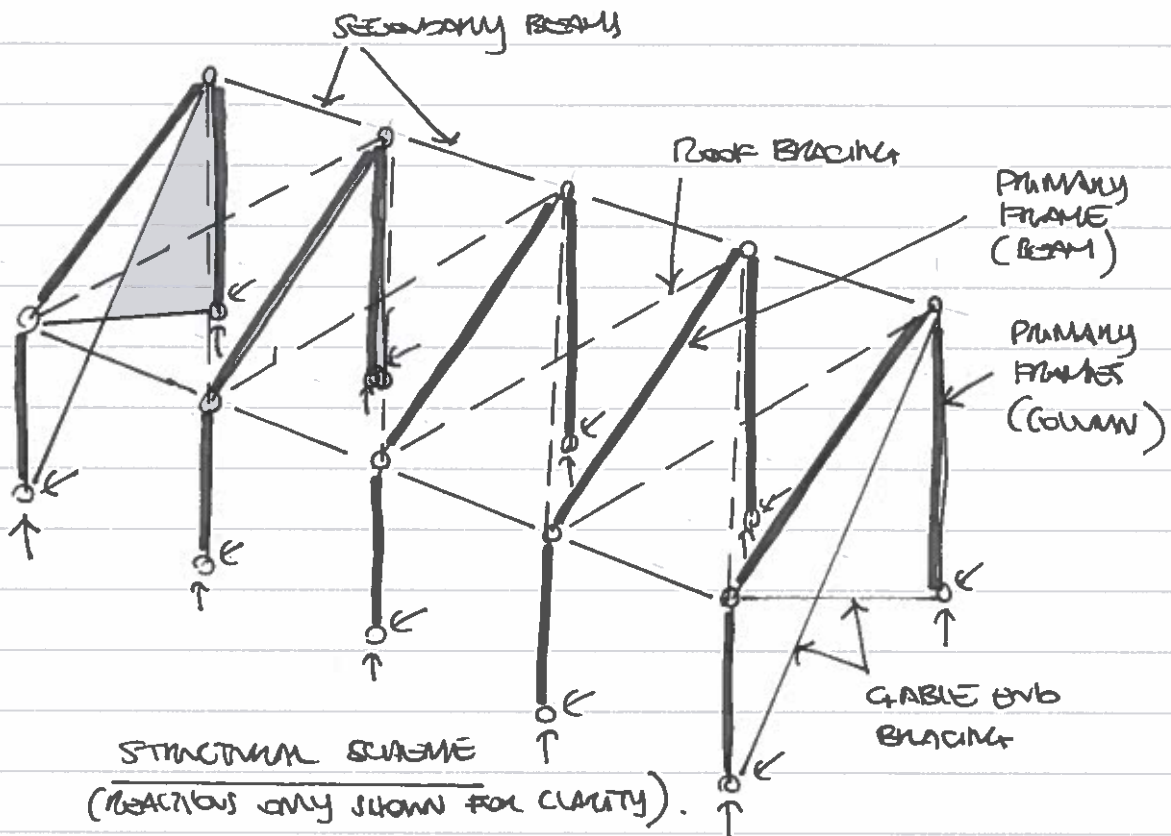


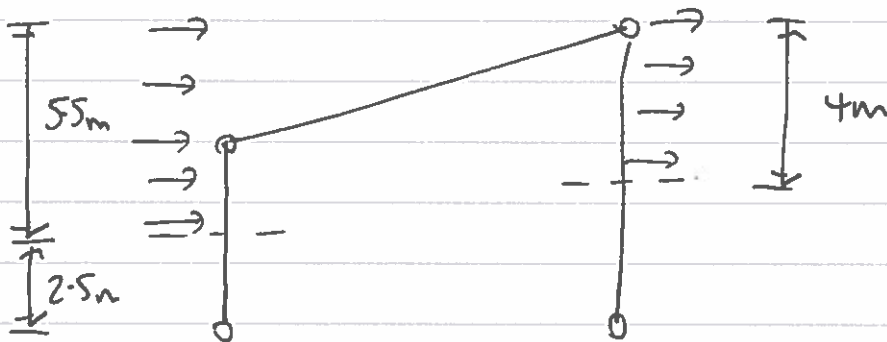
1a)

LOAD PATHS:

VERTICAL LOADS ARISING FROM SELF-WEIGHT OF CLADDING AND SNOW LOADS ARE CARRIED IN PURSUE OF THE METAL DECK WHICH SPANS BETWEEN THE PRIMARY FRAME BEAMS, WHICH ARE IN TURN SIMPLY SUPPORTED BY THE PRIMARY FRAME COLUMNS, THAT TRANSMIT THE AXIAL LOAD INTO THE FOUNDATIONS.

