

ENGINEERING TRIPOS

Datasheet: Environmental Engineering I Data Sheet

3D5 ENVIRONMENTAL ENGINEERING I.

Q1. a) Distrib %'s: 4, 15, 30, 25, 15, 11 for 3hr intervals  
 Cumulative 4, 19, 49, 74, 89, 100

Plot at 1/2 times: t = 1.5, 4.5, 7.5, 10.5, 13.5, 16.5 hrs.

Connect by smooth curve (see graph)

Lag by 1 hour

Subtract. i.e. measure difference between S-curves at new 1/2 time pts,  
 (i.e. t = 0.5, 1.5, 2.5, ... hrs ...)

From graph, these are

1, 2, 3, 4, 6, 8, 10, 12, 10, 9, 8, 7, 6, 5, 4, 3, 2

check  $\Sigma = 100$  ✓

Connect by smooth curve → hydrograph shape.

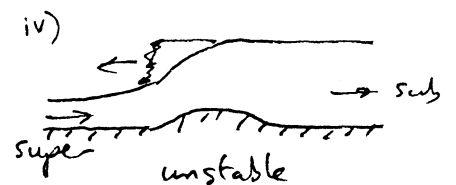
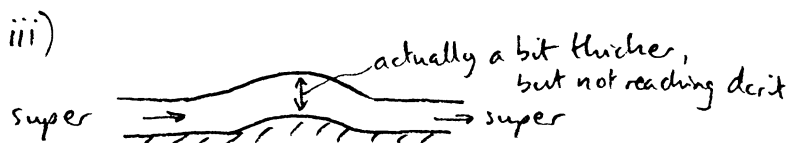
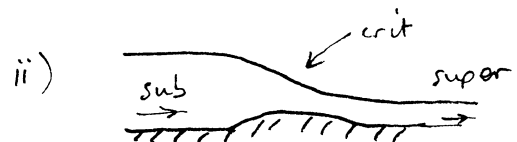
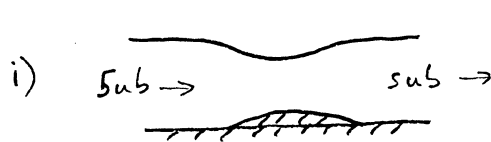
Because these are at 1/2 time points, then by piecewise linear approx these are the distribution percentages.

b) Subcritical  $Fr < 1$  } Froude No  $Fr \equiv \frac{U}{\sqrt{gd}} = \frac{U}{c}$   
 Critical  $Fr = 1$  }  
 Supercritical  $Fr > 1$  }

