxygen is consumed before the products leave the combustion chamber.
(a) What is the chemical equation for this process?
(b) Assuming that heat losses and kinetic energies of all streams are negligible, show that an exit temperature of 2443 K satisfies the SFEE for the combustion chamber. NB remember to include the latent heats for hydrogen and oxygen.
(c) The combustion products pass from the combustion chamber, where the velocity is negligible, into a nozzle where they are accelerated to produce thrust. Calculate the velocity of the products at the point in the nozzle where the temperature has fallen to 1000 K . Why do you think the combustion process is rich (i.e. an excess of hydrogen)?

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## ANSWERS

Note: Most of the answers depend upon the use of tabulated data, and are sensitive to rounding errors. The final results here are quoted to three significant figures for convenience, but some numerical error is to be expected and is acceptable.

S1. $51.25 \mathrm{MW}, 51.25 \%$
Q1. a) 5.8 ; b) $7^{\circ} \mathrm{C}$; c) 1.265
Q2. a) $42.0 \mathrm{~kJ} / \mathrm{kg}$; b) $52.5 \mathrm{~kJ} / \mathrm{kg}$
Q3. a) $39^{\circ} \mathrm{C},-18^{\circ} \mathrm{C}$; b) diagrams; c) $133 \mathrm{~kJ} / \mathrm{kg}$; d) $187 \mathrm{~kJ} / \mathrm{kg}$; e) 2.5 ; f) 4.47 ;
g) $1.88 \times 10^{-3} \mathrm{~kg} / \mathrm{s}$.

Q4. a) 11.6 bar, 1.33 bar ; b) 3.13 ; c) $7.6 \mathrm{~kJ} / \mathrm{kg}$; d) 3.61 .
$\mathrm{S} 2 . \quad \mathrm{CO}_{2}$ value, $\mathrm{N}_{2}$ value,
(a) $0.375,0.625,0.515,0.485$,
(b) 1.544 bar, 1.456 bar,
(c) $36.24 \mathrm{~kg} \mathrm{kmol}^{-1}, 229 \mathrm{Jkg}^{-1} \mathrm{~K}^{-1}$,
(d) $1.84 \mathrm{~m}^{3}, 4.36 \mathrm{kgm}^{-3}$.

Q5. $\quad 1130 \mathrm{~Pa}, 20.1 \%, 8.75^{\circ} \mathrm{C}$.
Q6. (a) $11.4^{\circ} \mathrm{C}$, (b) -27.67 kW .
Q7. (a) $\mathrm{C}_{3} \mathrm{H}_{8}+5.128\left(\mathrm{O}_{2}+3.762 \mathrm{~N}_{2}^{*}\right) \rightarrow 3 \mathrm{CO}_{2}+4 \mathrm{H}_{2} \mathrm{O}+0.128 \mathrm{O}_{2}+19.29 \mathrm{~N}_{2}^{*}$,
(b) $\mathrm{C}_{6} \mathrm{H}_{6}+7.143\left(\mathrm{O}_{2}+3.762 \mathrm{~N}_{2}^{*}\right) \rightarrow 5.286 \mathrm{CO}_{2}+3 \mathrm{H}_{2} \mathrm{O}+0.714 \mathrm{CO}+26.87 \mathrm{~N}_{2}^{*}$,
(c) $\mathrm{C}_{2} \mathrm{H}_{5} \mathrm{OH}+3\left(\mathrm{O}_{2}+3.762 \mathrm{~N}_{2}^{*}\right) \rightarrow 2 \mathrm{CO}_{2}+3 \mathrm{H}_{2} \mathrm{O}+11.28 \mathrm{~N}_{2}^{*}$,

Q8. (a) $12 \mathrm{H}_{2}+2.25 \mathrm{O}_{2} \rightarrow 4.5 \mathrm{H}_{2} \mathrm{O}+7.5 \mathrm{H}_{2}$, (c) $3744 \mathrm{~ms}^{-1}$.

