

3C7 2018

Q1 (a)  $\nabla^2 \psi = -2G\beta$ , when  $\beta$  is twist per unit length

$$\text{Here, } \nabla^2 \psi = \frac{\partial^2 \psi}{\partial x^2} + \frac{\partial^2 \psi}{\partial y^2} = 2(B+C)$$

$$\Rightarrow \beta = -(B+C)/G \quad (\text{constant } \checkmark) \quad \checkmark \quad (1)$$

Also require that  $\psi = 0$  on boundary

$$\Rightarrow \frac{\psi}{A} = 1 + \frac{B}{A}x^2 + \frac{C}{A}y^2 = 0$$

on boundary provided  $\frac{B}{A} = -\frac{1}{a^2}$  and  $\frac{C}{A} = -\frac{1}{b^2}$   
(from eqn. of the ellipse)

$$\Rightarrow \beta = A \left( \frac{1}{a^2} + \frac{1}{b^2} \right) / G$$

Conclusion:  $\psi$  satisfies  $\nabla^2 \psi = -2G\beta$  and  $\psi = 0$  on boundary if  $B = -A/a^2$  and  $C = -A/b^2$ ,  
i.e.  $\psi = A \left( 1 - \left( \frac{x}{a} \right)^2 - \left( \frac{y}{b} \right)^2 \right)$

$$(b) \quad T = 2 \int_A \psi \, dA$$

$$= 2A \int_A \left( 1 - \left( \frac{x}{a} \right)^2 - \left( \frac{y}{b} \right)^2 \right) dA$$

$$= 2A \pi ab / 2 = \pi ab A$$

$$\therefore \beta = A \left( \frac{a^2 + b^2}{a^2 b^2} \right) / G \quad (\text{from (1)})$$

$$\Rightarrow T = \frac{\pi a^3 b^3}{a^2 + b^2} G \beta$$

Stress:  $\tau_{zy} = -\partial \psi / \partial x = -2Bx = 2Ax/a^2$  ( $= 0$  when  $x=0$ )  
 $\tau_{zx} = \partial \psi / \partial y = 2Cy = -2Ay/b^2$

$$Q1 \quad (c) \quad T_{hole} = \frac{\pi a^3 b^3 (1/64) G \beta}{a^2 + b^2 (1/4)} = \frac{1}{16} T$$

$$T_{hollow} = T - T_{hole} = \frac{15}{16} \frac{\pi a^3 b^3}{a^2 + b^2} G \beta$$

Q2 (a)  $\phi = C_1 \sin 2\theta + C_2 \theta + C_3 \cos 2\theta$

To be an Airy stress function,  $\phi$  must satisfy:

$$\left( \frac{\partial^2}{\partial r^2} + \frac{1}{r} \frac{\partial}{\partial r} + \frac{1}{r^2} \frac{\partial^2}{\partial \theta^2} \right) \left( \frac{\partial^2 \phi}{\partial r^2} + \frac{1}{r} \frac{\partial \phi}{\partial r} + \frac{1}{r^2} \frac{\partial^2 \phi}{\partial \theta^2} \right) = 0$$

both zero

$$\frac{\partial^2}{\partial r^2} \left( \frac{-4C_1}{r^2} \sin 2\theta \right) = -24 C_1 \sin 2\theta / r^4$$

$$\frac{1}{r} \frac{\partial}{\partial r} \left( \frac{-4C_1}{r^2} \sin 2\theta \right) = 8 C_1 \sin 2\theta / r^4$$

$$\frac{1}{r^2} \frac{\partial^2}{\partial \theta^2} \left( \frac{-4C_1}{r^2} \sin 2\theta \right) = 16 C_1 \sin 2\theta / r^4$$

sum to zero

Similarly for the  $C_3$  term, sum = 0

$\Rightarrow$  Valid Airy stress function.

(b) i)  $\nabla_{r\theta} = \nabla_{\theta r} = 0$  on  $\theta = \pm \alpha$

ii)  $\nabla_{rr} = \frac{1}{r} \frac{\partial \phi}{\partial r} + \frac{1}{r^2} \frac{\partial^2 \phi}{\partial \theta^2} = -\frac{4C_1}{r^2} \sin 2\theta - \frac{4C_3}{r^2} \cos 2\theta$

$$\nabla_{\theta\theta} = \frac{\partial^2 \phi}{\partial r^2} = 0$$

$$\nabla_{r\theta} = \frac{\partial}{\partial r} \left( \frac{1}{r} \frac{\partial \phi}{\partial \theta} \right) = \frac{1}{r^2} (2C_1 \cos 2\theta + C_2) - \frac{1}{r^2} (2C_3 \sin 2\theta)$$

