## ENGINEERING TRIPOS PART IB

Friday 8 June 20129 to 11.30
9 to 10.30 Foreign Language Option
Paper 8

## SELECTED TOPICS

Answer one question from Section A. In addition:

If you are not taking the Foreign Language option, answer four questions, taken from only two of Sections $B-H$. Not more than two questions from each section may be answered.
If you are taking the Foreign Language option, answer two questions from one of Sections B-- $H$.

All questions carry the same number of marks.
The approximate number of marks allocated to each part of a question is indicated in the right margin.

Answers to questions in each section should be tied together and handed in separately.

Section A (Introductory Business Economics) 2
Section B (Civil and Structural Engineering) 3
Section C (Mechanics, Materials and Design) 6
Section D (Aerothermal Engineering) 10
Section E (Electrical Engineering) 13
Section F (Information Engineering) 16
Section G (Engineering for Life Sciences) 19
Section H (Manufacturing, Management and Design) 22
Attachments: Data Sheet for Section B (6 pages)
Data Sheet for Section E (2 pages)

STATIONERY REQUIREMENTS
Single-sided script paper
CUED approved calculated allowed

> | You may not start to read questions |
| :--- |
| printed on the subsequent pages of this |
| question paper until instructed that you |
| may do so by the Invigilator |

SECTION A Introductory Business Economics

Answer not more than one question from this section.

1 (a) Explain the concept of profit maximisation.
(b) Explain why firms may not seek to maximise profits.
(c) Using the Keynesian Consumption Function Model, explain the potential impact of a temporary increase in income tax on the level of aggregate demand.
(d) Using either the Life Cycle Model or the Permanent Income Model, explain the potential impact of a temporary increase in income tax on the level of aggregate demand.

2 (a) Outline the kinked demand curve theory of oligopoly.
(b) Under what circumstances might monopolies improve consumer welfare?
(c) Outline the accelerator model of investment.

SECTION B Civil and Structural Engineering

Answer not more than two questions from this section.

## Note Data Sheets at end of paper.

3 An underground railway is to be constructed beneath the centre of Cambridge. The ground conditions are typically 5 m of sand fill overlying Gault Clay, with the water table close to the ground surface. The undrained shear strength of the Gault clay can be taken to be $150 \mathrm{kN} \mathrm{m}^{-2}$, constant with depth. The unit weight of all soils can be taken to be $20 \mathrm{kN} \mathrm{m}^{-3}$.

You are to advise on the relative merits of two options: (i) a cut-and-cover tunnel, and (ii) a bored tunnel. The cut and cover tunnel would be a rectangular, 10 m wide and 6 m high, allowing for two tracks, whereas the bored tunnel would consist of two separate 8 m diameter tunnels. The depth to the bottom of the tunnels would vary between 12 m and 20 m . The tunnel route would be adjacent to a number of historic buildings.
(a) What methods of construction would be suitable for the bored tunnel? Would the tunnels need to be constructed by open-face or closed-face techniques?
(b) Suggest two methods by which the bored tunnels could be lined, briefly describing the processes.
(c) Discuss the advantages and disadvantages of cut-and-cover tunnel construction.
(d) Why does settlement potentially cause damage to masonry structures? How might the damage due to tunnel construction be minimised?
(e) Describe how instrumentation can be used to ensure the success of schemes to prevent damage to overlying structures.
4. A retaining wall, propped at the ground surface, is to be constructed in order to construct an 8 m deep station box for an underground railway, as shown in Fig. 1. The soil profile consists of a stiff clay with a unit weight $\gamma=20 \mathrm{kN} \mathrm{m}^{-3}$, an undrained shear strength of 50 kPa and a critical state friction angle $\varphi^{\prime}=20^{\circ}$. The retaining wall will penetrate a distance 8 m into the underlying clay. The water table is below the base of the retaining wall.

Sketch the earth pressure distribution on both sides of the retaining wall and calculate whether a penetration depth of 8 m is sufficient for stability of the wall with no surface surcharge for the following conditions:
(a) assuming short-term undrained conditions;
(b) assuming long-term drained conditions.

A surcharge of 25 kPa is applied to the ground surface. Is the wall stable:
(c) assuming short-term undrained conditions;
(d) assuming long-term drained conditions?

Surcharge


Fig. 1

## 5

5. A reinforced concrete wall AD supports a 5 m deep excavation and penetrates a further 3 m into the ground, as shown in Fig. 2. Tie rods, at 2 m intervals along the length of the wall, at position B , are connected to remote anchor blocks.

The cube strength of the concrete is 40 MPa and the yield strength of the reinforcing bars is 460 MPa . The earth pressures shown in Fig. 2 are those that act on the wall.
(a) If the earth pressure is zero at point $\mathrm{C}, x$ at point D and is assumed to vary linearly between these points, calculate $x$.
(b) What tension $T$ will each tie-rod carry?
(c) Calculate the shear force and bending moment at points A, B, C \& D and the maximum and minimum bending moments. Sketch the resulting shear force and bending moment diagrams.
(d) If the wall is to be singly reinforced, calculate the minimum effective depth of the section for the most heavily loaded part of the wall.
(e) If the actual effective depth is $50 \%$ greater than the value calculated in part (d) above, determine a suitable layout of reinforcement in the wall.


Fig. 2

SECTION C Mechanics, Materials and Design

Answer not more than two questions from this section.
6 (a) Consider bending vibrations of a wind turbine tower of height $L$ with a uniform cross-section of second moment of area $I$ made of a material with Young's modulus $E$. Figure 3(a) illustrates an idealisation of the tower as a continuous beam, encastred at its root, whose lowest frequency of vibration $f$ is given by the expression

$$
f=\frac{3.52}{2 \pi} \sqrt{\frac{E I}{m L^{4}}}
$$

where $m$ is the mass per unit length. Figure 3 (b) shows an alternative model where the tower is represented by a light rigid bar of length $L$ and a point mass $M$ at the top of the bar. The base of the bar is connected to the ground via a rotational spring of stiffness $k$.
(i) One modelling approach aims to match the lowest frequency of bending vibration of the tower and the stiffness associated with a transverse load at the top of the tower for the two idealisations of Figs. 3(a) and 3(b). Find expressions for $M$ and $k$ in terms of $E, I, L$ and $m$ which achieves this.
(ii) An alternative approach matches the mass and lowest frequency of vibration of the two tower idealisations, but not the stiffness. Give an advantage or disadvantage of such an approach, compared to the approach in part (i).
(a)

