

4C9 2019

Q1 (a)(i) • Correspondence principle : in the Laplace domain, the viscoelastic solution corresponds to the elastic solution, with the substitution $E \rightarrow S \bar{E}(s)$, $\nu \rightarrow S \bar{\nu}(s)$ etc (for any time-dependent moduli) [see Data Sheet]

• Applicable? Applies if there is no time-dependence in the boundary conditions (i.e. boundary between traction, displacement B.C.s). Loading patch radius here is fixed, so it applies.

(ii) For this material, ν is time-independent, so the only time dependence is $\sigma_{11}(t)$, $\sigma_{22}(t)$ and $\sigma_{33}(t)$ is via $p(t)$. So, $p \rightarrow p(t)$ gives required time dependence.

(b) If linear elastic:

$$\epsilon_{33} = \frac{1}{E} [\sigma_{33} - \nu (\sigma_{11} + \sigma_{22})]$$

Take Laplace transforms and apply correspondence principle: $E \rightarrow S \bar{E}(s)$, $\nu \rightarrow \nu$ (time independent)

$$\bar{\epsilon}_{33}(s) = \frac{1}{S \bar{E}(s)} [\bar{\sigma}_{33}(s) - \nu (\bar{\sigma}_{11}(s) + \bar{\sigma}_{22}(s))]$$

Data sheet: $\bar{E}_r(s) = \frac{1}{s^2 \bar{J}_c(s)}$

$$\therefore \bar{\epsilon}_{33}(s) = \bar{J}_c(s) [S \bar{\sigma}_{33}(s) - \nu S (\bar{\sigma}_{11}(s) + \bar{\sigma}_{22}(s))]$$

Inverse transform:

$$\epsilon_{33}(t) = \int_0^t J_c(t-\tau) \frac{d}{d\tau} [\sigma_{33}(\tau) - \nu (\sigma_{11}(\tau) + \sigma_{22}(\tau))] d\tau$$

(3)

(c) (i) Substitute for $\sigma_{11}(t)$, $\sigma_{22}(t)$ and $\sigma_{33}(t)$ into the constitutive equation:

$$\epsilon_{33}(t) = - \frac{1}{\pi a^2} \underbrace{\left(1 - 2\nu \left(\frac{1+2\nu}{2}\right)\right)}_{(1-\nu-2\nu^2)} \int_0^t J_c(t-\tau) \frac{\partial p(\tau)}{\partial \tau} d\tau$$

all constants \nearrow

Creep compliance, $J_c(t)$: use Laplace transforms

$$\epsilon_r(t) = \epsilon e^{-\frac{Et}{\eta}} \Rightarrow \bar{\epsilon}_r(s) = \frac{\epsilon}{s + \frac{E}{\eta}}$$

Data Sheet \rightarrow

$$\therefore \bar{J}_c(s) = \frac{1}{s^2 \bar{\epsilon}_r(s)} = \frac{s + \frac{E}{\eta}}{s^2 \epsilon} = \frac{1}{s^2 \eta} + \frac{1}{s \epsilon}$$

Inverse transform: $\boxed{J_c(t) = \frac{t}{\eta} + \frac{1}{\epsilon}}$

Substitute into constitutive equation:

$$\epsilon_{33}(t) = - \frac{(1-\nu-2\nu^2)}{\pi a^2} \int_0^t \left(\frac{t-\tau}{\eta} + \frac{1}{\epsilon}\right) \frac{\partial p(\tau)}{\partial \tau} d\tau$$

Loading, for $0 \leq t < t_0$:

• Step response: $p(t) = P_0 H(t)$

$$\therefore \boxed{\epsilon_{33}(t) = - \frac{(1-\nu-2\nu^2)}{\pi a^2} P_0 \left(\frac{t}{\eta} + \frac{1}{\epsilon}\right)}$$

Loading for $t > t_0$:

• Superposition of two step responses: $p(t) = P_0 [H(t) - H(t-t_1)]$

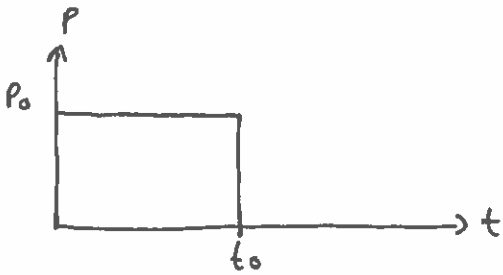
Solution is therefore:

$$\epsilon_{33}(t) = - \frac{(1-\nu-2\nu^2)}{\pi a^2} P_0 \left[\left(\frac{t}{r} + \frac{1}{E} \right) - \left(\frac{t-t_1}{r} + \frac{1}{E} \right) \right]$$

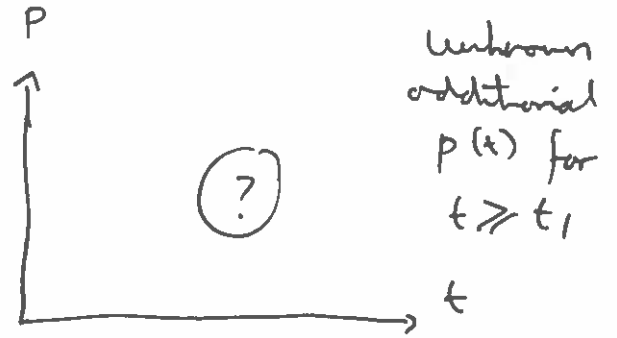
$$= - \frac{(1-\nu-2\nu^2)}{\pi a^2} P_0 \left(\frac{t_1}{r} \right)$$

(i.e. constant residual strain: $-\frac{r P_0}{E r}$ in depth)

(ii) Sketching the response:

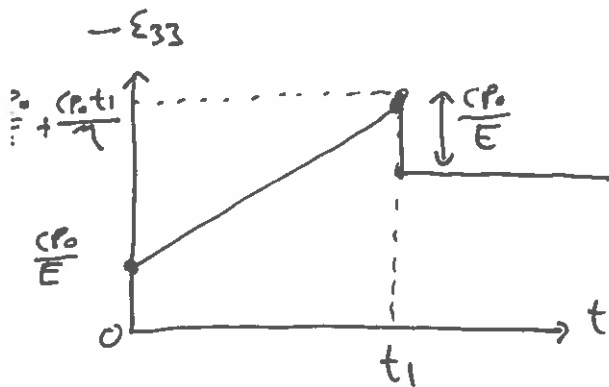


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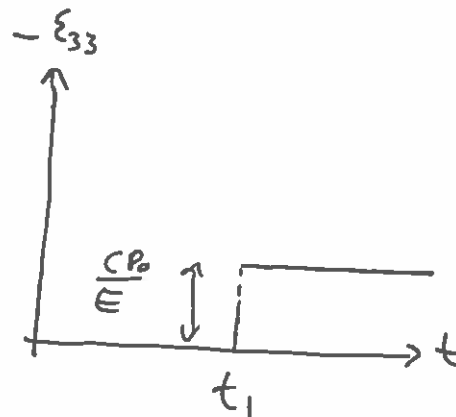


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required to hold strain at $\epsilon_{33}(t_1)$ for $t \geq t_1$

$$C = \frac{1}{\pi a^2} (1-\nu-2\nu^2)$$

what $p(t)$ will give $\epsilon_{33}(t) = - \frac{C P_0}{E} H(t-t_1)$?

(5)

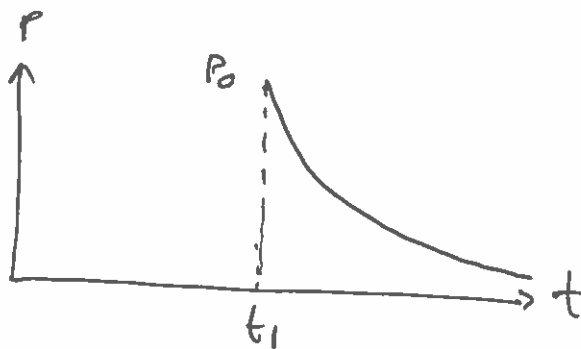
Conit to find using Laplace transform of constitutive relationship :

$$-\frac{CP_0}{E} H(t-t_1) = -C \int_0^t J_c(t-\tau) \frac{\partial P(\tau)}{\partial \tau} d\tau$$

$$\begin{aligned} \therefore \frac{P_0}{E} \frac{1}{s} e^{-st_1} &= \bar{J}_c(s) s \bar{P}(s) \\ &= \left(\frac{1}{s^2 \eta} + \frac{1}{sE} \right) s \bar{P}(s) \\ &= \frac{s^2 \eta + sE}{s^3 \eta E} s \bar{P}(s) \end{aligned}$$

$$\therefore \bar{P}(s) = \frac{P_0}{E} \frac{\eta E}{s^2 \eta + E} e^{-st_1} = P_0 \frac{1}{s + \frac{E}{\eta}}$$

