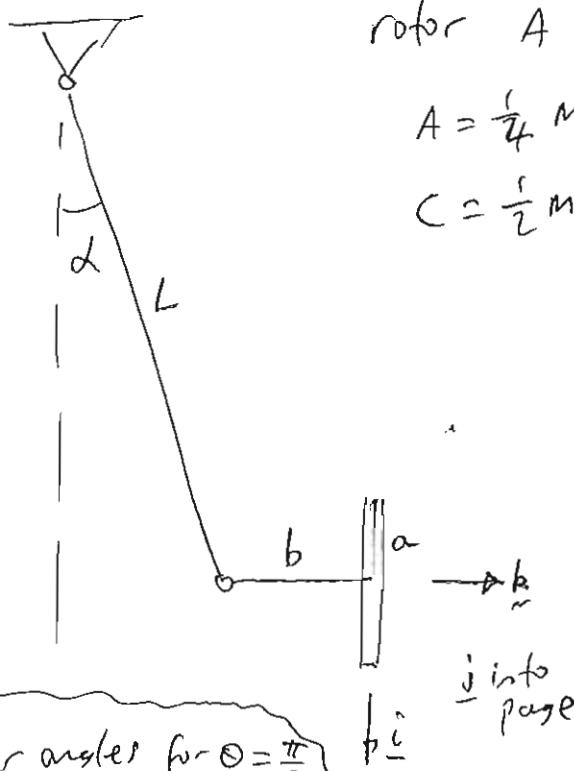


$$\dot{\phi} = \omega$$

3CS

Q1

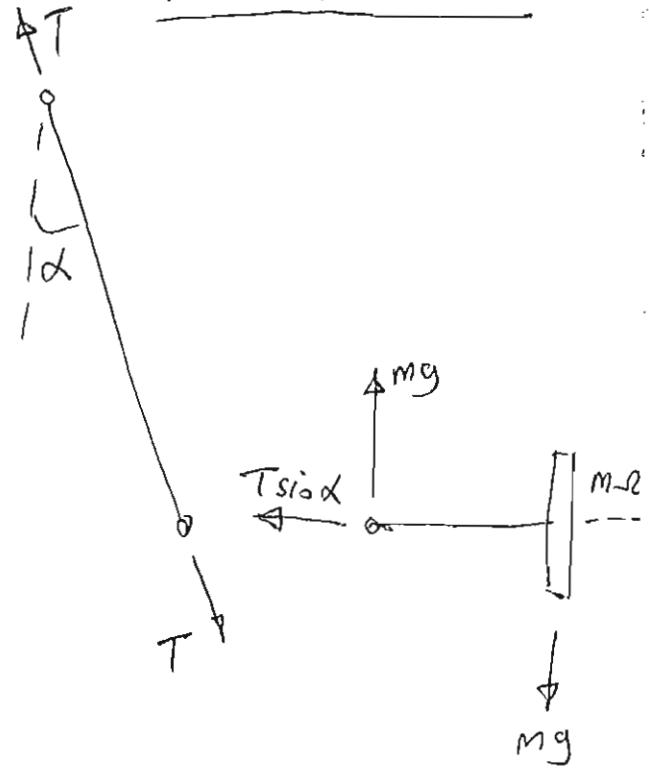


rotor A'AC

$$A = \frac{1}{4} M a^2$$

$$C = \frac{1}{2} M a^2$$

Free body diagrams



Euler angles for $\Theta = \frac{\pi}{2}$

$$\begin{aligned} R_1 &= -\dot{\phi} \sin \Theta = -\omega \\ R_2 &= \Theta = 0 \\ R_3 &= \dot{\phi} \cos \Theta = 0 \end{aligned}$$

$$T \cos \alpha = Mg$$

$$T \sin \alpha = m \cdot R^2 \cdot R$$

$$\therefore Mg \tan \alpha = m \cdot R^2 (b + L \sin \alpha)$$

$$\therefore g \tan \alpha = R^2 (b + L \sin \alpha)$$

Assume small α

$$\boxed{R^2 \approx \frac{g \alpha}{b}} \quad ①$$

Gyro ② $\therefore Q_2 = A \cdot \dot{\alpha} + (A \cdot R_3 - C \omega_3) \cdot R_1$

