Crib 4D6 Part IIB 4D6 ; Dynamics in Civil Engineering a)  $\overline{u} = 1 - \cos(\pi x) \implies \overline{u}(o) = 0$ So assumed mode captures the expected shape and boundary [2] conditions, b). Neglecting the vertical motion of the lack, the total potential energy is due to strain in the legs.  $\Rightarrow K_{eq} = 3x \int_{0}^{\infty} E_{\pm} \left(\frac{d^{2}\bar{u}}{dn^{2}}\right)^{2} dn$  $= 3EI \int_{0}^{L} \left( \frac{\pi^{2}}{R^{2}} \cos\left(\frac{\pi \chi}{L}\right) \right)^{2} dx = 3EI \frac{\pi^{+}}{L^{+}} \int_{0}^{L} \frac{1}{2} \left( 1 + \cos\frac{2\pi \chi}{L} \right) dx$  $=\frac{3ET}{2}\frac{\pi^{4}}{l^{4}}\left[\chi+\frac{l}{2\pi}m\frac{2\pi\chi}{l}\right]_{0}^{R}=\frac{3ET}{2}\frac{\pi^{4}}{l^{4}}.l.$ H  $=\frac{3\pi^4ET}{7.03}$ The kinetic energy comprises that of the submerged part of the legs, that of the legs above water and that of the leck  $= \mathcal{M}_{eq} = 3x \int_{0}^{\infty} \left(1 - \cos\left(\frac{\pi - x}{x}\right)\right)^{2} dx + 3x \int_{0}^{x} \left(1 - \cos\left(\frac{\pi - x}{x}\right)\right)^{2} dx$  $+ M(z)^2 = \overline{u}(z)$ For del: [As assured in question] = 3m [ 1-2cos Trx + cos Try dr + 4M = 3m 1-2cos #2 + 2 (1+ cos 2m2) dn + 4M

406,2017.

2

 $M_{ag} = 3m \left[ \frac{3n - 2\ell}{2} m \frac{\pi m}{2} + \frac{1}{2} \frac{\ell}{2\pi} m \frac{2\pi m}{2} \right]^{\ell} + 4M = 3m \frac{3\ell}{2} + 4M$ =  $\frac{9m\ell}{2} + 4M$ () continued.  $i_{1} \mathcal{O}_{n}^{2} = k_{eq} / M_{eq} \Rightarrow \mathcal{O}_{n} = \sqrt{\frac{3\pi^{4}ET}{2k^{3}}} \cdot \frac{2}{9ml+8M}$  $\Rightarrow \omega_n = \int \frac{3\pi^4 EI}{l^3 (9ml + 8M)}$ [2] C). WA = SXTT \* 1.8×10" - ROS(9×25×10°×80+8×9,6×106) = 1.37 ml/s = 0.218 Hz > TA= 4:59 5 11 th 2 0.2 => DAF 2 0.65 **B**]  $K_{q} = \frac{3 \times \pi 4 \times 1.8 \times 10^{11}}{2 \times 80^3} = 51.4 \times 10^6 \, \text{N/m}$  $F_{ay} = F_{u}(a=L) = 70 \times 2 = 140 MN$ :, Kmac 2 0.65x 140×10<sup>6</sup> × Ū(21=1) = 3.59M 8] d) The lower sea level reduces exposes the upper portion of the legs, climinating the entrained water mais and hence reducing the equivalent mass. This increases the natural frequency, decreasing the natural period and hence increasing the DAF. However, the wave loading is rolonger at deck level (the antirode of the sway node) and therefore the equivalent force is which to be lower, which may cancel out the effect of the increased DAF Danping way also be reduced (visions and radiation damping) but this is less significant in transient response.

