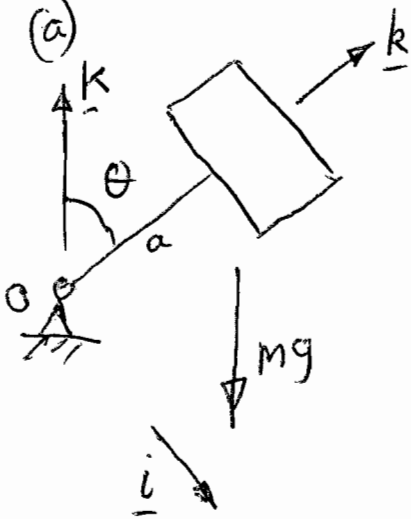


1. (a)



The only couple acting is due to  $mg$

$$\underline{Q} = mga \sin \theta \underline{j} \quad (1)$$

Gyro equation #3

$$Q_3 = C \dot{\omega}_3$$

$$\text{but } (1) \rightarrow Q_3 = 0$$

$$\therefore \dot{\omega}_3 = 0$$

$$\therefore \omega_3 = \text{const}$$

$$(b) \quad \underline{Q} = \dot{\underline{h}} \quad \therefore \underline{\hat{Q}} \cdot \underline{k} = \dot{\underline{h}} \cdot \underline{k}$$

$$\text{and from } (1) \quad \underline{Q} \cdot \underline{k} = 0 \quad \therefore \dot{\underline{h}} \cdot \underline{k} = 0$$

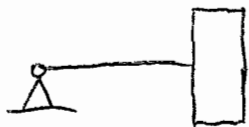
$$\text{but } \underline{k} \text{ is a constant vector} \quad \therefore \frac{d}{dt} (\underline{h} \cdot \underline{k}) = 0$$

$$\therefore \underline{h} \cdot \underline{k} = \text{const}$$

$$\text{Initial value} = C\omega$$

(c)

Gyro equation #2



$$Q_2 = A \dot{\Omega}_2 + (A \Omega_3 - C \omega_3) \Omega_1$$

$$\text{where } \Omega_1 = -\dot{\phi} \sin \theta$$

$$\Omega_2 = \dot{\theta}$$

$$\Omega_3 = \dot{\phi} \cos \theta$$

$$\text{Steady state precession} \quad \therefore \dot{\theta} = 0$$

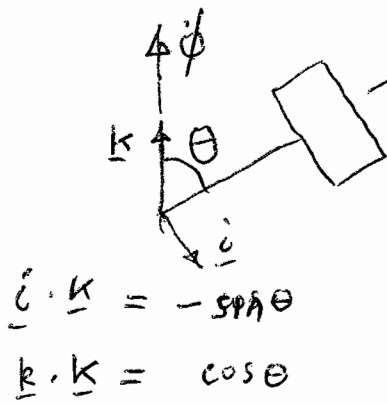
$$\therefore mga \sin \theta = -(A \dot{\phi} \cos \theta - C \omega_3) \dot{\phi} \sin \theta \quad (2)$$

$$\text{and put } \theta = \frac{\pi}{2} \quad \therefore mga = C \omega_3 \dot{\phi}$$

$$\therefore \dot{\phi}_{\text{final}} = \frac{mga}{C \omega_3}$$

ans

1(c) cont. Now prove  $\Theta = \frac{\pi}{2}$  by conservation of  $\underline{h} \cdot \underline{k}$



$$\underline{h} = A R_1 \underline{i} + A R_2 \underline{j} + C \omega_3 \underline{k}$$

$$\underline{h} \cdot \underline{k} = A R_1 (\underline{i} \cdot \underline{k}) + C \omega_3 (\underline{k} \cdot \underline{k})$$

$$\underline{i} \cdot \underline{k} = -\sin \theta$$

$$\underline{k} \cdot \underline{k} = \cos \theta$$

$$\therefore C \omega_3 = + A \dot{\phi} \sin \theta \sin \theta + C \omega_3 \cos \theta$$

$$\therefore C \omega_3 (1 - \cos \theta) = A \dot{\phi} \sin^2 \theta$$

$$= A \dot{\phi}^2 (1 - \cos^2 \theta)$$

$$= A \dot{\phi}^2 (1 - \cos \theta)(1 + \cos \theta)$$

$$\therefore C \omega_3 = A \dot{\phi}^2 (1 + \cos \theta) \quad (3)$$

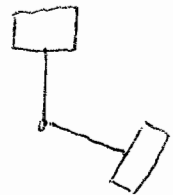
and with  $\dot{\phi}_{\text{final}} = \frac{mga}{C \omega_3}$ ,  $\Theta_{\text{final}} = \frac{\pi}{2}$

$$\therefore C \omega_3 = \frac{A m g a}{C \omega_3} \quad *$$

$$\text{but } (C \omega_3)^2 = (C \omega)^2 = \left( C \frac{\omega_{\text{crit}}}{2} \right)^2 = m g a A \quad (4)$$