(b)


Fig. 3
(b) Define the terms 'cut-in', 'rated' and 'stall' as applied to wind speeds for wind turbines and sketch a typical power versus wind speed characteristic.
(c) Give two methods of controlling the turbine power so that it remains fixed at its rated value between rated and stall wind speeds.
(d) Simplified wind data for an offshore wind turbine is given in the table below. A 'new generation' wind turbine has a cut-in wind speed of $3 \mathrm{~m} \mathrm{~s}^{-1}$, a stall wind speed of $20 \mathrm{~m} \mathrm{~s}^{-1}$ and a rated wind speed of $14 \mathrm{~m} \mathrm{~s}^{-1}$, at which it produces a rated output of 5 MW . The turbine operates at variable speed between cut-in and rated wind speeds so as to maintain its optimum tip-speed ratio of 10 , at which the power coefficient is 0.45 .

Find:
(i) the turbine diameter;
(ii) the rotational speed of the turbine at its rated wind speed;
(iii) the annual energy produced;
(iv) the capacity factor.

Take the power produced by a turbine as $P=0.5 C_{p} \rho A v^{3}$ and the density of air as $1.23 \mathrm{~kg} \mathrm{~m}^{-3}$.

| Wind speed, $v\left(\mathrm{~m} \mathrm{~s}^{-1}\right)$ | Number of days <br> per year |
| :---: | :---: |
| $<2.5$ | 10 |
| 7 | 185 |
| 12 | 100 |
| 16 | 50 |
| $>20$ | 20 |

7 (a) Discuss noise issues relating to wind turbines, including sources of noise, characterisation of noise and ways to manage the impact of noise.
(b) Figure 4(a) illustrates a wind turbine blade subject to a uniform torsional moment intensity $Q$ per unit length of blade. The blade has a thin-walled cross-section, which is uniform along its length. The wall thickness $t$ is uniform around the cross section and along the length of the blade, but is a design variable which can be varied. The area enclosed by the section $A_{e}$ and the cross-sectional perimeter $s$ are fixed. The blade is made of a material with shear modulus $G$.
(i) Derive an expression for the grouping of material parameters which should be maximised to minimise blade mass for a given torsional rotation of the tip of the blade relative to its root.
(ii) Why might torsional rigidity be an important design objective?
(c) Figure 4(b) illustrates the cross-section of a blade which is to be made of glass fibre composite with some regions of unidirectional material and other regions of $50 \% 0^{\circ}$ plies and $50 \%$ of $\pm 45^{\circ}$ plies.
(i) Use Fig. 4(c) to find the Young's moduli of the proposed lay-ups.
(ii) Identify which regions of the cross-section might be appropriate for each of the two lay-ups, explaining your reasoning.
(d) Describe the vacuum infusion process for making composite components, and comment on why it is widely used to make wind turbine blades.


8 A horizontal axis wind turbine has three blades of radius 20 m and turns with an angular velocity $\omega=30 \mathrm{rpm}$. Under normal operating conditions the aerodynamic loads on each blade may be assumed to be distributed as follows: the normal force intensity $F_{N}$ varies linearly from 0 at the hub $(r=0)$ to $1000 \mathrm{Nm}^{-1}$ at the blade tip ( $r=20 \mathrm{~m}$ ); the tangential force $F_{T}$ is uniformly distributed along the length of the blade with an intensity of $100 \mathrm{~N} \mathrm{~m}^{-1}$.
(a) What is the mechanical power generated by the turbine?
(b) Each blade has a structural spar with the box cross-section at the blade root $(r=0)$ shown in Fig. 5. The wall thickness of the spar is 20 mm . The aerodynamic surface of the blade may be neglected for structural calculations.

What is the maximum stress in the spar cross-section at the blade root due to the aerodynamic loads?
(c) The total mass of each blade per unit length may be assumed to vary linearly from $300 \mathrm{~kg} \mathrm{~m}^{-1}$ at the hub $(r=0)$ to 0 at the blade tip $(r=20 \mathrm{~m})$.

What is the maximum stress in the spar at the blade root due to:
(i) centrifugal loading;
(ii) self-weight loading?
(d) The spar material has a tensile strength of $\sigma_{T S}=400 \mathrm{MPa}$ and its fatigue behaviour may be calculated according to Basquin's Law and Goodman's Rule (Materials Data Book) with the constants $\alpha=0.08$ and $C_{1}=800 \mathrm{MPa}$.

What is the expected lifetime in years of the blades, assuming continuous operation under normal loading conditions? Detail any assumptions.


Fig. 5

## SECTION D Aerothermal Engineering

Answer not more than two questions from this section.

9 (a) Explain how take off conditions determine the wing area of a passenger aircraft.
(b) Why do passenger aircraft cruise at high altitude? Explain why the cruise altitude should increase during a long distance flight.
(c) Derive the following form of Breguet's range equation for the distance, $s$, between the start and end of cruise,

$$
s=\frac{V L / D}{g s f c} \ln \left(\frac{W_{\text {start }}}{W_{\text {end }}}\right)
$$

where $V$ is the flight speed, $s f c$ is the thrust specific fuel consumption, $L / D$ is the aircraft lift-to-drag ratio, $g$ is the acceleration due to gravity and $W_{\text {start }}$ and $W_{\text {end }}$ are the total aircraft weights at the start and end of cruise respectively. State any assumptions made in the derivation of this formula.
(d) Define the terms thermal efficiency and propulsive efficiency. Show that it is the product of these efficiencies which must be maximised in order to minimise the thrust specific fuel consumption.
(e) Describe how a modern turbofan engine is designed to achieve both high propulsive efficiency and high thermal efficiency. List the factors that limit the upper values of these efficiencies.