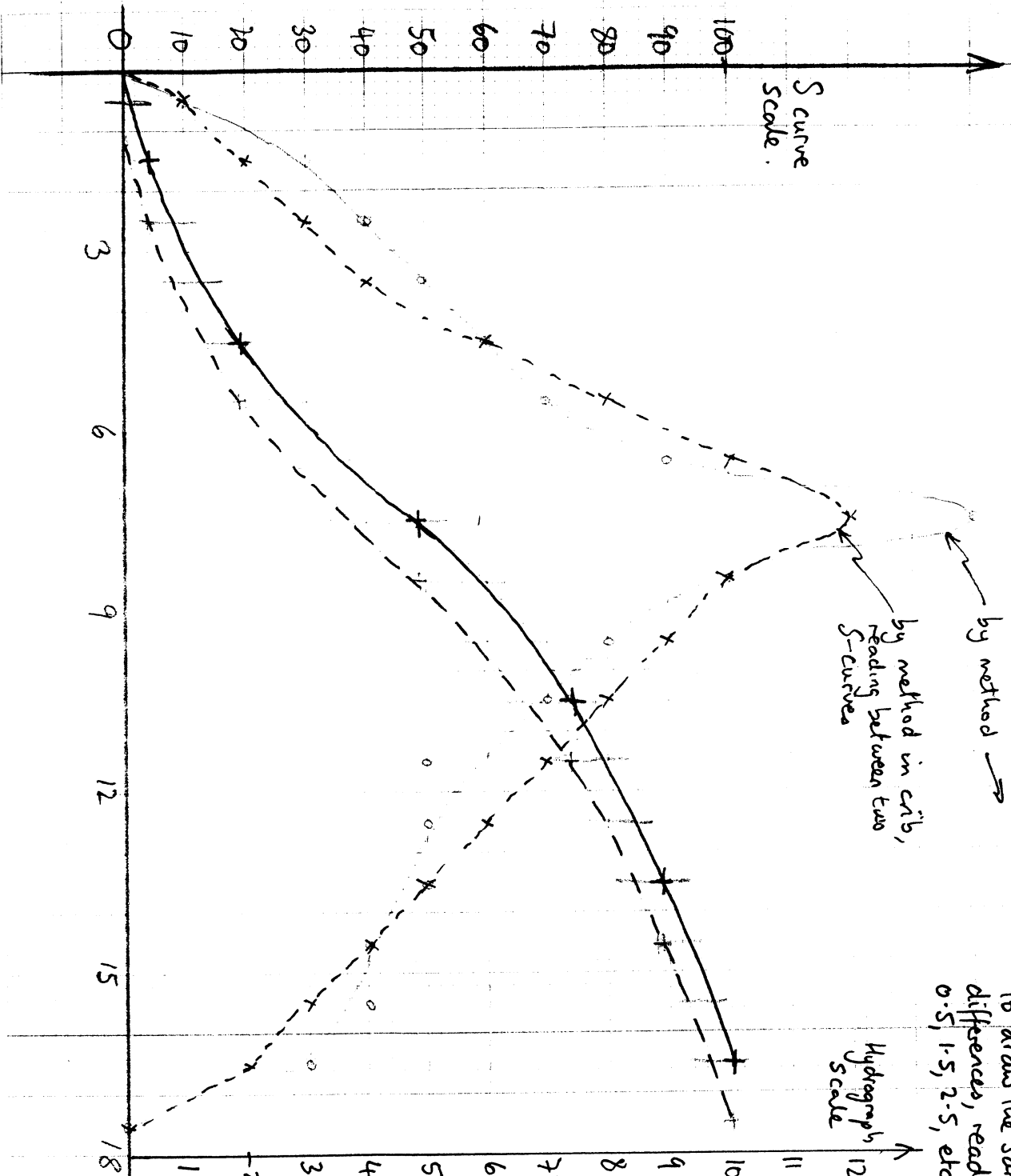
where U = velocity of flow relative to bed

c = velocity of propagation of small disturbances relative to flow

- hence, for example, small disturbances cannot travel upstream (relative to bed) for supercritical flow.



- hydraulic jump travels upstream until it stabilises.



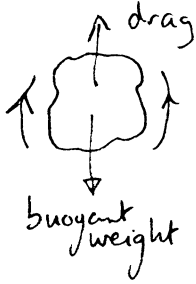
Alternative Procedure: Rather than trying to draw the same S-curve twice and read off the differences, read the y-coords of the S-curve at 0.5, 1.5, 2.5, etc, then lag and subtract numerically.

t	S	S lag	Diff
0.5	1	1	1
1.5	4	4	3
2.5	8	8	4
3.5	13	13	5
4.5	19	19	6
5.5	26	26	7
6.5	35	35	9
7.5	49	49	14
8.5	59	59	10
9.5	67	67	8
10.5	74	74	7
11.5	79	79	5
12.5	84	84	5
13.5	89	89	5
14.5	93	93	4
15.5	97	97	4
16.5	100	100	3
17.5	100	100	0

$\Sigma = 100$  ✓  
(guaranteed by this method).

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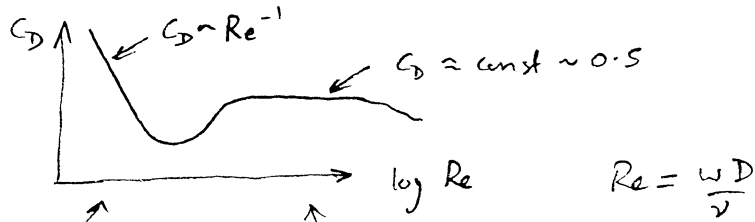
Q1 (c). Terminal velocity. No accel. Equate forces.



$$\frac{\pi D^3}{6} (\rho_s - \rho) g = C_D \frac{1}{2} \rho w^2 \frac{\pi D^2}{4} \quad (1)$$

submerged weight drag.