$$\therefore mg b = 0 + (0 - C \omega_3) \cdot R_1 = \frac{1}{2} M a^2 \omega \cdot R$$

$$\therefore \boxed{R = \frac{2gb}{a^2 \omega}} \quad ②$$

$$① \& ② \therefore \frac{g \alpha}{b} \approx \frac{4g^2 b^2}{a^4 \omega^2}$$

$$\therefore \boxed{\alpha = \frac{4gb^3}{a^4 \omega^2}}$$

Check small angle assumption is consistent with first spin,

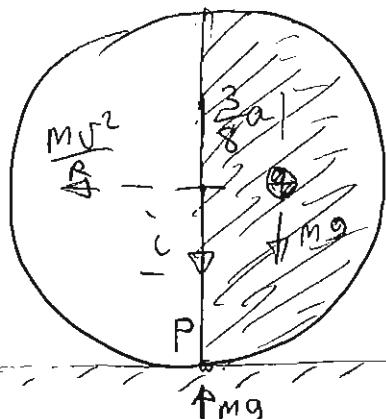
yes $\omega \rightarrow \infty \therefore \alpha \rightarrow 0$

3CS

Q2

 a, m

$$C = \frac{2}{5}ma^2$$

 $\rightarrow k$ j into page

$$\rightarrow \frac{m\omega^2}{R}$$

 $R \gg a$

$$v = -\Omega, R$$

$$\Omega_1 = -\dot{\phi} \cos \theta$$

$$\Omega_2 = 0$$

$$\Omega_3 = \dot{\phi} \cos \theta$$

$$\underline{\omega} = \omega_1 \underline{i} + \omega_2 \underline{j} + \omega_3 \underline{k}$$

$$\text{no slip } \underline{v}_p = \underline{\omega} \times \underline{a} \underline{i} + \underline{v} \underline{j} = 0$$

$$\therefore -a\omega_2 \underline{k} + a\omega_3 \underline{j} = -\underline{v} \underline{j}$$

$$\therefore \omega_2 = 0 \quad \omega_3 = -\frac{v}{a}$$

Gyro eq ② in steady state

$$Q_2 = (A\Omega_3 - C\omega_3) \Omega_1 \quad \text{fast spin } \Omega_3 \ll \omega_3$$

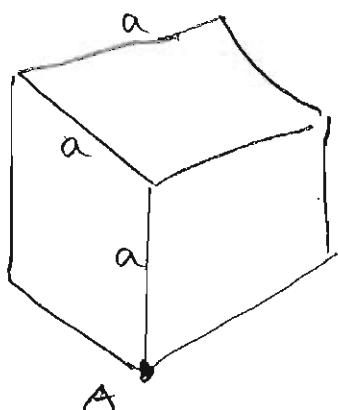
$$\therefore mg \frac{3}{8}a - \frac{m\omega^2}{R}a = -\frac{2}{5}ma^2 \left(-\frac{\omega}{a}\right) \frac{\omega}{R}$$

$$\therefore g \frac{3}{8} = \left(\frac{2}{5} + 1\right) \frac{\omega^2}{R}$$

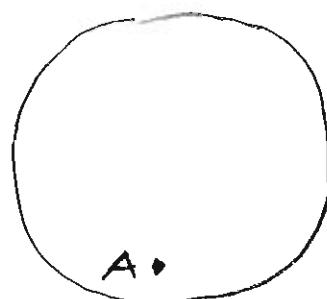
$$\therefore R = \frac{7}{5} \frac{8}{3} \frac{\omega^2}{g} = \frac{56}{15} \frac{\omega^2}{g}$$

$$\text{friction } \frac{m\omega^2}{Rmg} \leq \mu \quad \therefore \frac{\omega^2}{Rg} \leq \mu \quad \therefore \mu \geq \frac{15}{56}$$

(a) (i) The cube has principal moments of inertia $\frac{1}{6}ma^2$, $\frac{1}{6}ma^2$, $\frac{1}{6}ma^2$ at its centre
(straight from data book)