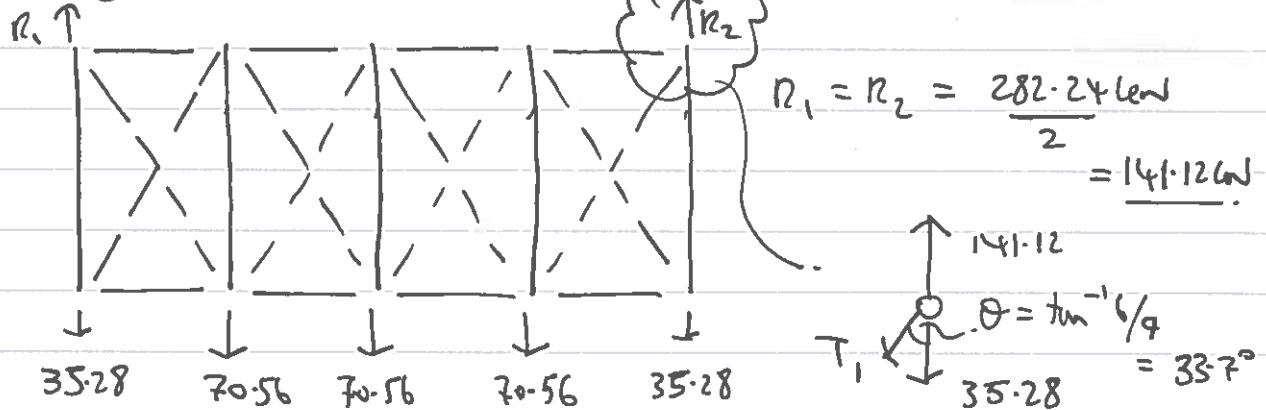
HORIZONTAL LOADS ARISING FROM WIND LOAD ARE CARRIED IN PURSUE OF THE METAL CLADDING THAT SPANS BETWEEN THE PRIMARY BEAMS AND COLUMNS. THIS WOULD CAUSE THE PIN-JOINED MAIN FRAMES TO SWAY. THIS IS PREVENTED BY THE BRACING IN THE ROOF PLANE WHICH FORMS A TRUSS SPANNING THE WIDTH OF THE BUILDING, ~~BEHIND~~ THE GABLE ENDS OF THE BUILDING PROVIDE A STIFF VERTICAL DIAPHRAGM TO SUPPORT THE ENDS OF THE ROOF TRUSS AND TRANSMIT THE FORCES INTO THE FOUNDATIONS.

1b i) WIND LOADS ON ROOF TRUSS:



$$\begin{aligned} \text{LATERAL LOAD } P \text{ PER FRAME} &= \left[(1.2 \text{ kN/m}^2) \times 5.5 \text{ m} \times 6 \text{ m} \right. \\ &\quad \left. + 0.8 \text{ kN/m}^2 \times 4 \text{ m} \times 6 \text{ m} \right] \times 1.2 \\ &= 70.56 \text{ kN} \end{aligned}$$

Plan view:

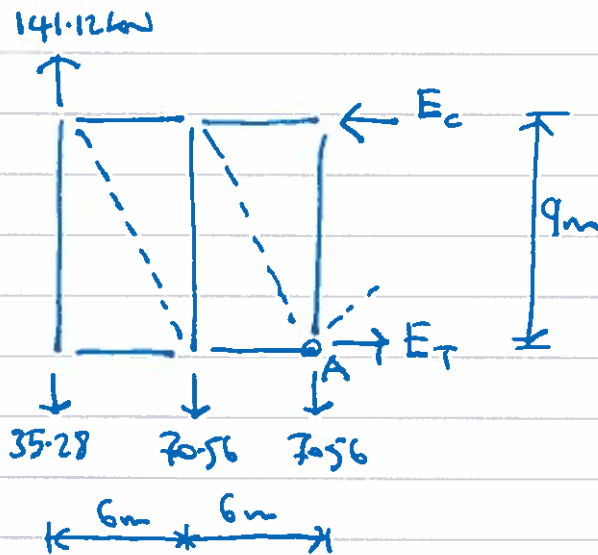


$$\begin{aligned} \therefore \text{TENSION IN ROOF MEMBER } T_1 &= \frac{(141.12 - 35.28)}{\cos 33.7} \\ &= 127.2 \text{ kN} \end{aligned}$$

STRENGTH CHECK:

$$\begin{aligned} P &= A \sigma_y / \gamma_m \quad \therefore A = \frac{P \gamma_m}{\sigma_y} \\ &= \frac{127.2 \times 10^3 \times 1.1}{355} \\ &= 394.1 \text{ mm}^2 \end{aligned}$$

$$\therefore \underline{\text{UJE CH 76} \times 38} \quad (A = 856 \text{ mm}^2)$$

COMPRESSION IN EAVES MEMBER.

$$E_c = \frac{1}{9} \left[(141.12 - 35.28) \cdot 12 - (70.56 \times 6) \right] = 94.08 \text{ kN}$$

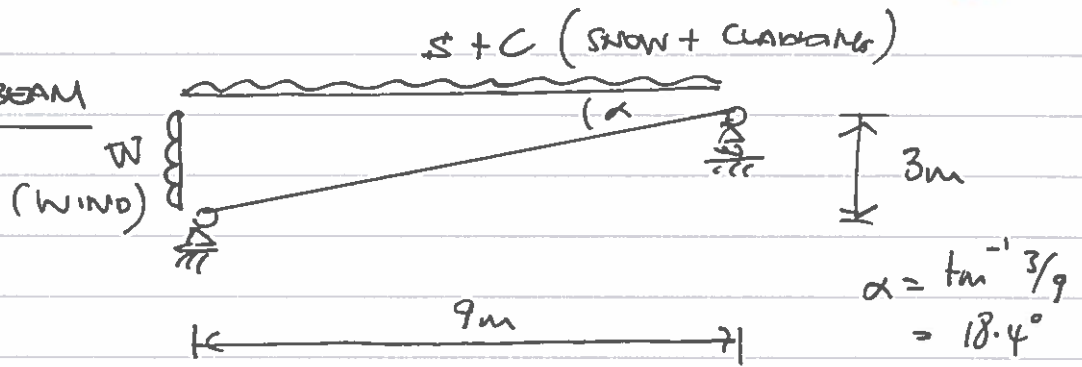
| COMPRESSION IN EAVES MEMBER E_c | <

| TENSION IN BRACING T_1 | \therefore IF EAVES MEMBER IS RESTRAINED AGAINST BUCKLING NO FURTHER CHECKS ARE NECESSARY (USE CH 78 x 38 TIRROCHOUT).