(2 a) i) 
$$k_1 = k_2 = 2\left(\frac{12EI}{L^3}\right) = 24.7 \times 10^6 \text{ Mm}$$

$$k_3 = \frac{3 E I_{\text{fower}}}{\left[ \left( \frac{2}{3} \right) L \right]^3} = 2.78 \times 10^6 N_{\text{m}}$$

$$M_{eq} = (1)^{2} (10,000) + (2)^{2} (10,000 + \frac{1}{3} (6000)) + (5)^{2} (\frac{2}{3} (6000)) = 158,000 \text{ kg}$$

$$K_{eq} = k_{1} (1)^{2} + k_{2} (2-1)^{2} + k_{3} (5-2)^{2} = 78.3 \times 10^{6} \text{ N/m}$$

$$w_{n} = \sqrt{\frac{k_{eq}}{m_{eq}}} = \sqrt{\frac{78.3 \times 10^{6}}{158 \times 10^{3}}} = 22.3 \text{ mal/s} = \frac{3.55 \text{ Hz}}{158 \times 10^{3}}$$

(ii) Try 
$$[1, 2, 6]$$
  
20 Meq = 202,000 ; Keq = 9.78 × 10<sup>7</sup> -  $P w_n = 22.0$  md/s  
 $\frac{f_n = 3.50}{5} \frac{f_n = 3.50}{5} \frac{f_n$ 

$$\begin{split} \dot{u}_{max} &= \prod S_{a} \rightarrow S_{a} = \frac{\tilde{u}_{max}}{\prod} \\ Bottom story! F = Ema = \left[ (m_{1}, m_{2}, m_{3}) \right] \left[ \frac{1}{2} \right] \tilde{u}_{max} \rightarrow \tilde{u}_{max} = \\ \tilde{\sigma} \circ S_{a} = \left[ \frac{(m_{1}(1) + m_{1}(2) + m_{3}(5))}{F \prod} \right] \tilde{u}_{max} \rightarrow \tilde{u}_{max} = \\ \tilde{\sigma} \circ S_{a} = \left[ \frac{(m_{1}(1) + m_{1}(2) + m_{3}(5))}{F \prod} \right] \tilde{u}_{max} \rightarrow \tilde{u}_{max} = 0.287 \\ \tilde{\sigma} = \left[ \frac{Emm}{2} \right] \tilde{u}_{max} = 0.287 \\ \tilde{\sigma} = \left[ \frac{Emm}{2} \right] \tilde{u}_{max} = 0.287 \\ \tilde{\sigma} = \left[ \frac{Emm}{2} \right] \tilde{u}_{max} = 0.287 \\ \tilde{\sigma} = \left[ \frac{Emm}{2} \right] \tilde{u}_{max} = 0.287 \\ \tilde{\sigma} = \left[ \frac{Emm}{2} \right] \tilde{u}_{max} = 0.287 \\ \tilde{\sigma} = \left[ \frac{Emm}{2} \right] \tilde{u}_{max} = 0.287 \\ \tilde{\sigma} = \left[ \frac{Emm}{2} \right] \tilde{u}_{max} = 0.287 \\ \tilde{\sigma} = \left[ \frac{Emm}{2} \right] \tilde{u}_{max} = 0.287 \\ \tilde{\sigma} = \left[ \frac{Emm}{2} \right] \tilde{u}_{max} = 0.287 \\ \tilde{\sigma} = \left[ \frac{Emm}{2} \right] \tilde{u}_{max} = 0.287 \\ \tilde{\sigma} = \left[ \frac{Emm}{2} \right] \tilde{u}_{max} = 0.287 \\ \tilde{\sigma} = \left[ \frac{Emm}{2} \right] \tilde{u}_{max} = 0.287 \\ \tilde{\sigma} = \left[ \frac{Emm}{2} \right] \tilde{u}_{max} = 0.287 \\ \tilde{\sigma} = \left[ \frac{Emm}{2} \right] \tilde{u}_{max} = 0.287 \\ \tilde{\sigma} = \left[ \frac{Emm}{2} \right] \tilde{u}_{max} = 0.287 \\ \tilde{\sigma} = \left[ \frac{Emm}{2} \right] \tilde{u}_{max} = 0.287 \\ \tilde{\sigma} = \left[ \frac{Emm}{2} \right] \tilde{u}_{max} = 0.287 \\ \tilde{\sigma} = \left[ \frac{Emm}{2} \right] \tilde{u}_{max} = 0.287 \\ \tilde{\sigma} = \left[ \frac{Emm}{2} \right] \tilde{u}_{max} = 0.287 \\ \tilde{\sigma} = \left[ \frac{Emm}{2} \right] \tilde{u}_{max} = 0.287 \\ \tilde{\sigma} = \left[ \frac{Emm}{2} \right] \tilde{u}_{max} = 0.287 \\ \tilde{\sigma} = \left[ \frac{Emm}{2} \right] \tilde{u}_{max} = 0.287 \\ \tilde{\sigma} = \left[ \frac{Emm}{2} \right] \tilde{u}_{max} = 0.287 \\ \tilde{\sigma} = \left[ \frac{Emm}{2} \right] \tilde{u}_{max} = 0.287 \\ \tilde{\sigma} = \left[ \frac{Emm}{2} \right] \tilde{u}_{max} = 0.287 \\ \tilde{\sigma} = \left[ \frac{Emm}{2} \right] \tilde{u}_{max} = 0.287 \\ \tilde{\sigma} = \left[ \frac{Emm}{2} \right] \tilde{u}_{max} = 0.287 \\ \tilde{\sigma} = \left[ \frac{Emm}{2} \right] \tilde{u}_{max} = 0.287 \\ \tilde{u}_{max} = 0.28$$