10 (a) An aircraft is fitted with two-shaft turbofan engines. At cruise conditions, after the core air stream has passed through the fan, the stagnation temperature $T_{023}$ is 293 K and the stagnation pressure $p_{023}$ is 70 kPa . The core compressor has a stagnation pressure ratio of 25 and an isentropic efficiency of 0.88 . Calculate the stagnation temperature $T_{03}$ at exit from the core compressor. Given that the mean blade speed $U_{\mathrm{m}}$ is $350 \mathrm{~m} \mathrm{~s}^{-1}$ and the compressor stage work coefficient $\Delta h_{0} / U_{m}^{2}$ is not to exceed 0.45 , determine the minimum number of core compressor stages required.
(b) The high-pressure turbine drives the core compressor and has an isentropic efficiency of 0.85 . The stagnation temperature $T_{04}$ at the inlet to the high-pressure turbine is 1650 K . Determine the stagnation temperature $T_{045}$ and the stagnation pressure $p_{045}$ at entry to the low-pressure turbine.
(c) The low-pressure turbine has an isentropic efficiency of 0.90 . The stagnation pressure ratio across the low-pressure turbine is 8.5 and atmospheric pressure is 28.7 kPa . Calculate the velocity $V_{9}$ of the core jet.
(d) The cruise speed of the aircraft is $240 \mathrm{~m} \mathrm{~s}^{-1}$ and the temperature rise for the core flow passing through the fan $\left(T_{023}-T_{02}\right)$ is 41 K . If the core jet velocity is equal to the bypass jet velocity, calculate the bypass ratio of the engine assuming that the fan is isentropic.

For the purposes of this question, neglect the mass flow rate of the fuel. Assume that the combustion products behave as a perfect gas with the same properties as air. Neglect any losses in the ductwork and the propelling nozzles.

Take $\gamma=1.4, R=287 \mathrm{~J} \mathrm{~kg}^{-1} \mathrm{~K}^{-1}$ and $c_{p}=1010 \mathrm{~J} \mathrm{~kg}^{-1} \mathrm{~K}^{-1}$ for air.

11 (a) A turbofan engine is fitted to an aircraft that cruises at Mach 0.80, at an altitude where the pressure and temperature are 26 kPa and 223 K , respectively. The engine has a total mass flow rate $\dot{m}_{a}$ of $400 \mathrm{~kg} \mathrm{~s}^{-1}$ and the jet velocity is $350 \mathrm{~m} \mathrm{~s}^{-1}$ for both the core and the bypass streams. Determine the flight velocity, the net thrust and the gross thrust from one engine. Assuming no loss in the inlet ducting, calculate the stagnation pressure $p_{02}$ and stagnation temperature $T_{02}$ at the engine inlet.
(b) The non-dimensional thrust $\widetilde{F}$ and the non-dimensional mass flow $\widetilde{\tilde{m}}$ for the engine are defined as

$$
\widetilde{F}=\frac{F_{G}+p_{a} A_{N}}{p_{02} A_{N}} \quad \text { and } \quad \widetilde{\tilde{m}}=\frac{\dot{m}_{a} \sqrt{c_{p} T_{02}}}{p_{02} A_{N}}
$$

where $A_{N}$ is the total nozzle exit area, $p_{a}$ is the ambient pressure and $F_{G}$ is the engine gross thrust. Explain carefully the significance of using the term $F_{G}+p_{a} A_{N}$ in the expression for $\widetilde{F}$. If the total nozzle exit area is $2.8 \mathrm{~m}^{2}$, calculate the values of $\widetilde{F}$ and $\widetilde{\dot{m}}$ for the engine at cruise.
(c) Use the relationship between net thrust $F_{N}$ and gross thrust $F_{G}$ to show that

$$
\frac{F_{N}}{p_{a} A_{N}}=\left\{\widetilde{F}\left(\frac{p_{02}}{p_{a}}\right)-1\right\}-\widetilde{\tilde{m}} M \sqrt{\gamma-1}\left(\frac{p_{02}}{p_{a}}\right)\left(\frac{T_{02}}{T_{a}}\right)^{-0.5}
$$

where $M$ is the flight Mach number and $T_{a}$ is the ambient temperature.
(d) The engine is operated at a new cruise condition at higher altitude where the pressure and temperature are 18 kPa and 216 K , respectively. The flight Mach number at this condition is 0.85 . Assuming that the non-dimensional operating point is unchanged, calculate the total mass flow through the engine and the net thrust at the new cruise condition.

For the purposes of this question, $\gamma=1.4, R=287 \mathrm{~J} \mathrm{~kg}^{-1} \mathrm{~K}^{-1}$ and $c_{p}=1010 \mathrm{~J} \mathrm{~kg}^{-1} \mathrm{~K}^{-1}$ for air. It can be assumed throughout that the propelling nozzles of the engine are choked.

SECTION E Electrical Engineering

Answer not more than two questions from this section.
Note the Data Sheet at end of paper

12 (a) Compare and contrast the use of aluminium, aluminium/copper and aluminium/silicon for application as the metallisation system in state-of-the-art integrated circuits.
(b) As device dimensions continue to shrink, multilevel interconnects are increasingly required to reduce the delay time. Estimate the delay time associated with two parallel aluminium wires of length 2.0 mm and of square cross section of side $0.35 \mu \mathrm{~m}$, separated by a 200 nm silicon dioxide layer. Assume the resistivity of aluminium is $2.7 \mu \Omega \mathrm{~cm}$ and that the relative dielectric constant for silicon dioxide is 3.9.
(c) Explain what is meant by a "self-aligned structure" including its benefits for transistor manufacture and explain why poly-silicon is often used as the gate contact material in such a process.

13 (a) Explain the difference between a metal, a semiconductor and an insulator in terms of their band structures.
(b) Explain with the aid of diagrams how conduction occurs in a doped semiconductor. What controls the number of carriers?
(c) Derive the relationship between the electrical conductivity and the mobility of a conductor. Then use the random walk model of electron scattering to relate the electrical conductivity to the effective electron mass, $\mathrm{m}^{*}$, and the time between collisions, $\tau$.
(d) The mobility of a particular semiconductor is $0.5 \mathrm{~m}^{2} \mathrm{~V}^{-1} \mathrm{~s}^{-1}$, its scattering limited velocity is $1.25 \times 10^{5} \mathrm{~m} \mathrm{~s}^{-1}$ and its effective electron mass $\mathrm{m}^{*}$ is 0.2 . This material, doped to a donor density of $4 \times 10^{21} \mathrm{~m}^{-3}$, is used to make a depletion mode FET, operating at a supply voltage of 0.5 V .
(i) At what source-drain length does the conduction become scattering limited?
(ii) What is the corresponding transit time?
(e) Calculate the scattering time between collisions. Compare this to the transit time and comment on the result.

