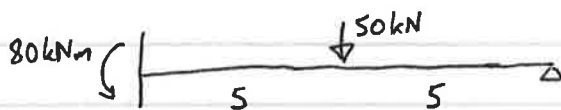


4D10 STRUCTURAL STEELWORK, 2014

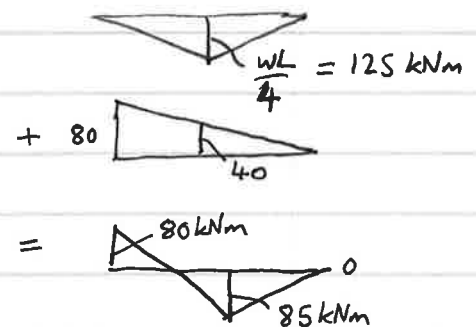
Q1 a) Elastic-plastic interaction is taken account of by Perry-Robertson by largely avoiding it - i.e. first yield of any point on any cross-section is taken as the design "failure" criterion. An initial imperfection is assumed which grows as axial load is increased leading to bending and axial stress combinations \rightarrow then check for axial load P which causes first yield.

b) The Perry-Robertson approach to LTB is complicated by the fact that there are two imperfections - a lateral one and a torsional one. One can pursue a Perry-Robertson analysis assuming two such imperfections and looking for first yield, but it gets complicated, and you need to make assumptions. Essentially EC3 handles this by just doing a curve fit to test data. - Also it rather cleverly uses $\sqrt{\text{Plastic/Elastic}}$ as the slenderness parameter such that column buckling curves can be used for LTB even though LTB does not fall away as $1/L^2$ as Euler buckling does. Residual stresses that arise due to differential cooling are treated similarly - i.e. it is just a curve fit, and there is no real attempt to actually look for first yield at any point.

c) 305 x 127 UB42, S275



BMD =



\therefore Critical span is RH span, as it is closer to "equal and opposite" end moments" case.

4D10 Q1 (c) i) cont'd

Compact?

305 x 127 UB 42

S275

$$\text{flange } \lambda = \frac{(124.3 - 8)/2}{12.1} \sqrt{\frac{275}{355}} = 4.2 < 8 \text{ OK}$$

$$\text{webs (bending)} \lambda = \frac{(307.2 - 2(12.1))}{8} \sqrt{\frac{275}{355}} = 31.1 < 56 \text{ OK.}$$

PLASTIC

$$M_{pl} = 275 \times 10^6 \text{ N/m}^2 \times 614 \times 10^{-6} \text{ m}^3 = \underline{\underline{168.9 \text{ kNm}}}$$

ELASTIC

$$M_{LTB} = \frac{\pi}{L} \sqrt{GJ EI_{min}} \left[1 + \frac{\pi^2 E \Gamma}{L^2 G J} \right]^{1/2}$$

$$J = 21.1 \text{ cm}^4$$

$$I_{min} = 389 \text{ cm}^4$$

$$M_{basic} = \frac{\pi}{L} \sqrt{GJ EI_{min}} = \frac{\pi}{5} (210 \times 10^9) \sqrt{\frac{81}{210} J \cdot I_y}$$

$$G = 81 \text{ GPa}$$

$$E = 210 \text{ GPa}$$

$$= \frac{\pi}{5} (210 \times 10^9) \sqrt{\frac{81}{210} (21.1)(389) \times 10^{-8}}$$

$$= \underline{\underline{74.2 \text{ kNm}}}$$

$$\Gamma = ID^2/4, \quad D = \text{dist. between flange centroids} = 307.2 - 12.1 = 295.1 \text{ mm}$$

$$\frac{\Gamma}{J} = \left(\frac{389}{21.1} \right) \frac{(0.2951)^2}{4} \text{ m}^2 = 0.401 \text{ m}^2$$

$$\frac{\pi^2 E \Gamma}{L^2 G J} = \frac{\pi^2}{25 \text{ m}^2} \cdot \frac{210}{81} \cdot 0.401 \text{ m}^2 = 0.4108$$

$$\sqrt{1 + \frac{\pi^2 E \Gamma}{L^2 G J}} = \sqrt{1 + 0.4108} = 1.1878$$

$$\therefore M_{LTB} = (74.2 \text{ kNm}) \times 1.1878 = \underline{\underline{88.1 \text{ kNm}}}$$

$$C_{unequal} = 0.6 \text{ for } \psi = 0 \quad (\text{DS3})$$

$$\therefore M_{cr} = \frac{M_{LTB}}{C_{unequal}} = \frac{88.1}{0.6} = \underline{\underline{146.9 \text{ kNm}}}$$