IF EAVES MEMBER NOT RESTRAINED AGAINST BUCKLING DESIGN EAVES MEMBER AS PIN-JOINED STUNT:



16ii) PRIMARY BEAM



$$2 \quad \text{UDL } \perp \text{ TO BEAM} = W \sin \alpha + S + C (\cos \alpha)$$

$$= 0.32W + 0.95(S + C)$$

$$W = 1.2 \text{ kN/m}^2 \times 3 \text{ m} \times 6 \text{ m} = 21.6 \text{ kN}$$

$$S + C = 0.95 \text{ kN/m}^2 \times 9 \text{ m} \times 6 \text{ m} = 51.3 \text{ kN}$$

$$\therefore \text{UDL @ SLS} = (0.32 \times 21.6) + (0.95 \times 51.3) = 55.65 \text{ kN}$$

$$\text{UDL @ ULS} = (0.32 \times 21.6 \times 1.2) + (0.95 \times 51.3 \times 1.2) = 66.78 \text{ kN}$$

BENDING CAPACITY (ULS):

$$M_{max} = 66.78 \times \frac{9}{8} = 75.12 \text{ kNm}$$

$$\therefore Z_p \geq \left(\frac{75.12 \times 10^6 \times 1.1}{355} \right) \times 10^{-3} = 233 \text{ cm}^3$$

$$2 \quad \therefore \text{SELECT UB } 254 \times 102 \times 22 \quad (Z_p = 259 \text{ cm}^3).$$

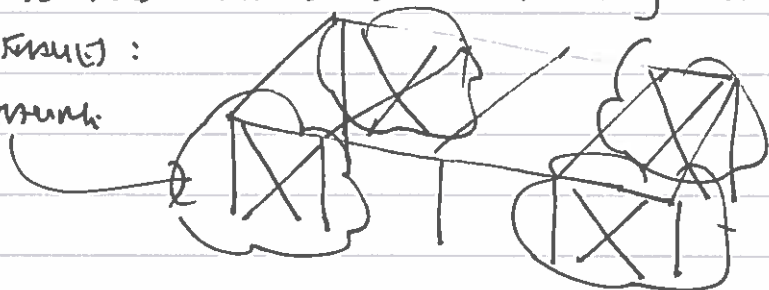
DEFLECTION CHECK (SLS)

$$1 \quad S = \frac{5 \times 55.65 \times 10^3 \times 9^3}{384 \times 210 \times 10^9 \times 2841 \times 10^{-8}} = 0.0885 \text{ m} = \frac{\text{SPAN}}{101}$$

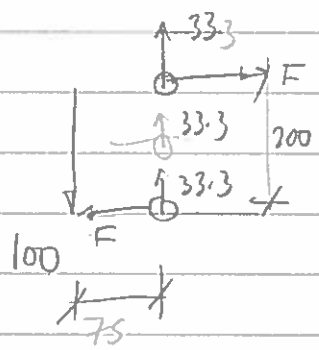
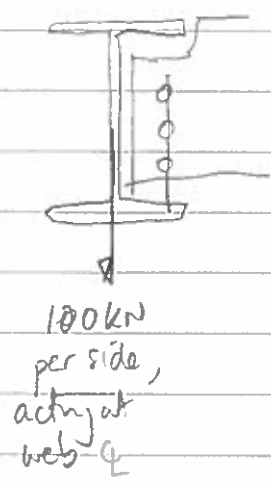
\(\therefore\) UB 254x102x22 IS ADEQUATE.

1c) CROSS BRACING SHOWN IN PART (a) CHECKS FOR TIVE AND -VE ~~MOMENTS~~ MOMENTS WITH DIRECTIONS PARALLEL TO THE PRIMARY BEAMS.

2 FOR WIND \perp TO FRAME:
INFLUENCE CROSS BRACING



2 a)

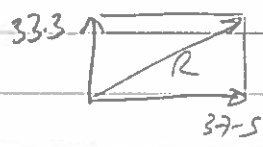


Moment equilib:

$$(100 \text{ kN}) / (75 \text{ mm}) = F (200 \text{ mm})$$

$$\Rightarrow F = \underline{\underline{37.5 \text{ kN}}}$$

∴ At bolt A:

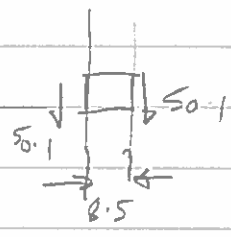


$$R = \sqrt{(33.3)^2 + (37.5)^2}$$

$$= \underline{\underline{50.1 \text{ kN}}}$$

on shear plane of bolt.

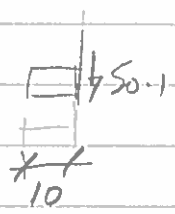
ii) Web bearing stress



$$\sigma_b = \frac{2(50.1 \times 10^3 \text{ N})}{20(8.5)}$$

$$= \underline{\underline{589 \text{ MPa}}}$$

iii) Cleat bearing stress



$$\sigma_b = \frac{50.1 \times 10^3}{20(10)}$$

$$= \underline{\underline{250 \text{ MPa}}}$$

Allowable load, i) Bolt $\Rightarrow \frac{39.4}{50.1} \times 200 \text{ kN} = (0.79)(200) = \underline{\underline{157 \text{ kN}}}$

ii), iii) Bearing $\Rightarrow \frac{1.5(275)}{589} \times 200 = 0.70(200) = \underline{\underline{140 \text{ kN}}}$

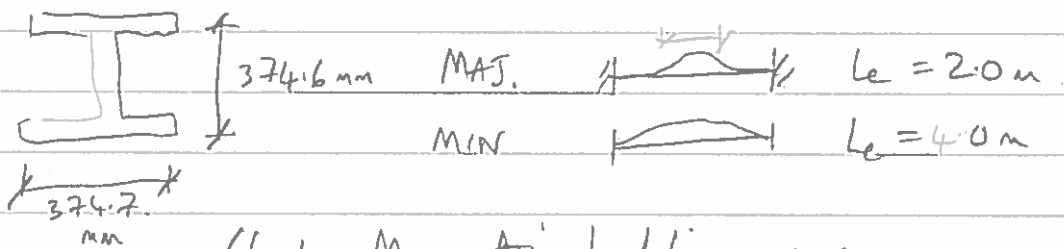
2 So bearing governs, and plate load = 140 kN

b) UC 356 x 368 x 202 in S275 steel.