Inverse transform :
$$P(t) = P_0 e^{-\frac{E(t-t_1)}{\eta}} H(t-t_1)$$



Q2

$$(a)(i) \nabla \cdot (\phi \underline{u}) = \frac{\partial}{\partial x_i} (\phi u_i) = \frac{\partial \phi}{\partial x_i} u_i + \phi \frac{\partial u_i}{\partial x_i} \\ = (\nabla \phi) \cdot \underline{u} + \phi \nabla \cdot \underline{u}$$

$$(ii) \nabla \cdot \nabla \times \underline{u} = \frac{\partial}{\partial x_i} [\nabla \times \underline{u}]_i = \frac{\partial}{\partial x_i} \left[\underset{\substack{\uparrow \\ \text{constant}}}{\epsilon_{ijk}} \frac{\partial}{\partial x_j} u_k \right] \\ = \epsilon_{ijk} \frac{\partial^2 u_k}{\partial x_i \partial x_j}$$

Given that: $\frac{\partial^2 u_k}{\partial x_i \partial x_j} = \frac{\partial^2 u_k}{\partial x_j \partial x_i}$ for any i, j

And: $\epsilon_{ijk} = -\epsilon_{ikj}$ for any i, j

$$\Rightarrow \epsilon_{ijk} \frac{\partial^2 u_k}{\partial x_i \partial x_j} = 0$$

$$(iii) [\nabla \times (\nabla \times \underline{u})]_i = \epsilon_{ijk} \frac{\partial}{\partial x_j} [\nabla \times \underline{u}]_k \\ = \epsilon_{ijk} \frac{\partial}{\partial x_j} [\epsilon_{kpq} \frac{\partial}{\partial x_p} u_q] \\ = \epsilon_{kij} \epsilon_{kpq} \frac{\partial^2 u_q}{\partial x_j \partial x_p}$$

data sheet \rightarrow $= (\delta_{ip} \delta_{jq} - \delta_{iq} \delta_{jp}) \frac{\partial^2 u_q}{\partial x_j \partial x_p}$

$$= \frac{\partial^2 u_j}{\partial x_j \partial x_i} - \frac{\partial^2 u_i}{\partial x_j \partial x_j}$$

$$= [\nabla (\nabla \cdot \underline{u}) - (\nabla \cdot \nabla) \underline{u}]_i$$

(b)(i) Minimum PE (Data Sheet):

$$\int \delta U dV - \int_S t_i^2 \delta u_i dS - \int b_i \delta u_i dV = 0$$

neglect

Strain energy density:

• non-zero strain components: ϵ_{11} only

$$\therefore U = \frac{E}{2(1+\nu)} \left[\epsilon_{11}^2 + \frac{\nu}{1-\nu} \epsilon_{11}^2 \right] = \frac{E}{2(1+\nu)(1-\nu)} \epsilon_{11}^2$$

• displacements required, so write $\epsilon_{11} = u_{1,1}$

• note $E = E(x_1)$

$$\therefore U = \frac{E(x_1)}{2(1+\nu)(1-\nu)} (u_{1,1})^2$$

• variations in U :

$$\delta U = \frac{\partial U}{\partial u_{1,1}} \delta u_{1,1} = \frac{E(x_1)}{(1+\nu)(1-\nu)} u_{1,1} \delta u_{1,1}$$

• per unit area of elastic layer:

$$\int \delta U dV = \int_0^H \frac{E(x_1)}{(1+\nu)(1-\nu)} u_{1,1} \delta u_{1,1} dx_1$$

• integrate by parts:

$$\int \delta U dV = \left[\frac{E(x_1)}{(1+\nu)(1-\nu)} u_{1,1} \delta u_{1,1} \right]_0^H - \int_0^H \frac{1}{(1+\nu)(1-\nu)} \frac{d}{dx_1} \left[E(x_1) u_{1,1} \right] \delta u_{1,1} dx_1$$

