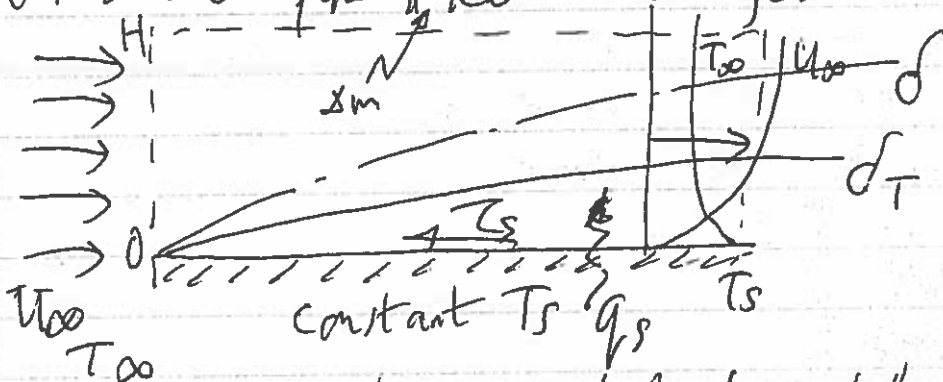


1. (a) Sketch flow & thermal boundary layers:



The relative thickness of the flow & thermal boundary layers,  $\delta$  &  $\delta_T$ , depends on the Prandtl number,  $Pr = \mu c_p / \lambda$ . The Prandtl number measures the viscous work, the thermal diffusion; for  $Pr > 1$ ,  $\delta > \delta_T$  & vice versa.

The Stanton number,  $St = \frac{h}{\rho c_p U_{\infty}}$ , Nusselt number,  $Nu = hx / \lambda$ , & Reynolds number,  $Re = \rho U_{\infty} x / \mu$  are related by:  $St = Nu / Pr Re$ , where  $h = q_s / (T_{\infty} - T_s)$

$St$  is "the heat transfer per unit convection"  
 $Re$  is "the convection vs. the viscous diffusion"  
 $Nu$  is "the non-dimensional heat transfer"

(b) There are various ways to derive the integral equations; here we use "A" for the  $L/N$  model:

$$\int_0^H (\rho U_{\infty} c_p T_{\infty} - \rho u c_p T) dy - \Delta m c_p T_{\infty} = \int_0^x q_s dx$$

$$\int_0^H (\rho U_{\infty} - \rho u) dy = \Delta m$$

$$\therefore \rho c_p T_{\infty} \int_0^H \frac{(T_{\infty} - T) u}{U_{\infty}} dy = \int_0^x q_s dx$$

$$\delta_{\theta} = \int_0^H \frac{(T_{\infty} - T) u}{(T_{\infty} - T_s) U_{\infty}} dy$$

$d/x:$   $q_s = \rho c_p u_{\infty} (T_{\infty} - T_s) \frac{d(\delta_{\theta})}{dx}$

$St = \frac{q_s / (T_{\infty} - T_s)}{\rho c_p u_{\infty}} \therefore \frac{d(\delta_{\theta})}{dx} = St$

(C) Given:  $\frac{u}{u_{\infty}} = \frac{3}{2} \left(\frac{y}{\delta}\right) - \frac{1}{2} \left(\frac{y}{\delta}\right)^3$  &  $\theta = \frac{T - T_s}{T_{\infty} - T_s} = \frac{3}{2} \left(\frac{y}{\delta_T}\right) - \frac{1}{2} \left(\frac{y}{\delta_T}\right)^3$

Let  $Pr = 1$

$\hookrightarrow$  assume  $\delta = \delta_T$   $\left. \begin{matrix} T = T_s \text{ at } y = 0 \\ T = T_{\infty} \text{ at } y = \delta \end{matrix} \right\}$

(otherwise we have to split & integrate - just algebra...)