4D10 Q1(c) cont'd.

$$\lambda = \sqrt{\frac{\text{Plastic}}{\text{Elastic}}} = \sqrt{\frac{168.9}{146.9}} = 1.072$$

Which curve? Rolled, $\frac{h}{b} = \frac{307.2}{124.3} = 2.47 > 2$
 \therefore curve (b) DS3

From DS1 $\chi \approx 0.55$ for $\lambda = 1.072$

$$\therefore M_{\text{DESIGN}} = \chi M_{\text{PL}} = 0.55(168.9) = \underline{93 \text{ kNm}}$$

\therefore OK for 85 kNm

c ii) But if end moment removed, central moment = 125 kNm
all other parameters remain the same \therefore 93 kNm strength
is NBT adequate.

c iii) There are many conservative assumptions throughout, but
in particular here we could argue that the LH ~~beam~~^{half} provides
some torsional restraint to the RH half across the central point.
This adds stiffness, so LTB moment is probably a bit higher.

4D10 2014

Q2 457 x 191 UB 82 beam column, S355, 10 m long

Assume compact.

$$\begin{aligned} a) \quad M_{PL(MAS)} &= 355 \times 10^6 \text{ N/m}^2 \times 1831 \times 10^{-6} \text{ m}^3 = 650 \text{ kNm} \\ N_{PL} &= 355 \times 10^6 \text{ N/m}^2 \times 104 \times 10^{-4} \text{ m}^2 = 3692 \text{ kN} \end{aligned}$$

Web fraction: Total area = 104 cm²

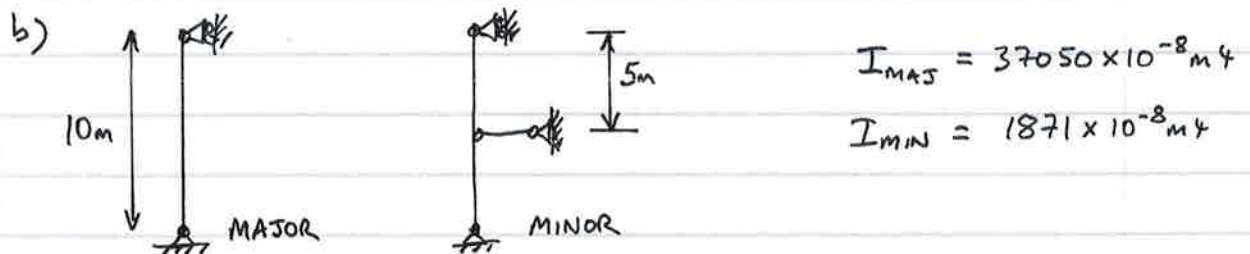
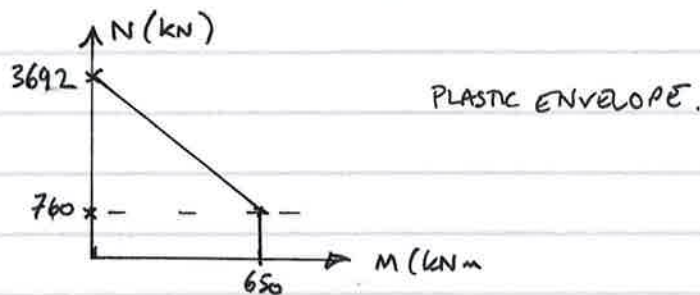
$$\text{Flange area} = 2 \times 191.3 \times 16 \text{ mm}^2 = 6122 \text{ mm}^2 = 61.2 \text{ cm}^2$$

$$\therefore \text{web area} = 104 - 61.2 = 42.8 \text{ cm}^2$$

$$\text{web fraction } a = \frac{42.8}{104} = 0.411$$

$$\text{half web fraction} = 0.206$$

$$\therefore \text{kink at } 0.206 \times 3692 = 760 \text{ kN.}$$



$$I_{MAS} = 37050 \times 10^{-8} \text{ m}^4$$

$$I_{MIN} = 1871 \times 10^{-8} \text{ m}^4$$

$$N_{Euler, MAS} = \frac{\pi^2 EI}{L^2} = \frac{\pi^2 (210 \times 10^9) (37050 \times 10^{-8})}{10^2} = \underline{\underline{7679 \text{ kN}}}$$

$$N_{Euler, MIN} = \frac{\pi^2 EI}{L^2} = \frac{\pi^2 (210 \times 10^9) (1871 \times 10^{-8})}{5^2} = \underline{\underline{1551 \text{ kN}}}$$

$$\text{Slenderness: MAJOR: } \lambda = \sqrt{\frac{\text{Plastic}}{\text{Elastic}}} = \sqrt{\frac{3692}{7679}} = 0.6934$$

$$\text{MINOR: } \lambda = \sqrt{\frac{\text{Plastic}}{\text{Elastic}}} = \sqrt{\frac{3692}{1551}} = 1.5429$$

