

3AS - Thermodynamics & Power
Generation
Crib - 2025.

①

Q1)

(a) $dg = vdp - sdT$

$$\Rightarrow \left(\frac{\partial g}{\partial p} \right)_T = v ; \quad \left(\frac{\partial g}{\partial T} \right)_p = -s = -S$$

$$\left(\frac{\partial^2 g}{\partial T \partial p} \right) = \left(\frac{\partial v}{\partial T} \right)_p ; \quad \frac{\partial^2 g}{\partial p \partial T} = - \left(\frac{\partial s}{\partial p} \right)_T$$

These must be equal

$$\Rightarrow \boxed{\left(\frac{\partial v}{\partial T} \right)_p = - \left(\frac{\partial s}{\partial p} \right)_T}$$

$$df = -p dv - s dT$$

$$\Rightarrow \left(\frac{\partial f}{\partial v} \right)_T = -p ; \quad \left(\frac{\partial f}{\partial T} \right)_v = -s$$

$$\frac{\partial^2 f}{\partial T \partial v} = - \left(\frac{\partial p}{\partial T} \right)_v ; \quad \frac{\partial^2 f}{\partial v \partial T} = - \left(\frac{\partial s}{\partial v} \right)_T$$

These two must be equal

$$\Rightarrow \boxed{\left(\frac{\partial s}{\partial v} \right)_T = \left(\frac{\partial p}{\partial T} \right)_v}$$

as required.

$$(b) \quad P_r = \frac{\beta T_r}{(\alpha V_r - 1)} - \frac{\alpha}{V_r^2} \quad (2)$$

(i) At critical point, there is no phase change if the substance is cooled isobarically or compressed isothermally.

$$\text{This implies } \left(\frac{\partial p}{\partial v} \right)_T = 0 \quad \& \quad \left(\frac{\partial^2 p}{\partial v^2} \right)_T = 0$$

Since the critical point is the point of inflection.

$$\left(\frac{\partial P_r}{\partial V_r} \right)_{T_r} = - \frac{\alpha \beta T_r}{(\alpha V_r - 1)^2} + \frac{2\alpha}{V_r^3}$$

$$\& \left(\frac{\partial^2 P_r}{\partial V_r^2} \right)_{T_r} = \frac{2\alpha^2 \beta T_r}{(\alpha V_r - 1)^3} - \frac{6\alpha}{V_r^4}$$

$$(a) \text{ Critical Point } P_r = V_r = T_r = 1$$

\& the above two derivatives are zero

$$\Rightarrow \frac{\beta}{2} = (\alpha - 1)^2 \quad \& \quad \frac{2\alpha\beta}{(\alpha - 1)^3} = 6$$

$$\Rightarrow \beta = \frac{3(\alpha - 1)^3}{\alpha}$$

$$\frac{3}{2} \frac{(\alpha - 1)^3}{\alpha} = (\alpha - 1)^2$$

$$\Rightarrow \boxed{\alpha = 3} \quad \text{using this gives} \quad \beta = 8.$$

$$\text{Thus @ Critical point } \boxed{\alpha = 3 \quad \& \quad \beta = 8}$$

$$(11) \quad dh = v dp + T ds \quad (\text{from data book}) \quad (3)$$

$$\left(\frac{\partial h}{\partial T}\right)_p = c_p = T \left(\frac{\partial s}{\partial T}\right)_p$$

$$\text{for } c_p = f(T) \text{ only } \left(\frac{\partial c_p}{\partial p}\right)_T = 0.$$

$$\left(\frac{\partial c_p}{\partial p}\right)_T = T \frac{\partial^2 s}{\partial p \partial T} = T \frac{\partial}{\partial T} \left(\frac{\partial s}{\partial p}\right)_T$$

Using the Maxwell relation in (a)

$$T \frac{\partial}{\partial T} \left(\frac{\partial s}{\partial p}\right)_T = T \frac{\partial}{\partial T} \left(\frac{\partial v}{\partial T}\right)_p$$

$$\left(\frac{\partial v}{\partial T}\right)_p = \frac{V_c}{T_c} \left(\frac{\partial V_r}{\partial T_r}\right)_{p_r} = \frac{V_c}{T_c} \frac{1}{(\partial T_r / \partial V_r)_{p_r}}$$

