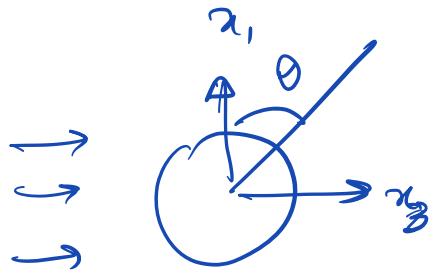


1. The conservation eq's can be written as

$$\frac{\partial \rho'}{\partial t} + \rho_0 \nabla \cdot \underline{u}' = 0 \quad (1)$$



$$\rho_0 \frac{\partial \underline{u}'}{\partial t} + \nabla p' = \Psi e_{x_1} \quad (2)$$

$\downarrow$  force per unit volume

$\frac{\partial}{\partial t} (1) - \nabla \cdot (2)$  gives:

$$\frac{\partial^2 \rho'}{\partial t^2} - \nabla^2 p' = - \frac{\partial \Psi}{\partial x_1} (x, t)$$

Using the Green's fn :

$$\rho' = - \iint \frac{\partial \Psi(y, t)}{\partial y_1} \delta \frac{(x-y - c_0(t-T))}{4\pi c_0 |x-y|} dy_1 dy$$

$$= - \frac{\partial}{\partial x_1} \iint \Psi(y, t) \delta \frac{(x-y - c_0(t-T))}{4\pi c_0 |x-y|} dy_1 dy$$

Carrying out the T integral,

$$\rho' = -\frac{\partial}{\partial x_1} \int \frac{\psi(y, t - \frac{|x-y|}{c_0})}{4\pi c_0^2 |x-y|} dy$$

Because the source is compact, assume  
 $y \approx 0$  above is in the argument of  
 $\psi$ :

$$\rho' = -\frac{\partial}{\partial x_1} \int \frac{\psi(y, t - \frac{r}{c_0})}{4\pi c_0^2 |x-y|} dy$$

$\downarrow$  (far-field).

$$\therefore \rho' \approx -\frac{1}{4\pi c_0^2 r} \frac{\partial}{\partial x_1} \left( f L \omega s(\omega(t - \frac{r}{c_0})) \right)$$

$$= -\frac{f L}{4\pi c_0^2 r} (\omega \sin(\omega(t - \frac{r}{c_0})) \cdot \left(\frac{1}{c_0}\right) \cos \theta$$

$$= -\frac{f L \omega \sin \omega(t - \frac{r}{c_0}) \cos \theta}{4\pi c_0^3 r}$$

$$p' = \omega^2 p'$$

$$\therefore p' = -\frac{1}{4\pi} \frac{L_{\text{oso}}}{r} \frac{\omega}{\omega_0} f \sin \omega(t - \frac{r}{c})$$

(b) If the far field,  $\frac{p'}{u'} \approx p_{\text{oso}}$

$$\therefore I = p' u' = \frac{p'^2}{p_{\text{oso}}}$$

$$\Rightarrow \bar{I} = \frac{1}{2} \frac{\hat{p}^* \hat{p}}{p_{\text{oso}}}$$

$$\hat{p}' = \underbrace{\text{Im} \left\{ -\frac{1}{4\pi} \frac{L_{\text{oso}} \omega}{r} f e^{-i\frac{\omega r}{c}} e^{i\omega t} \right\}}_{\hat{p}}$$

$$\therefore \bar{I} = \frac{1}{2} \frac{L^2 \omega^2 \theta \omega^2}{p_{\text{oso}} (4\pi)^2 r^2 c^2} f^2$$

$$\bar{P} = \int \bar{I} ds$$

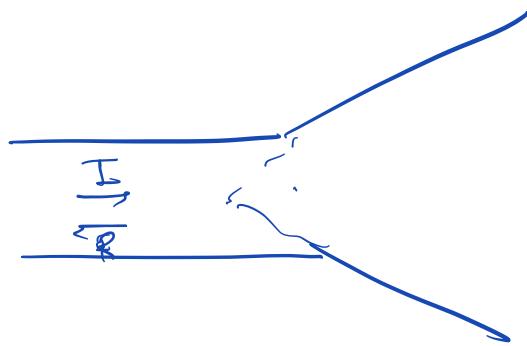
$$= \int_0^{\pi} \int_{-\infty}^{\infty} 2\pi r^2 \sin \theta d\theta$$

$$= \frac{1}{\rho_0 \omega (4\pi)^2} \frac{L^2}{\lambda^2} \frac{\omega^2 f^2}{c_0^2} \left(\frac{2}{3}\right)$$

$$= \frac{1}{24\pi} \frac{\omega^2 L^2 f^2}{\rho_0 \omega^3}$$

(C)  $L$  must be small enough so that  
the source can be treated as compact.