but  $C_D$



Small particles

large particles

Small particles: viscous drag dominates. no flow separation  
Low  $Re$   $C_D \propto \frac{1}{Re} \propto \frac{1}{wD}$



$$\therefore D^3 \propto \frac{1}{wD} w^2 D^2 \quad \text{from (1)}$$

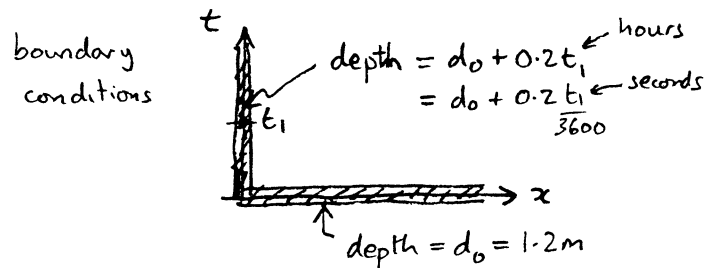
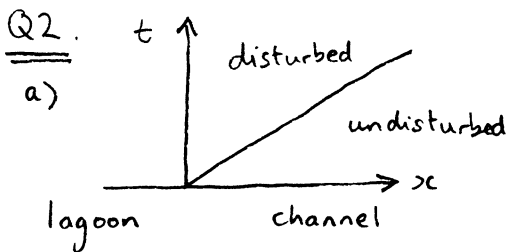
$$\therefore w \propto D^2 \quad \text{QED.}$$

Large particles: form (or pressure) drag dominates. flow separation  
High  $Re$   $C_D \approx \text{const}$



$$\therefore D^3 \propto w^2 D^2 \quad \text{from (1)}$$

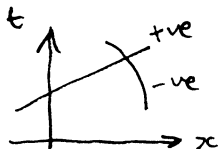
$$\therefore w \propto D^{1/2}$$



From data book

$$u - 2c = k_1 \text{ const on any -ve charac.} \quad (1)$$

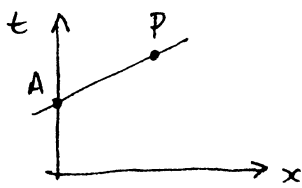
$$u + 2c = k_2 \text{ const on any +ve charac.} \quad (2)$$



Any -ve charac can be traced back to undisturbed region where  $u = u_0$  and  $c = c_0 = \sqrt{gd_0}$

$$\therefore u - 2c = k_1 = u_0 - 2c_0 \text{ everywhere in RH quadrant.}$$

Q2(a) cont'd.



On +ve charac AP, for any P on AP

$$u_A + 2c_A = u_P + 2c_P = k_2 \quad \text{from (2)}$$

$$\text{but } u_P - 2c_P = k_1 \quad \text{from (1)}$$

$$\text{Adding } \Rightarrow u_P = (k_1 + k_2)/2 = \text{const on AP}$$

$$\text{Subtracting } c_P = (k_2 - k_1)/4 = \text{const on AP}$$

$$\therefore \frac{1}{\text{slope of +ve charac}} = \left( \frac{dx}{dt} \right)_{AP} \text{ at P} = u_P + c_P = \text{const} + \text{const} = \text{const}$$

\(\therefore\) +ve characs are STRAIGHT lines.

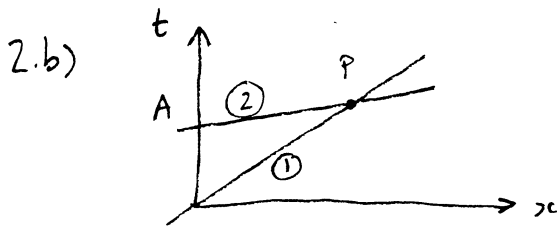
$$\begin{aligned} \text{So } \left( \frac{dx}{dt} \right)_{AP} \text{ at P} &= u_P + c_P = u_P - 2c_P + 3c_P \\ &= u_0 - 2c_0 + 3c_P \quad \text{from (1)} \\ &= u_0 - 2\sqrt{gd_0} + 3c_P \end{aligned}$$

$$\text{But } c_P = \text{const on AP} = c_A = \sqrt{gd_A}$$

$$\therefore \frac{1}{\text{+ve slope}} = u_0 - 2\sqrt{gd_0} + 3\sqrt{gd_A}$$

$$\text{slope} = \frac{dt}{dx} = \frac{1}{dx/dt} = \left[ u_0 - 2\sqrt{gd_0} + 3\sqrt{gd_A} \right]^{-1}$$

$$\text{with } d_A = d_0 + 0.2 \frac{t_1}{3600} = d_0 + \frac{t_1}{18 \times 10^3} \quad \underline{\underline{\text{QED}}}$$



