EGT3
ENGINEERING TRIPOS PART IIB

Module 4A10

## FLOW INSTABILITY

Answer not more than three questions.
All questions carry the same number of marks.
The approximate percentage of marks allocated to each part of a question is indicated in the right margin.

Write your candidate number not your name on the cover sheet.

## STATIONERY REQUIREMENTS

Single-sided script paper

## SPECIAL REQUIREMENTS TO BE SUPPLIED FOR THIS EXAM

CUED approved calculator allowed
Attachment: 4A10 data sheet (two pages)
Engineering Data Book

10 minutes reading time is allowed for this paper at the start of the exam.

You may not start to read the questions printed on the subsequent pages of this question paper until instructed to do so.

You may not remove any stationery from the Examination Room.

## Version GRH02

1 A thin annular layer of liquid, referred to as a liquid film, deposited on a wire of radius $a$, as depicted in Fig. 1, may become unstable due to surface tension $\gamma$ and lead to the formation of isolated drops. The thin film, of viscosity $\mu$, has thickness $h+\eta$, where $h$ is the thickness of the undisturbed annular film and $\eta(x, t)$ is an axisymmetric disturbance on the film. In the limit of small film thickness $(h \ll a)$ and ignoring gravity, you may assume that the equations governing the evolution of the velocity and pressure perturbations of the film are:

$$
-\frac{\partial p}{\partial x}+\mu \frac{\partial^{2} u}{\partial r^{2}}=0 \quad \text { where } p=p(x), \quad \frac{\partial v}{\partial r}=-\frac{\partial u}{\partial x}
$$

Here, $p$ denotes pressure, $x$ the horizontal coordinate along the wire, $r$ the radial coordinate, $u$ the horizontal velocity component and $v$ the radial velocity component. The boundary conditions are: no-slip on the wire, i.e. $u=v=0$ on $r=a$; and the following linearised conditions at the free surface where $r=a+h$ :

$$
\mu \frac{\partial u}{\partial r}=0, \quad p=\gamma\left(-\frac{\eta}{a^{2}}-\frac{\partial^{2} \eta}{\partial x^{2}}\right), \quad \frac{\partial \eta}{\partial t}=v
$$



Fig. 1
(a) By introducing normal mode perturbations to the film thickness of the form $\eta=\hat{\eta} e^{i k x+s t}$, where $k$ is the wavenumber in the $x$-direction, $t$ is time and $\hat{\eta}$ is a constant, show that the growth rate $s$ of the disturbances is:

$$
s=\left(\frac{h}{a}\right)^{3} \frac{\gamma}{3 \mu a}(k a)^{2}\left(1-(k a)^{2}\right)
$$

(b) Discuss the stability and, in particular, determine the most amplified and the cut-off wavenumbers.

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2 (a) Explain the principles and approach behind a linear stability analysis. Include a description of why a normal mode analysis may be regarded as enabling the influence of all possible small amplitude disturbances to be assessed.
(b) Consider a two-dimensional inviscid mixing layer with velocity profile

$$
\boldsymbol{u}= \begin{cases}U_{1} \boldsymbol{i} & \text { for } z>0 \\ U_{2} \boldsymbol{i} & \text { for } z<0\end{cases}
$$

as shown in Fig. 2.


Fig. 2

Investigate the temporal stability of this flow by considering small disturbances to the interface of the vortex sheet of the form

$$
\eta=\hat{\eta} e^{i k x+s t}
$$

to show that the dispersion relationship is given by

$$
s=-\frac{1}{2} i k\left(U_{1}+U_{2}\right) \pm \frac{1}{2} k\left(U_{1}-U_{2}\right) .
$$

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3 The inviscid, uniform density, flow behind a bluff body is to be modelled as an unconfined wake as shown in Fig. 3. The two shear layers are 2 units apart and the velocity ratio $\Lambda$ is equal to -1.4 , where $\Lambda \equiv\left(U_{1}-U_{2}\right) /\left(U_{1}+U_{2}\right)$. Sinuous and varicose waves are considered with wavenumber $k$ and angular frequency $\omega$. Figure 4 shows contours of $\omega_{i}$ (top) and $\omega_{r}$ (bottom) in the complex $k$-plane for sinuous waves (left) and varicose waves (right).
(a) With reference to Fig. 4, determine the stability of this flow and describe its motion if allowed to evolve from the steady flow shown in Fig. 3.
(b) A splitter plate is added at the line of symmetry such that vertical flow through the line of symmetry is prohibited. With reference to Fig. 4, describe the effect this will have on the flow.
(c) The splitter plate has Youngs Modulus $E$, moment of inertia per unit distance into the page $I$, and mass per unit length $m$. The reverse flow around the splitter plate exerts a shear stress $\tau$ on the splitter plate, causing a (negative) tension $T$. Defining $x=0$ at the downstream end of the bluff body, the equation for the vertical displacement, $Y(x, t)$, of the splitter plate is:

$$
E I \frac{\partial^{4} Y}{\partial x^{4}}-T \frac{\partial^{2} Y}{\partial x^{2}}+m \frac{\partial^{2} Y}{\partial t^{2}}=0
$$

If the splitter plate has length $L$, consider the local stability properties of the splitter plate in terms of $\omega$ and $k$, and estimate the length of the plate that is locally unstable to buckling.
(d) Assume that the plate becomes globally unstable to buckling if the locally unstable length, $D$, exceeds one quarter wavelength. By considering the local stability as a function of $k$, and using a graph, sketch and describe qualitatively the global stability of the splitter plate.


Fig. 3





Fig. 4

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4 The added mass of an object is the mass that is added to the object in order to account for the influence of the motion of the fluid around the object.
(a) When a cylinder with radius $a$ moves at speed $V$ through an otherwise stationary fluid, the kinetic energy per unit length of the fluid is $\rho V^{2} \pi a^{2} / 2$. By considering the work done when accelerating the cylinder, or otherwise, calculate the added mass per unit length of the cylinder.
(b) In a chemical plant, mercury with density $\rho$ flows down a channel at uniform speed, $U$. As shown in Fig. 5, a flexible cylindrical thermometer with negligible mass, length $l$, and radius $a$ is dipped to a depth $h$ into the mercury, where $h \ll l$. The thermometer's flexibility is modelled by a torsional spring at its top, which causes restoring moment $k \theta$. When $U=0$, show that the natural frequency, $\omega_{n}$, of vibrations of the thermometer satisfies $\omega_{n}^{2} \approx k /\left(\rho \pi a^{2} l h\right)$.


Fig. 5
(c) When flowing, the mercury has $U=0.01 \mathrm{~ms}^{-1}$. If $a=0.005 \mathrm{~m}$, calculate the frequency of vortex shedding around the cylinder. For mercury, the density is 13600 $\mathrm{kgm}^{-3}$ and the viscosity is $1.53 \times 10^{-3} \mathrm{Nsm}^{-2}$. The Strouhal number of vortex shedding in a flow at speed $U$ around a cylinder of diameter $D$ is approximately $f D / U=0.2$ for Reynolds numbers from $10^{2}$ to $10^{5}$.
(d) Model and discuss the behaviour of the thermometer as the depth, $h$, increases.

Discuss any defects in your model.

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