Consider boundary conditions:

$$\theta = \pm \alpha, \nabla_{r\theta} = 0 \Rightarrow 2C_1 \cos 2\alpha = -C_2 \text{ and } C_3 = 0$$

$$\theta = \pm \alpha, \nabla_{\theta\theta} = 0 \quad \checkmark$$

Summary:  $\nabla_{rr} = -\frac{4C_1}{r^2} \sin 2\theta$

$$\nabla_{r\theta} = \frac{2C_1}{r^2} (\cos 2\theta - \cos 2\alpha)$$

$$\nabla_{\theta\theta} = 0$$

$$Q2 \quad c) \quad M = \int_{-\alpha}^{\alpha} \sigma_{rr} \cdot r \cdot r d\theta = \int_{-\alpha}^{\alpha} \sigma_{rr} r^2 d\theta$$

$$= 2C_1 (\sin 2\alpha - 2\alpha \cos 2\alpha)$$

$$\Rightarrow C_1 = \frac{M}{2(\sin 2\alpha - 2\alpha \cos 2\alpha)}$$

Almost all candidates incorrectly attempted to integrate  $\sigma_{rr}$

$$\sigma_{rr} = \frac{-2M \sin 2\theta}{r^2 (\sin 2\alpha - 2\alpha \cos 2\alpha)}$$

$$\sigma_{r\theta} = \frac{M (\cos 2\theta - \cos 2\alpha)}{r^2 (\sin 2\alpha - 2\alpha \cos 2\alpha)}$$

$$\sigma_{\theta\theta} = 0$$

d) For  $\alpha$  small, simple beam theory would give:

$$\sigma = My/I \quad \text{where } y = r\theta$$

$$I = \frac{(2r\alpha)^3}{12} = \frac{2}{3} r^3 \alpha^3$$

$$\Rightarrow \sigma = \frac{3}{2} \frac{M\theta}{r^2 \alpha^3} \quad \neq$$

$$\sigma_{rr} = \frac{-2M \sin 2\theta}{r^2 (\sin 2\alpha - 2\alpha \cos 2\alpha)}$$

Expand trig functions,  $\alpha \rightarrow 0$

$$= \frac{-2M (2\theta - \frac{8\theta^3}{3!} + o(\theta^5))}{r^2 (2\alpha - \frac{8\alpha^3}{3!} - 2\alpha + 2\alpha \frac{4\alpha^2}{2!} + o(\alpha^5))}$$

$$\underbrace{\hspace{10em}}$$

must include these terms

$$= \frac{12M\theta}{8r^2 \alpha^3} \quad \neq \quad \checkmark$$

- Q3 a) 1. Stress state is symmetric about any diameter  
 2. Shear stress  $\tau_{r\theta} = 0$  along any radial line  
 3. Stresses  $\tau_{rr}$  and  $\tau_{\theta\theta}$  are function of  $r$  only  
 4. Isotropic solid

b) From data sheet:

$$\frac{d}{dr}(r\tau_{rr}) + \frac{d\tau_{r\theta}}{d\theta} - \tau_{\theta\theta} = 0 \quad (1)$$

$$\frac{d\tau_{\theta\theta}}{d\theta} + \frac{d}{dr}(r\tau_{r\theta}) + \tau_{r\theta} = 0 \quad (2)$$

Eqn(2) automatically satisfied for considered problem, eqn. (1) becomes:

$$\frac{d}{dr}(r\tau_{rr}) - \tau_{\theta\theta} = 0$$

Re-arranging,  $r \frac{d\tau_{rr}}{dr} = \tau_{\theta\theta} - \tau_{rr}$

c) Compatibility (data sheet)

$$\frac{d}{dr}\left(r \frac{d\tau_{r\theta}}{d\theta}\right) = \frac{d}{dr}\left(r^2 \frac{d\epsilon_{\theta\theta}}{dr}\right) - r \frac{d\epsilon_{rr}}{dr} + \frac{d^2\epsilon_{rr}}{d\theta^2}$$

For thin circular disk,  $\tau_{r\theta} = 0$ , and  $\epsilon_{rr}$  and  $\epsilon_{\theta\theta}$  do not depend on  $\theta$ , therefore:

$$\frac{d}{dr}\left(r^2 \frac{d\epsilon_{\theta\theta}}{dr}\right) - r \frac{d\epsilon_{rr}}{dr} = 0$$

$$\Rightarrow \underbrace{r^2 \frac{d^2\epsilon_{\theta\theta}}{dr^2}} + 2r \frac{d\epsilon_{\theta\theta}}{dr} - r \frac{d\epsilon_{rr}}{dr} = 0$$

$$\Rightarrow \frac{d}{dr}\left(r \frac{d\epsilon_{\theta\theta}}{dr}\right) - \frac{d\epsilon_{\theta\theta}}{dr} + 2 \frac{d\epsilon_{\theta\theta}}{dr} - \frac{d\epsilon_{rr}}{dr} = 0$$

$$\Rightarrow \frac{d}{dr}\left(r \frac{d\epsilon_{\theta\theta}}{dr}\right) = \frac{d}{dr}(\epsilon_{rr} - \epsilon_{\theta\theta})$$

$$\Rightarrow r \frac{d\epsilon_{\theta\theta}}{dr} = \epsilon_{rr} - \epsilon_{\theta\theta}$$

