

Q1

(a) Linear elasticity:  $U = \frac{1}{2} \sigma_{ij} \epsilon_{ij}$

• Beam is slender:  $\sigma_{22} = \sigma_{33} = 0$

• Kinematics: model assumes zero shear deformation,  $\epsilon_{12} = 0$  etc

• Only non-zero contributors to  $U$  is therefore  $\sigma_{11}, \epsilon_{11}$

• Linear elastic:  $\sigma_{11} = E_1 \epsilon_{11} \therefore U = \frac{1}{2} \sigma_{11} \epsilon_{11} = \frac{1}{2} E_1 \epsilon_{11}^2$

(b) Elastic beam only ( $h_2 = 0$ ):

• Kinetics (given):  $\epsilon_{11} = \frac{\partial u_1}{\partial x_1} = -w_{,311}(x_1) x_2$

• Strain energy density (from (a)):  $U = \frac{1}{2} E_1 (w_{,311} x_2)^2$

• Variations in PE (DOF = centre line deflection  $w$ ):

$$\delta \Pi = \int_V \delta U dV - F \delta w(x_1=L)$$

$$= \int_V \frac{\partial U}{\partial w_{,311}} \delta w_{,311} dV - F \delta w(x_1=L)$$

$$= \int_V E_1 w_{,311} x_2^2 \delta w_{,311} dV - F \delta w(x_1=L)$$

$$\begin{aligned} I_1 &= \frac{b(2h_1)^3}{12} \rightarrow \\ &= E_1 b \int_0^L \int_{-h_1}^{+h_1} w_{,311}(x_1) x_2^2 \delta w_{,311} dx_2 dx_1 - F \delta w(x_1=L) \\ &= E_1 I_1 \int_0^L w_{,311}(x_1) \delta w_{,311} dx_1 - F \delta w(x_1=L) \end{aligned}$$

• Integrate by parts:

$$\begin{aligned} \delta \Pi &= -E_1 I_1 \int_0^L w_{,311}(x_1) \delta w_{,311} dx_1 + E_1 I_1 \left[ w_{,311}(x_1) \delta w_{,311} \right]_0^L \\ &\quad - F \delta w(x_1=L) \end{aligned}$$

Q1 cont...

• Integrate by parts again:

$$\delta \Pi = E_1 I_1 \int_0^L w_{s''''}(x_1) \delta w \, dx_1 - E_1 I_1 \left[ w_{s''''}(x_1) \delta w \right]_0^L \\ + E_1 I_1 \left[ w_{s''}(x_1) \delta w_{s'} \right]_0^L - F \delta w(x_1=L)$$

• Require  $\delta \Pi = 0$  for any kinematically admissible  $\delta w$ :

$$\therefore \underline{w_{s''''} = 0} \quad \text{for } 0 \leq x_1 \leq L$$

• Boundary conditions:

$$\text{At } x_1 = L: (F + E_1 I_1 w_{s''''}) \delta w = 0 \quad \therefore \underline{w_{s''''} = -\frac{F}{E_1 I_1}}$$

$$\text{At } x_1 = L: w_{s''} \delta w_{s'} = 0 \quad \therefore \underline{w_{s''} = 0}$$

$$\text{At } x_1 = 0: w_{s''''} \delta w = 0 \rightarrow \text{satisfied by B.C. } \underline{w(x_1=0) = 0} \quad (\text{given})$$

$$\text{At } x_1 = 0: w_{s''} \delta w_{s'} = 0 \rightarrow \text{satisfied by B.C. } \underline{w_{s'}(x_1=0) = 0} \quad (\text{given})$$

(c) Add viscoelastic layers ( $h_2 \neq 0$ ):

Strategy: relate stress  $\sigma_{11}$  to strain  $\epsilon_{11}$ , and therefore curvature  $\kappa$  (from kinematics); then relate stresses  $\sigma_{11}$  to the moment; then relate moment to the tip force

• Elastic layer:

$$\epsilon_{11} = u_{1,1} = -w_{s'}(x_1) x_2 = -\kappa(x_1) x_2$$

$$\sigma_{11} = E_1 \epsilon_{11} = -\kappa(x_1) x_2 E_1$$

Q1 cont....