So \* is satisfied.

$$(d) \quad (KE + PE)_{\text{initial}} = (KE + PE)_{\text{final}}$$



$$\therefore \frac{1}{2} C \omega_3^2 + m g a = \frac{1}{2} A \omega_1^2 + \frac{1}{2} A \omega_2^2 + \frac{1}{2} C \omega_3^2 + m g a \cos \theta$$

$$\text{but at } \Theta_{\text{max}} \quad \omega_2 = 0 \quad \text{and} \quad \omega_1^2 = \dot{\phi}^2 \sin^2 \theta$$

$$\therefore m g a (1 - \cos \theta) = \frac{1}{2} A \dot{\phi}^2 \sin^2 \theta$$

$$= \frac{1}{2} A \dot{\phi}^2 (1 - \cos^2 \theta)$$

$$\therefore m g a = \frac{1}{2} A \dot{\phi}^2 (1 + \cos \theta)$$

$$\text{but } \textcircled{3} \rightarrow \dot{\phi}^2 = \left( \frac{C\omega_3}{A(1+\cos\theta)} \right)^2$$

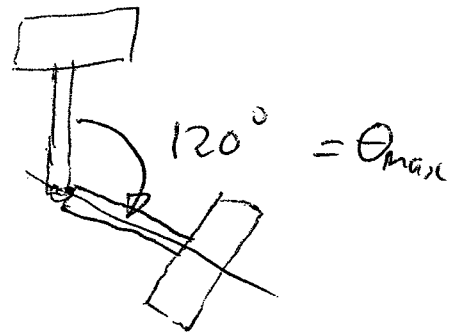
$$\therefore mga = \frac{1}{2} A \frac{(C\omega_3)^2}{A^2 (1+\cos\theta)}$$

$$\text{and from } \textcircled{4} = \frac{1}{2A} \frac{mga A}{1+\cos\theta}$$

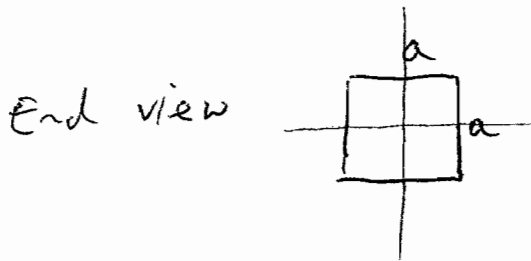
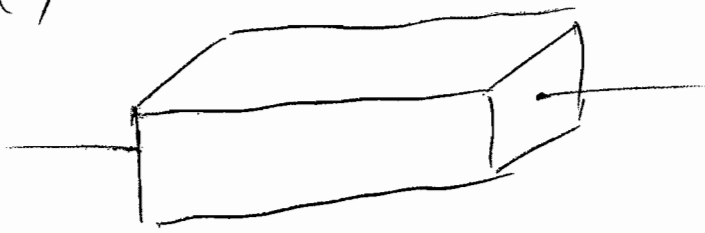
$$\therefore 1 + \cos\theta = \frac{1}{2}$$

$$\therefore \cos\theta = -\frac{1}{2}$$

$$\therefore \theta = \underline{\underline{\frac{2\pi}{3}}}$$

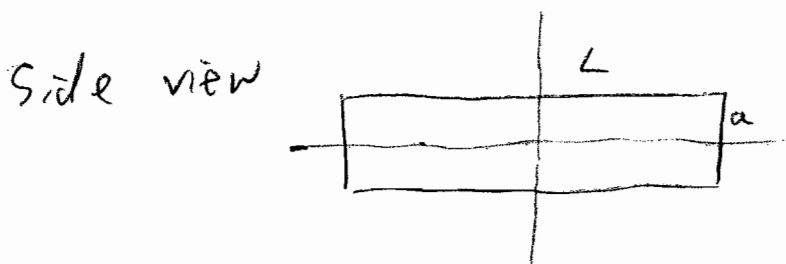


2. (a)



$$\therefore C = \frac{1}{12} M a^2 + \frac{1}{12} M a^2 = \underline{\underline{\frac{1}{6} M a^2}}$$

