Question 1

(a) Note the foce at x = 0 is zero and the force at x = L is -ku(x = L). Therefore, Boundary conditions:

$$EA\frac{\partial u}{\partial x}(0,t) = 0,$$
$$EA\frac{\partial u}{\partial x}(L,t) = -ku(L,t).$$

At t = 0, the bar at time zero travels at velocity v_0 , therefore the initial conditions are:

$$u(x,0) = 0,$$
$$\dot{u}(x,0) = v_0.$$

(b) The governing equation of the bar is given by

$$EA\frac{\partial^2 u}{\partial x^2} = \rho A \ddot{u}$$

Therefore, let w = w(x) be the weight function, the weak formulation is given by

$$\int_0^L EA \frac{\partial u}{\partial x} \frac{\partial w}{\partial x} dx + \int_0^L \rho A \ddot{u} w dx = w EA \frac{\partial u}{\partial x} \Big|_0^L.$$

Plug in the Neumann boundary condition from part (a), the weak formulation of the problem is given by

$$\int_0^L EA \frac{\partial u}{\partial x} \frac{\partial w}{\partial x} dx + \int_0^L \rho A \ddot{u} w dx = -wku(L,t).$$

(c) The stiffness matrix of a single element is given by

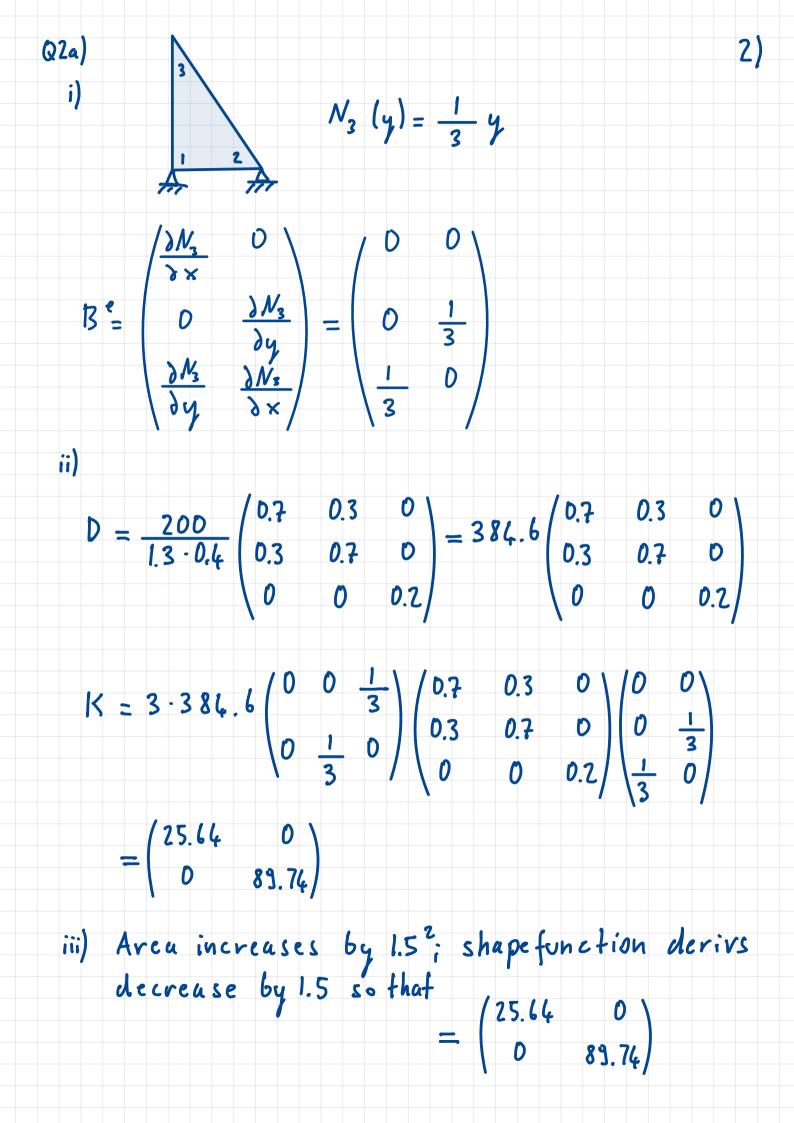
$$K = \frac{EA}{L} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix},$$