- Viscoelastic layers:

Given  $\epsilon_{11} = u_{1,1} = -\kappa(x_1) x_2$  (same as elastic layer  $\rightarrow$  given)

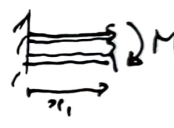
Use Boltzmann superposition principle to get stress:

$$\begin{aligned} \sigma_{11} &= \int_0^t \frac{\partial \epsilon_{11}(\tau)}{\partial \tau} E_r(t-\tau) d\tau \\ &= - \int_0^t x_2 \frac{\partial \kappa(x_1, \tau)}{\partial \tau} E_r(t-\tau) d\tau \end{aligned}$$

Given:  $E_r(t) = E_2 e^{-tE_2/\eta_2}$

Also noting:  $\kappa = \alpha t \Rightarrow \kappa(t=0) = 0 \Rightarrow$  initial discontinuity to allow for

- Total moment on the cross-section at position  $x_1$ :

$$M(x_1, t) = -b \int_{-h_1}^{+h_1+h_2} \sigma_{11}(x_1, x_2, t) x_2 dx_2$$


- Integrate separately across elastic and visco-elastic layers:

$$\begin{aligned} M(x_1, t) &= -b \int_{-h_1}^{h_1} [-\kappa(x_1, t) x_2 E_1] x_2 dx_2 \\ &\quad - b \int_{h_1}^{h_1+h_2} \left[ -x_2 \int_0^t \frac{\partial \kappa(x_1, \tau)}{\partial \tau} E_r(t-\tau) d\tau \right] x_2 dx_2 \\ &\quad - b \int_{-(h_1+h_2)}^{-h_1} \left[ -x_2 \int_0^t \frac{\partial \kappa(x_1, \tau)}{\partial \tau} E_r(t-\tau) d\tau \right] x_2 dx_2 \end{aligned}$$

$$\begin{aligned} &= E_1 I_1 \kappa(x_1, t) \\ &\quad + (I_2 - I_1) \int_0^t \frac{\partial \kappa(x_1, \tau)}{\partial \tau} E_r(t-\tau) d\tau \quad \textcircled{1} \end{aligned}$$

where:  $I_1 = \frac{b(z_{h_1})^3}{12}$ ,  $I_2 = \frac{b(z_{h_1} + z_{h_2})^3}{12}$

Q1 Cont...

• At the root of the cantilever ( $x_1 = 0$ )



Given:  $\kappa = \alpha t$

→ sub in to ①

Solve in the time domain,  
or take Laplace transforms

$$\therefore F(t)L = E_1 I_1 \alpha t$$

$$+ (I_2 - I_1) \int_0^t \alpha E_2 e^{-(t-\tau) \frac{E_2}{\eta_2}} d\tau$$

$$= E_1 I_1 \alpha t + E_2 (I_2 - I_1) \alpha \left[ \frac{\eta_2}{E_2} e^{-(t-\tau) \frac{E_2}{\eta_2}} \right]_0^t$$

$$\therefore F(t) = \frac{E_1 I_1 \alpha t}{L} + \frac{\eta_2 (I_2 - I_1) \alpha}{L} \left[ 1 - e^{-t \frac{E_2}{\eta_2}} \right]$$