~~$\int_0^H \frac{1}{(1+\nu)(1-\nu)} u_{1,1} \delta u_{1,1} dx_1$~~

$$\delta u_{1,1}(H) = \delta u = 0 \text{ (u fixed)}$$

$$\delta u_{1,1}(0) = 0$$

$$= \left(\frac{dE}{dx_1} u_{1,1} + E u_{1,1,1} \right)$$

External tractions and displacements:

- Let $\underline{F} = F \underline{e}_1$ be the force per unit area acting on the top block, to cause displacement $w \underline{e}_1$.
- Per unit area:

$$\int_S t_i^e \delta u_i dS = F \delta w = 0 \quad (w \text{ fixed})$$

Minimum PE

If $\delta \Pi = 0$:

~~$$\left[\frac{E(H)}{(1+\nu)(1-\nu)} u_{1,1}(H) - F \right] (\delta u) = 0$$~~

~~$$- \frac{1}{(1+\nu)(1-\nu)} \int_0^H \left(\frac{dE}{dx_1} u_{1,1} + E u_{1,11} \right) \delta u_1 dx_1 = 0$$~~

∴ Governing equations:

$$\boxed{\frac{dE}{dx_1} u_{1,1} + E u_{1,11} = 0} \quad \text{①} \quad 0 \leq x_1 \leq H$$

Boundary conditions:

~~$$\boxed{F = E(H) u_{1,1}(H)} \quad \text{②} \quad x_1 = H$$

$$u_1(H) = w \quad \text{③}$$

$$u_1(0) = 0 \quad \text{④} \quad \text{given}$$~~

$$\left. \begin{aligned} u_1(H) &= w \\ u_1(0) &= 0 \end{aligned} \right\} \text{given}$$

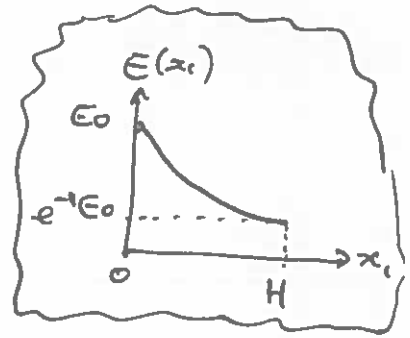
From constitutive equations, force per unit area on top plate, to displace w :

$$\sigma_{11} = F = \frac{\partial U}{\partial \epsilon_{11}} = \frac{E}{(1+\nu)(1-\nu)} \epsilon_{11} = \frac{E}{(1+\nu)(1-\nu)} u_{1,1}(H) \quad \text{②}$$

(ii) For $E(x_1) = E_0 e^{-\frac{x_1}{H}}$ } Sub. into ①

$$\frac{dE}{dx_1} = -\frac{E_0}{H} e^{-\frac{x_1}{H}}$$

④



$$\therefore -\frac{1}{H} u_{1,1} + u_{1,11} = 0$$

Try: $u_1 = a + b e^{-cx_1}$

$$\Rightarrow \frac{c}{H} e^{-cx_1} + c^2 e^{-cx_1} = 0 \quad \therefore c = -\frac{1}{H}$$

To satisfy B.C ③: $a + b e^{-1} = w$

B.C. ④ $a + b = 0$

$$\therefore a = \left(\frac{w}{1 - e^{-1}} \right) = -b$$

$$\therefore u_1 = \left(\frac{w}{1 - e^{-1}} \right) \left(1 - e^{-\frac{x_1}{H}} \right)$$

Using ②: $F = \left(\frac{E_0 e^{-1}}{(1+\nu)(1-\nu)} \right) \left(\frac{w}{1 - e^{-1}} \right) \frac{1}{H} e^{-1}$

$$\therefore F = \frac{E_0 w}{(1+\nu)(1-\nu) H} \left(\frac{e^{-2}}{1 - e^{-1}} \right) \approx 0.21 \left(\frac{w}{H} \right) \frac{E_0}{(1+\nu)(1-\nu)}$$

3.