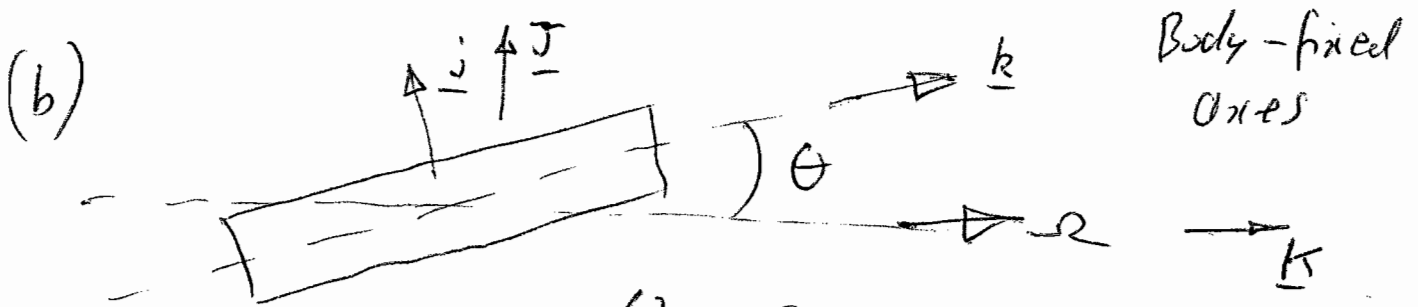
by h axis theorem & data book



$$A = \frac{1}{12} M a^2 + \frac{1}{12} M L^2$$

$$= \underline{\underline{\frac{1}{12} M (a^2 + L^2)}}$$

The body is an AAC body so behaves like a cylinder.



$$\omega_1 = 0$$

$$\omega_2 = -\Omega \sin \theta$$

$$\omega_3 = \Omega \cos \theta$$

i/

$$\therefore \underline{h} = A \omega_1 \underline{i} + A \omega_2 \underline{j} + C \omega_3 \underline{k}$$

$$= -A \Omega \sin \theta \underline{j} + C \Omega \cos \theta \underline{k}$$

and for ii/ resolve h into I J K

$$\begin{aligned}
 2(b) \text{ ii ant. } \quad \underline{h} \cdot \underline{k} &= -A \rho \sin \theta (\underline{j} \cdot \underline{k}) + C \rho \cos \theta (\underline{k} \cdot \underline{k}) \\
 &= -A \rho \sin \theta (-\sin \theta) + C \rho \cos \theta (\cos \theta) \\
 &= A \rho \sin^2 \theta + C \rho \cos^2 \theta
 \end{aligned}$$

$$\begin{aligned}
 \underline{h} \cdot \underline{j} &= -A \rho \sin \theta (\underline{j} \cdot \underline{j}) + C \rho \cos \theta (\underline{k} \cdot \underline{j}) \\
 &= -A \rho \sin \theta (\cos \theta) + C \rho \cos \theta (\sin \theta) \\
 &= (C - A) \rho \cos \theta \sin \theta
 \end{aligned}$$

$$\underline{h} \cdot \underline{i} = 0$$

$$\therefore \underline{h} = \frac{(C - A) \rho \cos \theta \sin \theta \underline{j} + (A \rho \sin^2 \theta + C \rho \cos^2 \theta) \underline{k}}{\text{ans}} \quad \textcircled{1}$$

$$\text{Now, } \underline{h} = \begin{bmatrix} \text{Inertia} \\ \text{Matrix} \end{bmatrix} \rho \underline{k}$$

$$= \begin{bmatrix} \dots & -e \\ \dots & -f \\ \dots & g \end{bmatrix} \begin{bmatrix} 0 \\ 0 \\ \rho \end{bmatrix}$$

$$= e \rho \underline{i} + f \rho \underline{j} + g \rho \underline{k}$$

compare with  $\textcircled{1}$

$$\therefore f = -(C - A) \rho \cos \theta \sin \theta$$

which is one of the products of inertia

(c) Use Euler's equations in body-fixed axes  $\underline{i}, \underline{j}, \underline{k}$

$$Q_1 = A\dot{\omega}_1 - (A-C)\omega_2\omega_3$$
$$= 0 - (A-C)(-\Omega\sin\theta)(\Omega\cos\theta)$$