2



$$(a) \quad p' = \frac{f(t - r/c_0)}{r}$$

$$= e^{i\omega(t - r/c_0)}$$

$$\rho_0 i \omega u_r' = + \left\{ \gamma \frac{1}{r^2} + \frac{i\omega}{c_0 r} \right\} e^{\{ 3 \}}$$

$$u_r' = \frac{1}{\rho_0 i \omega r^2} \left\{ \frac{1}{r} + ikr \right\} e^{\{ 3 \}}$$

$$Z = \frac{p'}{u'} = \frac{i \rho_0 \omega \rho_0}{(1 + ikr_0)} = \underline{\underline{\rho_0 c_0 i k r_0}} / \underline{\underline{1 + i k r_0}}$$

$$(b) P = \left\{ I e^{-i k x} + R e^{i k x} \right\} e^{\text{int}}$$

$$U = \underbrace{\int_{\text{Polo}} \left\{ I e^{-i k x} - R e^{i k x} \right\} e^{\text{int}}}_{\text{int}}$$

$$x=0: \\ P' = (I+R) e^{\text{int}}$$

$$U = \frac{(I-R)}{P_{\text{Polo}}} e^{\text{int}}$$

$$\frac{P'}{U} = \frac{(I+R) \beta \omega}{I-R}$$

$$\frac{(I+L) \cancel{R_0}}{(I-L)} = \frac{i k R_0 \cancel{R_0}}{(1+i k R_0)}$$

$$\begin{aligned} \frac{\cancel{I}}{R} + 1 &= \frac{i k R_0}{1+i k R_0} - 1 \\ \frac{I}{R} - 1 & \\ \Rightarrow \frac{Z}{\cancel{R}} &= \frac{-1}{1+i k R_0} \end{aligned}$$

$$\frac{\cancel{I}}{R} - 1 = - \left( \frac{1+i k R_0}{\cancel{R}} \right)$$

$$\begin{aligned} \frac{\cancel{I}}{R} &= -2(1+i k R_0) + 1 \\ &= -2 i k R_0 - 1 \end{aligned}$$

$$\frac{R}{I} = - \frac{1}{1 + 2ikr_0}$$

(5) In the absence of the horn,

$$\frac{R}{I} = -1$$

Therefore the horn reduces the amplitude of the reflected waves and hence aids transmission.

The efficiency of the sound transmission is frequency dependent.

If  $kr_0 \ll 1$ , the horn is not effective at all. However, if

$$kr_0 \gg 1,$$

$$\frac{R}{I} \approx 0,$$

i.e. we get perfect transmission.

Can you explain this result  
from a physical point of view?

Hint: think impedance and plane  
waves.

- 3 (a) An acoustic mode in a duct is said to be 'cut-off' if it is evanescent. Then the axial wavenumber is purely imaginary so that the amplitude of the disturbance decays exponentially along the duct.
- (b) The formula on the data card gives the pressure field as
- $$p'(\underline{x}, t) = e^{i(\omega t + n\theta)} J_n\left(\frac{z_{mn}r}{a}\right) (A e^{-ikx_3} + B e^{ikx_3})$$
- where  $z_{mn}$  is the  $m$ th zero of  $J_n(z)$  and  $k = (k_0^2 - z_{mn}^2/a^2)^{1/2}$ ,  $k_0 = \omega/c_0$ .
- We denote the rotation rate by  $\Omega$ .  
 Blade passing frequency is  $3\Omega$   
 At harmonics of blade passing frequency  $\omega = 3M\Omega$  M integer  
 $n = 3M$ ,  
 for cut-off modes we need  $k$  imaginary i.e.  $k_0 < \frac{z_{mn}}{a}$
- $$\frac{3M\Omega}{c_0} < \frac{z_{13M}}{a}$$
- For  $M=1$ , we require  $\Omega < \frac{c_0}{3a} z_{13} = \frac{c_0}{3a} 4.20119$
- $$= 2 \quad \Omega < \frac{c_0}{3a} \frac{z_{16}}{2} = \frac{c_0}{3a} \frac{7.50129}{2}$$
- $$= 3 \quad \Omega < \frac{c_0}{3a} \frac{z_{13M}}{M} = \frac{c_0}{3a} \left( \frac{3M + 0.80861(3M)^{2/3}}{M} \right)$$
- $$< \frac{c_0}{a} \left( 1 + \frac{0.80861}{3^{2/3}} M^{-2/3} \right)$$
- It is clear that the most stringent constraint is for large  $M$  and that we need  $\Omega < \frac{c_0}{a} = \frac{343}{0.15} = 2.2867 \times 10^3 \text{ rads}^{-1}$   
 $= \underline{\underline{21,836 \text{ rpm}}}$

- (c) (iii) The rotor pattern has the form  $e^{iM3(\Omega t - \theta)}$   
 and the stator pattern is  $e^{\mp iNS\theta}$  where  $S = \text{number of stator blades}$   
 So rotor-stator interaction modes are:  $e^{iM3(\Omega t - \theta) \mp iNS\theta}$   
 $= e^{i(M3\Omega t - (M3 \pm NS)\theta)}$
- Modes propagate if  $\frac{3M\Omega a}{c_0} > z_{13M \pm NS}$

Usually  $S \approx 2.4 \times$  number of rotor blades and  $S$  prime is a good choice.

