EGT3
ENGINEERING TRIPOS PART IIB

Tuesday 3 May 20229.30 to 11.10

## Module 4A15

## AEROACOUSTICS

Answer not more than three questions.
All questions carry the same number of marks.
The approximate percentage of marks allocated to each part of a question is indicated in the right margin.

Write your candidate number not your name on the cover sheet.

## STATIONERY REQUIREMENTS

Single-sided paper

## SPECIAL REQUIREMENTS TO BE SUPPLIED FOR THIS EXAM

CUED approved calculator allowed
Attachment: 4A15 Aeroacoustics data sheet (5 pages)
Engineering Data Book

10 minutes reading time is allowed for this paper at the start of the exam.

You may not start to read the questions printed on the subsequent pages of this question paper until instructed to do so.

You may not remove any stationery from the Examination Room.

## Version AA/7

1 Power lines sometimes 'sing' in windy conditions because, when the wind flows transversely to the wire, it can induce a periodic stream of vortices downstream of the wire. The periodic vortex stream generates a periodic force that produces sound. Assume that the force per unit length exerted by the wire on the fluid has a magnitude of $f \cos (\omega t)$ in a direction perpedicular to the flow, where $\omega$ is the angular frequency, and the wire can be treated as spatially compact in all directions, and thus a point source. Let $L$ be the length of the wire (into the page).
(a) Show that the farfield sound generated by the wire is given by

$$
p^{\prime}(\mathbf{x}, t)=-\frac{1}{4 \pi} \frac{L \cos \theta}{r} \frac{\omega}{c_{0}} f \sin \left(\omega\left(t-r / c_{0}\right)\right)
$$

where $c_{0}$ is the speed of sound, $r=|\boldsymbol{x}|$ and $\theta$ is the angle between $\boldsymbol{x}$ and $\boldsymbol{f}$ (see Fig. 1).
(b) Find the time-averaged power of the sound radiated by the wire.
(c) Would this result be valid for any value of $L$ ? Explain your answer.


Fig. 1

## Version AA/7

2 A conical horn is attached to a cylindrical duct to increase the sound transmission out of the duct, as shown in Fig. 2. Assume that the waves propagating in the duct are plane with angular frequency $\omega$, and in the horn are segments of spherical waves that originate from a virtual apex, $A$. The distance of $A$ from the connection between the duct and the horn is $r_{0}$. The terminating impedance of the duct can be assumed to be the impedance presented by the waves in the horn.
(a) Show that the terminating impedance of the duct is given by

$$
Z=\rho_{0} c_{0} \frac{i k r_{0}}{1+i k r_{0}}
$$

where $k=\omega / c_{0}$ and $\rho_{0}$ and $c_{0}$ are mean density and speed of sound, respectively.
(b) The reflection coefficient, $R$ is defined as the ratio of the amplitude of the reflected wave to the incident wave. Show that, for waves travelling down the tube toward the horn, $R=-1 /\left(1+i 2 k r_{0}\right)$
(c) Using the result in part (b), comment on the effectiveness of the horn in aiding the transmission of the waves out of the cylindrical duct.


Fig. 2

## Version AA/7

3 (a) Explain the meaning of the term "cut-off" in connection with acoustic modes in a duct.
(b) A 3-bladed fan of diameter 300 mm is to be operated in a cylindrical duct of circular cross-section of the same diameter. Table 1 shows the values of $z_{m n}$, the $m^{\text {th }}$ zero of $d J_{n}(z) / d z$, where $J_{n}$ is the $n^{\text {th }}$ order Bessel function of the first kind. For $|n|>6$, use $z_{1 n} \approx|n|+0.80861|n|^{1 / 3}$. Use the data in Table 1 to determine $R_{\max }$, the maximum number of revolutions per minute if all rotor-alone modes are to be cut-off at atmospheric conditions. Formulae on the data sheet may be used without proof.
(c) The fan rotor in (b) is operated at $10,000 \mathrm{rpm}$. Choose a suitable number of blades for a downstream stator row, explaining clearly the reasons for your choice. With your choice of stator blade number which, if any, of the rotor-stator interaction modes at the blade-passing frequency (bpf) and at 2 bpf propagate?