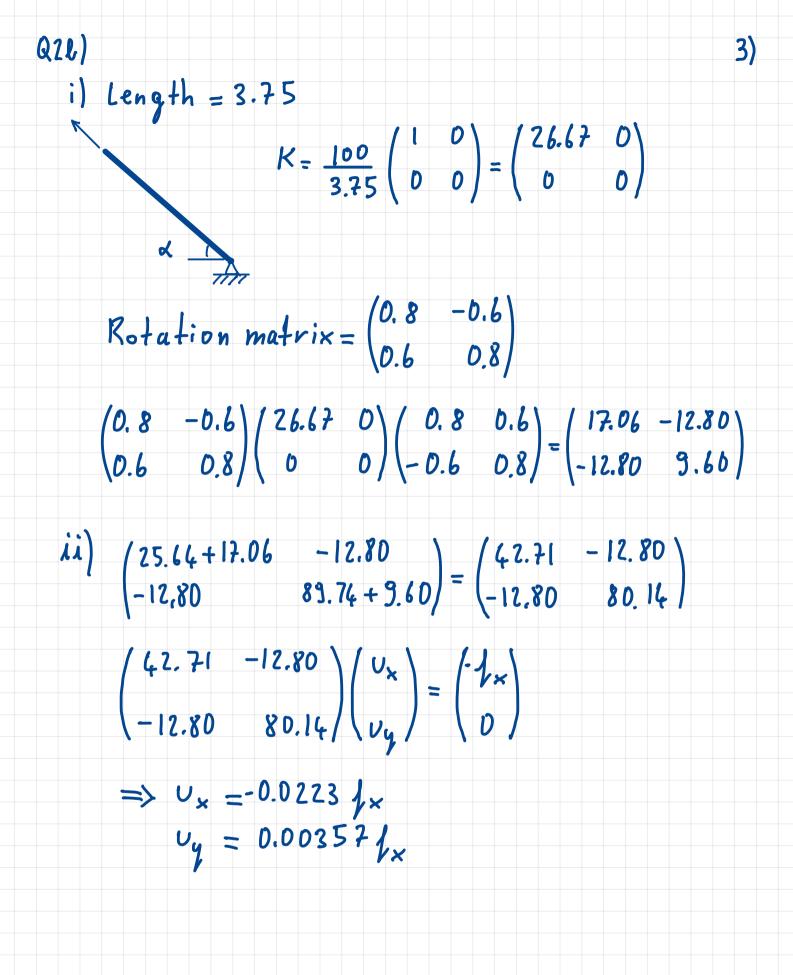
while the element mass matrix is given by

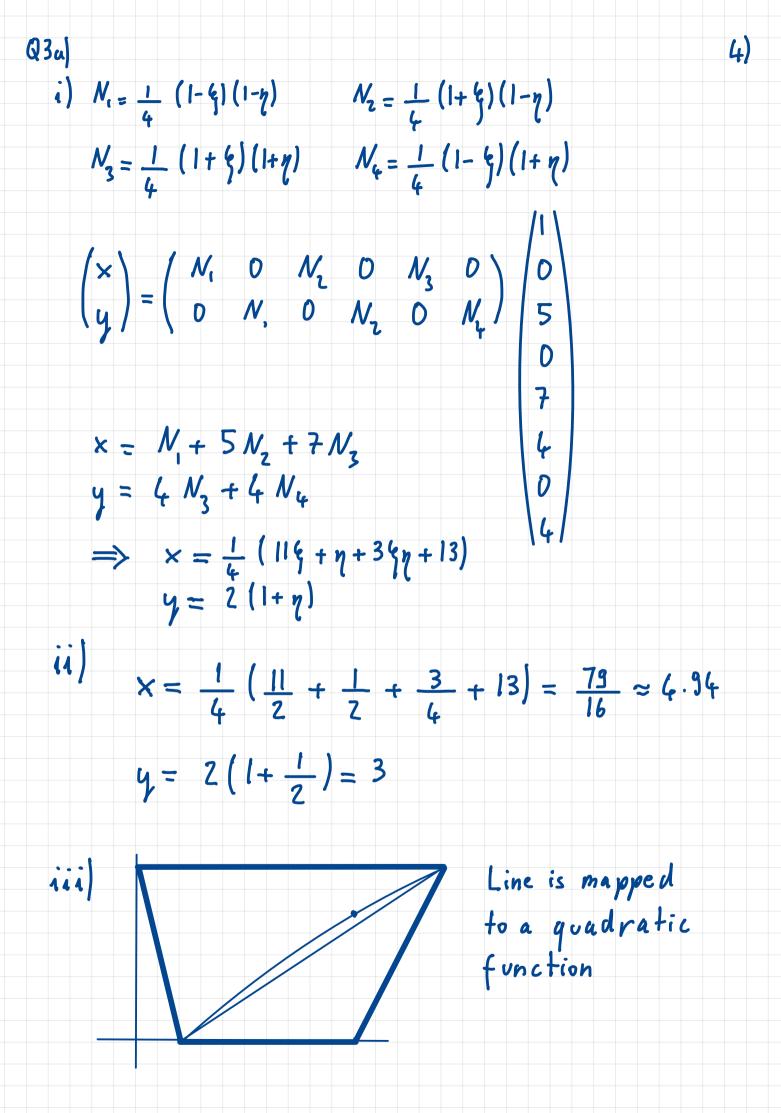
$$M = \frac{\rho AL}{6} \begin{bmatrix} 2 & 1\\ 1 & 2 \end{bmatrix}.$$