·: ·.

(3 b) ii) Equal energy rule for plateau partion of spectra.



# 406

(3) a) It possible to solve a dynamic problem by obtaining the response of the system at individual frequency components and the input motion and then superposing these to obtain the overall response. buch an analytis is called frequency drucin any this. It has the advantage of requiring less computational effort and can be carried out quickly. However it has the disadvantage of nort being appliable for non-linear systems due to non-orthogonal rucks. Time domain analysis on the other hand attempts to solve direct by the equations of motion. This is normally dance by using an integration I cheme much as Newmark - & meltiod. a However the computational effort can be quite large but non-linear tystems [15 12] an be analysed using this method. b) It is well known that soil behavior is highly non-linear especially at large strains. Earthquake boading normaly involves this non-liner behaviour due to the large cyclic strains it induces in the soil. Hence to include soil - behaviour while analyning soil-structure systems, time domain analysis is required . Further it is also poinible to Carry ant analysis of coupled par Solid and flind phases to capture the behaviour of both solud and finid pleases present in Saturated sits [10%] c) Calculate the small. Strain Schenn modules amage Vertical stress belues the tunnel = 5v = 5v' Lin Wight (8x4-7.5x3.5) x24x1 = 138 kn/m. 1-8m->  $5.5v = 6v = \frac{138}{8x1} = 17.25 kpa$ Porson's late for End = 0.3  $K_0 = \frac{V}{1-V} = \frac{0.3}{0.7} = 0.428$  $p' = \sigma_v' \left(\frac{1+2k_0}{3}\right) = 17.25 \times \left(\frac{1+2k_0}{3}\right) = 0.619 \times 1725$ = 10.6786 kpa  $i. C_{max} = 100 \left[ \frac{3 - e^2}{1 + e^2} \right]^2 \sqrt{p^2} = 100 \left( \frac{3 - 0.67}{1.67} \right)^2 \left( \frac{10 \cdot 67}{1.000} \right)^{0.5^2}$ = 33.59 NPa

3 c) contri . Now consider the demensions of the trinnel Euse Wolf's formulae

$$2 \{ = 8 \text{ m} \quad 2b = 1 \text{ m} \quad e = 4 \text{ m} :$$

$$\therefore \ U_{1b} = 8 \qquad e_{1b} = \frac{L_{1}}{e_{1}s} = 8$$
From data Stutic: - Horizand Styler :  

$$K_{1bx} = G_{1b} \frac{b}{1-v} \left( 6.8 \left( \frac{U_{1b}}{V_{1b}} \right)^{0.65} + 2.44 \right) \left[ 1 + \left( 0.33 + \frac{1.34}{4!4} \right) \right] \left( \frac{e}{b} \right)^{0.8} \right]$$

$$= G_{1} \frac{0.5}{(1-0.3)} \left( 6.8 \times 8^{0.65} + 2.44 \right) \left[ 1 + \left( 0.33 + \frac{1.34}{9} \right) 8^{0.8} \right]$$