|  | $n=0$ | $n= \pm 1$ | $n= \pm 2$ | $n= \pm 3$ | $n= \pm 4$ | $n= \pm 5$ | $n= \pm 6$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| $m=1$ | 0.00000 | 1.84118 | 3.05424 | 4.20119 | 5.31755 | 6.41562 | 7.50127 |
| $m=2$ | 3.83170 | 5.33144 | 6.70613 | 8.01524 | 9.28240 | 10.51986 | 11.73494 |
| $m=3$ | 7.01558 | 8.53632 | 9.96947 | 11.34592 | 12.68191 | 13.98719 | 15.26818 |
| $m=4$ | 10.17346 | 11.70600 | 13.17037 | 14.58585 | 15.96411 | 17.31284 | 18.63744 |
| $m=5$ | 13.32369 | 14.86359 | 16.34752 | 17.78875 | 19.19603 | 20.57551 | 21.93172 |

Table 1

## Version AA/7

4 To attenuate sound of angular frequency $\omega$ travelling as plane waves in a duct of cross-sectional area $S$, a Helmholtz resonator with volume $V$ is connected to the side wall, as shown in Fig. 3. The neck of the resonator has a length $l$ and cross-sectional area $A$. Show that the transmission loss, $L_{T}$, is given by

$$
L_{T}=10 \log _{10}\left(\frac{|I|^{2}}{|T|^{2}}\right)=10 \log _{10}\left(1+\frac{1}{4 S^{2}}\left(\frac{c_{0}}{\omega V}-\frac{\omega l}{c_{0} A}\right)^{-2}\right)
$$

where $I$ and $T$ are the strengths of the incident and transmitted waves and $c_{0}$ is the speed of sound.

Hint: apply conditions of 1) continuity of mass flow rate into and out of the control volume across the neck of the Helmholtz resonator and the duct upstream and downstream of it, and 2) matching of pressure at $x=0$.


Fig. 3

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Version AA/7

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Page 6 of 6

## Module 4A15 Aeroacoustics Data Sheet

## USEFUL DATA AND DEFINITIONS

## Physical Properties

Speed of sound in an ideal gas $\sqrt{\gamma R T}$, where $T$ is temperature in Kelvins

## Units of sound measurement

$$
\begin{aligned}
\text { SPL (sound pressure level) } & =20 \log _{10}\left(\frac{p_{r m s}^{\prime}}{2 \times 10^{-5} \mathrm{Nm}^{-2}}\right) \mathrm{dB} \\
\text { IL (intensity level) } & =10 \log _{10}\left(\frac{\text { intensity }}{10^{-12} \text { watts } \mathrm{m}^{-2}}\right) \mathrm{dB} \\
\text { PWL (power level) } & =10 \log _{10}\left(\frac{\text { sound power output }}{10^{-12} \text { watts }}\right) \mathrm{dB}
\end{aligned}
$$

## Definitions

Surface impedance $Z_{s}$, relates the pressure perturbation applied to a surface, $p^{\prime}$, to its normal velocity $v^{\prime} ; p^{\prime}=Z_{s} v^{\prime}$

Characteristic impedance of a fluid $\rho_{0} c_{0}$
Specific impedance of a surface $Z_{s} /\left(\rho_{0} c_{0}\right)$
Wavenumber $k=\omega / c_{0}=2 \pi / \lambda$, where $\lambda$ is the wavelength
Helmholtz number (or compactness ratio) $=k D$, where $D$ is a typical dimension of the source.

Strouhal number $=\omega D /(2 \pi U)$ for sound of frequency $\omega($ in $\mathrm{rad} / \mathrm{s})$, produced in a flow with speed $U$, length scale $D$.

## Basic equations for linear acoustics

## Conservation of mass

$$
\frac{\partial \rho^{\prime}}{\partial t}+\rho_{0} \nabla \cdot \mathbf{v}^{\prime}=0
$$

## Conservation of momentum

$$
\rho_{0} \frac{\partial \mathbf{v}^{\prime}}{\partial t}+\nabla p^{\prime}=0
$$

## Isentropic

$$
c_{0}^{2}=\left.\frac{d p}{d \rho}\right|_{S}
$$

## Wave equation

$$
\frac{1}{c_{0}^{2}} \frac{\partial^{2} p^{\prime}}{\partial t^{2}}-\nabla^{2} p^{\prime}=0
$$