4D10 2014, Q2 cont'd.

$$DS2: \quad h/b = 460/191.3 = 2.4 > 1.2, \quad \text{flange } T = 16 \text{ mm} < 40 \text{ mm}$$

S355. Major - about y-y \rightarrow curve (a)
Minor - about z-z \rightarrow curve (b)

$$DS1. \text{ Major: } \lambda = 0.6934 \rightarrow \chi \approx 0.85 \text{ curve (a)}$$

$$\text{Minor: } \lambda = 1.5429 \rightarrow \chi \approx 0.33 \text{ curve (b)}$$

$$\therefore N_{cr, \text{maj}} = 0.85 (3692) = \underline{\underline{3138 \text{ kN}}}$$

$$N_{cr, \text{min}} = 0.33 (3692) = \underline{\underline{1218 \text{ kN}}}$$

Moments: PLASTIC $M_{PL, \text{maj}} = 650 \text{ kNm}$ (earlier)

$$\text{ELASTIC} \quad M_{\text{basic}} = \frac{\pi}{L} \sqrt{G J E I_{\text{min}}}$$

$$= \frac{\pi}{5} (210 \times 10^9) \sqrt{\frac{81}{210} \cdot 69.2 \cdot 1871 \times 10^{-8}}$$

$$= \frac{\pi}{5} (2100) (223.5) = \underline{\underline{294.9 \text{ kNm}}}$$

$$J = 69.2 \text{ cm}^4$$

$$I_{\text{minor}} = 1871 \text{ cm}^4$$

$$L = 5 \text{ m (LTB eff. length)}$$

$$G = 81 \text{ GPa}$$

$$E = 210 \text{ GPa}$$

$$\Gamma = \frac{I D^2}{4}, \quad D = 460 - 16 = 444 \text{ mm}$$

$$I_{\text{minor}} = 1871 \text{ cm}^4$$

$$\frac{\Gamma}{J} = \frac{1871 \text{ cm}^4}{69.2 \text{ cm}^4} \frac{(0.444)^2 \text{ m}^2}{4} = 1.333 \text{ m}^2$$

$$\frac{\pi^2 E \Gamma}{L^2 G J} = \frac{\pi^2}{25 \text{ m}^2} \frac{210}{81} 1.333 = 1.3639$$

$$\sqrt{1 + \frac{\pi^2 E \Gamma}{L^2 G J}} = \sqrt{1 + 1.3639} = 1.5375$$

$$M_{\text{LTB}} = M_{\text{basic}} \times \sqrt{\quad} = 294.9 \text{ kNm} \times 1.5375 = \underline{\underline{453.4 \text{ kNm}}}$$

$C_{\text{unequal}} = 1$ (equal + opp end moments)

$$\therefore M_{\text{cr}} = M_{\text{LTB}} = \underline{\underline{453.4 \text{ kNm}}}$$

ELASTIC

4D10 2014

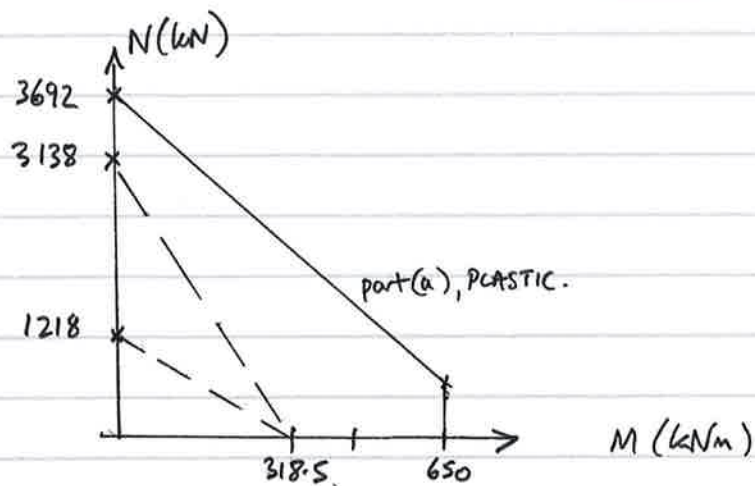
Q2 (cont'd).