Rewriting the state equation as

$$\frac{(\alpha V_r - 1) p_r}{\beta} + \frac{\alpha}{\beta} \frac{(\alpha V_r - 1)}{V_r^2} = T_r$$

$$\Rightarrow \left(\frac{\partial T_r}{\partial V_r}\right)_{p_r} = \frac{\alpha p_r}{\beta} + \frac{\alpha}{\beta} \left(\frac{2}{V_r^3} - \frac{\alpha}{V_r^2}\right) \equiv F$$

$$\therefore \left(\frac{\partial v}{\partial T}\right)_p = \frac{V_c}{T_c} \frac{1}{F}$$

but F is not a function of T or T_r

$$\text{Thus } \frac{\partial}{\partial T} \left(\frac{\partial v}{\partial T}\right)_p = 0.$$

$$\Rightarrow \left(\frac{\partial c_p}{\partial p}\right)_T = 0 \Rightarrow \boxed{c_p = f(T) \text{ only.}}$$

(iii) $du = Tds - pdv$

④

$$\left(\frac{\partial u}{\partial v}\right)_T = T \left(\frac{\partial s}{\partial v}\right)_T - p$$

from part (a) $\left(\frac{\partial s}{\partial v}\right)_T = \left(\frac{\partial p}{\partial T}\right)_v$

from the given state equation

$$p = \frac{p_c \beta T}{T_c (\alpha v_r - 1)} - \frac{\alpha p_c}{v_r^2}$$

$$\Rightarrow \left(\frac{\partial p}{\partial T}\right)_v = \frac{p_c}{T_c} \frac{\beta}{(\alpha v_r - 1)}$$

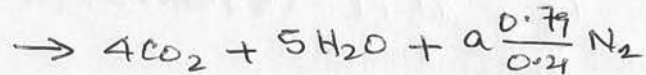
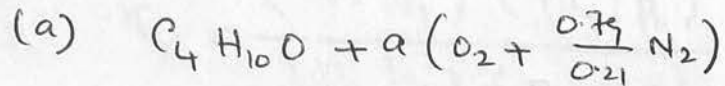
$$\begin{aligned} \Rightarrow \left(\frac{\partial u}{\partial v}\right)_T &= \frac{T p_c \beta}{T_c (\alpha v_r - 1)} - \frac{T p_c \beta}{T_c (\alpha v_r - 1)} + \frac{\alpha p_c v_c^2}{v^2} \\ &= \frac{\alpha p_c v_c^2}{v^2} \end{aligned}$$

integrating from v_0 to v

$$\Rightarrow \boxed{\Delta u = \alpha p_c v_c^2 \left[\frac{1}{v_0} - \frac{1}{v} \right]}$$

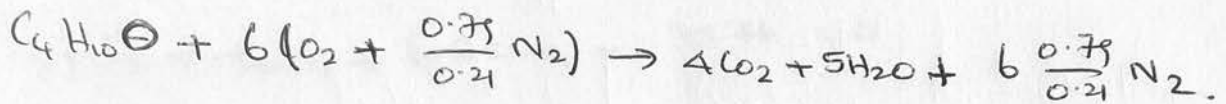
Q2

(5)

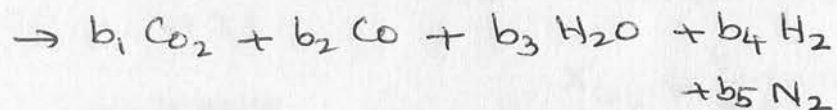
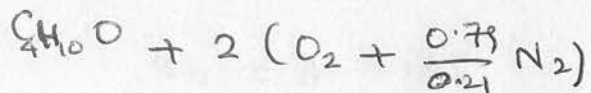


O atom balance gives $a = 6$.

So, the stoichiometric reaction is



(b) Fuel: air = 1:2 molar ratio



$$\Rightarrow b = 2 \frac{0.79}{0.21} N_2 = 7.5238$$

C-atom: $b_1 + b_2 = 4 \Rightarrow b_2 = 4 - b_1$

H-atom: $2b_3 + 2b_4 = 10 \Rightarrow b_3 + b_4 = 5$
 $b_4 = 5 - b_3$

O-atom: $2b_1 + b_2 + b_3 = 5$

$$\Rightarrow b_1 + b_3 = 1 \Rightarrow b_3 = 1 - b_1$$

$$b_4 = 4 + b_1$$

4 unknowns & 3 relations, So we need 1 Kp relation.