Undisturbed bdy (1)

$$\frac{dx}{dt} = u_0 + \sqrt{gd_0}$$

$$u_0 = -0.5 \text{ m/s}$$

$$\sqrt{gd_0} = \sqrt{9.81(1.2)} = 3.43 \text{ m/s}$$

$$\therefore \frac{dx}{dt} = -0.5 + 3.43 = 2.93 \text{ m/s}$$

$$\left( \frac{dt}{dx} \right)_{(1)} = \frac{1}{2.93} = \underline{\underline{0.341 \text{ s/m}}}$$

On AP

$$d_A = 1.2 + 0.2 = 1.4 \text{ m}$$

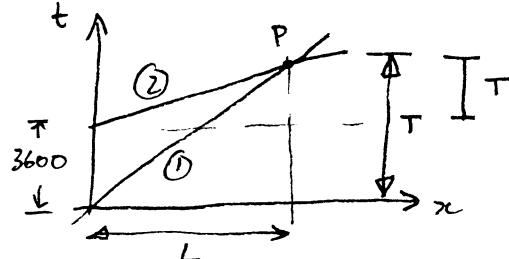
$$\sqrt{gd_A} = \sqrt{9.81(1.4)} = 3.71 \text{ m/s}$$

$$\frac{dx}{dt} = -0.5 - 2(3.43) + 3(3.71) = 3.76 \text{ m/s}$$

$$\left( \frac{dt}{dx} \right)_{(2)} = 0.266 \text{ s/m}$$

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Q2(b) cont'd



$$T = 3600 + T_1$$

$$T = L \left( \frac{dt}{dx} \right)_1 \quad T_1 = L \left( \frac{dt}{dx} \right)_2 \quad \therefore L(0.341) = 3600 + L(0.266)$$

$$\therefore L = 3600 / (0.341 - 0.266) = 48 \times 10^3 \text{ m}$$

$$T = L(0.341) = 16.4 \times 10^3 \text{ seconds} = \underline{\underline{4.55 \text{ hrs}}}$$

At intersection point (and beyond) Method of Characteristics no longer applies (- it predicts two values of depth at P). Will get a hydraulic jump, bore, surge here (if one has not occurred sooner, (which it probably has)). Momentum and continuity considerations may help track further evolution of the surge as it progresses into the undisturbed region.

Q3  $\frac{1}{\sqrt{\lambda}} = -2 \log \left[ \frac{k}{3.7D} + \frac{2.51}{Re \sqrt{\lambda}} \right]$

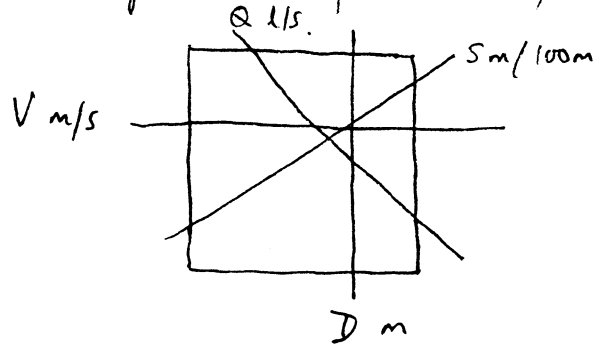
Darcy:  $h_f = \frac{\lambda L v^2}{2gD}$   
 $\Rightarrow \sqrt{\lambda} = \frac{\sqrt{2gDS}}{v}$  with  $S = \frac{h_f}{L}$  hydraulic grad.

$$\therefore \frac{v}{\sqrt{2gDS}} = -2 \log \left[ \frac{k}{3.7D} + \frac{2.51 v}{Re \sqrt{2gDS}} \right] \quad Re = vD/\nu$$

$$v = -2 \sqrt{2gDS} \log \left[ \frac{k}{3.7D} + \frac{2.51 \nu}{D \sqrt{2gDS}} \right]$$

Five variables  $v, D, S, \nu, k$  (also  $Q = \frac{\pi D^2}{4} \cdot v$ )

Assume Temp = 15°C  $\Rightarrow$  kinematic viscosity constant.  
 Create separate chart for each  $k$ , relating  $v, D, S$  (and  $Q$ ).