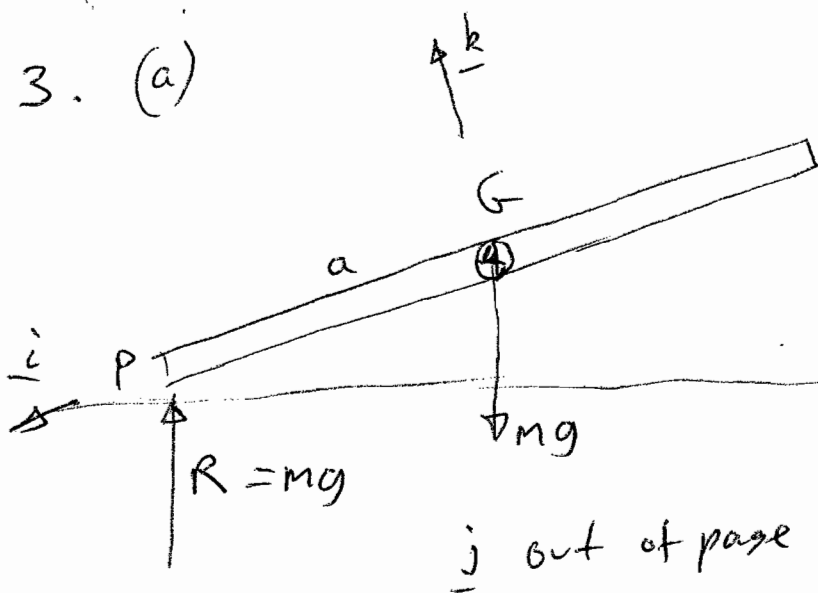
$$Q_1 = \underline{(A-C)\Omega^2\sin\theta\cos\theta} \quad \text{ans}$$

(d)  $Q_1 = 0$  if  $A = C$

$$\therefore a = L$$

This is when the rotor is a cube which is an AAA body, identical to a sphere which can spin about any axis with no dynamic moment.

3. (a)



G is at rest  
 $\therefore R = mg$   
 (no net force on the coin)

$$\underline{Q} = -mga \cos \theta \underline{j}$$

taking moments about G

(b) Gyro equation "2"

$$Q_2 = A \dot{\Omega}_2 + (A \Omega_3 - C \omega_3) \Omega_1$$

with  $\Omega_1 = -\dot{\phi} \sin \theta$

$\Omega_2 = \dot{\theta} = 0$  (insteads slate)

$\Omega_3 = \dot{\phi} \cos \theta$

$$\therefore -mga \cos \theta = (A \dot{\phi} \cos \theta - C \omega_3) (-\dot{\phi} \sin \theta) \quad (1)$$

No slip at P:

$$\underline{\omega} \times a \underline{i} = 0 \quad \therefore (\Omega_1 \underline{i} + \Omega_2 \underline{j} + \omega_3 \underline{k}) \times a \underline{i} = 0$$

$$\therefore \Omega_2 = 0$$

and  $\omega_3 = 0$

3 (b) cont. so (1)  $\rightarrow m g a \cos \theta = A \dot{\phi} \sin \theta$

$$\therefore \dot{\phi}^2 = \frac{m g a}{A} \cot \theta$$

but for a thin coin  $C = \frac{1}{2} m a^2$

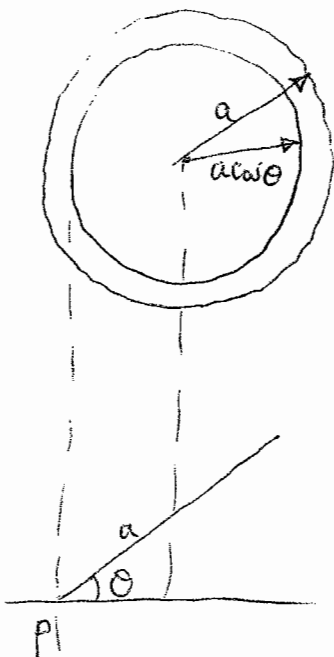
$$A = \frac{1}{4} m a^2$$

$$\therefore \dot{\phi}^2 = \frac{4 m g a \cot \theta}{m a^2} = 4 \frac{g}{a} \cot \theta$$

$$\therefore \dot{\phi} = \underline{2 \sqrt{\frac{g}{a} \cot \theta}} \quad \text{ans}$$

This is the rate of turning of the reference frame  
ie the wobbling rate

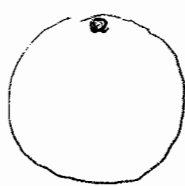
3(c) For the turning rate of the head of the coin  
consider the circle on which point P rolls.