So,  $\delta_{\theta} = \int_0^{\delta} \frac{\left(\frac{3}{2} \eta - \frac{1}{2} \eta^3\right) \left(1 - \frac{3}{2} \eta + \frac{1}{2} \eta^3\right) d\eta}{\frac{u}{u_{\infty}}}$

$= \left[ \frac{3}{2} \frac{\eta^2}{2} - \frac{9}{4} \frac{\eta^3}{3} + \frac{3}{4} \frac{\eta^5}{5} - \frac{1}{2} \frac{\eta^4}{4} + \frac{3}{4} \frac{\eta^5}{5} - \frac{1}{4} \frac{\eta^7}{7} \right]_0^{\delta}$

~~...~~  $\therefore \delta_{\theta} = 0.14 \delta$  [Levee (3), side 21]

(d) Given: Blasius,  $\delta/x = 5.0/\sqrt{Re_x}$

Given,  $Nu = St Pr Re_x$

$= Pr \frac{\rho u_{\infty} \delta}{\mu} \cdot 0.14 \frac{d}{dx} \left( 5.0 \frac{\sqrt{\mu x}}{\sqrt{\rho u_{\infty} \mu}} \right)$   $St$  from (b)

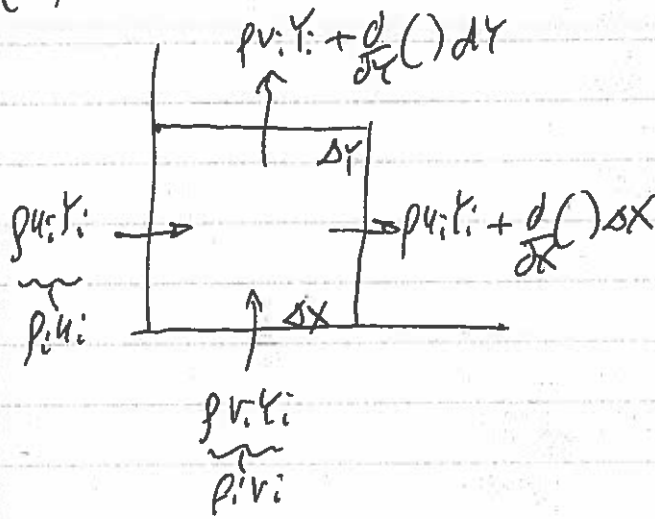
$= Pr \frac{\rho u_{\infty} x}{\mu} \cdot 0.14 \times 5.0 \times \frac{\sqrt{\mu}}{\sqrt{\rho u_{\infty}}} \cdot \frac{1}{2} \frac{1}{x^{1/2}}$

$\therefore Nu = 0.35 Pr Re_x^{1/2}$

cf. correlation ( $Pr > 0.6$ ),  $Nu = 0.45 Pr^{1/3} Re^{1/2}$

2

(a) Consider  $\Delta x, \Delta t$  control volume:



contents:  $\Delta x \Delta y \frac{d(p y_i)}{dx}$

net flux:  $\frac{d(p y_i)}{dx} \Delta x \Delta y + \frac{d(p y_i)}{dy} \Delta y \Delta x$

source:  $\dot{w}_i \Delta x \Delta y$

$\therefore \frac{d(p y_i)}{dt} + V \cdot p y_i = \dot{w}_i$        $G_{di} = \rho_i (u_i - u)$

$\bar{u}_i = u + u_{di}$   
 $\bar{u}_{bulke}$       diffusive fluxes

$\therefore \frac{d(p y_i)}{dt} + V \cdot p y_i = -V \cdot G_{di} + \dot{w}_i$

(b) Sum over all species:

$$\frac{d}{dt} \rho \sum_{i=1}^N y_i + V \cdot \rho u \sum_{i=1}^N y_i = -V \cdot \sum_{i=1}^N G_{di} + \sum_{i=1}^N \dot{w}_i$$

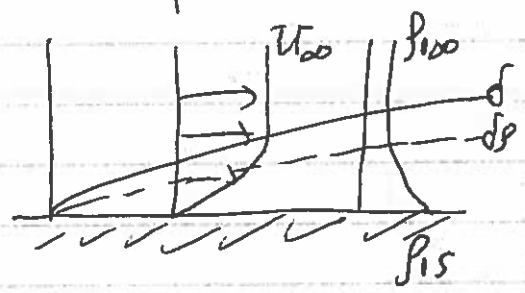
$\quad \quad \quad = 1 \quad \quad \quad = 1 \quad \quad \quad = 0 \quad \quad \quad = 0 \quad \quad \quad !$

$\therefore \frac{d\rho}{dt} + V \cdot \rho u = 0$ , standard bulk mass conservation eq<sup>n</sup>.