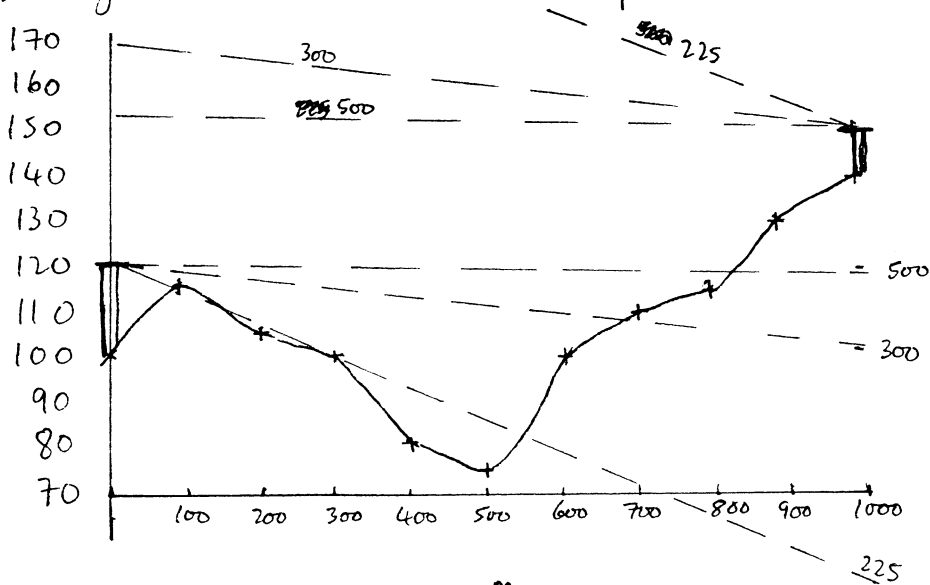


If any two are known, the other two may be found from intersection point.

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Q3 cont'd.

b) Longitudinal section. Draw at compressed horiz. scale.



Available pipes (dia, mm)	Head loss / 100 m at $Q = 200 \text{ l/s}$
<del>500</del> 225 mm $\phi$	7 m / 100 m
<del>300</del> 300 mm $\phi$	1.8 m / 100 m
<del>225</del> 500 mm $\phi$	0.15 m / 100 m

Draw on hydraulic grade lines from either end.

Difference between hydraulic grade lines =  $(150 \text{ m} - 120 \text{ m}) + \text{head loss over 1 km}$

$$500 \text{ mm} \Rightarrow 30 + 1.5 = 31.5 \text{ m}$$

$$300 \text{ mm} \Rightarrow 30 + 18 = 48 \text{ m}$$

$$225 \text{ mm} \Rightarrow 30 + 70 = 100 \text{ m}$$

Booster pump gives 200 l/s for 59 m head, so pick 300 mm dia pipe, and put pump at 600 m point.

Reason for 300 mm  $\phi$ :

i) should really draw pipeline characteristic on  $(H, Q)$  diagram for each diameter, and then pick the diam. whose characteristic ~~etc~~ crosses the pump characteristic nearest the required flow rate.

ii) can see 225 mm  $\phi$  gives negative pressures near 100 m mark - unless one pumps from right next to treatment works - but that would give huge pressures ( $\sim 100 \text{ m}$  head) in the pipeline, and 100 m head seems to be beyond capabilities of the pump anyway.

iii) The 500 mm  $\phi$  is an option. Could have gravity flow to 800 m then a 31.5 m lift by the pump.

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Q3 cont'd.

→ But

it seems more sensible to have a 300 mm  $\phi$  pipe to flow under gravity to near the 600 m mark, and then have a 48 m lift there.

Why 600 m?  $\begin{cases} \text{if pick 500 m} \rightarrow \text{large pressures beyond pump} \\ \text{if pick 700 m} \rightarrow \text{negative pressures before pump.} \end{cases}$

So, pick pump speed to give 200 l/s at 48 m head.

i) Pump characteristic is plotted on graph paper

ii) Pipeline characteristic is sketched on graph paper.