It is like a sphere hence

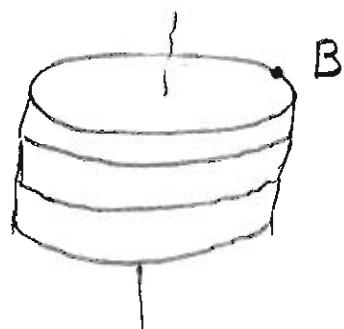


the point A is defined only by its distance
= $a\sqrt{3}/2$
from the centre

Hence the principal moments of inertia at A are $\frac{1}{6}ma^2$, $\frac{11}{12}ma^2$, $\frac{11}{12}ma^2$
(using parallel axis theorem $\frac{1}{6}ma^2 + m\left(\frac{\sqrt{3}a}{2}\right)^2$
 $= \frac{11}{12}ma^2$)

(ii) The cube can be visualised as three equal discs stacked

The top disc (note that a square plate is "AAC" just like a circular disc) can be rotated without changing any inertial properties hence the principal moments of inertia at B are $\frac{1}{6}ma^2$, $\frac{11}{12}ma^2$, $\frac{11}{12}ma^2$ as before



Q3

$$(b) (i) \frac{d}{dt} \left(\frac{\partial T}{\partial q_i} \right) - \frac{\partial T}{\partial q_i} + \frac{\partial V}{\partial q_i} = Q_i$$

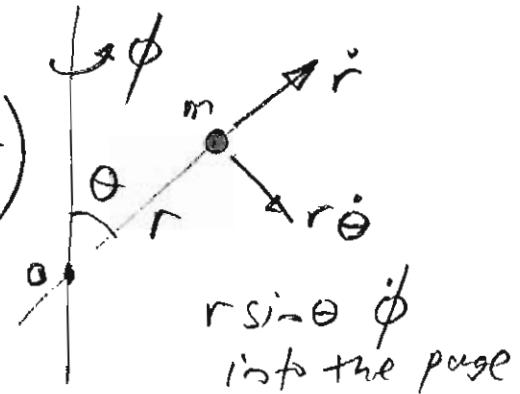
↑ ↓ ↗

if these are zero

then $\frac{\partial T}{\partial q_i} = \text{const}$ is a conservation law

$$(ii) T = \frac{1}{2} m v^2$$

$$= \frac{1}{2} m \left(\dot{r}^2 + (r\dot{\theta})^2 + (r \sin \theta \dot{\phi})^2 \right)$$



VELOCITIES

$$\frac{\partial T}{\partial \dot{r}} = m \dot{r} \quad \frac{\partial T}{\partial \dot{\theta}} = m r^2 \dot{\theta} \quad \frac{\partial T}{\partial \dot{\phi}} = m r^2 \sin^2 \theta \dot{\phi}$$

$$\frac{\partial T}{\partial r} = m \dot{\theta}^2 r + m \sin^2 \theta \dot{\phi}^2 r \quad \frac{\partial T}{\partial \theta} = m r^2 \dot{\phi}^2 \sin \theta \cos \theta$$

$$\frac{\partial T}{\partial \phi} = 0$$

$$\frac{\partial V}{\partial r} = \frac{GMm}{r^2} - \frac{3J_2a^2GM}{2r^4} (3\cos^2 \theta - 1)$$

$$\frac{\partial V}{\partial \theta} = -\frac{GMm J_2 a^2}{r^3} 3 \cos \theta \sin \theta \quad \frac{\partial V}{\partial \phi} = 0$$

Conservation law from ϕ

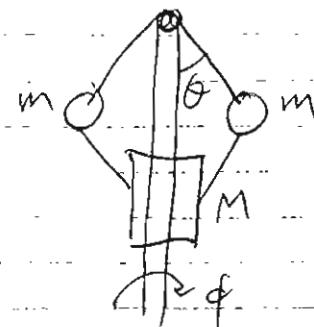
$$\therefore r^2 \sin^2 \theta \dot{\phi} = \text{const}$$

The other equations are

$$r : \ddot{r} = \dot{\theta}^2 r + \sin^2 \theta \dot{\phi}^2 r + \frac{GM}{r^2} - \frac{3GMJ_2a^2}{2r^4} (3\cos^2 \theta - 1) = 0$$

$$\theta : r^2 \ddot{\theta} - r^2 \dot{\phi}^2 \sin \theta \cos \theta + \frac{3GMJ_2a^2}{r^3} \cos \theta \sin \theta =$$

4(a) Use θ , $\dot{\theta}$ as generalised coordinates, where $\dot{\theta} = \omega$.
 Masses m have velocity $a\dot{\theta}$ in the plane of the diagram and $a\sin\theta\dot{\theta}$ into the page. Mass M has velocity $2a\sin\theta\dot{\theta}$ vertically.