The electronic charge $=1.6 \times 10^{-19} \mathrm{C}$. Mass of electron $=9.104 \times 10^{-31} \mathrm{~kg}$.

14 (a) Describe what is meant by 'scaling' in CMOS integrated circuits. In simple scaling, if a feature length scales by a factor $k$, how much do the following properties scale by: device density, transit time, voltage, power per chip, wafer diameter? Make clear any assumptions needed for your answer.
(b) What are the main factors that are presently limiting scaling?
(c) Draw a typical carrier velocity vs. electric field curve for a III-V semiconductor such as GaAs, labelling the various regions and giving typical numerical values.
(d) Derive the transit time for an InGaAs FET with source-drain distance of $0.05 \mu \mathrm{~m}$, if the mobility is $4 \mathrm{~m}^{2} \mathrm{~V}^{-1} \mathrm{~s}^{-1}$, the scattering limited velocity is $2 \times 10^{5} \mathrm{~m} \mathrm{~s}^{-1}$ and $\mathrm{V}_{\text {SD }}$ is 0.5 V . Comment on the result.
(e) Calculate a suitable dopant density so that the transistor will operate with a gate voltage swing of 0.1 V . Assume that the dielectric constant for InGaAs is $1.5 \times 10^{-10} \mathrm{~F} \mathrm{~m}^{-1}$ and the channel depth is $0.02 \mu \mathrm{~m}$.

SECTION F Information Engineering

Answer not more than two questions from this section.

15 (a) Derive the formula for performing 1-dimensional linear interpolation to obtain the intensity $x_{p}$ of a pixel at location $p$, where $a<p<b$ and $x_{a}$ and $x_{b}$ are the intensities of pixels located at $a$ and $b$ respectively.
(b) Extend this formula to 2-dimensional bi-linear interpolation to obtain the pixel intensity $x_{p, q}$ at location ( $p, q$ ) in terms of the four pixels at locations (a,c), (a,d), (b,c) and (b,d).
(c) For fun, it is required to adjust the locations of the features of a facial image. Describe the image-processing technique to achieve this and explain why a Gaussian function is often used to produce the displacement field. Derive an expression for the displacement field $\mathbf{d}(s, t)$ at arbitrary position $(s, t)$ in the image. The maximum of the displacement is centred at location $(p, q)$ and the field is such that the pixel at $(p, q)$ in the modified image comes from the location $(u, v)$ in the original image, while the other pixels around it are displaced less than this.
(d) Explain why it is necessary to control the standard deviation (spread) of the Gaussian function so that it maintains the sense of the image after the image has been distorted (i.e. features should not be distorted in an unrecognisable way). Hence determine how the deviation should relate to the maximum displacement required, $|(u-p, v-q)|$.

16 (a) A grey scale image, $I(x, y)$, is to be smoothed and differentiated as part of the edge detection process.
(i) Why is smoothing required? Describe a smoothing filter used in practice, and explain the effects of increasing the size of the filter.
(ii) Give an expression for computing the intensity of a smoothed pixel, $S(x, y)$, in terms of two discrete 1D convolutions.
(iii) Derive expressions for computing the spatial derivatives after smoothing. Include details of the two 1D convolutions required.
(b) Image features such as blob-like shapes can be localised in an image by filtering with a band-pass filter over multiple scales.
(i) Show how different resolutions of the image can be represented efficiently in an image pyramid. Your answer should include details of the implementation of smoothing to avoid convolution with large filters.
(ii) What is meant by band-pass filtering and how is this implemented for this purpose.
(iii) Show how image features such as blob-like shapes can be localised in an image and describe how to determine an appropriate scale and feature size.

17 (a) A mobile phone application is used to match an image of a planar target such as a book cover by searching images of known planar objects in its database. A finite number of interest points are extracted automatically from each image. Each feature is to be characterised by its SIFT descriptor.
(i) Give details of how the SIFT descriptor is computed. What image structure is the descriptor attempting to describe?
(ii) How are the descriptors used to match image features to features of known objects in the database?
(iii) How is the target recognised?
(b) The matching scheme is to be extended to describe the similarity of images with different categories of objects using histograms of visual words.
(i) What is meant by visual words and how can they be computed from SIFT descriptors of interest points in the image?
(ii) Show how an arbitrary image can be represented by a vector of fixed length.
(iii) How can the category of the object in the image be assigned?

## SECTION G Engineering for the Life Sciences

Answer not more than two questions from this section.

18 The cornea of the eye is largely comprised of the natural material collagen.
(a) Describe the structure and self-assembly of collagen.
(b) How is collagen in the cornea organized? How does this differ from the collagen organization in the sclera? How does this relate to the function of the cornea?
(c) Describe LASIK surgery in relation to the cornea.
(d) Both the collagen density and the collagen fibril diameter can influence different mechanical properties of a soft collagenous tissue such as the cornea.
(i) Consider a hypothetical disease in which the collagen fibril diameter is unchanged, but in which the collagen density increases. What effect does this change have on the elastic modulus of the tissue? What effect does this change have on the intrinsic permeability? How do the elastic modulus and intrinsic permeability relate to the function of the cornea?
(ii) Now consider a different hypothetical disease, in which the collagen density overall does not change, but for which the diameter of individual collagen fibrils is decreased. What effect does this change have on the elastic modulus and intrinsic permeability, and how does this differ from the change in part (d)(i) above?

19 (a) Describe, with an appropriate diagram, how the confocal optics in a Scanning Laser Ophthalmoscope (SLO) allows depth sectioning of the fundus.
(b) A light beam is focused at a thin confocal aperture of diameter $d$. The beam cross-section is a disc of radius $r$ given by:

$$
r^{2}(z)=r_{0}^{2}\left(1+\left(\frac{\lambda z}{\pi r_{0}^{2}}\right)^{2}\right)
$$

where $z$ is the axial distance from the confocal aperture, $r_{0}$ is the radius at the focal point, and $\lambda$ is the wavelength of light. Assume that the lens is placed in air.
(i) What is the approximate radial resolution of the lens, $\Delta x=2 r_{0}$, in terms of the numerical aperture $N A$ and wavelength $\lambda$ ?
(ii) The axial resolution $\Delta z$ is defined as the range in $z$ over which at least half of the beam cross-sectional area would pass through the aperture. Show that:

$$
\Delta z=\frac{2 \lambda}{\pi(N A)^{2}} \sqrt{\frac{1}{2}\left(\frac{d \pi N A}{\lambda}\right)^{2}-1}
$$

and evaluate $\Delta z$ for $N A=0.4, \lambda=800 \mathrm{~nm}$ and $d=20 \mu \mathrm{~m}$.
(iii) First by increasing $N A$ alone, then by decreasing $d$ alone, find the minimum achievable $\Delta z$ (in each case keeping other values as in (ii)). Also list in each case the values of $N A$ and $d$ which you used.
(iv) What are the problems with the improvements outlined in (iii)?
(c) What factors, other than lens resolution and aperture size, limit the radial resolution and the depth range of an SLO?