$$I_{\text{MAJ}} = 66260 \times 10^{-8} \text{ m}^4$$

$$I_{\text{MIN}} = 23690 \times 10^{-8} \text{ m}^4$$

$$A = 257 \times 10^{-4} \text{ m}^2$$

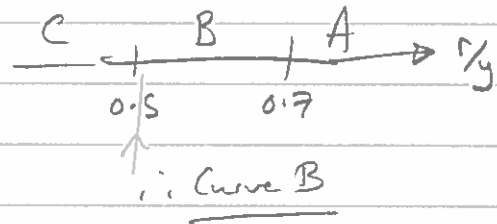


Clearly Minor Axis buckling governs.

$$r = \sqrt{\frac{I_{\text{min}}}{A}} = \sqrt{\frac{23690 \text{ cm}^4}{257 \text{ cm}^2}} = 9.6 \text{ cm} = \underline{\underline{96 \text{ mm}}}$$

$$y_{\text{EXT FIBRE}} = \frac{374.7}{2} = 187.4 \text{ mm}$$

$$\frac{y}{r} = \frac{96}{187.4} = \underline{\underline{0.51}}$$



$$\lambda = \frac{L}{r} = \frac{4000}{96} = \underline{\underline{41.7}}$$

$$\lambda_0 = \pi \sqrt{\frac{E}{G_y}} = \pi \sqrt{\frac{210 \times 10^3}{275}} = 86.8$$

$$\bar{\lambda} = \frac{\lambda}{\lambda_0} = \frac{41.7}{86.8} = \underline{\underline{0.48}}$$

Graph $\bar{\lambda} = 0.48$, Curve B $\rightarrow \chi \approx 0.9$.

$$N_{\text{pl}} = A_{\text{st}} G_y = 257 \times 10^2 \text{ mm}^2 (275 \text{ N/mm}^2) = 7067 \text{ kN}$$

$$N_{\text{Design}} = 0.9 (7067) = \underline{\underline{6361 \text{ kN}}}$$

$$c) \quad u_s 305 \times 165 \times 54 \approx \underline{\underline{5275}}$$

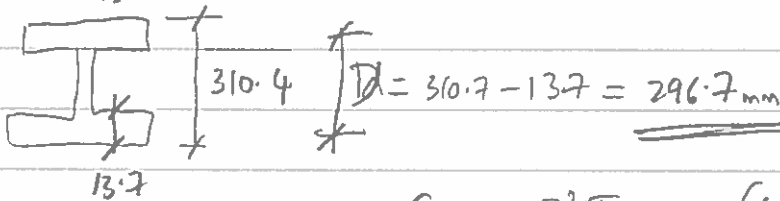
Uniform moment \Rightarrow no shear.

$$I_{max} = 11706 \times 10^{-8} \text{ m}^4$$

$$I_{min} = 1063 \times 10^{-8} \text{ m}^4$$

$$J = 348 \times 10^{-8} \text{ m}^4$$

$$Z_{max} = 846 \times 10^{-6} \text{ m}^3$$



$$C_r = \frac{J^2 I_{yy}}{4} = \frac{(0.2967)^2 (1063)}{4} \times 10^{-8} \text{ m}^6$$

$$= 23.4 \times 10^{-8}$$

$$M_c = \frac{\pi}{6} \left[(210 \times 10^9) (1063 \times 10^{-8}) \left[\frac{210 \times 10^9}{2.6} \times \frac{\pi^2}{36} (210 \times 10^9) 23.4 \times 10^{-8} \right] \right]^{1/2}$$

$$= \frac{\pi}{6} (210 \times 10^9) 10^{-8} \left[1063 \left(\frac{348}{2.6} + \frac{\pi^2}{36} 23.4 \right) \right]^{1/2}$$

$$= \frac{\pi}{6} (2100) 145$$

$$= \underline{\underline{159.5 \text{ kNm}}}$$

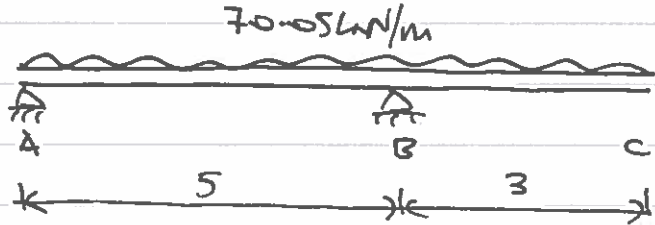
$$M_p = \frac{(275)(846 \times 10^{-6})}{\times 10^3} = \underline{\underline{233 \text{ kNm}}}$$

$$\lambda = \sqrt{\frac{M_p}{M_c}} = \sqrt{\frac{233}{159.5}} = 1.21 \Rightarrow \chi_{LT} = 0.5$$