There is no need to calculate it. We already know that

- it passes through point O ( $Q=0$  with static head = 30 m)

- it passes through req'd point A ( $Q=200$  l/s with head = 48 m

(= 30 m + 18 m)  
static friction

Now pick any point  $Q_1, H_1$  on pump characteristic at  $N_1 = 1000$  rpm.

If pump runs at  $N_2$  r.p.m. then the points

$$Q_2 = \frac{N_2}{N_1} Q_1 \quad \text{and} \quad H_2 = \frac{N_2^2}{N_1^2} H_1 \quad \text{will lie on pump characteristic curve for } N_2 \text{ r.p.m.}$$

So, want to find  $N_2$  such that  $(Q_2, H_2) = \text{point A} = (200 \text{ l/s}, 48 \text{ m})$ .

Starting at any  $Q_1, H_1$  on  $N_1$  curve, and letting  $N_2$  be a variable defines a curve  $(Q_2, H_2) = (Q_2(N_2), H_2(N_2))$

$$= \left( \left( \frac{N_2}{N_1} \right) Q_1, \left( \frac{N_2^2}{N_1^2} \right) H_1 \right)$$

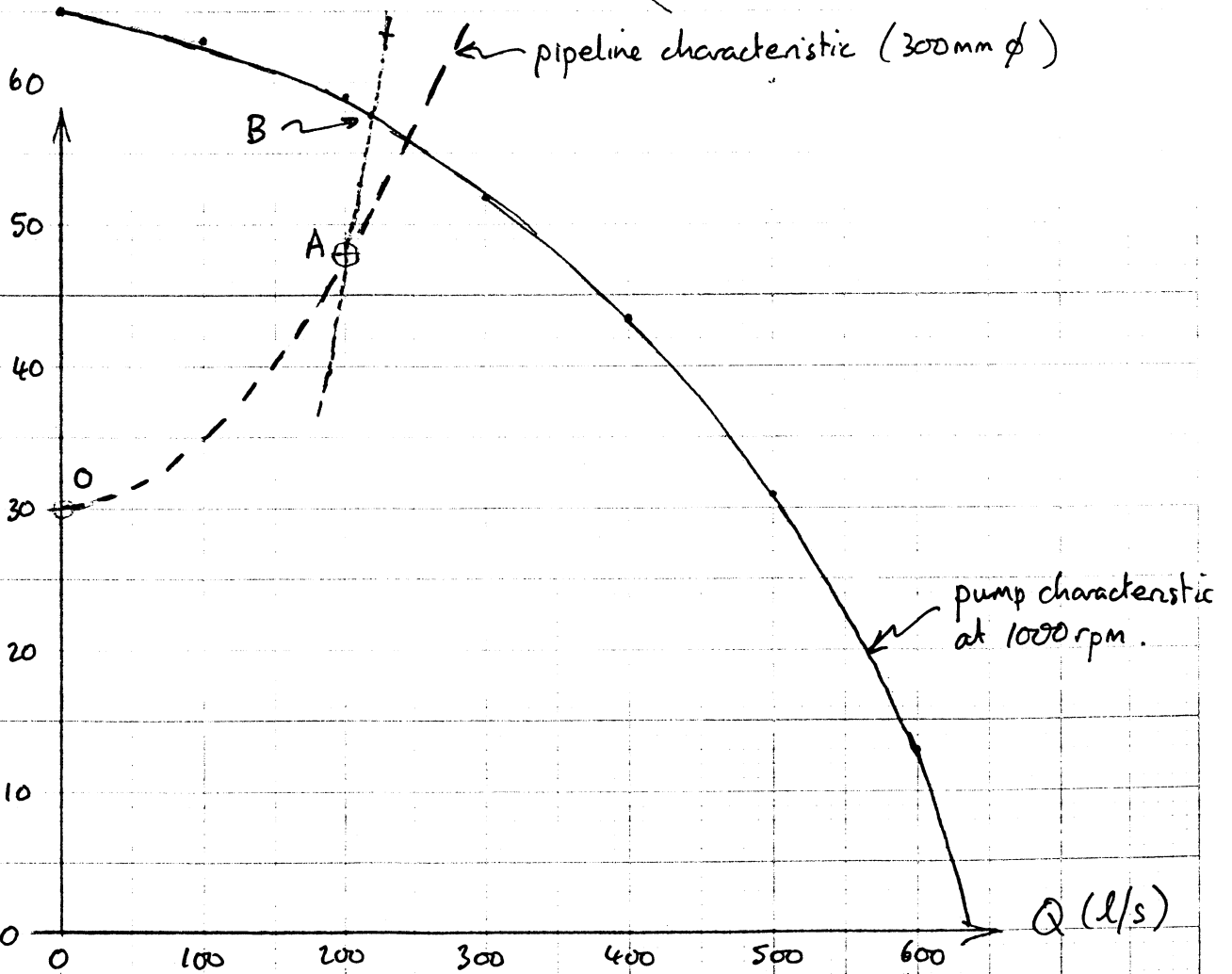
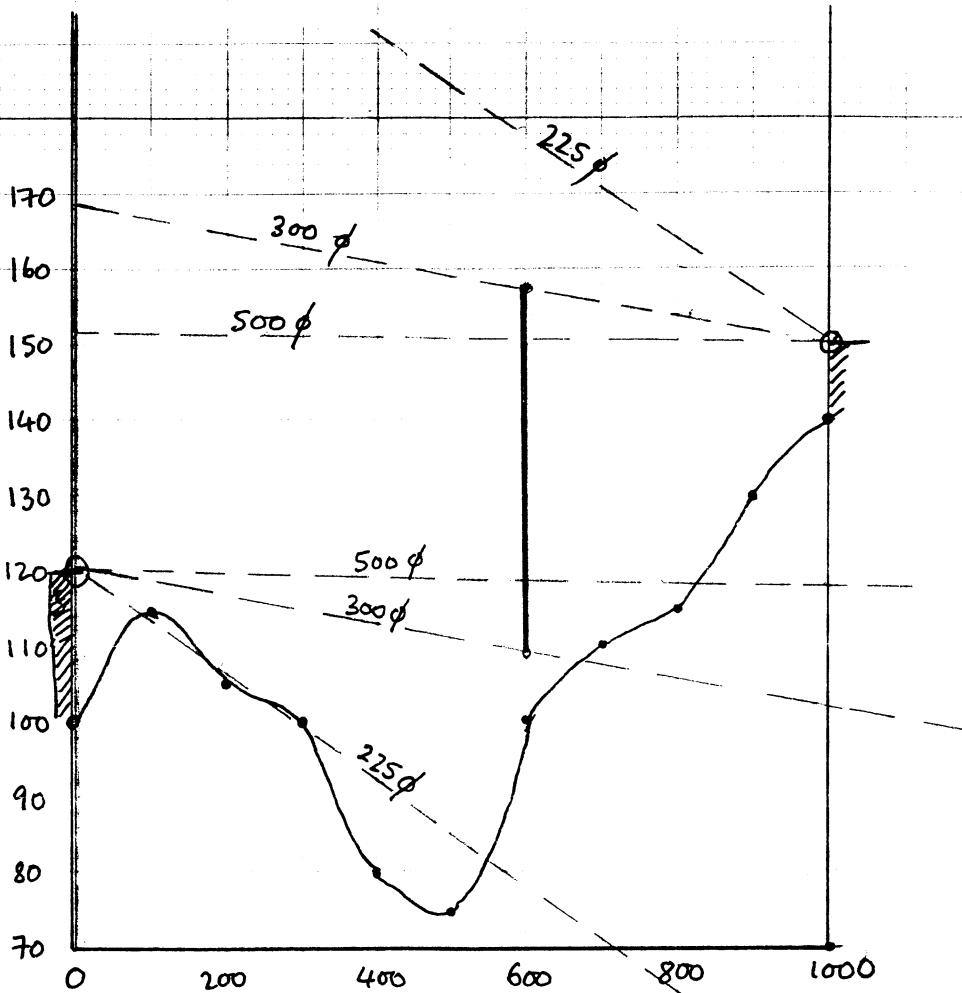
$$= (x, y)$$

$$\text{So } x^2 = \frac{N_2^2}{N_1^2} Q_1^2 = \frac{y}{H_1} Q_1^2 \quad \therefore y = \left( \frac{H_1}{Q_1^2} \right) x^2$$

$\left( \frac{H_1}{Q_1^2} \right) \leftarrow \text{known, if } Q_1, H_1 \text{ are known.}$

i.e. the set of points "generated" by  $Q_1, H_1$  lie on a parabola. We want that parabola to pass through A.

$$y = kx^2 \quad \text{with } y = 48 \text{ m}, \quad x = 200 \text{ l/s}$$
$$\therefore k = \frac{48}{(200)^2} = 0.0012$$





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Q3 b) cont'd.