## Energy density

$$
e=\frac{1}{2} \rho_{0} v^{\prime 2}+\frac{1}{2 \rho_{0} c_{0}^{2}} p^{\prime 2}
$$

Intensity $\mathbf{I}=p^{\prime} \mathbf{v}^{\prime}$
Velocity potential $\phi^{\prime}$ satisfies the wave equation and $p^{\prime}=-\rho_{0} \frac{\partial \phi^{\prime}}{\partial t}, \mathbf{v}^{\prime}=\nabla \phi^{\prime}$.
Autocorrelation $F(\xi)$, the autocorrelation of $f(y)$ is given by

$$
\begin{gathered}
F(\xi)=\overline{f(y) f(y+\xi)} \\
F(0)=\overline{f^{2}}
\end{gathered}
$$

## Integral length scale, $l$

$$
l \overline{f^{2}}=\int_{-\infty}^{\infty} F(\xi) d \xi
$$

## Sound power

Sound power from a source is defined as

$$
P=\int_{S} \overline{\mathbf{I}} \cdot \mathbf{d S}=\int_{S_{\infty}} \frac{\overline{p^{\prime 2}}}{\rho_{0} c_{0}} \mathbf{d S}
$$

for a statistically stationary source. For an outward propagating spherically symmetrical sound field $P=\frac{\overline{p^{\prime 2}}}{\rho_{0} c_{0}} 4 \pi r^{2}$, where $p^{\prime}$ is the acosutic pressure at radius $r$.

For a sound field, which is a function of spherical polar coordinates $r, \boldsymbol{\theta}$ only, and is independent of the azimuthal angle,

$$
P=2 \pi r^{2} \int_{0}^{\pi} \frac{\overline{p^{\prime 2}}}{\rho_{0} c_{0}} \sin \theta d \theta
$$

## Simple wave fields

## 1D or plane wave

The general solution of the 1D wave equation is $p^{\prime}(x, t)=f\left(t-x / c_{0}\right)+g(t+$ $x / c_{0}$ ), where $f$ and $g$ are arbitrary functions. In a plane wave propagating to the right $p^{\prime}=\rho_{0} c_{0} u^{\prime}$; in a plane wave propagating to the left $p^{\prime}=-\rho_{0} c_{0} u^{\prime}, u^{\prime}$ being the particle velocity.

## Spherically symmetric sound fields

The general spherically symmetric solution of the 3D wave equation is

$$
\phi^{\prime}(r, t)=\frac{f\left(t-r / c_{0}\right)}{r}+\frac{g\left(t+r / c_{0}\right)}{r}
$$

where $r$ is the distance from the source; $f$ and $g$ are arbitrary functions.

## $\cos \theta$ dependence

The general solution of the 3 D wave equation with $\cos \theta$ dependence is

$$
p^{\prime}(\mathbf{x}, t)=\frac{\partial}{\partial x}\left[\frac{f\left(t-r / c_{0}\right)}{r}+\frac{g\left(t+r / c_{0}\right)}{r}\right]=\cos \theta \frac{\partial}{\partial r}\left[\frac{f\left(t-r / c_{0}\right)}{r}+\frac{g\left(t+r / c_{0}\right)}{r}\right]
$$

## Useful mathematical formulae

Spherical polar coordinates $(r, \theta, \psi)$

## Gradient

$$
\nabla p^{\prime}=\left(\frac{\partial p^{\prime}}{\partial r}, \frac{1}{r} \frac{\partial p^{\prime}}{\partial \theta}, \frac{1}{r \sin \theta} \frac{\partial p^{\prime}}{\partial \psi}\right)
$$

## Divergence

$$
\nabla \cdot \mathbf{v}^{\prime}=\frac{1}{r^{2}} \frac{\partial}{\partial r}\left(r^{2} v_{r}^{\prime}\right)+\frac{1}{r \sin \theta} \frac{\partial}{\partial \theta}\left(\sin \theta v_{\theta}^{\prime}\right)+\frac{1}{r \sin \theta} \frac{\partial v_{\phi}^{\prime}}{\partial \psi}
$$

