

IA 1999 Answers

Paper 1: Mechanical Engineering

ANSWERS

Section A

1. (a) $w_0, Q^+; w^+, Q^+; w_0, Q^-; w^-, Q^-$
- (b) $Q = bv_1(p_3 - p_1); Q = (b+1)p_3(v_3 - v_1); \left((b+1)\frac{p_3}{p_3 - p_1} + b\frac{v_1}{v_3 - v_1} \right)^{-1}$
- (c) $w_x = -v_1(p_3 - p_1), Q = bv_1(p_3 - p_1); w_x = 0, Q = (b+1)p_3(v_3 - v_1)$
2. (b) 9.54 kg
3. (a) 13.86 kg
4. (b) 270.5K; 1229.5K; 3.76 bar; 348.8K; 37.5%

Section B

5. Answers approximate
 - (a) (ii) 1.1×10^{-3} rad/s anti-clockwise, 1.0×10^{-3} rad/s clockwise,
 0.45×10^{-3} rad/s anti-clockwise, 0.058 mm/s,
 - (iii) 94 N
6. (a) (i) $\Omega_1 d \underline{k}, \omega R \underline{k}, \Omega_2 d \underline{k}$
(ii) $2R \cos \alpha \underline{i}, -2R \omega \cos^2 \alpha \underline{k}$
- (b) $\Omega_1 d - \omega R, \omega R - 2R \omega \cos^2 \alpha - \Omega_2 d$
- (c) $1 - 2 \cos^2 \alpha, -1$
7. (b) $\underline{v} = \alpha \lambda S \underline{e}_t$ where $S = \sqrt{1 + \sin^2 \lambda}$
 $\underline{a} = (\alpha \lambda \ddot{S} + \alpha \lambda^2 \sin \lambda \cos \lambda / S) \underline{e}_t - \alpha \lambda^2 \cos \lambda / S \underline{e}_n$
- (c) (i) $F = m \underline{a} \cdot \underline{e}_t, N = m \underline{a} \cdot \underline{e}_n$
(ii) $F = -N \sin \lambda$
8. (b) (i) 5747 m/s
(c) 9.52×10^5 Ns, 3367 m/s

Section C

9. (a) $B = T = \frac{\rho c R D}{2h(D + R)}$
(b) 17.7 mins
(c) Working in minutes:
For $t \leq 12$, $\theta = (15T - 25)\exp(-t/T) + 20 - 15T + 15t$
For $t > 12$, $\theta = 200 - 282.4 \exp(-t/T)$
56.5°C, 30.7 mins.
10. (a) $\theta = (y_2 - y_1)/L$, $y_g = (y_2 + y_1)/2$
(b) $m\ddot{y}_g = -k(y_1 - x_1) - k(y_2 - x_2)$
(d) 5 μm
11. (b) $c = \frac{1}{\sqrt{8}} \left(\frac{1}{R} \sqrt{\frac{L}{C}} + R \sqrt{\frac{C}{L}} \right)$
(c) $\sqrt{2} - 1 < \frac{1}{R} \sqrt{\frac{L}{C}} < \sqrt{2} + 1$

Part 1A Engineering Tripos 1999

Paper 2 Section A, Structures

Answers

1 (a) (i) Sliding will occur.

(b) $W_{\max} = 61.4 \text{ kN}$; $W_{\min} = 4.78 \text{ kN}$

2 (a) (i) $2W$

(ii)

Bar	Tension
AC, EG	$-2W/\sqrt{3}$
BC, EF	$2W/\sqrt{3}$
CD, DE	$2W/\sqrt{3}$

(b) (i) vertical = $10WL/3EA$, horizontal = 0

(ii) reduced by $2WL/\sqrt{3}EA$

3 (a) $T_{CD} = \sqrt{2}wa/4$; reactions at A: $\uparrow 3wa/4$, $\leftarrow wa/4$;
reactions at B: $\uparrow wa/4$, $\rightarrow wa/4$

(b) $M_B = wa^2/4$

(c) $M_B = wa^2/4$, $M_{\min} = -wa^2/32$, $M_C = 0$

4 (a) $M_C = 2wa^2$

(b) Using clockwise +ve, $\theta_B = 3wa^3/24EI$, $\theta_C = 21wa^3/24EI$

(c) $\delta_A = 3wa^4/24EI \uparrow$, $\delta_D = 90wa^4/24EI \downarrow$

5 (b) $S_{\max} = 1 \text{ kN}$, $S_{\min} = -1 \text{ kN}$; $M_{\max} = 0$, $M_{\min} = -300 \text{ Nm}$

(c) $\sigma_{\max} = 1.64 \times 10^6 \text{ N/m}^2$

(d) 182 N

Engineering Tripos Part 1A 1999
Paper 2 Section B MATERIALS
Answers

6 (a) $\epsilon_t = \ln(A_0/A)$
 $\sigma_{TS} = \sigma_t$ (at necking) $\times (A / A_0)$.

