

1)

$$a) \quad \sigma_{\theta\theta} = \frac{d}{dr} (r\sigma_{rr}) + \rho\omega^2 r^2$$

$$r\sigma_{rr} = \frac{\rho\omega^2}{32} (3D^2 r - 12r^3) + \frac{\nu\rho\omega^2}{32} (D^2 r - 4r^3)$$

$$\frac{d}{dr} (r\sigma_{rr}) = \frac{\rho\omega^2}{32} (3D^2 - 36r^2) + \frac{\nu\rho\omega^2}{32} (D^2 - 12r^2)$$

$$= \sigma_{\theta\theta} - \rho\omega^2 r^2 \quad \checkmark \quad (\text{in equilibrium})$$

Check for compatibility

$$\epsilon_{rr} = \frac{1}{E} (\sigma_{rr} - \nu\sigma_{\theta\theta})$$

$$= \frac{\rho\omega^2}{32E} (3D^2 - 12r^2) + \frac{\nu\rho\omega^2}{32E} \overbrace{(D^2 - 4r^2 - 3D^2 + 4r^2)}^{-2D^2}$$

$$- \frac{\nu^2\rho\omega^2}{32E} (D^2 - 12r^2)$$

$$u = r\epsilon_{\theta\theta} = \frac{r}{E} (\sigma_{\theta\theta} - \nu\sigma_{rr})$$

$$= \frac{\rho\omega^2}{32E} (3D^2 r - 4r^3) + \frac{\nu\rho\omega^2}{32E} (D^2 r - 12r^3 - 3D^2 r + 12r^3)$$

$$= \frac{\rho\omega^2}{32E} (D^2 r - 4r^3)$$

$$\frac{du}{dr} = \frac{\rho \omega^2}{32E} (3D^2 - 12r^2) + \frac{\nu \rho \omega^2}{32E} (-2D^2) - \frac{\nu^2 \rho \omega^2}{32} (D^2 - 12r^2)$$

$$= \epsilon_{rr} \quad \text{ie compatibility } \checkmark$$

Boundary conditions

$$\sigma_{rr} \Big|_{r=0} = \frac{3D^2 \rho \omega^2}{32} + \frac{\nu D^2 \rho \omega^2}{32} = \sigma_{\theta\theta} \Big|_{r=0}$$

$$\& \sigma_{rr} \Big|_{r=\frac{D}{2}} = \frac{\rho \omega^2}{32} (3D^2 - 3D^2) + \frac{\nu \rho \omega^2}{32} (D^2 - D^2) = 0$$

(b)  $\sigma_{rr}$  &  $\sigma_{\theta\theta} > 0$  ~~the~~ for all  $r$ . This is plane stress  
 $\Rightarrow$  active yield condition

$$\begin{array}{c} \sigma_{rr} - \sigma_{zz} = \tau \\ \parallel \\ \sigma_{\theta\theta} \end{array} \quad \text{at } r=0$$

$$\text{Yield} \quad \frac{\rho \omega^2}{32} 3D^2 + \frac{\nu \rho \omega^2}{32} D^2 = \tau$$

$$\omega^2 = \frac{32\tau}{\rho D^2(3+\nu)}$$

$$\Rightarrow \omega = 1865.1 \text{ rad/s}$$

Displacement  $u$  at  $r = \frac{D}{2}$

$$u \Big|_{r=\frac{D}{2}} = \frac{\rho \omega^2 D^3}{32E} \left[ \frac{3}{2} - \frac{4}{8} + \nu(-1) - \nu^2 \left( \frac{1}{2} - \frac{1}{2} \right) \right]$$

$$= \frac{\rho \omega^2 D^3}{27E} (1-\nu)$$

For die to touch casing  $u_c = 0.005 \text{ m}$

$$\Rightarrow \omega^2 = \frac{32E}{\rho D^3(1-\nu)} u_c$$

$$\omega = 9780.8 \text{ rad/s}$$

ie yielding occurs first

2

(a) Must satisfy  $\nabla^4 \phi = 0$

$$\frac{\partial^2 \phi}{\partial x^2} = 0; \quad \frac{\partial^3 \phi}{\partial y^2} = 2A + 6Cy + 6Dxy$$

$$\frac{\partial}{\partial x^2} (2A + 6Cy + 6Dxy) = 0$$

$$\frac{\partial^2}{\partial y^2} (2A + 6Cy + 6Dxy) = 0 \Rightarrow \nabla^4 \phi = 0$$

$$\sigma_{xx} = \frac{\partial^2 \phi}{\partial y^2} = 2A + 6Dxy + 6Cy$$

$$\sigma_{yy} = \frac{\partial^2 \phi}{\partial x^2} = 0$$

$$\begin{aligned} \sigma_{xy} &= -\frac{\partial^2 \phi}{\partial x \partial y} = -\frac{\partial}{\partial y} (By + Dy^3) \\ &= -B - 3Dy^2 \end{aligned}$$