$$= G_{1} \times 72.2469$$

$$\therefore K_{1bx} = 2426.776 \text{ MN/m}$$
Usubal Stiffners:  $K_{1b} = \frac{G_{1b}}{2-v} \left( 3.1 \left( \frac{U_{1b}}{9} \right)^{0.717} \right) \left[ 1 + \left( 0.25 + 0.27 \frac{b}{2} \right) \left( \frac{e}{b} \right)^{0.8} \right]$ 

$$= G_{1} \times 0.27 \times (3.1 \times 8^{0.75} + 1.6) \left[ 1 + \left( 0.25 + 0.27 \frac{b}{2} \right) \left( \frac{e}{b} \right)^{0.8} \right]$$

$$= G_{1} \times 0.29441 \times 16.3462 \times 2.4844$$

$$= G_{1} \times 11.94357$$

$$\therefore K_{1b} = \frac{401 \cdot 1838}{1-v} \left[ 3.73 \left( \frac{b}{2} \right)^{2.4} + 0.27 \right] \left( 1 + \frac{e}{b} + \frac{1.4}{0.357} + \frac{1.4}{0.357}$$

3 d) Mars ythe tunnel par mindth = 138 km = 14067.200 mg . For Horizontal mode, Soil participaty will 30×14067 = 422018.3486 kg Horizontal stypnen = Kmx = 2426.776 MN/m . Horizontal Stypnen = Kmx = 2426.776 MN/m .  $h = \frac{1}{2\pi}\sqrt{\frac{Kmx}{m}} = \frac{1}{2\pi}\sqrt{\frac{24.26.776 \times 10^6}{422018.3486}} = 12.069 Hz$ 

<sup>1</sup>3 d) Cutat.  
Vertical Vibration mode: Suit participaty ≈ 10 × 14 067.27 ×  
= 140672.78 kg.  
KB = 4.01. 1838 MN/m.  
<sup>1</sup> 
$$\frac{1}{20} = \frac{1}{2\pi} \sqrt{\frac{KU}{MB}} = \frac{1}{2\pi} \sqrt{\frac{4041828 \times 10^6}{140672.78}} = 8.5 Hz$$
  
Rocking mode :- Mern Meerled weeken = 954,600 kg.mt.  
Kry = 29703.2681 min M/m.  
<sup>1</sup>  $\frac{1}{2\pi} \sqrt{\frac{KU}{3}} = \frac{1}{2\pi} \sqrt{\frac{29703.2681 \times 10^6}{1554.600}} = 27.64 Hz [00.6]
3 e) Shear wave velocity = 20 m/s. After inpodential sciple stram.
Set  $U_3 = \sqrt{\frac{2}{3}} = 3$   $G = 3$   $D_1^2$ .  
Sol cut weight = 15 km/m<sup>3</sup> = 3 Uig der kit f=1529.05 kg/m<sup>3</sup>.  
 $\therefore$  G = 0.6116 HPa  
 $\therefore$  Kry = 884.239 G = 540.7331 mam/m.  
 $\frac{1}{16} \frac{1}{8} = \frac{1}{2\pi} \sqrt{\frac{2020}{12}385} = \frac{1.6285}{12} Hz.$   
 $\frac{1}{16} \frac{1}{8} = \frac{1}{2\pi} \sqrt{\frac{2020}{12}(138.0)}} = \frac{1.6285}{12} Hz.$   
 $\frac{1}{16} \frac{1}{8} = \frac{1}{2\pi} \sqrt{\frac{540.506}{12}} = \frac{1.1468}{12} Hz.$   
 $\frac{1}{16} \frac{1}{8} = \frac{1}{2\pi} \sqrt{\frac{540.506}{12}} = \frac{3.73}{12} Hz.$   
 $\frac{1}{16} \sqrt{12} \frac{1}{27} \sqrt{\frac{540.506}{12}} = \frac{3.73}{12} Hz.$   
 $\frac{1}{16} \sqrt{12} \sqrt{10} \sqrt{10} \sqrt{10}$   $\frac{1}{381600} = \frac{3.73}{12} Hz.$   
 $\frac{1}{16} \sqrt{12} \sqrt{10} \sqrt{10} \sqrt{10} \sqrt{10}$   
 $\frac{1}{3} \frac{1}{6} C = 10.5 Hz.$   
 $\frac{1}{10} \sqrt{12} \sqrt{10} \sqrt{10} \sqrt{10} \sqrt{10} \sqrt{10} \sqrt{10} \sqrt{10}$   
 $\frac{1}{3} \sqrt{10} \sqrt{1$$ 