20 (a) This question regards the mechanisms of visual processing.
(i) Describe the differences between ganglion cell, simple cell and complex cell receptive fields.
(ii) What is colour opponency and why might it be a useful way for the visual system to code colour?
(iii) Describe the evidence that the visual system processes different spatial frequencies within an image independently.
(iv) Describe the cues to object depth that are missing from a photograph.
(b) This question regards optimality in visual processing.
(i) Describe the source of the resolution limit of compound eyes, including the corresponding mathematical formula.
(ii) Derive the formula for the response of a neuron, $r$, as a function of the stimulus (contrast level of light), $s$. Assume that the neuron encodes this stimulus with an information theoretically optimal tuning curve, with negligible noise in its response. Assume that its response lies between $r_{\text {min }}$ and $r_{\text {max }}$ and is a monotonically increasing function of the stimulus, and that the stimulus is uniformly distributed between $s_{\min }$ and $s_{\max }$. In your derivation:

- explain what makes a tuning curve optimal in this case (i.e. what is the quantity that the tuning curve needs to optimise),
- state what is the response distribution of the neuron in this case,
- explain the steps taken to derive the final formula for the tuning curve.
(iii) Describe the conditions under which two line segments may appear to belong to the same physical object despite not being connected, and how this reflects statistically optimal computations in the brain.

SECTION H Manufacturing, Management and Design

Answer not more than two questions from this section.

21 (a) List and describe four sources of invention that could be classified as technology push, and four sources of invention that could be classified as market pull. Provide one example for each of the sources.
(b) Describe the four generic ways in which a market may be segmented using a perceptual map.
(c) List the two characteristics of a well-defined market segment.
(d) Sketch three different perceptual maps that could be used to segment the markets for a low cost electrical bicycle. Explain your choice of axes used in each of your perceptual maps.

22 A team of university researchers have developed a novel device for detecting very small quantities of explosives in the air with very high levels of reliability. The researchers believe that this device could be used to improve the effectiveness of airport security systems.
(a) Describe who would be the potential stakeholders in the market for such an airport security device.
(b) Describe how interviews and observations could be used to understand the needs of customers and users of such a device.
(c) The device combines standard and bespoke components, and the reliability of the device is highly dependent on the quality of the components and the assembly process. It is anticipated that the market for this device will be around 100 units per year. Discuss the issues that would need to be considered in designing a production system for this device.

## 23

23 (a) List the four tests an invention must satisfy in order for it to be patentable.
(b) Explain why it is desirable to have more than one claim in a patent.
(c) Provide one strength and one weakness for each of the following types of business model:
(i) product;
(ii) service;
(iii) service enabled by a product;
(iv) product with consumables.
(d) The website of the UK semiconductor firm ARM plc states that its business model "[..] involves the designing and licensing of IP [intellectual property] rather than the manufacturing and selling of actual semiconductor chips".
(i) Describe what is meant by 'licensing of IP' in this context.
(ii) Discuss the relative merits of a business model based around licensing of IP in comparison to a business model that is focused upon the manufacturing and selling of semiconductor chips.

## END OF PAPER

## Data sheet - Soil Mechanics

## General definitions



Specific gravity of solid
$\mathrm{G}_{\mathrm{s}}$
Voids ratio
Specific volume
Water content
$\mathrm{e}=\mathrm{V}_{\mathrm{v}} / \mathrm{V}_{\mathrm{s}}$
$\mathrm{v}=\mathrm{V}_{\mathrm{t}} / \mathrm{V}_{\mathrm{s}}=1+\mathrm{e}$

Degree of saturation
$\mathrm{w}=\left(\mathrm{W}_{\mathrm{w}} / \mathrm{W}_{\mathrm{s}}\right)$
$S_{r}=V_{w} / V_{v}=\left(w_{s} / e\right)$
Unit weight of water
$\gamma_{\mathrm{w}}=9.81 \mathrm{kN} / \mathrm{m}^{3}$ (although we assume $10 \mathrm{kN} / \mathrm{m}^{3}$ )

Unit weight of soil
$\gamma=W_{\mathrm{t}} / \mathrm{V}_{\mathrm{t}}=\left(\frac{\mathrm{G}_{\mathrm{s}}+\mathrm{S}_{\mathrm{r}} \mathrm{e}}{1+\mathrm{e}}\right) \gamma_{\mathrm{w}}$
Buoyant (effective or submerged) unit weight $\quad \gamma^{\prime}=\gamma-\gamma_{w}=\left(\frac{\mathrm{G}_{\mathrm{s}}-1}{1+\mathrm{e}}\right) \gamma_{\mathrm{w}} \quad$ (soil saturated)

Unit weight of dry soil
$\gamma_{\mathrm{d}}=\mathrm{W}_{\mathrm{s}} / \mathrm{V}_{\mathrm{t}}=\left(\frac{\mathrm{G}_{\mathrm{s}}}{1+\mathrm{e}}\right) \gamma_{\mathrm{w}}$
$I_{d}=\frac{\left(e_{\text {max }}-e\right)}{\left(e_{\text {max }}-e_{\text {min }}\right)}$
where $e_{\text {max }}$ is the maximum voids ratio achievable in the quick tilt test (for sands), and $e_{\min }$ is the minimum voids ratio achievable by vibratory compaction (for sands).