## Laplacian

$$
\nabla^{2} p^{\prime}=\frac{1}{r^{2}} \frac{\partial}{\partial r}\left(r^{2} \frac{\partial p^{\prime}}{\partial r}\right)+\frac{1}{r^{2} \sin \theta} \frac{\partial}{\partial \theta}\left(\sin \theta \frac{\partial p^{\prime}}{\partial \theta}\right)+\frac{1}{r^{2} \sin ^{2} \theta} \frac{\partial^{2} p^{\prime}}{\partial \psi^{2}}
$$

## Delta functions

## Kronecker Delta

$$
\delta_{i j}= \begin{cases}1 & i=j \\ 0 & i \neq j\end{cases}
$$

1D $\delta$-function $\delta(x)=0$ for $x \neq 0$ and $\int_{-\infty}^{\infty} \delta(a x-b) f(x) d x=f(b / a) /|a|$
3D $\delta$-function $\delta(\mathbf{x})=\boldsymbol{\delta}\left(x_{1}\right) \boldsymbol{\delta}\left(x_{2}\right) \boldsymbol{\delta}\left(x_{3}\right)$

## Convolution algebra

Convolution of $f(\mathbf{x})$ and $g(\mathbf{x})$

$$
(f \star g)(\mathbf{x})=\int_{-\infty}^{\infty} f(\mathbf{y}) g(\mathbf{x}-\mathbf{y}) d \mathbf{y}
$$

## Commutative properties

$$
\begin{gathered}
f \star g=g \star f \\
\frac{\partial}{\partial x_{i}}(f \star g)(\mathbf{x})=f \star \frac{\partial g}{\partial x_{i}}=\frac{\partial f}{\partial x_{i}} \star g
\end{gathered}
$$

## Green's function

3D Green's function for wave equation

$$
\begin{aligned}
\left(\frac{\partial^{2}}{\partial t^{2}}-c_{0}^{2} \nabla^{2}\right) g(\mathbf{x}, t \mid \mathbf{y}, \tau) & =\delta(t-\tau) \boldsymbol{\delta}(\mathbf{x}-\mathbf{y}) \\
g(\mathbf{x}, t \mid \mathbf{y}, \tau) & =\frac{\delta\left\{|\mathbf{x}-\mathbf{y}|-c_{0}(t-\tau)\right\}}{4 \pi c_{0}|\mathbf{x}-\mathbf{y}|}
\end{aligned}
$$

## Lighthill's Acoustic Analogy

## Lighthill's equation

$$
\left(\frac{\partial^{2}}{\partial t^{2}}-c_{0}^{2} \nabla^{2}\right) \rho^{\prime}=\frac{\partial^{2} T_{i j}}{\partial x_{i} \partial x_{j}} .
$$

For cold, isentropic, low Mach-number jets, $T_{i j}$ can be approximated as:

$$
T_{i j}=\rho_{0} u_{i} u_{j}
$$

Lighthill eight power law Acoustic power,

$$
P_{a} \sim \frac{\rho_{o} d_{j}^{2}}{c_{0}^{5}} u_{j}^{8}
$$

where $d_{j}$ and $u_{j}$ are the jet exit diameter and velocity, respectively.

## In a cylindrical duct of radius $a$

The pressure field is given by

$$
p^{\prime}(\mathbf{x}, t)=e^{i(\omega t+n \theta)} J_{n}\left(z_{m n} r / a\right)\left(A e^{-i k x_{3}}+B e^{i k x_{3}}\right),
$$

where $z_{m n}$ is the $m$ th zero of $\dot{J}_{n}(z)$ and $k=\left(k_{0}^{2}-z_{m n}^{2} / a^{2}\right)^{1 / 2}$.
For large azimuthal wavenumber, $n$

$$
z_{1 n} \approx n+1.85 n^{1 / 3}
$$

## In a duct of varying area $A(x)$

Webster horn equation

$$
\frac{1}{c_{0}^{2}} \frac{\partial^{2} p^{\prime}}{\partial t^{2}}-\frac{1}{A} \frac{\partial}{\partial x}\left(A \frac{\partial p^{\prime}}{\partial x}\right)=0
$$

Modified Webster horn equation $\psi(x)=\hat{p}(x) A^{1 / 2}, A=\pi a^{2}$

$$
\frac{d^{2} \psi}{d x^{2}}+\left(k^{2}-\frac{1}{a} \frac{d^{2} a}{d x^{2}}\right) \psi=0
$$