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Q2 (a)

$$(i) \int_V \nabla \times \underline{a} \, dV = \int_V \epsilon_{ijk} \frac{\partial}{\partial x_j} a_k \, dV$$

$$= \oint_S \epsilon_{ijk} a_k n_j \, dS \quad (\text{Data Sheet} \rightarrow \text{divergence theorem})$$

$$= \oint_S \underline{n} \times \underline{a} \, dS$$

$$(ii) \nabla(\sqrt{\underline{a} \cdot \underline{a}}) = \frac{\partial}{\partial x_i} (a_j a_j)^{\frac{1}{2}} \underline{e}_i \quad (\underline{a} = a_i \underline{e}_i)$$

$$= \frac{1}{2} (a_j a_j)^{-\frac{1}{2}} (2a_i) \underline{e}_i \quad \leftarrow \frac{\partial}{\partial x_i} (\dots)$$

$$= (a_j a_j)^{-\frac{1}{2}} a_i \underline{e}_i = \frac{\partial}{\partial a_j} (\dots) \frac{\partial a_j}{\partial x_i}$$

$$= \frac{\underline{a}}{\sqrt{\underline{a} \cdot \underline{a}}} = \frac{\partial}{\partial a_j} (\dots) \delta_{ij}$$

$$(iii) \nabla \times (\underline{a} \times \underline{b}) = \epsilon_{ijk} \frac{\partial}{\partial x_j} (\epsilon_{kpq} a_p b_q)$$

$$= (\epsilon_{kij} \epsilon_{kpq}) \frac{\partial}{\partial x_j} (a_p b_q)$$

$\epsilon$ - $\delta$  identity  
(Data Sheet)  $\rightarrow$

$$= (\delta_{ip} \delta_{jq} - \delta_{iq} \delta_{jp}) \frac{\partial}{\partial x_j} (a_p b_q)$$

$$= \frac{\partial}{\partial x_j} (a_i b_j) - \frac{\partial}{\partial x_j} (a_j b_i)$$

$$= a_i \frac{\partial b_j}{\partial x_j} - b_i \frac{\partial a_j}{\partial x_j} + b_j \frac{\partial a_i}{\partial x_j} - a_j \frac{\partial b_i}{\partial x_j}$$

$$= \underline{a} (\nabla \cdot \underline{b}) - \underline{b} (\nabla \cdot \underline{a}) + \underline{b} \cdot \nabla \underline{a} - \underline{a} \cdot \nabla \underline{b}$$

2 b i

$$\frac{\partial f}{\partial x_i} = \frac{\partial f}{\partial X_k} \frac{\partial X_k}{\partial x_i} = F_{ki}^{-1} \frac{\partial f}{\partial X_k} = \mathbf{F}^{-T} \nabla_0 f$$

ii

$$\frac{\partial u_i}{\partial x_j} = \frac{\partial u_i}{\partial X_k} \frac{\partial X_k}{\partial x_j} = \frac{\partial u_i}{\partial X_k} F_{kj}^{-1} = (\nabla_0 \mathbf{u}) \mathbf{F}^{-1}$$

iii

$$\begin{aligned} \int_{\Omega_0} \nabla_0 \cdot (J \mathbf{F}^{-T}) dV &= \int_{\partial \Omega_0} J \mathbf{F}^{-T} \mathbf{N} ds \\ &= \int_{\partial \Omega} \mathbf{n} ds \\ &= \int_{\Omega} (\nabla \cdot \mathbf{I}) \mathbf{n} ds \\ &= 0 \end{aligned}$$

This must hold for any domain, therefore can be localised and integrals removed, proving the result.

iv

$$\begin{aligned} \frac{\partial [J \mathbf{F}^{-1} \mathbf{v}]_i}{\partial X_i} &= \frac{\partial J F_{ik}^{-1} v_k}{\partial X_i} \\ &= \frac{\partial J F_{ik}^{-1}}{\partial X_i} v_k + J F_{ik}^{-1} \frac{\partial v_k}{\partial X_i} \\ &= \underbrace{\frac{\partial J F_{ki}^{-T}}{\partial X_i}}_0 v_k + J F_{ik}^{-1} \frac{\partial v_k}{\partial X_i} \\ &= J \frac{\partial X_i}{\partial x_k} \frac{\partial v_k}{\partial X_i} \\ &= J \nabla \cdot \mathbf{v} \end{aligned}$$