Consider the water-gas shift reaction



Rxn 8.

@ $T = 1400K$

$$\ln(K_p) = -0.767$$

$$K_p = \frac{(P_{CO_2}/P_0)(P_{H_2}/P_0)}{(P_{CO}/P_0)(P_{H_2O}/P_0)} = \frac{b_1 b_4}{b_2 b_3} \frac{(P/P_0)^2}{(P/P_0)^2} \quad (6)$$

$$0.4644 = \frac{b_1 (4+b_1)}{(4-b_1)(1-b_1)}$$

$$\Rightarrow 0.5356 b_1 + 6.322 b_1 - 1.8576 = 0$$

$$\Rightarrow b_1 = 0.2869 \quad \text{physically meaningful answer.}$$

$$b_2 = 3.7131$$

$$b_3 = 0.7131$$

$$b_4 = 4.2869$$

$$b_5 = 7.5238$$

$$\text{Sum} = \underline{\underline{16.5238}}$$

mole fractions are

$$X_{CO_2} = \frac{b_1}{\text{Sum}} = 0.0174$$

$$X_{CO} = 0.2247$$

$$X_{H_2O} = 0.0432$$

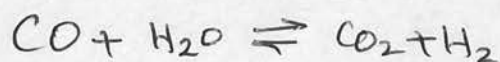
$$X_{H_2} = 0.2594$$

$$X_{N_2} = 0.4553$$

$$\underline{\underline{\Sigma = 1.000}}$$

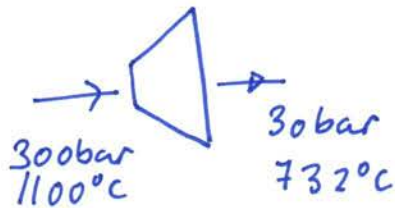
(c) No changes to X_i because of pressure change, since no pressure dependence appears in the K_p relation.

(d) H_2 Yield by shifting the water-gas shift reaction to the right by spraying water.



$H_2O \uparrow \Rightarrow H_2 \uparrow$
by Le-Chatelier principle.

3(a)
i)



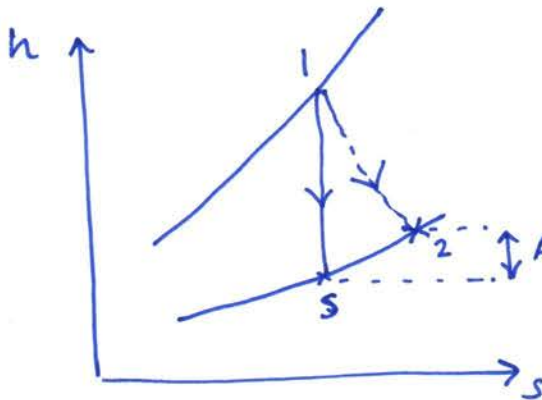
$$\dot{W}_t = 360 \text{ MW}$$

$$\dot{m}(1839 - 1353) = 360 \times 10^3$$

$$\dot{m} = 740.74 \text{ kg/s} \quad [2]$$

$$\begin{aligned} \text{ii) } \text{Loss} &= T_0 \Delta S_{\text{irr}} = (25 + 273) \times (-3.494 + 3.549) \times 740.74 \\ &= 12.1 \text{ MW} \quad (\text{or } 16.4 \text{ kJ/kg}) \end{aligned} \quad [2]$$

iii)



$$\text{Exergetic loss} = T_0 \Delta S_{\text{irr}}$$

$$\text{Power loss due to irreversibility} \approx T_2 \Delta S_{\text{irr}}$$

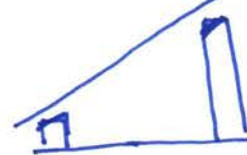
$$T_2 \gg T_0$$

$$\left[\begin{aligned} &\text{if turbine were reversible} \\ &\text{additional power would be} \approx (732 + 273) \\ &\quad \times (3.549 - 3.494) \\ &= +55.3 \text{ MW} \end{aligned} \right] \quad [2]$$

iv)