Which curve?

Rolled I $\frac{h}{b} = 2.4 > 2 \rightarrow$ curve (b) DS3

$$\lambda = 0.49 \text{ (DS1)}$$

$$M_{cr} = 0.49(650) = \underline{\underline{318.5 \text{ kNm}}}$$



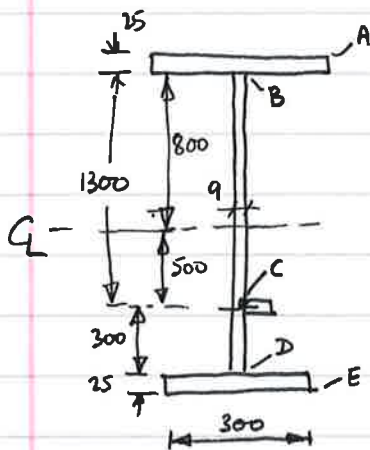
Q3 a)



$$3.2(1200) = 9F \Rightarrow F = \frac{3.2(1200)}{9} = 426.7 \text{ kN}$$

$$R = F + 1200 = \underline{1627 \text{ kN}}$$

BM = 1200 kN x 3.2m = 3840 kNm
at support



$$I = \frac{1}{12} (1650^3(300) - (1600)^3(291)) = 12.98 \times 10^9 \text{ mm}^4$$

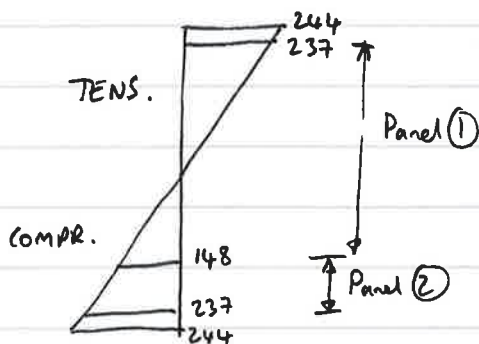
$$\sigma = \frac{M_y}{I} = \frac{(3840 \times 10^6 \text{ Nmm}) y}{12.98 \times 10^9 \text{ mm}^4} = 0.2958 y$$

(y in mm, σ in MPa)

At A, E; y = 825 mm $\sigma = 244 \text{ MPa}$

B, D; y = 800 mm $\sigma = 237 \text{ MPa}$

C; y = 500 mm $\sigma = 148 \text{ MPa}$



Shear stress $\tau = \frac{1200 \times 10^3 \text{ N}}{1600 \times 9} = 83.3 \text{ N/mm}^2$

Bottom flange in compression.

Is it compact?

$$\lambda = \frac{b}{t} \sqrt{\frac{\sigma_y}{355}} = \frac{150 - 4.5}{25} = 5.82 < 8$$

\therefore OK

and $\sigma < \sigma_y$

\therefore OK.

Check local panel strength at B and D:

$$\sigma \leq \sqrt{\sigma_y^2 - 3\tau^2} \quad \text{(DS4)} \quad \bullet \quad \sigma^2 + 3\tau^2 \leq \sigma_y^2 \quad ?$$

$$\left(\frac{\sigma}{\sigma_y}\right)^2 + 3\left(\frac{\tau}{\sigma_y}\right)^2 \leq 1 \quad ?$$

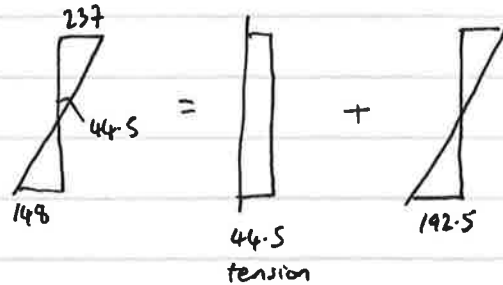
$$\left(\frac{237}{355}\right)^2 + 3\left(\frac{83.3}{355}\right)^2$$

$$= 0.4457 + 3(0.055) = 0.61 < 1 \quad \therefore \text{OK}$$

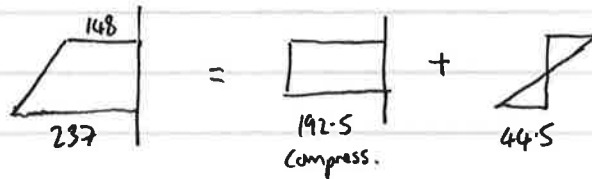
✓

4D10 Q3 cont'd.