-3-

 $(\hat{\mathbf{A}})$ 

- Instability that occurs when the motion-induced Fluctuating lift force gets into phase with structural velocities, adding energy to system @ each cycle. (See notes for sketches & Further explanation)

b) SOOF Flutter (torsion) - no vertical movement:  
I is + C\_x is + K\_x is = C\_F is + C\_G is  
If C\_x - C\_G becomes negative, then motion is negatively  
damped, so torsional oscillations grow.  
c) Restaining force = k\_x is ; moment = 
$$\frac{1}{2}$$
 Cm  $V^2b^2L$   
Angle increases by dx:  $k_x dx = \frac{1}{2}$  Cm  $V^2b^2L$   
Angle increases by dx:  $k_x dx = \frac{1}{2}$  Cm  $V^2b^2L$   
Assume half the wave =  $\phi = \sin \frac{\pi x}{L}$ ,  $a = a_0 B$   
 $Text = \int_{0}^{L} I \phi^2 dx = I \frac{L}{2}$   
 $I = Mv^2 = 16.0 \times 10^3 (4.5)^2 = 1.44 \times 10^6 \text{ kg} \text{ m}^2/m$ 

$$k_{eq} = T_{eq} (w^{2}) = 8.64 \times 10^{8} (1.70^{2}) = 25.0 \times 10^{8} \text{ N·m}$$

$$M_{obe} = generalized \quad \text{brgue} = \int_{0}^{L} T \neq d_{X} = T \int_{0}^{L} \sin \frac{\pi x}{L} d_{X} = \frac{2}{T} LT$$

$$k_{eq} = \frac{2}{\pi} LT = \frac{2}{\pi} L \left( \frac{1}{L_{d}} \frac{U}{L_{d}} \sqrt{\frac{1}{L}} \sqrt{\frac{1}{L}} \frac{1}{L_{d}} \right)$$

$$V^{2} = \frac{k_{eq} \pi}{\frac{1}{L_{d}}} = \frac{(25.0 \times 10^{8}) \pi}{1.23 (30^{2}) 1200 (1.1)}$$

$$V = 73.3 \text{ M/s} = 164 \text{ mph}$$

## Q1

All candidates attempted Q1, and generally produced good solutions. The vast majority of candidates correctly assessed the assumed mode shape as satisfying the required boundary conditions. Rayleigh's method was correctly applied, in general, although marks were lost for failing to account correctly for the three legs and/or the mass of the deck. A significant number of candidates lost marks on the forced response by failing to calculate correctly the equivalent force (by scaling by the mode shape) and/or the maximum displacement (again, by mode scaling).

## Q2

31 candidates attempted Q2, and most did quite well. Most students used mass lumping methods in part (a), although the appropriateness of assumptions varied significantly. No student observed that a mass lumping approach would add mass to the second storey. In part a(i), about half of students quickly checked another mode shape to see if the natural frequency was lower; the other half attempted to solve for the "real" eigenmodes, which no one did successfully. In part (b), the majority of students used an appropriate method, with a minority forgetting the modal participation factor. About 2/3 of students correctly calculated the ductility, though many overcomplicated the solution.

#### Q3

35 candidates attempted Q3, and most did quite well. The initial parts on differences between frequency and time domain analyses were well recognised and most candidates identified soil nonlinearity as the main reason for requiring time domain analyses. The quantitative parts of the question required the students to calculate the horizontal, vertical and rotational stiffness of the tunnel. Almost all candidates had the right approach but a few made arithmetic errors which promulgated into calculation of natural frequencies. With the strong earthquake event and liquefaction, this time round many candidates could see floatation as a mode of failure apart from tunnel wall resonance.

#### Q4

This question was attempted by only 6 candidates. Most candidates performed very well on this question, explaining flutter in parts (a) and (b) in a clear fashion, and calculating the critical wind speed for static torsional divergence using an appropriate method. The errors that did occur were primarily related to calculating the mode-generalised torque.

JPT/MSPG/MJD/FAM