$$Q_{\text{in}} = \frac{740.74}{104} = 7.12 \text{ m}^3/\text{s}$$

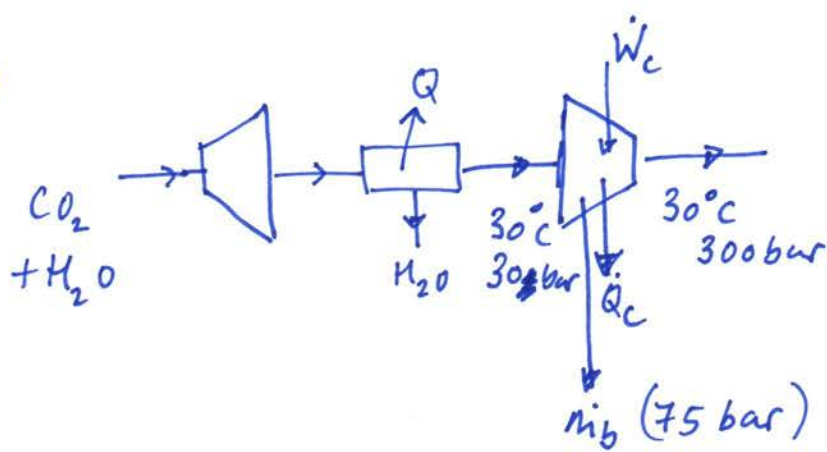
$$Q_{\text{out}} = \frac{740.74}{15.1} = 49 \text{ m}^3/\text{s}$$



Blade height increases to accommodate larger volume flow

fluid densities much larger than for GT or ST
→ higher power density (small blade heights)

[2]



i) Enthalpy drops because the fluid does not behave like an ideal gas
 $\rightarrow h = f(P, T)$, The fluid is compressed from a gas to a super-critical state. Heat transfer outweighs shaft work input.

$$\text{ii) SFEE} \quad \dot{m}_b(290) + [\dot{m}_c - \dot{m}_b](250) - \dot{m}_c(480) = \dot{W}_c - \dot{Q}_c$$

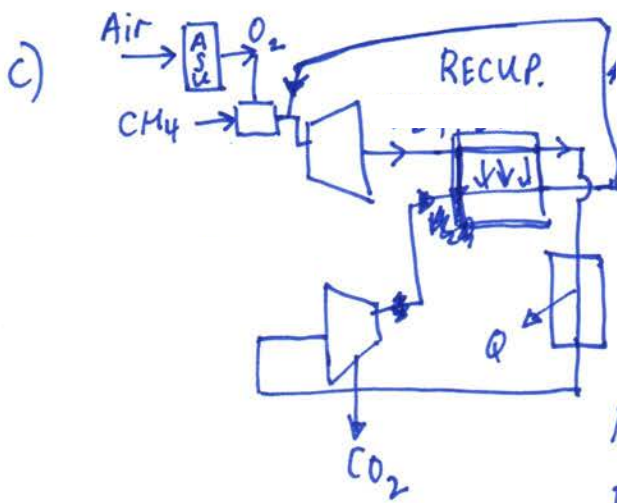
$$\dot{m}_c = 740.74 \times 0.97 = 718.5 \text{ kg/s}$$

$$\dot{m}_b = 26.6 \text{ kg/s}$$

$$\rightarrow -\dot{Q}_c = -164.2 \text{ MW} - 56 \text{ MW} \quad \dot{Q}_c = \underline{\underline{220 \text{ MW}}} \quad [2]$$

iii) if reversible $\dot{W}_c = \dot{m}_b \left[290 - (\frac{303}{290})(1.296) - 480 + (\frac{303}{290})(2.039) \right]$
 $+ (\dot{m}_c - \dot{m}_b) \left[250 - (\frac{303}{290})(1.077) - 480 + (\frac{303}{290})(2.039) \right]$
 $(\dot{W}_c)_{REV} = \underline{43.5 \text{ MW}}$

→ Work to overcome irrev = 12.5 MW [4]



Produces high concentration CO₂ → easy to capture CO₂ compared with conventional power plant.