## Classification of particle sizes

| Boulders | larger than |  | 200 mm |
| :---: | :---: | :---: | :---: |
| Cobbles | between | 200 mm and | 60 mm |
| Gravel | between | 60 mm and | 2 mm |
| Sand | between | 2 mm and | 0.06 mm |
| Silt | between | 0.06 mm and | 0.002 mm |
| Clay | smaller than | 0.002 mm (two |  |
| D | equivalent diameter of soil particle |  |  |
| $\mathrm{D}_{10}, \mathrm{D}_{60} \mathrm{etc}$ | particle size such that $10 \%$ (or $60 \%$ ) etc.) by weight of a soil sample is |  |  |

## Stress components

Principle of effective stress (saturated soil):

$$
\begin{array}{lll}
\text { total stress } & \sigma=\text { effective stress } \sigma^{\prime}+ & \text { pore water pressure } u \\
& \tau= & \tau^{\prime}+
\end{array}
$$

and

$$
\begin{aligned}
& \sigma_{\mathrm{v}}=\text { vertical stress } \\
& \sigma_{\mathrm{h}}=\text { horizontal stresss } \\
& \tau=\text { shear stress }
\end{aligned}
$$

## Strength of clays (undrained behaviour only)

Under constant volume (undrained) conditions only, the strength of clays can be characterised by the undrained shear strength $c_{u}$ which is mobilized when the shear stress $\tau=c_{u}$. This conforms to Tresca's criterion, and the active and passive total horizontal stresses, $\sigma_{\mathrm{a}}$ and $\sigma_{\mathrm{p}}$ respectively, are given by
$\sigma_{a}=\sigma_{v}-2 c_{u}$
$\sigma_{p}=\sigma_{v}+2 c_{u}$
where $\sigma_{\mathrm{v}}$ is the total vertical stress.


## Strength of sands

Mobilised angle of shearing $\phi^{\prime}$
where

$$
\tau=\sigma^{\prime} \tan \phi^{\prime}
$$



$$
\begin{aligned}
\sin \phi^{\prime} & =\mathrm{TS} / \mathrm{OS} \\
& =\frac{\left(\sigma_{1}^{\prime}-\sigma_{3}^{\prime}\right) / 2}{\left(\sigma_{1}^{\prime}+\sigma_{3}^{\prime}\right) / 2} \\
\therefore \phi^{\prime} & =\sin ^{-1}\left[\frac{\left(\sigma_{1}^{\prime} / \sigma_{3}^{\prime}\right)-1}{\left(\sigma_{1}^{\prime} / \sigma_{3}^{\prime}\right)+1}\right]
\end{aligned}
$$

Earth pressure coefficient K:

$$
\sigma_{\mathrm{h}}^{\prime}=K \sigma_{\mathrm{v}}^{\prime}
$$

Active pressure:

$$
\sigma_{\mathrm{v}}^{\prime}>\sigma_{\mathrm{h}}^{\prime}
$$

$$
\therefore \quad \sigma_{1}^{\prime}=\sigma_{v}^{\prime}
$$

$$
\sigma_{3}^{\prime}=\sigma_{\mathrm{h}}^{\prime}
$$

$$
\mathrm{K}_{\mathrm{a}}=\left(1-\sin \phi^{\prime} / 1+\sin \phi^{\prime}\right)
$$

Passive pressure: $\quad \sigma_{\mathrm{h}}^{\prime}>\sigma_{\mathrm{v}}^{\prime}$

$$
\therefore \sigma_{1}^{\prime}=\sigma_{\mathrm{h}}^{\prime}
$$

[We assume principal stresses

$$
\begin{aligned}
& \sigma_{3}^{\prime}=\sigma_{\mathrm{v}}^{\prime} \\
& \mathrm{K}_{\mathrm{p}}=\left(1+\sin \phi^{\prime}\right) /\left(1-\sin \phi^{\prime}\right)=\frac{1}{\mathrm{~K}_{\mathrm{a}}}
\end{aligned}
$$

are horizontal and vertical]

Angle of shearing resistance:
at peak strength $\phi_{\max }^{\prime}$ at $\left(\sigma_{1}^{\prime} / \sigma_{3}^{\prime}\right)_{\max }$
at critical state $\phi_{\text {crit }}^{\prime}$ after large strains.

## Sand strength data: friction hypothesis

In any shear test on soil, failure occurs when $\phi^{\prime}=\phi_{\max }^{\prime}$ and

$$
\phi_{\max }^{\prime}=\phi_{\text {crit }}^{\prime}+\phi_{\text {dilatancy }}^{\prime}
$$

where $\phi_{\text {crit }}^{\prime}$ is the ultimate angle of shearing resistance of a random aggregate which deforms at constant volume, so the dilatancy, which indicates an increase in volume during shearing, approaches zero ( $\phi_{\text {dilatancy }}^{\prime}->0$ ) as $\phi_{\text {max }}^{\prime}->\phi_{\text {crit }}^{\prime}$.
$\phi_{\text {crit }}^{\prime}$ is a function of the mineralogy, size, shape and distribution of particles. For a particular soil it is almost independent of initial conditions. Typical values ( $\pm 2^{\circ}$ ):

$$
\phi_{\text {crit }}^{\prime} \quad \phi_{\max }^{\prime}
$$

feldspar $\quad 40^{\circ}$
quartz $33^{\circ}$
$53^{\circ}$
( $\mathrm{I}_{\mathrm{d}}=1$, and mean effective stress $\mathrm{OS}<150 \mathrm{kPa}$ ) mica $25^{\circ}$

## Seepage

Excess pore water pressure

Head

$$
\mathrm{h}=\mathrm{u} / \gamma_{\mathrm{w}}
$$

Potential $\quad \overline{\mathrm{h}}=\mathrm{h}+\mathrm{z}$


Total pore water pressure head at $\mathrm{A}: \quad \mathrm{u}=\gamma_{\mathrm{w}} \mathrm{h}=\gamma_{\mathrm{w}}(\overline{\mathrm{h}}-(-\mathrm{z}))$
B: $\quad u+\Delta u=\gamma_{w}(h+\Delta h)=\gamma_{w}(\bar{h}+z+\Delta \bar{h}+\Delta z)$
[Excess pore water pressure at
A: $\overline{\mathbf{u}}=\gamma_{w} \overline{\mathrm{~h}}$
B: $\left.\overrightarrow{\mathrm{u}}+\Delta \overline{\mathrm{u}}=\gamma_{\mathrm{w}}(\overline{\mathrm{h}}+\Delta \overline{\mathrm{h}})\right]$

Hydraulic gradient $\mathrm{A}-\mathrm{B}$

$$
\mathrm{i}=-\frac{\Delta \overline{\mathrm{h}}}{\Delta \mathrm{~s}}=-\frac{\Delta \overline{\mathrm{u}}}{\gamma_{\mathrm{w}} \Delta \mathrm{~s}}
$$