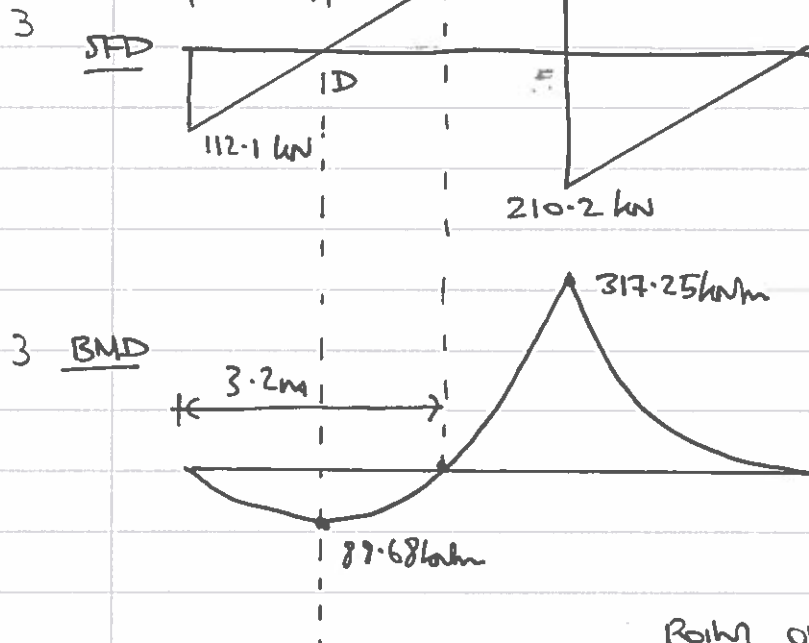
6

$$M_{\text{design}} = 0.5(233) = \underline{\underline{116.5 \text{ kNm}}}$$

3a) DESIGN LIVE LOAD = $40 \text{ kN/m} \times 1.6 = 64 \text{ kN/m}$
 DESIGN DEAD LOAD = $24 \text{ kN/m}^3 \times 0.6 \times 0.3 \times 1.4 = 6.05 \text{ kN/m}$
 Total design load = 70.05 kN/m



B) $\sum R_A = 70.05 \times 8 \times 1$
 $\therefore R_A = 112.1 \text{ kN}$
 $R_B = 448.3 \text{ kN}$

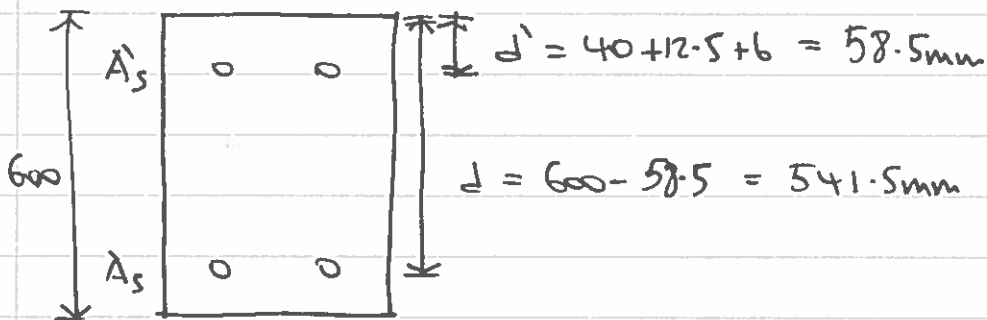


D) $M_D = -70.5 \times 1.6 \times \frac{1.6}{2} + 112.1 \times 1.6 = 89.68 \text{ kNm}$
 or
 $M = \int_0^{1.6} -70.5 dx = -112.1 \times 1.6/2 = 89.68 \text{ kNm}$

E) $M_B = -70.5 \times 3 \times 1.5 = -317.25$

Point of contraflexure $l_1 = 1.6 \times 2 = 3.2 \text{ m}$

3 b) Consider cross-section AT B: ($M = 317.25 \text{ kNm}$; $V = 238.2 \text{ kN}$)



Longitudinal reinforcement:
 If $A'_s = 0$; $M_u = 0.225 f_{cu} b d^2$

$$M_u = 0.225 \times 50 \times 300 \times 541.5^2 / 1.5$$

$$= 659.8 \text{ kNm} > 317.25 \text{ kNm}$$

\therefore NO COMPRESSION STEEL REQUIRED.

$$x/d = \gamma_c A_s f_y / \gamma_s 0.6 f_{cm} b d$$

$$= \frac{1.5}{1.15} \cdot \frac{460 \times 10^6}{0.6 \times 50 \times 10^6} \cdot \frac{1}{0.3 \times 0.5415} \cdot A_s$$

$$\therefore x/d = 123 A_s \quad - (1)$$

$$\text{AS6 } M_u = A_s f_y \frac{d}{\gamma_s} \left(1 - \frac{x}{2d} \right) \quad - (2)$$

$$\therefore M_u = A_s \cdot 460 \times 10^6 \cdot \frac{0.5415}{1.15} \left(1 - \frac{123 A_s}{2} \right) \quad \leftarrow \text{SUBSTITUTE (1) INTO (2)}$$

$$M_u = 216.6 \times 10^6 A_s (1 - 61.5 A_s) \quad - (3)$$

AT CROSS-SECTION B, $M_u = 317.25 \text{ kNm}$

$$\therefore (3) \text{ BECOMES: } 61.5 A_s^2 - A_s + 1.46 \times 10^{-3} = 0$$

SOLVING QUADRATIC FOR LOWER ROOT:

$$A_s = \frac{+1 \pm 0.8}{123} = 1.626 \times 10^{-3} \text{ m}^2$$

$$= 1626 \text{ mm}^2$$

4

1 IN NO. 25mm ϕ BAR = 491 mm²

\therefore PROVIDE 4T25 (1964 mm²).