(c) Fick's Law:  $G_{di} = -\rho D_{ij} \nabla y_i$

$y_i = p_i/p$  so  $G_{di} = -D_{ij} \nabla p_i$       "D<sub>ij</sub> is a coefficient"

Hence, for mass transfer from a flat plate:



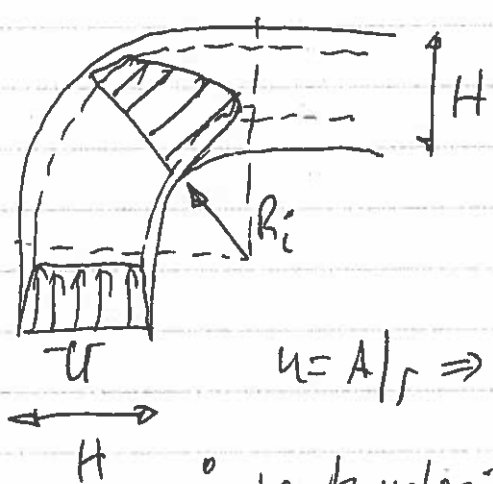
mass transfer coefficient:

$$h_m = - \frac{D (dp/dy)_{y=0}}{(P_{is} - P_{\infty})}$$

cf. heat transfer coefficient,  $h = - \frac{\lambda (dT/dy)_{y=0}}{\rho_p U_{\infty} (T_s - T_{\infty})}$

cf. momentum transfer coeff.  $C_f = \frac{\mu (du/dy)_{y=0}}{\frac{1}{2} \rho U_{\infty}^2}$

(d) Consider developing flow in a rectangular channel with thin, turbulent b'layers and



$$Sh_x \sim Sc^{1/3} Re_x^{4/5}$$

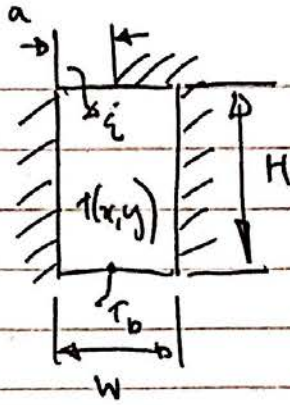
assuming free-vortex flow outside the kin b'layers

$$u = A/r \Rightarrow \int uH = \int_{R_i}^{R_i+H} \frac{A}{r} dr = A \log_e \left( 1 + \frac{H}{R_i} \right)$$

∴ peak velocity at inside of bend  $u_{pk} = \frac{uH}{R_i} \cdot \frac{1}{\log_e(1 + H/R_i)}$

∴ ratio of mass transfer coefficient  $\frac{h_{mH}}{h_{m2H}} = \frac{2 \log_e(1.5)}{\log_e(2)} = 1.13$

So, 13% more mass transfer



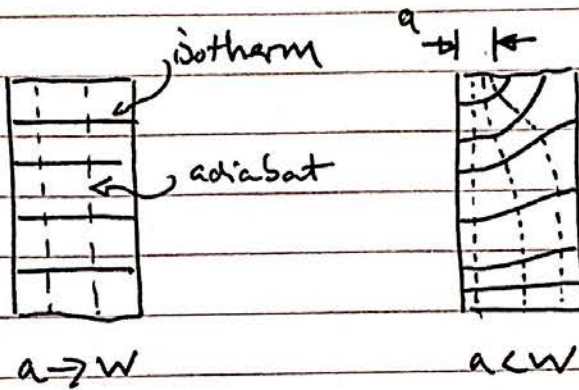
i) As  $a \rightarrow w$  the problem can be approximated as 1-D.

$$\dot{Q} = -\lambda \text{Area} \frac{dT}{dy} \Rightarrow -qa = -\lambda w \frac{d\theta}{dy}$$

$$\text{so } \theta = \frac{q}{\lambda} \frac{a}{w} y //$$

this is a lower bound on the full 2-D temperature field, and will be the 0<sup>th</sup> term in the series.

ii)