Darcy's law $\quad \mathrm{v}=\mathrm{ki}$
$\mathrm{v}=$ average or superficial seepage velocity
$\mathrm{k}=$ coefficient of permeability

## Typical permeabilities

| $\mathrm{D}_{10}>10 \mathrm{~mm}$ | $:$ | non-laminar flow |
| :--- | :--- | :--- |
| $10 \mathrm{~mm}>\mathrm{D}_{10}>1 \mu \mathrm{~m}$ | $:$ | $\mathrm{k} \cong 0.01\left(\mathrm{D}_{10} \text { in } \mathrm{mm}\right)^{2} \mathrm{~m} / \mathrm{s}$ |
| clays | $:$ | $\mathrm{k} \cong 10^{-9}$ to $10^{-11} \mathrm{~m} / \mathrm{s}$ |

## Design of reinforced concrete

Data sheet for use in Part IB Civil Engineering Elective Course.


## Design Stresses

Cube strength for concrete $f_{c u}$. At failure in bending, stress in concrete $=0.4 f_{c u}$ over whole area of concrete in compression.

Tensile yield stress of steel $f_{y}$. At failure in bending, stress in bars in tension $=$ $0.87 f_{y}$, stress in bars in compression $=0.75 \mathrm{f}_{\mathrm{y}}$.

## Design Equations

Moment capacity of singly reinforced beam

$$
\begin{aligned}
& M \leq 0.15 f_{c u} b d^{2} \\
& M=0.87 f_{y} A_{s} d(1-x / 2) \\
& x=2.175\left(f_{y} / f_{c u}\right) \cdot\left(A_{s} / b d\right) \quad(\leq 0.5 \text { to avoid over reinforcement })
\end{aligned}
$$

## Moment capacity of doubly reinforced beam

$$
\begin{aligned}
& M=0.15 f_{c u} b d^{2}+0.75 f_{y} A_{s}^{\prime}\left(d-d^{\prime}\right) \\
& 0.87 f_{y} A_{s}=0.75 f_{y} A_{s}^{\prime}+0.2 f_{c u} b d
\end{aligned}
$$

## Shear capacity of all beams

Total shear capacity $V=\left(v_{c}+v_{s}\right) b d$
Where, $\mathrm{v}_{\mathrm{c}}=0.68\left(100 \mathrm{~A}_{\mathrm{s}} / \mathrm{bd}\right)^{0.33} \cdot(400 / \mathrm{d})^{0.25} \quad\left(\mathrm{~N} / \mathrm{mm}^{2}\right)$
and $v_{s}=0.87 f_{y} \mathrm{~A}_{\mathrm{sq}} /(\mathrm{bs})$
in which $s=$ shear link spacing, $A_{s q}$ is total area of all shear bars in a link and $A_{s}$ is the total area of effective longitudinal tension steel at the section.

Standard bar sizes

| Diameter (mm) | 6 | 8 | 10 | 12 | 16 | 20 | 25 | 32 | 40 |
| :--- | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: |
| 50 | 28 | 50 | 78 | 113 | 201 | 314 | 491 | 804 | 1256 |
| Area $\left(\mathrm{mm}^{2}\right)$ <br> 1963 |  |  |  |  |  |  |  |  |  |

## Available steel types

Deformed high yield steel

$$
\begin{aligned}
& \mathrm{f}_{\mathrm{y}}=460 \mathrm{~N} / \mathrm{mm}^{2} \\
& \mathrm{f}_{\mathrm{y}}=250 \mathrm{~N} / \mathrm{mm}^{2}
\end{aligned}
$$

Plain mild steel

Lap and anchorage lengths 40 bar diameters
Density of reinforced concrete: $24 \mathrm{kN} / \mathrm{m}^{3}$
Reinforcement areas per metre width

|  | Spacing of bars (mm) |  |  |  |  |  |  |  |  |  |
| :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- |
|  | 75 | 100 | 125 | 150 | 175 | 200 | 225 | 250 | 275 | 300 |
| Bar Dia. (mm) |  |  |  |  |  |  |  |  |  |  |
| 6 | 377 | 283 | 226 | 189 | 162 | 142 | 126 | 113 | 103 | 94.3 |
| 8 | 671 | 503 | 402 | 335 | 287 | 252 | 224 | 201 | 183 | 168 |
| 10 | 1050 | 785 | 628 | 523 | 449 | 393 | 349 | 314 | 285 | 262 |
| 12 | 1510 | 1130 | 905 | 754 | 646 | 566 | 503 | 452 | 411 | 377 |
| 16 | 2680 | 2010 | 1610 | 1340 | 1150 | 1010 | 894 | 804 | 731 | 670 |
| 20 | 4190 | 3140 | 2510 | 2090 | 1800 | 1570 | 1400 | 1260 | 1140 | 1050 |
| 25 | 6550 | 4910 | 3930 | 3270 | 2810 | 2450 | 2180 | 1960 | 1790 | 1640 |
| 32 | 10700 | 8040 | 6430 | 5360 | 4600 | 4020 | 3570 | 3220 | 2920 | 2680 |
| 40 | 16800 | 12600 | 10100 | 8380 | 7180 | 6280 | 5580 | 5030 | 4570 | 4190 |
| 50 | 26200 | 19600 | 15700 | 13100 | 11200 | 9820 | 8730 | 7850 | 7140 | 6540 |
|  |  |  |  |  |  |  |  |  |  |  |
| Areas calculated to 3 significant figures according to B.S.I recommendations |  |  |  |  |  |  |  |  |  |  |

April 2010

## Part IB Data Sheet: Electrical Engineering Elective Transistor Design Summary Sheet

## Gauss's Theorem

$\varepsilon_{0} \varepsilon_{\mathrm{r}} \mathrm{E}_{1}-\varepsilon_{0} \varepsilon_{\mathrm{r}} \mathrm{E}_{2}=$ charge per unit area enclosed between upper surface 1 and lower surface 2 .