In one wobble the circumference  
of length  $2\pi a$  rolls on a circle  
of circumference  $2\pi a \cos \theta$

So a spot on the edge of the coin  
has advanced a distance

$$2\pi a - 2\pi a \cos \theta$$



after one wobble



So the apparent angle  $\beta$  through which the coin has turned is  $\frac{2\pi a(1 - \cos\theta)}{a}$

The denominator could equally be  $a \cos\theta$ , but at this stage assume small  $\theta$

$$\therefore \cos\theta \approx 1 - \frac{1}{2}\theta^2$$

Ignore the  $\theta^2$  bit in the denominator, but not in the numerator

$$\therefore \beta \approx 2\pi \frac{1}{2}\theta^2 = \pi\theta^2$$

The time taken for one wobble of the coin

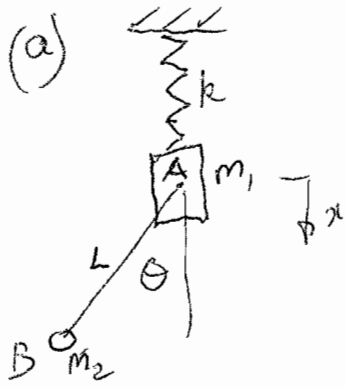
$$T = \frac{2\pi}{\dot{\phi}} = \frac{2\pi}{2\sqrt{\frac{g}{a\theta}}} = \pi\sqrt{\frac{a\theta}{g}}$$

again for small  $\theta$   
 $\cot\theta \approx \frac{1}{\theta}$

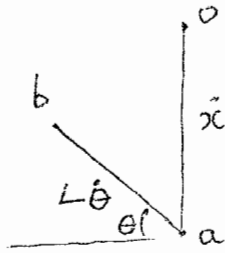
$$\text{So } \dot{\beta} = \frac{\beta}{T} = \frac{\pi\theta^2}{\pi\sqrt{\frac{a\theta}{g}}} = \underline{\underline{\sqrt{\frac{g\theta^3}{a}}}} \quad \text{ans}$$

This is the rate of turning of the head on the coin.  $\dot{\beta} \rightarrow 0$  as  $\theta \rightarrow 0$  as observed.

4. (a)



Velocity diagram



$$T = \frac{1}{2} M_1 \dot{x}^2 + \frac{1}{2} M_2 \left[ (\dot{x} - L\dot{\theta} \sin\theta)^2 + (L\dot{\theta} \cos\theta)^2 \right]$$

$$= \frac{1}{2} (M_1 + M_2) \dot{x}^2 - \frac{1}{2} M_2 (2\dot{x} L\dot{\theta} \sin\theta) + \frac{1}{2} M_2 L^2 \dot{\theta}^2 (\sin^2\theta + \cos^2\theta)$$

$$T = \frac{1}{2} \left[ (M_1 + M_2) \dot{x}^2 + M_2 (L^2 \dot{\theta}^2 - 2L\dot{x} \dot{\theta} \sin\theta) \right] \quad \underline{\underline{\text{ans}}}$$

$$V = \frac{1}{2} k x^2 - M_1 g x - M_2 g \left( x + \frac{L}{\sin\theta} (1 - \cos\theta) \right) \quad \underline{\underline{\text{ans}}}$$

(b) x-equation

$$\text{Lagrange} = \frac{d}{dt} \left( \frac{\partial T}{\partial \dot{x}_j} \right) - \frac{\partial T}{\partial x_j} + \frac{\partial V}{\partial x_j} = Q_j$$

$$\frac{\partial T}{\partial \dot{x}} = (M_1 + M_2) \dot{x} - M_2 L \dot{\theta} \sin\theta$$

$$\frac{\partial T}{\partial x} = 0$$

$$\frac{\partial V}{\partial x} = kx - (M_1 + M_2)g$$

$$\therefore \underline{\underline{(M_1 + M_2) \ddot{x} - M_2 L \ddot{\theta} \sin\theta - M_2 L \dot{\theta}^2 \sin\theta \cos\theta + kx - (M_1 + M_2)g = 0}} \quad \underline{\underline{\text{ans.}}}$$

check: put  $\theta = 0$

$$\therefore (M_1 + M_2) \ddot{x} + kx = (M_1 + M_2)g \quad \text{usual equation for mass on spring} \quad \checkmark$$