Q3 d)  $\nabla_{rr} = A - \frac{B}{r^2} - \frac{E\alpha}{r^2} \int_0^r r T dr$

$$\nabla_{\theta\theta} = A + \frac{B}{r^2} + \frac{E\alpha}{r^2} \int_0^r r T dr - E\alpha T$$

$$(T = T(r) - T_0)$$

e)  $\frac{\partial}{\partial r} \left[ \frac{1}{r} \frac{\partial(ur)}{\partial r} \right] = (1+\nu)\alpha \frac{\partial(T-T_0)}{\partial r}$

Integrate once,

$$\frac{1}{r} \frac{\partial(ur)}{\partial r} = (1+\nu)\alpha (T-T_0) + 2C_2$$

$$\Rightarrow \frac{\partial(ur)}{\partial r} = (1+\nu)\alpha (T-T_0)r + 2C_2r$$

Integrate again,

$$ur = (1+\nu)\alpha \int_0^r r (T-T_0) dr + C_2 r^2 + C_1$$

$$\Rightarrow u = C_2 r + \frac{C_1}{r} + (1+\nu)\frac{\alpha}{r} \int_0^r r (T-T_0) dr$$

Q4 a) Stress tensor (matrix)

$$\underline{\underline{\sigma}} = \begin{bmatrix} Y/2 & \tau & 0 \\ \tau & 0 & 0 \\ 0 & 0 & 0 \end{bmatrix}$$

Mohr circle:



$$|\sigma_2 - \sigma_1| = 2\sqrt{\tau^2 + Y^2/16} = Y$$

$$\begin{aligned} \tau^2 + Y^2/16 &= Y^2/4 \\ \Rightarrow \tau &= \pm \frac{\sqrt{3}Y}{4} \end{aligned}$$

Principal strain increments (from deformed sheet)

$$d\varepsilon_1 : d\varepsilon_2 : d\varepsilon_3 = \lambda (1 : -1 : 0)$$

$$b) i) \sigma_{rr} = A - B/r^2, \quad \sigma_{\theta\theta} = A + B/r^2$$

$$\text{Plane strain: } 0 = \varepsilon_{zz} = \frac{\sigma_{zz}}{E} - \frac{\nu}{E} (\sigma_{rr} + \sigma_{\theta\theta})$$

$$\Rightarrow \sigma_{zz} = \nu (\sigma_{rr} + \sigma_{\theta\theta})$$

Far field boundary condition:

$$\text{As } r \rightarrow \infty \quad \sigma_{rr} = \sigma_{\theta\theta} = 0 \Rightarrow A = 0$$

$$\text{At } r = a: \sigma_{rr} = -p \Rightarrow B = pa^2$$

$$\Rightarrow \sigma_{rr} = -pa^2/r^2, \quad \sigma_{\theta\theta} = pa^2/r^2$$

Since  $\sigma_{rr} < \sigma_{zz} < \sigma_{\theta\theta}$ , yield occurs  
when  $\sigma_{\theta\theta} - \sigma_{rr} = Y$

Q4 (b) (cont)

i)  $2pa^2/r^2 = Y \Rightarrow p = Yr^2/(2a^2) \quad (1)$

$$\Rightarrow \sigma_{rr} = \frac{-Ya^2}{2r^2}, \quad \sigma_{\theta\theta} = \frac{Ya^2}{2r^2}$$

$$(\sigma_{rr} = -Y/2, \quad \sigma_{\theta\theta} = Y/2 \quad @ \quad r=a)$$

c) Equilibrium must always be satisfied,

$$\therefore \frac{d\sigma_{rr}}{dr} = \frac{\sigma_{\theta\theta} - \sigma_{rr}}{r} = \frac{Y}{r}$$

Consider a plastic tube with radius  $c$  nested in an elastic hole. The interface pressure between them is  $-p_i$ .

$$\int_{-p}^{-p_i} \frac{d\sigma_{rr}}{dr} dr = Y \int_a^c \frac{dr}{r}$$

$$p - p_i = Y \ln\left(\frac{c}{a}\right)$$

$$p_i = p - Y \ln\left(\frac{c}{a}\right)$$

This is still elastic for  $r > c$ , hence from (1)

$$p_i = Y/2$$

Equating (1) and (2),

$$p = Y\left(\frac{1}{2} + \ln\left(\frac{c}{a}\right)\right)$$

GNW