(b) Boundary conditions:  $\sigma_{yy} = 0$  @  $y = \pm \frac{d}{2}$  ✓

$$\& \sigma_{xy} = 0 \text{ @ } y = \pm \frac{d}{2}$$

$$-B - 3\frac{Dd^2}{4} = 0$$

$$B = -\frac{3Dd^2}{4}$$

$$\sigma_{xy} = -B \left( 1 + \frac{3D}{B} y^2 \right) = \frac{3D}{4} (d^2 - 4y^2)$$

$$e_{xz} = 0 \quad T = \int_{-d/2}^{d/2} \sigma_{xx} b \, dy$$

$$= b \left[ 2Ay + 3Cy^2 \right]_{-d/2}^{d/2}$$

$$= b \left[ 2Ad \right]$$

$$A = \frac{T}{2bd}$$

$$M = b \int_{-d/2}^{d/2} \sigma_{xx} y \, dy$$

$$= b \left[ 2Ay^2 + 2Cy^3 \right]_{-d/2}^{d/2}$$

$$M = \frac{bCd^3}{2}$$

$$C = \frac{2M}{bd^3}$$

$$S = b \int_{-d/2}^{d/2} \sigma_{xy} \, dy$$

$$S = bD \left[ \frac{3d^2y}{4} - y^3 \right]_{-d/2}^{d/2}$$

$$= bD \left[ \frac{3}{4}d^3 - \frac{d^3}{4} \right] = \frac{Dbd^3}{2} \Rightarrow D = \frac{2S}{bd^3}$$

(c)  
For  $M = T = 0$

$$\sigma_{xx} = \frac{12S}{b-d^3} xy \quad ; \quad \sigma_{xy} = \frac{6S}{4bd^3} (d^2 - 4y^2)$$
$$= \frac{3S}{2bd^3} (d^2 - 4y^2)$$

Ⓒ  $x = l$  &  $y = d/2$

$$\sigma_{xx} = \frac{12S}{bd^3} \frac{6S}{bd^2} l \quad ; \quad \sigma_{xy} = 0$$

$$Y = \frac{6Sl}{bd^2} \Rightarrow S = \frac{Ybd^2}{6l}$$

Ⓒ  $x = l$  &  $y = 0$

$$\sigma_{xx} = 0 \quad ; \quad \sigma_{xy} = \frac{3S}{2bd} = \frac{Y}{2}$$

$$S = \frac{Ybd}{3}$$

For yield to occur simultaneously

$$\frac{d}{6l} = \frac{1}{3} \Rightarrow l = \frac{d}{2}$$

Result not meaningful as (i) root close to loading

(ii) Displacement boundary conditions at  $x = l$  most probably are not satisfied.

3

$$(a) \quad \nabla^2 \psi = -2G\beta$$

$$\frac{\partial^2 \psi}{\partial x^2} = -\frac{2m}{a^2} ; \quad \frac{\partial^2 \psi}{\partial y^2} = -\frac{2m}{b^2}$$

$$\nabla^2 \psi = \frac{-2m(a^2 + b^2)}{a^2 b^2} = -2G\beta$$

$$\text{ie } m = \frac{G\beta a^2 b^2}{a^2 + b^2} \quad \checkmark$$

also must satisfy traction = 0 on boundary ie  $\psi = \text{constant}$

$$\text{on } \frac{x^2}{a^2} + \frac{y^2}{b^2} = 1 \quad \text{Yes with } \psi = 0. \quad \checkmark$$

$$(b) \quad T = 2 \int_A \psi dA$$

$$= 2m \int_A \left(1 - \frac{x^2}{a^2} - \frac{y^2}{b^2}\right) dA = \frac{2m\pi ab}{2}$$