Here  $2.4 \times 3 = 7.2$  so try  $S=7$ ,  $S=11$  is another possibility.

For 10,000 rpm  $\frac{\omega_0}{c_0} = 0.458 < 1$  so all rotor-alone modes are cut-off

Mode of frequency  $M \times$  bpf propagate if  $0.458 \times 3M > z_{1/3M+N7}$

i.e. if  $0.458 > \frac{z_{1/3M+N7}}{3M}$

For 7 stator blades

For bpf,  $M=1$

$\frac{z_{1/3+N7}}{3} > 1$  for all  $N$  and so certainly  $> 0.458$

All modes at bpf are cut-off

For 2bpf,  $M=2$

$\frac{z_{1/6+N7}}{6}$  is  $> 1$  for all  $N$  except  $N=1$

For  $N=1$   
 $|6-7|=1$

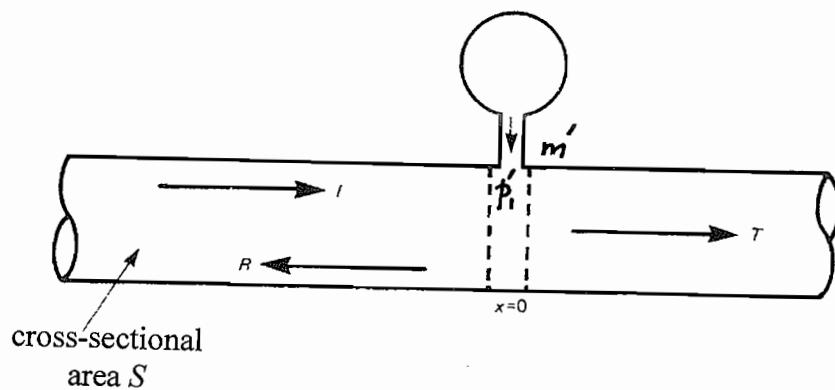
$$\frac{z_{11}}{6} = \frac{1.84118}{6} = 0.3068 < 0.458$$

So the first order stator mode propagates at 2 bpf with  $S=7$

A choice of  $S=11$  would have been better for 1 and 2 bpf,  
Then the minimum value of  $|6+N11|$  is 5 and  $z_{15}=6.41562$

$$\frac{z_{15}}{6} = \frac{6.41562}{6} > 0.458$$

and all modes are cut-off at 2 bpf as well as bpf.



Continuity of pressure at  $x=0$

$$(I+R)e^{i\omega t} = p'_i = T e^{i\omega t} \quad (1)$$

Continuity of mass flow rate

$$\frac{(I-R)}{C_0} S e^{i\omega t} + m' = \frac{T S}{C_0} e^{i\omega t} \quad (2)$$

where  $m'$  = mass flow rate out of Helmholtz resonator.

From (1),  $R = T - I$  and substituting into (2) gives

$$(2I-T) \frac{S}{C_0} e^{i\omega t} + m' = \frac{TS}{C_0} e^{i\omega t} \quad (3)$$

From equation (1.20) in the Lecture Notes (a full solution would include deriving the Helmholtz resonator result as in Lecture Notes)

$$\begin{aligned} m' &= \frac{i\omega A}{l} \frac{1}{\omega^2 - C_0^2 A/Vl} p'_i \\ &= \frac{i\omega A}{l} \frac{1}{\omega^2 - C_0^2 A/Vl} T e^{i\omega t} \quad \text{from equation (1)} \end{aligned}$$

Substituting for  $m'$  in equation (3) leads to

$$(2I-T) \frac{S}{C_0} + \frac{i\omega A}{l} \frac{1}{\omega^2 - C_0^2 A/Vl} T = \frac{TS}{C_0}$$

$$\text{i.e. } 2I = 2T + \frac{i\omega A C_0}{S l} \frac{1}{\omega^2 - C_0^2 A/Vl} T$$

Ratio of incident to transmitted acoustic power =  $\frac{|I|^2}{|T|^2}$

$$= \frac{1}{1 + \left( \frac{\omega A C_0}{2 S l} \frac{1}{\omega^2 - C_0^2 A/Vl} \right)^2} = \frac{1}{1 + \frac{1}{4 S^2} \left( \frac{1}{\frac{\omega l}{C_0 A}} - \frac{C_0}{\omega V} \right)^2}$$