But air separation unit reduces plant efficiency (requires work)

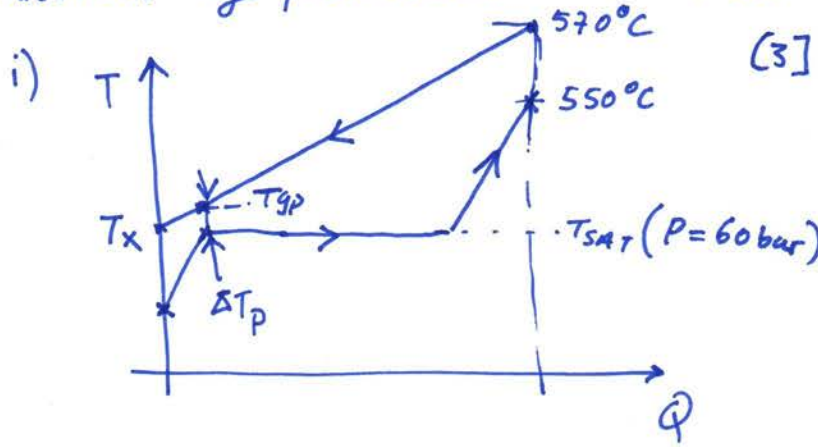
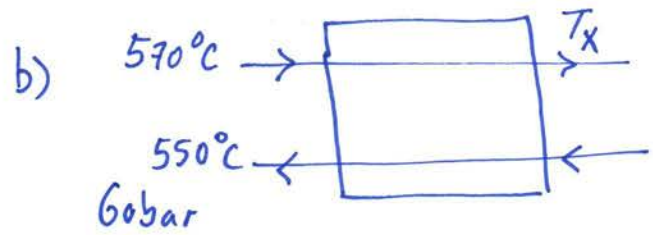
Potential to have ~~more~~ smaller plant size compared to conventional CCS.

4

- a) 'Grey' $H_2 \rightarrow$ Steam methane reformation using natural gas
 'Blue' $H_2 \rightarrow$ SMR + carbon capture
 'Green' $H_2 \rightarrow$ Electrolysis with renewables

Using H_2 as replacement for CCGT \rightarrow used for long term energy storage

Need to consider both direct ~~impact~~ emissions and also 'round trip' emissions of production + distribution of fuel.



i) $T_{gp} = T_{sat} + \Delta T_p = 275.58 + 15 = 290.58^\circ C$ [2]

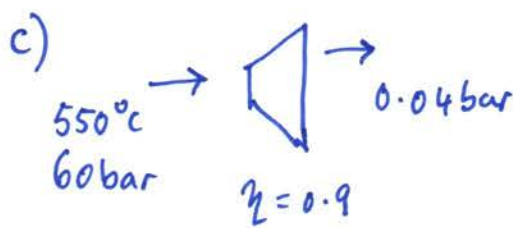
ii) $1.1 (570 - 290.58) \times \dot{m}_g = \dot{m}_s (-1213.9 + 3541.3)$ [2]
 $\rightarrow \dot{m}_s / \dot{m}_g = 0.1321$

iii) $(T_x + 290.58) \times 1.1 = \left(\frac{\dot{m}_s}{\dot{m}_g} \right) \times (1213.9 - 1214)$
 $\rightarrow T_x = 159.4^\circ C$

Need to ensure T_x above dew-point but scope to reduce T_x and increase heat transfer efficiency

$\eta_b = \frac{Q}{Q_{max}} = \frac{570 - 159.4}{570 - 28.96} = 75.9\%$

[3]



from chart

$$h_s = 2130 \text{ kJ/kg}$$

$$\rightarrow \frac{h_2 - h_1}{h_s - h_1} = 0.9$$

$$h_2 = 0.9(2130 - 3550) + 3550$$

$$= 2272 \text{ kJ/kg}$$

$$\dot{m}_s = 0.1321 \times 300 = 39.63 \text{ kg/s}$$

$$\text{Power} = 39.63 \times [-2272 + 3550] = \underline{50.6 \text{ MW}}$$

$$\text{Max Power} = \dot{m}(b_2 - b_1) = 39.63 \left[(h - T_0 s)_2 - (h - T_0 s)_1 \right]$$

$$= 39.63 \left[\left[3541.3 - \left(\frac{28.96}{+273} \right) (7.0307) \right] - \left[121.4 - \left(\frac{28.96}{+273} \right) (0.422) \right] \right] = \underline{56.4 \text{ MW}}$$

losses mostly due to wetness + viscous dissipation in turbine

$$\eta_{ov} = \eta_b \eta_2 + \eta_1 (1 - \eta_b \eta_2)$$

d) ~~feed heating will reduce η_b~~

$$\rightarrow \eta_{ov} \downarrow, T_x \uparrow$$

~~(ii) triple or dual steam levels will increase η_2~~

$$\rightarrow \eta_{ov} \uparrow$$