Draw the parabola  $H = kQ^2$  above the point A until it intersects pump 1000 rpm curve.

$Q$	200	210	220	230
$kQ^2 = H$	48	52.9	58.0	63.5

This crosses  $N_1 = 1000$  rpm pump curve at B, where  $Q_1 \approx 220$  l/s,  $H_1 = 58.0$  m.

Can thus solve for  $N_2 = N_1 \frac{Q_2}{Q_1} = (1000) \left( \frac{200}{220} \right) = 909.1$  rpm

Check  $N_2 = N_1 \sqrt{\frac{H_2}{H_1}} = (1000) \left( \frac{48}{58} \right)^{1/2} = 909.7$  rpm ✓

∴ Run pump at 910 rpm.

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Q4. a)  $u_* = \left(\frac{\tau_0}{\rho}\right)^{1/2}$      $\tau_0 = \rho g R S$      $u_* = \sqrt{g R S}$

Wide channel  $R \approx d \Rightarrow u_* \approx \sqrt{g d S} = \sqrt{9.81(1)(0.001)} = 0.1 \text{ m/s}$ .

$\frac{u_* k_s}{\nu} = \frac{(0.1)(0.01)}{10^{-6}} = 1000 > 70 \therefore$  hydraulically rough.

b) Use  $u = u_* 2.5 \log\left(\frac{30.2 y}{k_s}\right)$  for all heights (although only really applies to overlap layer).

y	0.01	0.1	0.2	0.5	1.0 (m)
u	0.85	1.43	1.6	1.83	2.0 (m/s)

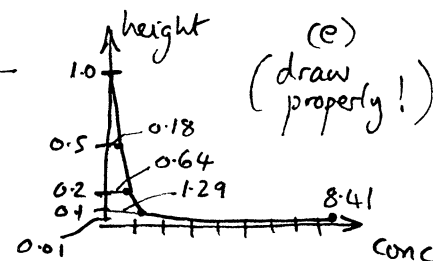
c)  $W = (56 \times 10^4) (0.00021)^2 (1.65) = 0.0407$

$\frac{W}{K u_*} = \frac{0.0407}{(0.4)(0.1)} = 1.02 \approx 1.0$

d)  $\frac{C}{C_a} = \left[ \left(\frac{d-y}{y}\right) \left(\frac{a}{d-a}\right) \right]^i$      $C_a = 9.9 \text{ kg/m}^3$  at  $a = 0.01 \text{ m}$

$\therefore C = (9.9) \left(\frac{0.01}{0.99}\right) \left(\frac{1-y}{y}\right) = 0.1 \left(\frac{1-y}{y}\right)$

y	0.01	0.1	0.2	0.5	1.0
C	9.9	0.9	0.4	0.1	0
u	0.85	1.43	1.6	1.83	2.0
uC	8.41	1.29	0.64	0.18	0



(f) i)  $\int_{0.01}^1 u C dy = 0.5 \left(\frac{0.183}{2}\right) + 0.3 \left(\frac{0.64 + 0.183}{2}\right) + 0.1 \left(\frac{1.29 + 0.64}{2}\right) + \left(\frac{8.41 + 1.29}{2}\right) (0.09) = \underline{\underline{0.7 \text{ kg/ms}}}$

ii)  $\int_b^d u C dy = 11.6 u_* C_b b [I_1 \log Ad + I_2]$

$\frac{b}{d} = \frac{0.01}{1} = 0.01$      $I_1 = 0.788$      $I_2 = 2.107$

$A = \frac{30.2}{k_s} = 3020 \Rightarrow \int_b^d u C dy = (11.6)(0.1)(9.9)(0.01) [0.788 \log 3020 + 2.107] = \underline{\underline{0.97 \text{ kg m}^{-1} \text{ s}^{-1}}}$