6 (b) $\epsilon_t = n - B$at necking.
 $\sigma_{TS} = \sigma_0(n)^n \exp (n - B)$
 $B = 1.86$.

7 $\epsilon = t/2R$ and $\sigma = E(t/2R)$.
 R must satisfy $R \geq (t/2) (E/\sigma_y)$.
The merit index M (for a given t) is $M = \sigma_y/E$.
Best choices include PE, PP, Nylon, elastomers.

8 (a) The elastic stress concentration factor $K_t = \sigma_{local}/\sigma_{nominal}$
For a hole, $K_t = 1 + 2 (a/\rho)^{1/2} = 3$ (a is notch length, ρ is notch tip radius)
The stress intensity factor $K = \sigma_{nominal} (\pi a)^{1/2}$
($K = K_c$ when $\sigma_{nominal} = \sigma_f$).
The local tensile stress $\sigma_{local} = K/(2\pi r)^{1/2}$ (r is distance)

8 (b) The steel's endurance fatigue limit is roughly half its yield strength, i.e., 150 MPa. Since the cyclic stress is ± 25 MPa, the likelihood of a crack nucleating elsewhere away from any pre-existing flaw is negligible.
 $\Delta K_{(threshold)}$ is 6 MPam^{1/2}; K_{IC} is 50 MPam^{1/2}, approximately.
 $\Delta\sigma = 50$ MPa and $\Delta K_{(threshold)} = 6$ MPam^{1/2}; acceptable "a" is 3.65mm.

9 (b) $\sigma_y = \sigma_0 + 2(2Gb/l)$ ($2\tau_y = \sigma_y$, approximately)
 $\sigma_y = 421$ MPa.

9 (c) The new T_D is 148 K.

10 (b) $T_1 = 1308$ K = 1035 C, an increase in 35 C.

Paper 3: Electrical and Information Engineering

Answers

1. (a) $R_2 = 3 \text{ k}\Omega$, $V_{DD} = 23 \text{ V}$, $R_1 = 100 \text{ k}\Omega$ to $10 \text{ M}\Omega$ typically.
 (b) $R_{in} = R_1$ as in (a), $R_{out} = 8.33 \text{ k}\Omega$, Gain = - 41.65 (i.e. inverting).
 (c) $R_{load} = R_{out} = 8.33 \text{ k}\Omega$.
 (d) $C = 95.5 \text{ nF}$.
2. (b) $R_{IN} = 1099 \Omega$, $G = - 9.01$
 (c) $v_o = - 4.72 v_s$.
 (d) $f_{3dB} = 1.67 \text{ MHz}$ (*not* 1.59 MHz).
3. (b) $V_{th} = -5 \text{ V}$, $R_{th} = 20 \Omega$.
 (c) $R_A = 20 \Omega$, $R_B = 40 \Omega$.
 (d) chain 3 circuits as in (c).
4. (a) $P_{loss} = 590.8 \text{ W}$, $V_{in} = 247.8 \text{ V}$.
 (b) $C = 902 \mu\text{F}$, ($X_c = 3.53 \Omega$).
 (c) $V_{load} = 242.6 \text{ V}$.

Section B

5. (b) Sum-of-products:

$$\begin{aligned} O_0 &= I_0 \cdot \overline{I_1} + \overline{I_0} \cdot I_1 = I_0 \oplus I_1 \\ O_1 &= \overline{I_1} \cdot I_2 + \overline{I_0} \cdot I_2 + I_0 \cdot I_1 \cdot \overline{I_2} = I_2 \cdot (\overline{I_0} + \overline{I_1}) + I_0 \cdot I_1 \cdot \overline{I_2} \\ O_2 &= I_0 \cdot I_1 \cdot I_2 \end{aligned}$$

Product-of-sums:

$$\begin{aligned} O_0 &= (I_0 + I_1) \cdot (\overline{I_0} + \overline{I_1}) \\ O_1 &= (I_1 + I_2) \cdot (I_0 + I_2) \cdot (\overline{I_0} + \overline{I_1} + \overline{I_2}) \\ O_2 &= I_0 \cdot I_1 \cdot I_2 \end{aligned}$$