4(b) cont.  $\theta$  equation

$$\frac{\delta T}{\delta \dot{\theta}} = m_2 L^2 \dot{\theta} - m_2 L \dot{x} \sin \theta$$

$$\frac{\delta T}{\delta \theta} = -m_2 L \dot{x} \dot{\theta} \cos \theta$$

$$\frac{\delta V}{\delta \theta} = +m_2 g L \sin \theta$$

$$\therefore m_2 L^2 \ddot{\theta} - m_2 L \ddot{x} \sin \theta - m_2 L \dot{x} \dot{\theta} \cos \theta + m_2 L \dot{x} \dot{\theta} \cos \theta + m_2 g L \sin \theta = 0$$

$$\therefore \underline{m_2 L \ddot{\theta} - \ddot{x} \sin \theta + g \sin \theta = 0}$$

ans.

check: put  $x=0 \quad \therefore \ddot{\theta} + \frac{g}{L} \sin \theta = 0$

, usual pendulum equation ✓

4(c) put  $\sin \theta \sim \theta$   
 $\cos \theta \sim 1$

and ignore small terms.

$$\therefore (m_1 + m_2) \ddot{x} + kx = (m_1 + m_2)g$$

$$\text{and } \underline{L \ddot{\theta} + g \theta = 0}$$

ans

4(d) Put  $\sin \theta \approx \theta$  and  $\cos \theta \approx 1 - \frac{\theta^2}{2}$   
 into  $T$  &  $V$ , keeping up to order  $\theta^2$

$$T \approx \frac{1}{2} \left[ (m_1 + m_2) \dot{x}^2 + m_2 L^2 \dot{\theta}^2 \right]$$

$$V \approx \frac{1}{2} k x^2 - (m_1 + m_2) g x + \frac{1}{2} m_2 g L \theta^2$$

Make a substitution  $x = y + x_0$   
 $x^2 = y^2 + 2x_0 y + x_0^2$

$$\therefore V = \frac{1}{2} k (y^2 + 2x_0 y + x_0^2) - (m_1 + m_2) g (y + x_0) + \frac{1}{2} m_2 g L \theta^2$$

$$\text{Choose } x_0 = \frac{(m_1 + m_2) g}{k}$$

and arbitrarily shift datum for  $V$  to  $\frac{1}{2} k x_0^2 - (m_1 + m_2) g x_0$

$$\therefore V = \frac{1}{2} k y^2 + \frac{1}{2} m_2 g L \theta^2$$

and since  $\dot{x} = \dot{y}$

$$T = \frac{1}{2} \left[ (m_1 + m_2) \dot{y}^2 + m_2 L^2 \dot{\theta}^2 \right]$$

$T$  &  $V$  give diagonal mass and stiffness matrices

$$[M] = \begin{bmatrix} m_1 + m_2 & 0 \\ 0 & m_2 L^2 \end{bmatrix} \quad \text{and} \quad [K] = \begin{bmatrix} k & 0 \\ 0 & m_2 g L \end{bmatrix}$$

4(d) Put  $\sin \theta \approx \theta$  and  $\cos \theta \approx 1 - \frac{\theta^2}{2}$   
 into  $T$  &  $V$ , keeping terms up to order<sup>2</sup>

$$T \approx \frac{1}{2} \left( (m_1 + m_2) \dot{x}^2 + m_2 L^2 \dot{\theta}^2 \right)$$

$$V \approx \frac{1}{2} k x^2 - (m_1 + m_2) g x + \frac{1}{2} m_2 g L \theta^2$$

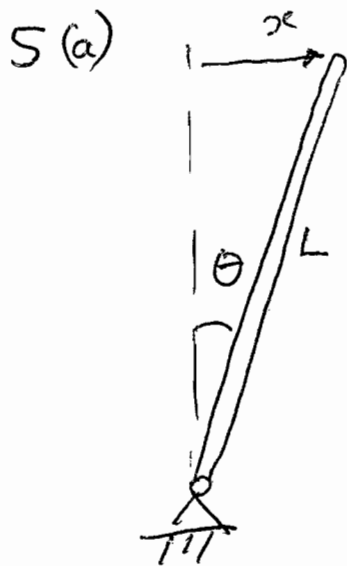
only quadratic terms count in the mass & stiffness matrices so

$$[M] = \begin{bmatrix} m_1 + m_2 & 0 \\ 0 & m_2 L^2 \end{bmatrix} \quad [k] = \begin{bmatrix} k & 0 \\ 0 & m_2 g L \end{bmatrix}$$

$$\text{and } [M] \begin{bmatrix} \ddot{x} \\ \ddot{\theta} \end{bmatrix} + [k] \begin{bmatrix} x \\ \theta \end{bmatrix} = \begin{bmatrix} -(m_1 + m_2)g \\ 0 \end{bmatrix}$$

↖

this comes from  $\frac{\partial V}{\partial x} = -(m_1 + m_2)g$   
 which is a constant throughout  
 the motion.