$a \rightarrow w$

$a \rightarrow w$

iii)

$$\frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2} = 0 \Rightarrow \frac{\partial^2 \theta}{\partial x^2} + \frac{\partial^2 \theta}{\partial y^2} = 0 \quad \text{where } \theta(x, y) = T(x, y) - T_b$$

$$\text{let } \theta = XY \text{ so } Y \frac{\partial^2 X}{\partial x^2} + X \frac{\partial^2 Y}{\partial y^2} = 0 \Rightarrow \frac{1}{X} \frac{\partial^2 X}{\partial x^2} = -\frac{1}{Y} \frac{\partial^2 Y}{\partial y^2} = -m^2$$

(we need a negative separation constant for homogeneous  $x$  boundaries)

$$\Rightarrow \theta = (C_1 \sin(mx) + C_2 \cos(mx)) (C_3 \sinh(my) + C_4 \cosh(my))$$

$$\text{for } \theta(0, y) = \theta(w, y) = 0$$

$$\theta(x, y) = \sum_{n=0}^{\infty} \cos\left(\frac{n\pi x}{w}\right) (A_n \sinh\left(\frac{n\pi y}{H}\right) + B_n \cosh\left(\frac{n\pi y}{H}\right)) //$$

iv) Consider:  $\theta(x, 0) = 0$

$$\theta(x, 0) = \sum_{n=0}^{\infty} \cos(m_n x) (A_n \sinh(0) + B_n \cosh(0)) = 0$$

$$\Rightarrow B_n = 0$$

Now look at  $0^{\text{th}}$  term carefully:  $m_n y$

$$\theta(x, y) = \underbrace{\lim_{n \rightarrow 0} \left\{ \cos(m_n x) A_n \sinh(m_n y) \right\}}_{A_0} + \sum_{n=1}^{\infty} \cos(m_n x) A_n \sinh(m_n y)$$

$$A_0 = \lim_{n \rightarrow 0} \left\{ A_n m_n \right\} y \quad \text{so we need to find } \lim_{n \rightarrow 0} \left\{ A_n m_n \right\}$$

Consider:  $\lambda \frac{\partial \theta(x, H)}{\partial y} = \begin{cases} q & x < a \\ 0 & \text{otherwise} \end{cases}$

$$\frac{\partial \theta(x, H)}{\partial y} = \lim_{n \rightarrow 0} \left\{ A_n m_n \right\} + \sum_{n=1}^{\infty} \cos(m_n x) A_n m_n \cosh(m_n H) = \begin{cases} q/\lambda \\ 0 \end{cases}$$

to find  $\lim_{n \rightarrow 0} \left\{ A_n m_n \right\}$  and  $A_n$  we need to

multiply both sides by  $\cos(m_n x)$  and integrate

from  $x=0 \Rightarrow W$ .

$$\int_0^w \lim_{n \rightarrow \infty} \{A_n m_n\} \cos(m_n x) dx + \sum_{n=1}^{\infty} \cos(m_n x) \cos(m_n x) A_n m_n \cosh(m_n H) dx \dots$$

$$= \int_0^a \frac{q}{\lambda} \cos(m_n x) dx + \int_a^w 0 \cos(m_n x) dx$$

For  $k=0$  we have:

$$\int_0^w \lim_{n \rightarrow \infty} \{A_n m_n\} dx = \int_0^a \frac{q}{\lambda} \cos(m_n x) dx$$

$$\lim_{n \rightarrow \infty} \{A_n m_n\} w = \frac{q a}{\lambda} \Rightarrow \lim_{n \rightarrow \infty} \{A_n m_n\} = \frac{q a}{\lambda w} //$$

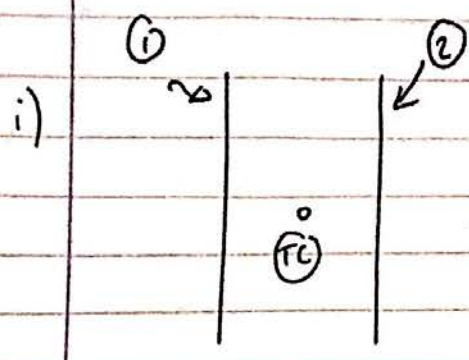
For  $k=1, 2, \dots$  we have: for  $n=k$  only

$$\int_0^w \cos^2(m_n x) A_n m_n \cosh(m_n H) dx = \int_0^a \frac{q}{\lambda} \cos(m_n x) dx$$

$$\int_0^w \cos^2(m_n x) dx = \frac{w}{2}, \quad \int_0^a \cos(m_n x) dx = \frac{\sin(m_n a)}{m_n}$$

$$\Rightarrow A_n = 2 \frac{q}{\lambda w} \frac{\sin(m_n a)}{m_n^2} \frac{1}{\cosh(m_n H)} //$$