SHAPE REINFORCEMENT

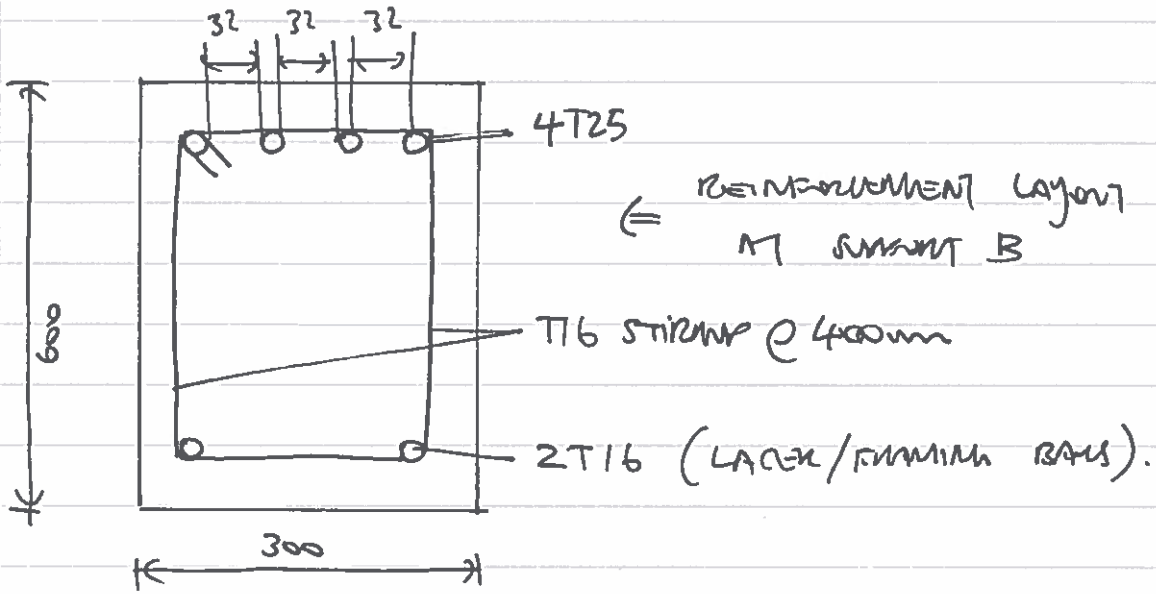
ASSUME $\cot \theta = 2.5$; $s = 0.75 \times 541.5 \rightarrow$ SAY 400mm

$$\therefore A_{sv} = \frac{V_{ud} s \gamma_s}{f_y 0.9 d \cot \theta} \quad - (4)$$

$$= \frac{238.2 \times 10^3 \times 400 \times 1.15}{460 (100.9 \times 541.5) 2.5} = 147.8 \text{ mm}^2$$

2

∴ PROVIDES 1 STIRRUP (2 LEGS) OF T12 (226mm²)

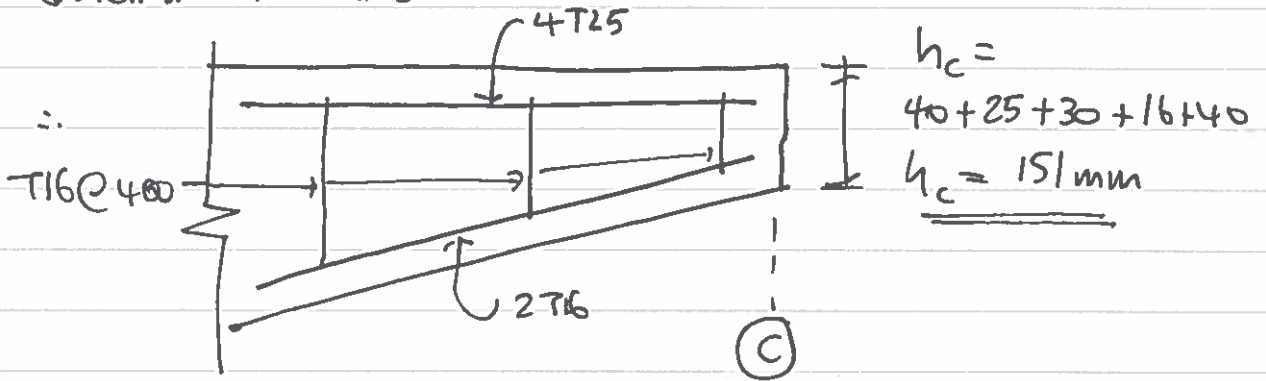


3c)

$M_u \approx 0.8 f_y A_s$ and $V_{rd,s}, V_{rd,c} \propto 1/d$

∴ BY INSPECTION OF BMD $h_c = 0$, BUT THIS WOULD NOT PROVIDE REQUIRED COVER OF 40mm AND SPACING OF BARS

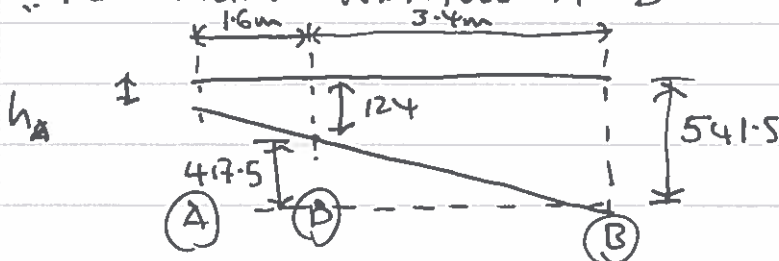
2



h_A IS LIMITED BY SHEAR CRITERIA REINFORCEMENT AT A AND BY MINIMUM DEPTH OF BEAM FOR BENDING AT D.

∴ EFFECTIVE DEPTH $d_D \approx M / 0.8 f_y A_s = 89.68 \times 10^6 / 0.8 \times 460 \times 1964 = 124 \text{ mm}$

∴ FOR BENDING UTILIZATION AT D



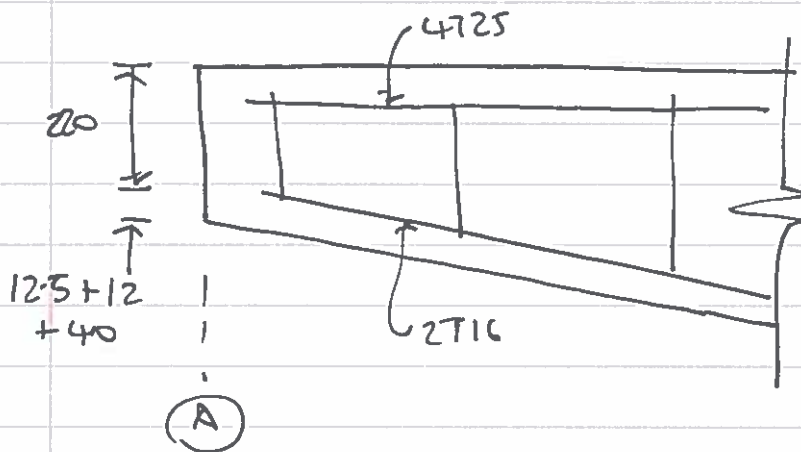
$h_A = 541.5 - \left(\frac{5 \times 417.5}{34} \right)$
 $= -0.5$
 ∴ NOT OK