Let u_1, u_2 denote the nodal displacement of the element at x = 0 and x = L, respectively. The semi discrete formulation of the problem is then

$$\frac{EA}{L} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix} \begin{bmatrix} u_1 \\ u_2 \end{bmatrix} + \frac{\rho AL}{6} \begin{bmatrix} 2 & 1 \\ 1 & 2 \end{bmatrix} \begin{bmatrix} \ddot{u}_1 \\ \ddot{u}_2 \end{bmatrix} = \begin{bmatrix} 0 \\ -ku_2 \end{bmatrix}.$$







iv) The mapping from the physical element 5) to the parent element requires the solution of a nonlinear equation. That is we must compute the roots of a nonlinear function, which can be done by e.g. plotting and trial and error.

Q31) In index notation: $\int_{\Sigma} U_{i} W_{i} + \int_{\Sigma} C_{i} U_{i} W + \int_{\Sigma} f_{i} W = 0$ $\int_{\Omega} (v_{ii} w)_{ii} = \int_{\Sigma} v_{iii} w + \int_{\Sigma} v_{ii} w_{ii}$ $-\int_{\mathcal{N}} u_{i\hat{e}\hat{e}} W + \int W u_{i\hat{e}} n_{\hat{e}} + \int c_{\hat{e}} u_{i\hat{e}} W + \int \int W = 0$ $\int (-v_{ii} + c_{i}v_{i} + f_{i}w + \int w v_{i}n_{i} = 0$ $-v_{iii} + c_{i}v_{ii} + f = 0$ on N $v_{i} = 0$ on Γ_N υ - Ο on _p

Question 4

(a) A priori error estimation is the analysis of the errors in a finite element simulation before actually performing an analysis. It can tell whether the finite element solution will converge to the exact solution as the mesh is refined, and how quickly it will converge if the element type is changed or the mesh is refined.

A posterior error estimation involves quantifying the error after a simulation has been performed. With an estimate of the error, a judgement can be made on the reliability of the result.

(b) The two strategies are the h-adaptivity and p-adaptivity. H-adaptivity changes the finite element mesh, while p-adaptivity changes the polynomial order of the finite element shape functions to reduce the error.

(c) The error estimate is given by

$$||u - u_h||_s \le Ch^{\beta} ||u||_{k+1}.$$

Therefore, with polynomial order k = 3, L_2 -norm s = 2, $\beta = k + 1 - s = 2$, the a-priori error estimate is

$$||u - u_h||_2 \le Ch^2 ||u||_4$$

(d) The Crank-Nicolson approximation of \dot{y} is given by

$$y_{n+1} = y_n - 0.5\lambda\Delta t(y_n + y_{n+1}),$$

which can be rewritten as

$$y_{n+1} = \frac{1 - 0.5\lambda\Delta t}{1 + 0.5\lambda\Delta t} y_n$$

Hence, the amplitude factor is given by

$$A = \left| \frac{1 - 0.5\lambda \Delta t}{1 + 0.5\lambda \Delta t} \right| < 1.$$

Consequently, the solution is unconditionally stable.