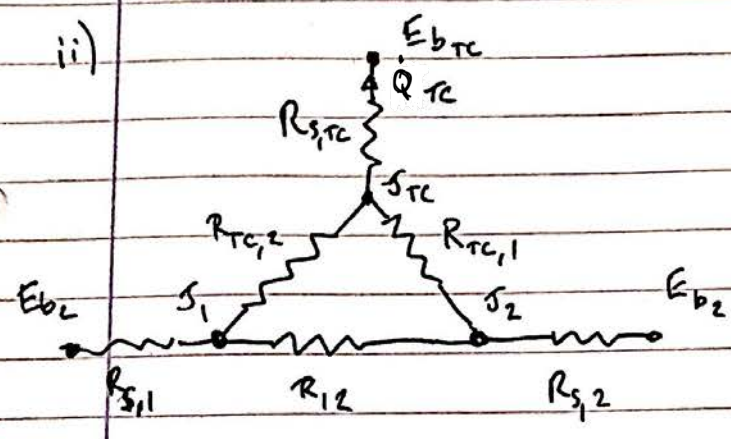
$$\theta(x, y) = \frac{q}{\lambda w} \left\{ a y + 2 \sum_{n=1}^{\infty} \cos(m_n x) \frac{\sinh(m_n y)}{\cosh(m_n H)} \frac{\sin(m_n a)}{m_n^2} \right\} //$$



$$F_{TC,TC} + F_{TC,1} + F_{TC,2} = 1$$

By symmetry  $F_{TC,1} = F_{TC,2} = 0.5$

ii)



$$R_{S,TC} = \frac{1 - \epsilon_{TC}}{\epsilon_{TC} A_{TC}}$$

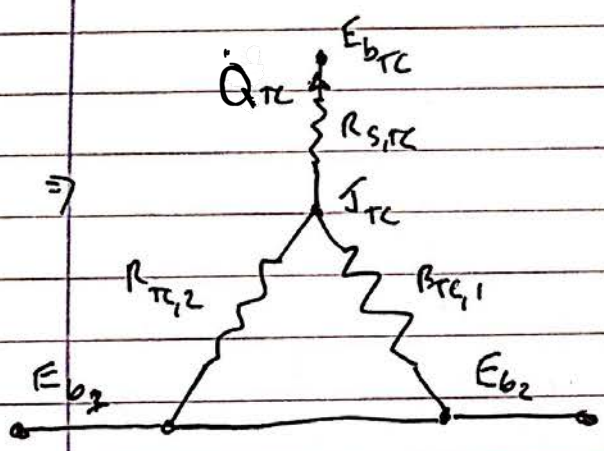
$$R_{TC,1} = \frac{1}{F_{TC,1} A_{TC}} = \frac{2}{A_{TC}}$$

$$R_{TC,2} = \frac{1}{F_{TC,2} A_{TC}} = \frac{2}{A_{TC}}$$

$$R_{12} = \frac{1}{\tau_{12} A_1} \rightarrow 0$$

$$R_{S,1} = \frac{1 - \epsilon_1}{\epsilon_1 A_1} \rightarrow 0$$

$$R_{S,2} = \frac{1 - \epsilon_2}{\epsilon_2 A_2} \rightarrow 0$$



iii) Energy balance at  $J_{TC}$

$$\frac{E_{b2} - J_{TC}}{R_{TC,2}} + \frac{E_{b1} - J_{TC}}{R_{TC,1}} = \frac{J_{TC} - E_{bTC}}{R_{S,TC}}$$



$$\Rightarrow J_{TC} = \frac{E_{b2} + E_{b1} + E_{brc}}{R_{TC2} + R_{TC1} + R_{S,TC}}$$