4 $\therefore h_A$ is limited by ribon controlling members at A.

use-allowance Eq (4): $d = \frac{V_{uds} S^* \gamma_s}{f_y A_s} \cdot 0.9 \cot \theta$

$$= \frac{112.1 \times 10^3 \times 400 \times 1.15}{460 \times 226 \times 0.9 \times 2.5}$$

$$= 220 \text{ mm}$$



$$\therefore h_A = 220 + 12.5 + 12 + 40$$

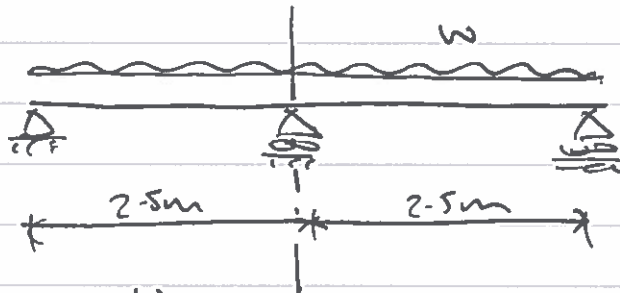
$$= 284.5 \text{ mm}$$

say 285 mm.

4a) LOADS @ ULS:

$$\text{LONG TERM: } 3.5 \text{ kN/m} \times 1.5 = 5.25 \text{ kN/m} = w_L$$

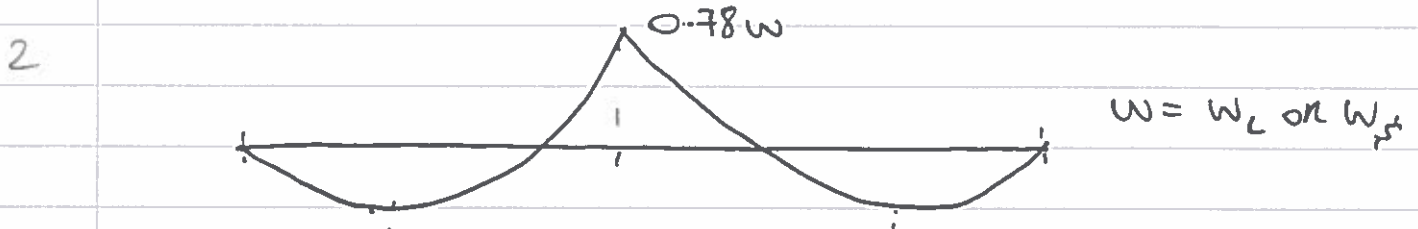
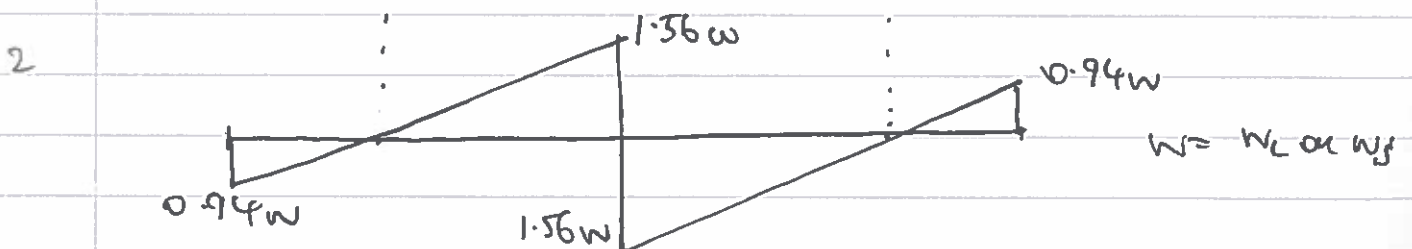
$$\text{SHORT TERM: } (3.5 + 1.2) \times 1.5 = 7.05 \text{ kN/m} = w_S$$

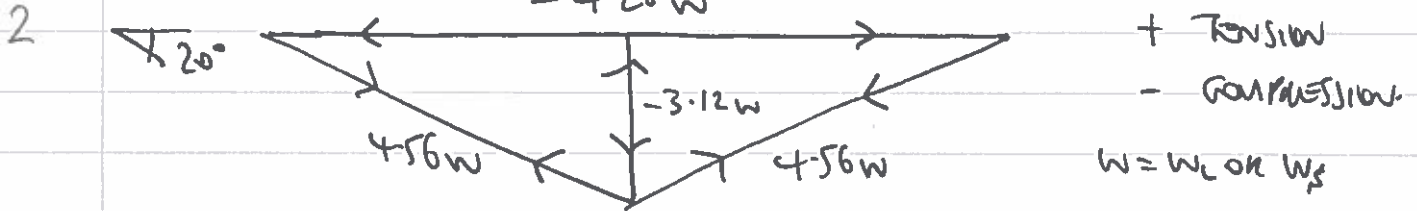
CONSIDER TIMBER BEAM:

① $M_1 = \frac{wl^2}{8}$; $\theta_1 = \frac{wl^3}{24EI}$

② $\theta_2 = \frac{M_2 l}{3EI} = \theta_1$ due to symmetry
 $\therefore M_2 = \frac{wl^2}{8} = \underline{0.78w}$

2 $\sum B: R_A l - wl \frac{l}{2} + \frac{wl^2}{8} = 0$
 $\therefore R_A = \frac{3wl}{8} = \underline{0.94w}$
 $R_B = \frac{5wl}{8} = \underline{1.56w}$