Think in terms of  $\theta$

$$T = \frac{1}{2} I \dot{\theta}^2$$

Now do a coordinate transformation

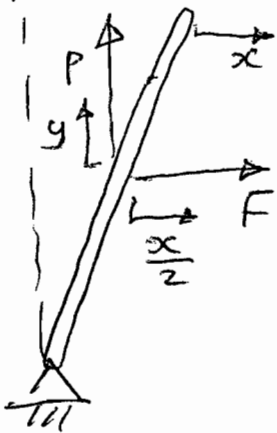
$$x = L \sin \theta$$

$$\dot{x} = L \dot{\theta} \cos \theta$$

$$\begin{aligned} \dot{x}^2 &= L^2 \dot{\theta}^2 \cos^2 \theta = \dot{\theta}^2 (L^2 - L^2 \sin^2 \theta) \\ &= \dot{\theta}^2 (L^2 - x^2) \end{aligned}$$

$$\therefore T = \frac{1}{2} I \frac{\dot{x}^2}{L^2 - x^2} \quad \underline{\text{ans}}$$

For generalized forces use a work argument



$$dW = F \frac{dx}{2} + P dy$$

easy by inspection

needs thought

$$y = \frac{L}{2} (\cos \theta - 1)$$

$$dy = \frac{L}{2} (-\sin \theta d\theta) = -\frac{x}{2} d\theta$$

and  $dx = L \cos \theta d\theta = L \sqrt{1 - \sin^2 \theta} d\theta$

$$\therefore d\theta = \frac{dx}{\sqrt{L^2 - x^2}}$$

$$\therefore dW = Q dx = \frac{F}{2} dx - \frac{Px}{2\sqrt{L^2 - x^2}} dx$$

$$\therefore Q = \frac{F}{2} - \frac{Px}{2\sqrt{L^2 - x^2}} \quad \underline{\text{ans}}$$

5(b)

$$\frac{\partial T}{\partial \dot{x}} = \frac{I \dot{x}}{L^2 - x^2}$$

$$\frac{\partial T}{\partial x} = \frac{I \dot{x}^2 x}{(L^2 - x^2)^2}$$

$V = 0$  because all forces are included in  $Q$

$$\text{so } \frac{\partial V}{\partial x} = 0$$

5(b) cont'd Lagrange :  $\frac{d}{dt} \left( \frac{I \dot{x}}{L^2 - x^2} \right) - \frac{I \dot{x}^2 x}{(L^2 - x^2)^2} = Q$

$\therefore \frac{I \ddot{x} (L^2 - x^2) - I \dot{x} (-2x)}{(L^2 - x^2)^2} - \frac{I \dot{x}^2 x}{(L^2 - x^2)^2} = Q$

$\therefore \frac{I \ddot{x}}{L^2 - x^2} + \frac{I \dot{x}^2 x}{(L^2 - x^2)^2} = \frac{F}{2} - \frac{Px}{2\sqrt{L^2 - x^2}}$  ans

Small vibrations  $\therefore x \ll L$  and ignore  $\dot{x}^2 x$  terms

$\therefore \frac{I \ddot{x}}{L^2} + \frac{Px}{2L} = \frac{F}{2} \quad \therefore \ddot{x} + \frac{PL}{2I} x = \frac{FL^2}{2I}$

$\therefore \omega_n = \sqrt{\frac{PL}{2I}}$  ans  $\omega_n^2$

5(c) Use  $\theta$   $\therefore T = \frac{1}{2} I \dot{\theta}^2$   
 $V = 0$  as before

$dW = Q d\theta = F \frac{dx}{2} + p dy$

$\left. \begin{aligned} dx &= L \cos \theta d\theta \\ dy &= -\frac{L}{2} \sin \theta d\theta \end{aligned} \right\} \text{as before}$

$\therefore Q = F \frac{L}{2} \cos \theta - P \frac{L}{2} \sin \theta$

$\frac{\delta T}{\delta \dot{\theta}} = I \dot{\theta} \quad \frac{\delta T}{\delta \theta} = 0 \quad \frac{\delta V}{\delta \theta} = 0$