Panel ①



Panel ②



$$\lambda = \frac{b}{t} \sqrt{\frac{G_y}{355}}$$

① $\lambda = \frac{1300}{9} = 144 (> 56)$

② $\lambda = \frac{300}{9} = 33.3 (> 24 \text{ but } < 56)$

K_c	K_b	K_g
0^* (n/a)	0.91	0.36
0.82	1.21	1.0

DS3

Stability:

$$\frac{\sigma_c}{K_c \sigma_y} + \left(\frac{\sigma_b}{K_b \sigma_y} \right)^2 + \left(\frac{\tau}{K_g \sigma_y / \sqrt{3}} \right)^2$$

① $= 0^* + \left(\frac{192.5}{0.91(355)} \right)^2 + \left(\frac{83.3\sqrt{3}}{0.36(355)} \right)^2$

$= 0^* + 0.355 + 1.27 = 1.63 > 1$
FAIL.

② $= \frac{192.5}{0.82(355)} + \left(\frac{44.5}{1.21(355)} \right)^2 + \left(\frac{83.3\sqrt{3}}{1.0(355)} \right)^2$

$= 0.66 + 0.01 + 0.16 = 0.83 \therefore \text{OK}$

So panel ① fails. **SUBTLETY:** Panel ① is in TENSION + BENDING so K_c factors inappropriate - so best set first term in stability eqn to zero, rather than trying to use K_c factors with $\sigma_c = \text{negative}$.

To strengthen, add another stiffener into panel ①

┌ Could also try moving the original stiffener up a bit, since panel ② has a bit of spare capacity - but adding a stiffener is preferred ┘.

4D10 Q3 cont'd.

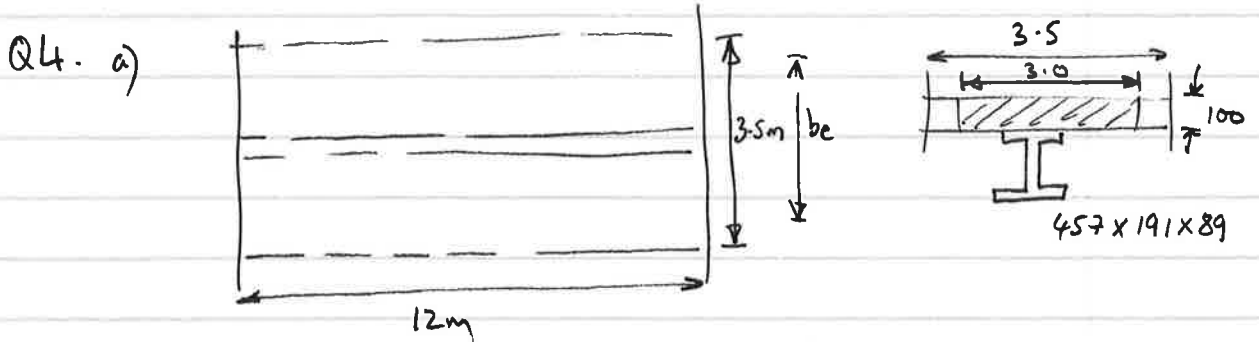
Stress at stiffener = 148 MPa (from earlier)

Longitudinal fillet weld with transverse crack = Class D (DSS)

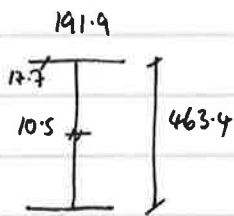
$$\begin{array}{ccc} m & K_2 & G_0 \\ 3 & 1.52 \times 10^{12} & 53 \end{array} \quad G_r > G_0$$

$$N G_r^m = K_2$$

$$N = \frac{1.52 \times 10^{12}}{(148)^3} = 0.469 \times 10^6 \text{ cycles} \\ (\approx \frac{1}{2} \text{ a million}).$$



$$\text{effective slab width} = \min\left(3.5, \frac{\text{span}}{4}\right) = \underline{\underline{3.0\text{m}}}$$



$$\lambda_{\text{flange}} = \frac{b}{t} \sqrt{\frac{355}{355}} = \frac{191.9 - 10.5}{(17.7)(2)} = 5.1 < 8 \quad \checkmark \text{ OK}$$

$$\lambda_{\text{web}} = \frac{b}{t} \sqrt{1} = \frac{463.4 - 2(17.7)}{10.5} = 40.8 > 25 \text{ but } < 56$$

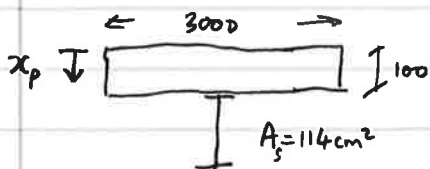
\therefore web not compact in compression but compact in bending, so OK here.