So kinetic energy

$$T = 2\left\{ \frac{1}{2}m(a^2\dot{\theta}^2 + a^2\sin^2\theta\dot{\theta}^2) \right\} + \frac{1}{2}M(2a\sin\theta\dot{\theta})^2$$

$$\text{Potential energy } V = 2mga(F\cos\theta) + M \cdot 2ga(F\cos\theta)$$

Lagrange equations

$$\ddot{\theta} = \frac{d}{dt}(2ma^2\sin^2\theta\dot{\theta}) = Q = \text{driving torque on shaft}$$

$$\begin{aligned} \ddot{\theta} &= \frac{d}{dt}[a^2(m+2M\sin^2\theta)\cdot 2\dot{\theta}] - 4Ma^2\sin\theta\cos\theta\dot{\theta}^2 \\ &\quad - 2ma^2\sin\theta\cos\theta\dot{\theta}^2 + 2(m+M)g\sin\theta = 0 \end{aligned}$$

(b) Put $\dot{\theta} = \omega = \text{constant}$ and simplify:

$$\begin{aligned} (m+2M\sin^2\theta)\ddot{\theta} + 2M\sin\theta\cos\theta\dot{\theta}^2 - m\sin\theta\cos\theta\omega^2 \\ + (m+M)\frac{g}{a}\sin\theta = 0 \quad (1) \end{aligned}$$

(c) If $\theta = \text{constant}$, require $m\sin\theta\cos\theta\omega^2 = (m+M)\frac{g}{a}\sin\theta$

so either $\sin\theta = 0$, so $\theta = 0$ or π (impossible)

$$\text{or } \cos\theta = \frac{(m+M)g}{ma\omega^2}, \text{OK provided } \leq 1$$

Case 1: $\omega^2 < (m+M)g/ma$: only root is $\theta = 0$

So perturb: let θ be small, and linearise (1)

$$\text{Then } m\ddot{\theta} - m\theta\omega^2 + (m+M)\frac{g}{a}\theta \approx 0$$

$$\text{i.e. } \ddot{\theta} + \left[\frac{(m+M)g}{ma} - \omega^2 \right] \theta \approx 0$$

4(c) contd: So $\theta=0$ solution is stable, and the motion for small θ is oscillatory with angular frequency

$$\omega = \sqrt{\frac{(m+M)g}{ma} - \Omega^2}$$

Case 2: $\Omega^2 > \frac{(m+M)g}{ma}$

Previous analysis shows that $\theta=0$ solution is now unstable.

Investigate the other solution: let $\theta = \alpha + \delta$
where $\cos \alpha = \frac{(m+M)g}{ma\Omega^2}$ and $|\delta| \ll 1$

Linearise ① again:

$$(m+2M\sin^2\alpha)\ddot{\delta} - m(\sin\alpha + \delta\cos\alpha)(\cos\alpha - \delta\sin\alpha)\Omega^2 + (m+M)\frac{g}{a}(\sin\alpha + \delta\cos\alpha) \approx 0$$

$$\therefore (m+2M\sin^2\alpha)\ddot{\delta} + \left\{ m\Omega^2\sin^2\alpha - m\Omega^2\alpha^2 + (m+M)\frac{g}{a}\cos\alpha \right\} \delta \approx 0$$

$$\therefore (m+2M\sin^2\alpha)\ddot{\delta} + m\Omega^2\sin^2\alpha\delta \approx 0$$

So this solution is always stable when it exists,
and the frequency of small oscillations in δ is

$$\omega = \sqrt{\frac{m\Omega^2\sin^2\alpha}{(m+2M\sin^2\alpha)}}$$