$\therefore I \ddot{\theta} = F \frac{L}{2} \cos \theta - P \frac{L}{2} \sin \theta$  ans

(Check, small  $\theta$   $\therefore I \ddot{\theta} + \frac{PL}{2} \theta = \frac{FL}{2}$   
as before with  $x = L\theta$ )

5(c) cont'd

Substitute

$$x = L \sin \theta$$

$$\dot{x} = L \dot{\theta} \cos \theta$$

$$\ddot{x} = L \ddot{\theta} \cos \theta - L \dot{\theta}^2 \sin \theta$$

into the  $x$ -equation from 5(b)

$$\begin{aligned} \frac{I}{L^2 - x^2} (L \ddot{\theta} \cos \theta - L \dot{\theta}^2 \sin \theta) + \frac{I (L \dot{\theta} \cos \theta)^2 L \sin \theta}{(L^2 - x^2)^2} &= \\ &= \frac{F}{2} - \frac{PL \sin \theta}{2 \sqrt{L^2 - (L \sin \theta)^2}} \end{aligned}$$

$$\therefore \frac{I}{L^2 \cos^2 \theta} (L \ddot{\theta} \cos \theta - L \dot{\theta}^2 \sin \theta) + \frac{I \dot{\theta}^2 \sin \theta}{L \cos^2 \theta} = \frac{F}{2} - \frac{P \sin \theta}{2 \cos \theta}$$

$$\therefore I \ddot{\theta} = \frac{FL}{2} \cos \theta - \frac{PL}{2} \sin \theta \quad \underline{\text{as required}}$$

Notes • Several candidates obtained full marks, simply by not making mistakes.

• Fudging the "show that" was obvious

• "V=0 because buoyancy forces are equal and opposite to gravity earned" no marks



## Engineering Tripos Part IIA 2007

### Answers

1. (c)  $\dot{\phi} = \frac{mga}{C\omega} = \sqrt{\frac{mga}{A}} = \frac{C\omega}{A}$  (d)  $120^\circ$
2. (a)  $m(a^2+L^2)/12, m(a^2+L^2)/12, ma^2/6$   
 (b)  $\omega = 0 \mathbf{i} - \Omega \sin \theta \mathbf{j} + \Omega \cos \theta \mathbf{k}$   
 (i)  $\mathbf{h} = -A\Omega \sin \theta \mathbf{j} + C\Omega \cos \theta \mathbf{k}$   
 (ii)  $\mathbf{h} = (C-A)\Omega \cos \theta \sin \theta \mathbf{J} + (A\Omega \sin^2 \theta + C\Omega \cos^2 \theta) \mathbf{K}$   
 product of inertia =  $-(C-A)\cos \theta \sin \theta$   
 (c)  $Q_1 = (A-C)\Omega^2 \cos \theta \sin \theta$   
 (d)  $L = a$
3. (a)  $\mathbf{Q} = -mga \cos \theta \mathbf{j}$  (b)  $2\sqrt{\frac{g}{a \tan \theta}}$  (c)  $\sqrt{\frac{g\theta^3}{a}}$
4. (a)  $V = \frac{1}{2}k\theta^2 - m_1gx - m_2g(x - L(1 - \cos \theta))$   
 (b)  $(m_1 + m_2)\ddot{x} - m_2L\ddot{\theta} \sin \theta - m_2L\dot{\theta}^2 \sin \theta \cos \theta + kx - (m_1 + m_2)g = 0$   
 $L\ddot{\theta} - \ddot{x} \sin \theta + g \sin \theta = 0$   
 (c)  $(m_1 + m_2)\ddot{x} + kx = (m_1 + m_2)g$   
 $L\ddot{\theta} + g\theta = 0$   
 (d)  $M = \begin{bmatrix} m_1 + m_2 & 0 \\ 0 & m_2L^2 \end{bmatrix}$   $K = \begin{bmatrix} k & 0 \\ 0 & m_2gL \end{bmatrix}$
5. (b)  $\frac{E\ddot{x}}{L^2 - x^2} + \frac{E\dot{x}^2x}{(L^2 - x^2)^2} = \frac{F}{2} - \frac{Px}{2\sqrt{L^2 - x^2}}$   
 (c)  $T = \frac{1}{2}I\dot{\theta}^2$   $Q = \frac{FL \cos \theta}{2} - \frac{PL \sin \theta}{2}$