BENDING MOMENT DIAGRAM (TIMBER BEAM):SHEAR FORCE DIAGRAM (TIMBER BEAM):

AXIAL FORCES:

$$\begin{aligned} \text{STRUT} &= -1.56W \times 2 = -3.12W \\ \text{TIE ROD} &= 1.56W / \cos 70^\circ = 4.56W \\ \text{TIMBER} &= -4.56W \cos 20^\circ = -4.28W \end{aligned}$$

4b)

LONG TERM

$$\begin{aligned} M_{\max} &= 0.78 \times 5.25 = 4.1 \text{ kNm} \\ V_{\max} &= 1.56 \times 5.25 = 8.2 \text{ kN} \\ f_{m,d} &= 0.7 \times 1 \times 16 / 1.3 = 8.62 \text{ MPa} \\ f_{v,d} &= 0.7 \times 1 \times 1.8 / 1.3 = 0.97 \text{ MPa} \end{aligned}$$

BENDING: $\sigma = M/z$ WHERE $z = bd^2/6$ (maximum section)

3

$$\therefore d \geq \sqrt{6M/\sigma_b} = \sqrt{\frac{6 \times 4.1 \times 10^6}{8.62 \times 125}} = 151 \text{ mm}$$

3

shear: $d \geq \frac{V}{\tau_b} = \frac{8.2 \times 10^3}{0.97 \times 125} = 68 \text{ mm}$

SHORT TERM

$$\begin{aligned} M_{\max} &= 5.5 \text{ kNm} \\ V_{\max} &= 11 \text{ kN} \\ f_{m,d} &= 0.9 \times 1 \times 16 / 1.3 = 11.1 \text{ MPa} \\ f_{v,d} &= 0.9 \times 1 \times 1.8 / 1.3 = 1.25 \text{ MPa} \end{aligned}$$

3

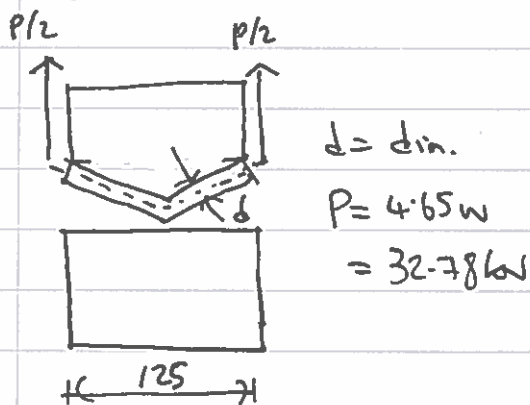
BENDING: $d \geq \sqrt{6M/\sigma_b} = \sqrt{\frac{6 \times 5.5 \times 10^6}{11.1 \times 125}} = 154 \text{ mm}$

SHEAR: $d \geq V/\tau_b = 11 \times 10^3 / (1.25 \times 125) = 70.4 \text{ mm}$

\(\therefore\) SHORT TERM BENDING GOVERNS $d \geq 154 \text{ mm}$

4(c) TWO POSSIBLE FAILURE MODES:

① PLASTIC HINGE IN BOLT



$$M_{pl} = 0.8 \sigma_y d^3 / 6 \text{ Nm}$$

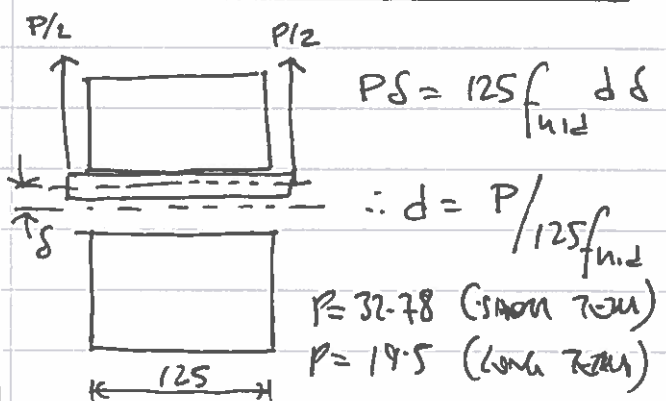
$$\frac{125P}{8} = \frac{0.8 \times 355 \times d^3}{6 \times 1.15}$$

$$\therefore d = \sqrt[3]{\frac{32.78 \times 10^3 \times 125}{8} \times \frac{6 \times 1.15}{0.8 \times 355}}$$

$$= \underline{23.1 \text{ mm}}$$

3

② BERMUNG FAILURE IN TIMBER



$$PS = 125 f_{h1d} d \delta$$

$$\therefore d = P / 125 f_{h1d}$$

$$P = 32.78 \text{ (SHORT TERM)}$$

$$P = 19.5 \text{ (LONG TERM)}$$

$$f_{h1d} = k_{mod} f_{h0k} / (k_{90} \sin^2 \alpha + \cos^2 \alpha) \text{ N/mm}^2$$

$$f_{h0k} = 0.082 (1 - 0.01 \times 25) 310$$

$$= 19 \text{ MPa}$$

$$k_{90} = 0.40 + 0.015 d = 1.275$$

$$\therefore f_{h1d} = k_{mod} 19 / (1.275 \times 0.117 + 0.883) 1.3$$

$$= 14.16 k_{mod} \text{ MPa}$$

$$\therefore \text{SHORT TERM } d = 32.78 \times 10^3 / 125 \times 14.16 \times 0.9$$

$$= \underline{20.6 \text{ mm}}$$

$$\text{LONG TERM } d = 19.5 \times 10^3 / 125 \times 14.16 \times 0.7$$

$$= \underline{15.7 \text{ mm}}$$

3

\therefore PLASTIC HINGE IN BOLT GOVERNS. ($d = 23.1 \text{ mm}$)

$$\text{say } \underline{\underline{d = 25 \text{ mm}}}$$