1.

$$a) \quad e_{ijk} \nabla_{jk} = \begin{bmatrix} e_{ijk} \nabla_{jk} \\ e_{2jk} \nabla_{jk} \\ e_{3jk} \nabla_{jk} \end{bmatrix} = \begin{bmatrix} \nabla_{23} - \nabla_{32} \\ \nabla_{31} - \nabla_{13} \\ \nabla_{12} - \nabla_{21} \end{bmatrix}$$

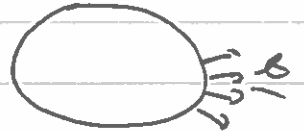
$$= \underline{0} \quad \text{if} \quad \nabla_{ij} = \nabla_{ji}$$

b) Linear balance

$$\int_{\partial \Omega} \underline{\nabla} n \, ds + \int_{\Omega} \underline{f} \, d\Omega = 0$$

$$\int_{\partial \Omega} \nabla \cdot \underline{\nabla} \, d\Omega + \int_{\Omega} \underline{f} \, d\Omega = 0$$

$$\Rightarrow -\nabla \cdot \underline{\nabla} = \underline{f} \quad \text{by localisation argument.}$$



$$\underline{n} = \underline{\nabla} n$$

Angular balance (moment)

$$\int_{\partial \Omega} \underline{x} \times \underline{\nabla} n \, ds + \int_{\Omega} \underline{x} \times \underline{f} \, d\Omega = 0$$

$$(e_{ijk} x_j \nabla_{ke}) n_e$$

$$\int_{\partial \Omega} \frac{\partial}{\partial n_e} (e_{ijk} x_j \nabla_{ke}) \, d\Omega + \int_{\Omega} \underline{x} \times \underline{f} \, d\Omega = 0$$

$$\int_{\partial \Omega} e_{ijk} x_j \frac{\partial \nabla_{ke}}{\partial n_e} + e_{ijk} \delta_{je} \nabla_{ice} \, d\Omega + \int_{\Omega} \underline{x} \times \underline{f} \, d\Omega = 0$$

$$\int_{\partial \Omega} \underline{x} \times (\nabla \cdot \underline{\nabla} + \underline{f}) \, d\Omega + \int_{\partial \Omega} e_{ijk} \nabla_{kj} \, d\Omega$$

$$= 0 \quad \text{by linear balance}$$

$$\underline{\underline{\underline{\nabla}}} \cdot \underline{\underline{\underline{\nabla}}}^T = 0 \quad \text{if} \quad \underline{\underline{\underline{\nabla}}} = \underline{\underline{\underline{\nabla}}}^T$$

$$c) \underline{\underline{k}} = \frac{\partial \underline{\underline{U}}}{\partial \underline{\underline{x}}} = \frac{\partial \underline{\underline{U}}}{\partial X} \cdot \frac{\partial X}{\partial \underline{\underline{x}}} = \underline{\underline{\dot{F}}} \underline{\underline{F}}^{-1}$$

$$d) i) \int_{\Omega} \underline{\underline{\sigma}} : \underline{\underline{d}} \, d\Omega = \int_{\Omega_0} J \underline{\underline{\sigma}} : \underline{\underline{d}} \, dX$$

$$\therefore \underline{\underline{z}} = J \underline{\underline{\sigma}} \quad \text{where} \quad J = \det \underline{\underline{F}}$$

$$\begin{aligned} ii) \int_{\Omega} \underline{\underline{\sigma}} : \underline{\underline{d}} \, d\Omega &= \int_{\Omega_0} J \underline{\underline{\sigma}} : (\dot{\underline{\underline{F}}} \underline{\underline{F}}^{-1} + \underline{\underline{F}}^{-T} \dot{\underline{\underline{F}}}^T) \, dX \\ &= \int_{\Omega_0} J \underline{\underline{\sigma}} : \dot{\underline{\underline{F}}} \underline{\underline{F}}^{-1} \, dX \quad (\text{by symmetry of } \underline{\underline{\sigma}}) \\ &= \int_{\Omega_0} J \underline{\underline{\sigma}} \underline{\underline{F}}^{-T} : \dot{\underline{\underline{F}}} \, dX \\ &= \int_{\Omega_0} J \underline{\underline{\sigma}} \underline{\underline{F}}^{-T} : \underline{\underline{F}} \underline{\underline{F}}^{-T} \dot{\underline{\underline{F}}} \, dX \\ &= \int_{\Omega_0} \underbrace{J \underline{\underline{F}}^{-T} \underline{\underline{\sigma}} \underline{\underline{F}}^{-T}}_{\underline{\underline{S}}} : \underline{\underline{F}}^T \dot{\underline{\underline{F}}} \, dX \\ &= \underline{\underline{S}} \end{aligned}$$