$$\left( \frac{1}{R_{S,TC}} + \frac{1}{R_{TC2}} + \frac{1}{R_{TC1}} \right)$$

$$= \frac{A_{TC} \frac{1}{2} \sigma (T_2^4 + T_1^4)}{1 - \epsilon_{TC}} + \frac{\epsilon_{TC} A_{TC} \sigma T_{TC}^4}{1 - \epsilon_{TC}}$$

$$\frac{\epsilon_{TC} A_{TC}}{1 - \epsilon_{TC}} + \frac{A_{TC}}{2} + \frac{A_{TC}}{2}$$

$$= \frac{\frac{1}{2} \sigma (T_2^4 + T_1^4)}{1 - \epsilon_{TC}} + \frac{\epsilon_{TC} \sigma T_{TC}^4}{1 - \epsilon_{TC}}$$

$$\frac{\epsilon_{TC}}{1 - \epsilon_{TC}} + 1$$

$$J_{TC} = \frac{(1 - \epsilon_{TC}) \frac{1}{2} \sigma (T_2^4 + T_1^4) + \epsilon_{TC} \sigma T_{TC}^4}{\epsilon_{TC} + 1 - \epsilon_{TC}}$$

$$\epsilon_{TC} + 1 - \epsilon_{TC}$$

$$Q_{TC} = \frac{J_{TC} - E_{brc}}{R_{S,TC}} = \frac{(1 - \epsilon_{TC}) \frac{1}{2} \sigma (T_2^4 + T_1^4) + (\epsilon_{TC} - 1) \sigma T_{TC}^4}{\frac{(1 - \epsilon_{TC})}{\epsilon_{TC} A_{TC}}}$$

$$= \frac{(1 - \epsilon_{TC})}{(1 - \epsilon_{TC})} \left[ \frac{1}{2} \sigma (T_2^4 + T_1^4) - \sigma T_{TC}^4 \right] \epsilon_{TC} A_{TC}$$

$$Q_{TC}$$

$$= A_{TC} \epsilon \sigma \left[ \frac{1}{2} (T_2^4 + T_1^4) - T_{TC}^4 \right]$$

iv) In equilibrium:  $\dot{q}_{TC} = 0$

$$\Rightarrow \frac{1}{2}(T_2^4 + T_1^4) = T_{TC}^4$$

$$\Rightarrow T_{TC} = \left( \frac{1}{2}(T_2^4 + T_1^4) \right)^{1/4}$$

- b) Convective cooling reduces temperature at  
 i) the bead until the radiation is in balance,  
 so result from (a)(iii) still valid:

$Q_{\text{radiation}} = Q_{\text{convection}}$

$$A_{TC} h (T_{\text{air}} - T_{TC}) = A_{TC} \frac{\epsilon_{TC} \sigma}{h} \left[ \frac{1}{2} (T_2^4 + T_1^4) - T_{TC}^4 \right]$$

$$T_{TC} = T_{\text{air}} - \frac{\epsilon_{TC} \sigma}{h} \left[ \frac{1}{2} (T_2^4 + T_1^4) - T_{TC}^4 \right]$$

or  $T_{\text{air}} - T_{TC} = \frac{\epsilon_{TC} \sigma}{h} \left[ \frac{1}{2} (T_2^4 + T_1^4) - T_{TC}^4 \right]$

$$T_{TC} = T_{\text{air}} - \frac{\epsilon_{TC} \sigma}{h} \left[ \frac{1}{2} (T_2^4 + T_1^4) - T_{TC}^4 \right]$$

ii)	$h = 50 \text{ W m}^{-2} \text{ K}^{-1}$ $T_1 = 500 \text{ K}$ $T_2 = 300 \text{ K}$	Guess 1: $T_{TC} = 400 \text{ K}$	$394.5 \text{ K}$
		Guess 2: $T_{TC} = 394.5$	$393.72 \text{ K}$
		Guess 3: $T_{TC} = 393.72$	$393.61 \text{ K}$
		Guess 4: $T_{TC} = 393.61$	$393.6 \text{ K}$

so  $\Delta T = 7.4 \text{ K}$

- iii) Reduce  $\epsilon_{TC}$ , increase  $h$ , use a radiation shield at an intermediate temperature.