Permanent Load:	Concrete	$24 \text{ kN/m}^3 \times 0.1 \text{ m} \times 3.5 \text{ m}$	$= 8.4 \text{ kN/m}$
	Steel	89.3 kg/m	$= 0.88 \text{ kN/m}$
	Finishes	$1 \text{ kN/m}^2 \times 3.5 \text{ m}$	$= 3.5 \text{ kN/m}$
			<u><u>12.78 kN/m</u></u>

Imposed Load	$6 \text{ kN/m}^2 \times 3.5 \text{ m}$	$= \underline{\underline{21 \text{ kN/m}}}$
--------------	-----------------------------------------	---------------------------------------------

$$\text{Design load} = 1.35(12.78) + 1.5(21) = 17.25 + 31.5 = \underline{\underline{48.8 \text{ kN/m}}}$$

$$\text{BM} = \frac{wL^2}{8} = \frac{48.8 \times 12^2}{8} = \underline{\underline{878.4 \text{ kNm}}}$$



Assume N/A in slab:

$$x_p = \frac{A_s f_{sy}}{0.6 b_c f_{cd}} = \frac{114 \times 10^2 \times 355}{0.6 (3000) (30)} = \underline{\underline{74.9 \text{ mm}}} \quad \therefore \text{in slab } \checkmark$$

$$M_{ult} = (114 \times 10^2) \text{ mm}^2 \times 355 \left[\frac{463.4}{2} + 100 - \frac{74.9}{2} \right] \text{ mm} \times \text{N/mm}^2$$

$$= \underline{\underline{1191 \text{ kNm}}} > 878.4 \text{ kNm} \quad \therefore \underline{\underline{\text{OK}}}$$

4D1 Steel 2014

Q4 b) Studs. For full strength need studs in each half span to be capable of carrying the UB axial load ($A_s G_y$)

$$A_s G_y = (114 \times 10^2)(355) \quad \text{mm}^2 \text{ N/mm}^2$$

$$= 4047 \text{ kN}$$

Studs: 13mm dia, 65mm high \rightarrow 47kN each DSB

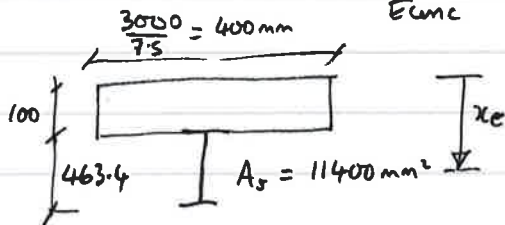
$$\therefore \text{No. req'd for whole beam} = \frac{2 \times 4047}{47} = 172$$

$$\text{Spacing} = \frac{12000}{172} = 70 \text{ mm.} \quad \text{Singly}$$

(or place (more) in pairs at $140 \text{ mm} \times 0.8 = 112$ say 110mm)
 \uparrow
 80% efficient

c) Deflection: $E_{\text{conc}} = 28 \text{ kN/mm}^2$ (short term) DSB

$$\frac{E_{\text{steel}}}{E_{\text{conc}}} = \frac{210}{28} = 7.5$$



$$\text{centroid } x_c = \frac{\int x dA}{\int dA}$$

$$= \frac{(400)(100)(50) + (11400)(100 + 463.4/2)}{40000 + 11400}$$

$$= 112.5 \text{ mm}$$

$$I_s = \underbrace{41020 \times 10^4}_{\text{steel}} + \underbrace{11400 \left(\frac{463.4}{2} - 112.5 \right)^2}_{\text{steel // axis}} + \underbrace{\frac{(400)(100)^3}{12}}_{\text{conc}} + \underbrace{400(100)(112.5 - 50)^2}_{\text{conc // axis}}$$

$$= 1147.6 \times 10^6 \text{ mm}^4.$$

$$\delta = \frac{5}{384} \frac{(21 \times 12)(12)^3 \times 10^9}{(210)(1147 \times 10^6)} = 23.5 \text{ mm}$$

(cf $\frac{\text{span}}{250} = \frac{12000}{250} = 48$, so deflection OK).