~~$$\text{since } \dot{\underline{\underline{F}}} = \underline{\underline{F}}^T \dot{\underline{\underline{F}}} + \dot{\underline{\underline{F}}}^T \underline{\underline{F}}$$~~

since $\underline{\underline{S}} = \underline{\underline{F}}^T \dot{\underline{\underline{F}}} + \dot{\underline{\underline{F}}}^T \underline{\underline{F}}$ is symmetric.

$$d) \dot{\tilde{V}} = \underline{Q} \underline{V} \underline{Q}^T$$

$$\dot{\tilde{V}} = \dot{\underline{Q}} \underline{V} \underline{Q}^T + \underline{Q} \dot{\underline{V}} \underline{Q}^T + \underline{Q} \underline{V} \dot{\underline{Q}}^T \neq \underline{Q} \dot{\underline{V}} \underline{Q}^T$$

$\therefore \dot{\tilde{V}}$ is not objective.

$$e) \tilde{\underline{l}} = \dot{\tilde{F}} \tilde{F}^{-1}, \quad \underline{l} = \underline{F} \underline{F}^{-1}, \quad \tilde{F} = \underline{Q} \underline{F}$$

~~\tilde{F}~~

$$\dot{\tilde{F}} = \dot{\underline{Q}} \underline{F} + \underline{Q} \dot{\underline{F}}$$

$$\tilde{\underline{l}} = (\dot{\underline{Q}} \underline{F} + \underline{Q} \dot{\underline{F}}) (\underline{Q} \underline{F})^{-1}$$

$$= (\dot{\underline{Q}} \underline{F} + \underline{Q} \dot{\underline{F}}) (\underline{F}^{-1} \underline{Q}^{-1})$$

$$= \dot{\underline{Q}} \underline{F} \underline{F}^{-1} \underline{Q}^{-1} + \underline{Q} \dot{\underline{F}} \underline{F}^{-1} \underline{Q}^{-1}$$

$$= \dot{\underline{Q}} \underline{Q}^{-1} + \underline{Q} \underline{l} \underline{Q}^{-1} \neq \underline{Q} \underline{l} \underline{Q}^{-1}$$

\therefore not objective.

$$\tilde{\underline{d}} = (\tilde{\underline{l}} + \tilde{\underline{l}}^T) / 2 = (\dot{\underline{Q}} \underline{Q}^{-1} + \underline{Q} \dot{\underline{Q}}^{-1}) / 2 + \underline{Q} (\underline{l} + \underline{l}^T) \underline{Q}^{-1} / 2$$

$$= (\dot{\underline{Q}} \underline{Q}^{-1} + \underline{Q} \dot{\underline{Q}}^{-1}) / 2 + \underline{Q} \underline{d} \underline{Q}^{-1}$$

Consider $\underline{Q} \underline{Q}^{-1} = \underline{I}$

$$\frac{d}{dt} (\underline{Q} \underline{Q}^{-1}) = \dot{\underline{Q}} \underline{Q}^{-1} + \underline{Q} \dot{\underline{Q}}^{-1} = 0$$

$$\therefore \tilde{\underline{d}} = \underline{Q} \underline{d} \underline{Q}^{-1} \Rightarrow \text{objective.}$$

MODULE 4C9: Examiner's comments

Question 1 (viscoelasticity)

Parts (a) and (b) were generally well-answered. Few candidates answered (c)(i) correctly, with problems with the inverse Laplace transform and failure to spot the superposition of two step responses common. Fewer answered (c)(ii) correctly and many did not attempt this part. A surprising number of answers for (c) did not involve time.

Question 2 (variational methods)

The tensor manipulations in (a) were generally performed without difficulty. There was general discomfort with process behind variational methods, with few able to derive the governing differential equation. A number of solutions failed to account for the contribution of the spatially varying Young's modulus.

Question 3 (nonlinear continuum mechanics)

This question was very well done by almost all who attempted it. The number and quality of the attempts was pleasing given that this is the first year that the topic has been included in the course.