## FET Design Summary

- $\quad \tau_{\mathrm{t}} \rightarrow$ switching time as 1st approx. (scattering limited transit time).
- $\quad \tau_{\text {eff }}=\tau_{\mathrm{t}}+\mathrm{R}_{\text {load }} \mathrm{C}_{\text {eff(output) }} \rightarrow$ switching time as 2 nd approx.
- $L=v_{s} \tau_{t}$ (source-drain spacing).
- $\quad I_{\text {sat }}=e \mathrm{Nv}_{\mathrm{s}} W \mathrm{~d}_{\mathrm{s}}=\mathrm{eNWL} \mathrm{d}_{\mathrm{s}} / \tau_{\mathrm{t}}$
- Aspect ratio $W / L$ (technology?).
- ( $1 / 2$ ) $\mathrm{eN}\left(\mathrm{d}_{\mathrm{s}}\right)^{2} / \varepsilon_{0} \varepsilon_{\mathrm{r}}=$ (Max Gate Voltage)
- $\quad E_{\text {peak }}^{\prime}=\mathrm{eNd}_{\mathrm{s}} / \varepsilon_{\mathrm{o}} \varepsilon_{\mathrm{r}}<\mathrm{E}_{\text {breakdown }}$
- Minimum Drain Source Voltage $\sim \mathrm{E}_{s} \mathrm{~L} \quad$ ( $\mathrm{E}_{\mathrm{s}}$ is the field required to reach limiting velocities).


## Mutual Conductance

## $\mathrm{g}_{\mathrm{mo}} \sim \mathrm{I}_{\text {sat }} / \mathrm{V}_{\text {gate(max) }}$

Mutual conductance reduces with frequency as $\mathrm{g}_{\mathrm{m}}(\omega) \approx \mathrm{g}_{\mathrm{mo}} /\left(1+\mathrm{j} \omega \tau_{\mathrm{t}}\right)$;

$$
\begin{aligned}
\mathrm{v}_{\text {out }}=\mathrm{g}_{\mathrm{m}}(\omega) \mathrm{R}\left(1+j \omega R C_{\text {eff(out })}\right) & \approx g_{\mathrm{mo}} R /\left[1+j \omega\left(\tau_{\mathrm{t}}+R C_{\text {eff(out })}\right)\right. \\
& =g_{\mathrm{mo}} R /\left[1+j \omega \tau_{\text {eff }}\right]
\end{aligned}
$$

## Capacitances for FET

Parallel Plate Capacitance: $\varepsilon_{0} \varepsilon_{\mathrm{r}}$ Area/spacing
Used for rough estimates of parasitic capacitance.
Effective Capacitances for FET

$$
\begin{aligned}
& \mathrm{C}_{\text {eff(out }} \rightarrow \mathrm{C}_{\text {gate/drain }}+\mathrm{C}_{\text {drain/source }}+\mathrm{C}_{\text {load; }} \\
& \mathrm{C}_{\text {eff(in) }} \rightarrow \mathrm{MC}_{\text {gate/drain }}+\mathrm{C}_{\text {gate/source (proximity) }}+\mathrm{C}_{\text {gate/source (electronic) }} ; \\
& \mathrm{C}_{\text {electronic }}=\mathrm{g}_{\mathrm{mo}} \tau_{\mathrm{t}} \quad ; \mathrm{M}=\left(1+\mathrm{g}_{\mathrm{mo}} \mathrm{R}_{\text {load }}\right)
\end{aligned}
$$

## Time Constants for FET

$\mu=\mathrm{e} \tau / \mathrm{m}^{*}$ relates mean free time $\tau$ and mobility.
Transit time $\tau_{\mathrm{t}}$ over distance L and scattering limited velocity $\mathrm{v}_{\mathrm{s}}$ are related by $\tau_{\mathrm{t}}=L / v_{s}$.
$v_{\text {out }} \approx g_{\text {mo }} R /\left[1+j \omega \tau_{\text {eff }}\right]=g_{m o} R /\left[1+j \omega /\left(2 \pi f_{t}\right)\right]$
$1 /\left(2 \pi f_{\tau}\right)=\tau_{t}+R C_{\text {eff }}$ (out) $=\tau_{\text {eff }} \quad$ The transition frequency is $f_{t}$.
$10 \%$ to $90 \%$ rise time is $T=2.2 \tau_{\text {eff }}=(2.2 / 2 \pi)\left(1 / f_{t}\right)=0.35 / f_{t}$.

## Electrical Engineering Elective: <br> Tunnel Barrier Design Summary Sheet

## Schrodinger's Equation

Complex Wave $\quad \Psi=A \exp (-\mathrm{j} 2 \pi \mathrm{ft}+\mathrm{j} 2 \pi \mathrm{x} / \lambda)=\mathrm{A} \exp (-\mathrm{j} \omega \mathrm{t}) \exp (\mathrm{jkx})$;
<momentum> $\quad \Psi=\mathrm{p} \Psi=(\mathrm{h} / \lambda) \Psi=-\mathrm{j}(\mathrm{h} / 2 \pi) \partial \Psi / \partial \mathrm{x} ;$
$<$ Total energy> $\Psi=E \Psi=\mathrm{h} \Psi=\mathrm{j}(\mathrm{h} / 2 \pi) \partial \Psi / \partial \mathrm{t}$
$(\mathrm{h} / 2 \pi) \rightarrow \hbar ; \mathrm{h}=6.625 \times 10^{-34} \mathrm{~J} / \mathrm{s}$.
Schrodinger's equation:-
$E \Psi=(1 / 2 \mathrm{~m})[-\mathrm{j} \hbar \partial / \partial \mathrm{x}]^{2} \Psi+\mathrm{e} \not \mathscr{C}^{\Psi}$

## Tunneling (Rectangular barrier $\varepsilon \phi$ )

Propagating waves outside barrier with incident kinetic energy $U_{\text {incident }}=\left(\hbar \mathrm{k}_{\mathrm{i}}\right)^{2} / 2 \mathrm{~m}^{*}$
Evanescent waves inside barrier: $-(\hbar \mathrm{k})^{2} / 2 \mathrm{~m}^{*}=\left(\hbar \mathrm{k}_{\mathrm{i}}\right)^{2} / 2 \mathrm{~m}^{*}-\mathrm{e} \phi$

## Technology Design Summary

## Diffusion

Constant Surface Concentration:

$$
C(x, t)=C_{s} \operatorname{erfc}\left[\frac{x}{2(D t)^{1 / 2}}\right]
$$

Constant Total Dopant:

$$
C(x, t)=\frac{S}{(\pi D t)^{1 / 2}} \exp \left[\frac{-x^{2}}{4 D t}\right]
$$