(c) 2-input NAND implementation

$$\begin{aligned} O_0 &= \overline{\overline{(I_0 \cdot I_1)} \cdot \overline{\overline{(I_0 \cdot I_1)}}} \\ O_1 &= \overline{\overline{I_2 \cdot (I_0 \cdot I_1)} \cdot \overline{\overline{I_0 \cdot I_1 \cdot I_2}}} \\ O_2 &= \overline{\overline{\overline{I_0 \cdot I_1 \cdot I_2}}} \end{aligned}$$

6. (b) i) Unused states: 0010, 0110, 1110, 1010
 ii)

$$\begin{aligned} J_A &= Q_B \cdot Q_C + Q_B \cdot \overline{Q_D} \\ K_A &= \overline{Q_B} \cdot \overline{Q_D} + Q_B \cdot \overline{Q_C} \cdot Q_D \\ J_B &= \overline{Q_A} \cdot \overline{Q_D} + \overline{Q_A} \cdot Q_C \\ K_B &= \overline{Q_A} \cdot \overline{Q_C} \cdot Q_D + Q_A \cdot Q_C \end{aligned}$$

7. b) iii) $10.9 \mu\text{s}$

8. b) ii) 65536 iii) 4

Section C

9. b) 0.53 T

c) the inductance doubles

10. b) i) 8 pF

ii) 16 pF

c) $8 + 40d$ pF

d) rises 0.043 mm up the tube

11. a)

$$\frac{N_A d}{\pi (1/\mu_A + 1/\mu_B)} \ln \frac{R_2}{R_1}$$

b) peak flux VT

d) flux density is zero

Numerical answers to 1999 Part Ia, Paper 4 Mathematical Methods

Section A

Question 1.

b)

(i) $\sqrt{3}x + y + \frac{z}{\sqrt{2}} = \sqrt{3}$

(ii) $x = \sqrt{3}y = \sqrt{6}z$

(iii) $x = \frac{1}{2}, y = \frac{1}{2\sqrt{3}}, z$ arbitrary.

c) $\left(\frac{1}{2}, \frac{1}{2\sqrt{3}}, \frac{1}{2\sqrt{6}}\right)$.

Question 2.

a) $5^{\frac{1}{4}} \exp(i0.25\theta + 0.5n\pi i)$ for $n = 0, 1, 2, 3$, where $\theta = \tan^{-1} 4/3$. Solutions of polynomial are $(3 \pm 4i)^{\frac{1}{4}}$, so above roots together with their complex conjugates.

b) 1

c) $1 - \frac{x^2}{2} + \frac{5x^4}{24} + \dots$

Question 3.

a) $y = \frac{\exp(-4x)}{5} - \frac{\exp(-2x)}{5} - \frac{x \exp(-4x)}{7}$.

b) $y_n = \frac{3}{5}2^n + \frac{2}{5}\left(-\frac{1}{2}\right)^n$.

Question 4.

a)

$$Q(\theta) = \begin{pmatrix} \cos \theta & -\sin \theta & 0 \\ \sin \theta & \cos \theta & 0 \\ 0 & 0 & 1 \end{pmatrix}$$

Question 5.

b) eigenvalues 2, 1, -1, normalised eigenvectors $(1, 0, 0)^t$, $(0, 1, 1)^t/\sqrt{2}$, $(0, 1, -1)^t/\sqrt{2}$.

d)

$$A - 2I = \begin{pmatrix} 0 & 0 & 0 \\ 0 & -2 & 1 \\ 0 & 1 & -2 \end{pmatrix}$$

Section B

6. (b) $t - \frac{t^2}{2T}$ for $0 \leq t \leq T$, $\frac{T}{2}$ for $t \geq T$

(c) (i) $\frac{t^2}{2} - \frac{t^3}{6T}$ (ii) $\frac{5}{6}T^2$

7. (a) $\frac{1}{2} - \frac{2}{\pi} \sum_{n=1}^{\infty} \frac{\sin(2n-1)\theta}{2n-1}$

(b) $a_0 = \frac{1}{2}$, $a_n = \frac{1}{n^2\pi^2} [1 - (-1)^n]$ $b_n = -\frac{1}{n\pi}$

(d) $\lambda = 1$ $\mu = \frac{1}{2}$

8. $\frac{q}{p+q}$

9. (a) $x(t) = \frac{1}{2} \sin t + e^{-t/2} \cos \frac{\sqrt{3}}{2}t$ $y(t) = \frac{1}{2} \sin t - \frac{1}{\sqrt{3}} e^{-t/2} \sin \frac{\sqrt{3}}{2}t$ for $t \geq 0$.

(b) 0, 0.

10. $(-1, 2)$, $(-1, -\frac{4}{3})$ and $(1, 0)$ are saddle points, $(-\frac{1}{3}, \frac{2}{9})$ is a maximum