v

$$\begin{aligned} \frac{\partial [J \mathbf{A} \mathbf{F}^{-T}]_i}{\partial X_i} &= \frac{\partial J A_{ik} F_{kj}^{-T}}{\partial X_j} \\ &= \underbrace{\frac{\partial J F_{kj}^{-T}}{\partial X_j}}_0 A_{ij} + J F_{kj}^{-T} \frac{\partial A_{ik}}{\partial X_j} \\ &= J \frac{\partial X_j}{\partial x_k} \frac{\partial A_{ik}}{\partial X_j} \\ &= J \nabla \cdot \mathbf{A} \end{aligned}$$

c Requires  $\nabla \cdot \mathbf{P} = J \nabla \cdot \sigma$ . Above result implies  $\mathbf{P} = J \sigma \mathbf{F}^{-T}$ .

3 a Shape is a unit square undergoing pure shear in the  $\mathbf{e}_1$ -direction.

$$\mathbf{F} = \begin{bmatrix} 1 & \alpha \\ 0 & 1 \end{bmatrix}$$

$$\mathbf{E} = \frac{1}{2} \begin{bmatrix} 0 & \alpha \\ \alpha & 0 \end{bmatrix}$$

b i

$$E = (1/2)(U^T R^T R U - I) = (1/2)(U^T U - I)$$

by orthogonality of the rotation tensor. Therefore the strain tensor depends on the stretch only.

ii Eigenvalues of a tensor/matrix do not change under a rotation.

iii

$$\begin{aligned} E &= (1/2)(U^T U - I) \\ &= (1/2)\left(\sum_i \lambda_i^2 \mathbf{u}_i \otimes \mathbf{u}_i - \mathbf{I}\right) \end{aligned}$$

were  $\lambda_i$  and  $\mathbf{u}_i$  are the eigenvalues and eigenvectors of  $\mathbf{U}$ . Note the  $\mathbf{U}^T = \mathbf{U}$  and multiplication changes the eigenvalues but not the eigenvectors.

iv The eigenvalues are the ‘principal stretches’, i.e. the ratio by which a fibre aligned with a particular eigenvector is stretched. The eigenvectors of  $\mathbf{U}$  are the stretch directions in the material configuration. The eigenvectors of  $\mathbf{U}$  are the stretch directions in the spatial (current) configuration.

v For an incompressible problem,  $\det \mathbf{F} = \det(\mathbf{R}\mathbf{U}) = \det \mathbf{R} \det \mathbf{U} = 1$ .  $\det \mathbf{R} = 1$  since  $\mathbf{R}$  is a rotation, hence  $\det \mathbf{U} = 1$  must hold. This requires that  $\lambda_1 \lambda_2 \lambda_3 = 1$ .

vi

$$\begin{aligned} \mathbf{V} &= \mathbf{R}\mathbf{U}\mathbf{R}^T \\ &= \mathbf{R} \left( \sum_i \lambda_i \mathbf{u}_i \otimes \mathbf{u}_i \right) \mathbf{R}^T \\ &= \sum_i \lambda_i \mathbf{R}\mathbf{u}_i \otimes \mathbf{R}\mathbf{u}_i \\ &= \sum_i \lambda_i \mathbf{v}_i \otimes \mathbf{v}_i, \end{aligned}$$

therefore  $\mathbf{v}_i = \mathbf{R}\mathbf{u}_i$ .

$$\begin{aligned} \mathbf{F} &= \mathbf{R}\mathbf{U} \\ &= \sum_i \lambda_i \mathbf{R}\mathbf{u}_i \otimes \mathbf{u}_i \\ &= \sum_i \lambda_i \mathbf{v}_i \otimes \mathbf{u}_i \end{aligned}$$