$$T = \frac{G\beta a^2 b^2}{a^2 + b^2} \pi ab$$

$$\beta = \frac{T}{G} \frac{a^2 + b^2}{\pi a^3 b^3}$$

Max shear stress  $x=0, y=b$

$$\frac{\partial \psi}{\partial y} = -\frac{2my}{b^2} \Big|_{y=b} = -\frac{2m}{b}$$

ie magnitude of shear stress  $\frac{2m}{b} = \frac{2G\beta a^2 b^2}{a^2 + b^2}$

$$= \frac{2T}{\pi a b^2}$$

(c) Consider  $\psi = m \left(1 - \frac{y^2}{b^2}\right)$  over rectangle

$$\nabla^2 \psi = \frac{\partial^2 \psi}{\partial y^2} = -\frac{2m}{b^2} = -2G\beta$$

$$m = G\beta b^2$$

$$T = 2 \int \psi dA = 2m \cdot 2a \cdot \frac{4b}{3} = \frac{16mab}{3}$$

$$T = \frac{16G\beta a b^3}{3}$$

~~For ellipse~~ For ellipse  $\frac{T}{\beta} = \frac{G\pi a^3 b^3}{a^2 + b^2}$

$$a \gg b = G\pi a b^3$$

$$\Rightarrow \text{rectangle is } \frac{16}{3\pi} = 1.7 \text{ stiffer}$$

m same in both cases when  $a \gg b$

Peak stress =  $2m$  in both cases.



4)

(a) Use Lamé's equations

$$\sigma_{rr} = A - \frac{B}{r^2}, \quad \sigma_{\theta\theta} = A + \frac{B}{r^2}$$

$$r \rightarrow \infty \quad \sigma_{rr} = A = T$$

$$r = a \quad \sigma_{rr} = 0$$

$$\Rightarrow 0 = T - \frac{B}{a^2} \Rightarrow B = Ta^2$$

$$\sigma_{rr} = T \left( 1 - \frac{a^2}{r^2} \right); \quad \sigma_{\theta\theta} = T \left( 1 + \frac{a^2}{r^2} \right)$$

$$\sigma_{\theta\theta} > \sigma_{rr} > \sigma_{zz} = 0$$

At edge of hole by Tresca  $\sigma_{\theta\theta} - \sigma_{zz} = T$

$$2T = T$$

$$T = \frac{T}{2} \quad \checkmark$$

(b) In elastic region  $\sigma_{rr} = A_1 - \frac{B_1}{r^2}$

$$\sigma_{\theta\theta} = A_1 + \frac{B_1}{r^2}$$

$$r \rightarrow \infty, \sigma_{rr} \rightarrow A_1 \Rightarrow A_1 = T$$

In plastic region  $\sigma_{\theta\theta} = T$  (from a)

$$\frac{d}{dr} (r \sigma_{rr}) = \sigma_{\theta\theta} = T$$

$$\sigma_{rr} = T + \frac{D}{r}$$

$$\text{at } r=0 \quad \sigma_{rr}=0 \Rightarrow 0 = T + \frac{D}{a}$$

$$\sigma_{rr} = T \left(1 - \frac{a}{r}\right)$$

$$\sigma_{\theta\theta} = T$$

at  $r=c$

$$\sigma_{rr}: \quad T \left(1 - \frac{a}{c}\right) = T - \frac{B_1}{c^2} \quad (1)$$

$$\sigma_{\theta\theta}: \quad T = T + \frac{B_1}{c^2} \quad (2)$$

$$(1) + (2) \Rightarrow T \left(2 - \frac{a}{c}\right) = 2T$$

$$\frac{c}{a} = \frac{T}{2(T-T)}$$

$$\& B_1 = \frac{Tac}{2}$$

(c)

$$\text{For } T = \frac{3}{4} \gamma$$

$$\frac{c}{a} = 2, \quad B_1 = \gamma a^2$$

In plastic zone:  $a \leq r \leq 2a$

$$\sigma_{rr} = 1 - \frac{a}{r}; \quad \sigma_{\theta\theta} = T$$

In elastic zone  $r > 2a$

$$\frac{\sigma_{rr}}{\gamma} = 0.75 - \frac{a^2}{r^2}; \quad \frac{\sigma_{\theta\theta}}{\gamma} = 0.75 + \frac{a^2}{r^2}$$

