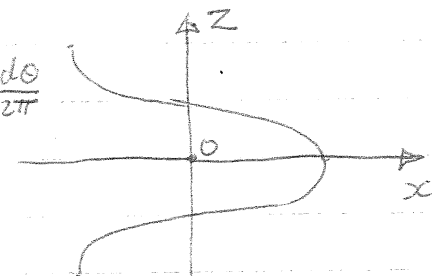


1. (a) Three views

(centre of mass is at

$(0, 0, 0)$ by inspection,

$$dm = m \frac{d\theta}{2\pi}$$



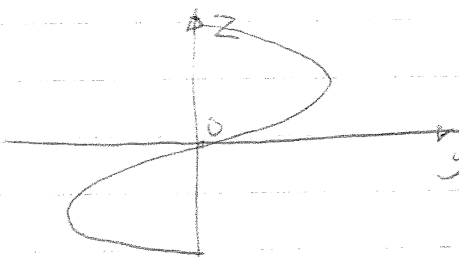
$$I_{yy} = \int (x^2 + z^2) dm = \int_{-\pi}^{\pi} \left(R^2 \cos^2 \theta + \left(\frac{\theta L}{2\pi} \right)^2 \right) \frac{m d\theta}{2\pi}$$

$$= \frac{R^2 M}{2\pi} (\pi) + \frac{m L^2}{(2\pi)^2} \left(\frac{2}{3} \pi^3 \right)$$

$$= \frac{m}{12} (6R^2 + L^2)$$

$$I_{xz} = \int xz dm = \int_{-\pi}^{\pi} R \cos \theta \frac{\theta L}{2\pi} m \frac{d\theta}{2\pi}$$

$$= \frac{mRL}{4\pi^2} \int_{-\pi}^{\pi} \theta \cos \theta d\theta = 0$$

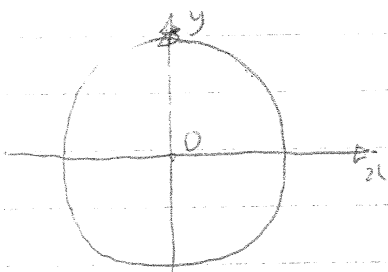


$$I_{zz} = \int (y^2 + z^2) dm = \frac{m}{12} (6R^2 + L^2) \text{ again}$$

$$I_{yz} = \int yz dm = \frac{mRL}{4\pi^2} \int_{-\pi}^{\pi} \theta \sin \theta d\theta = \frac{2\pi mRL}{4\pi^2}$$

$$I_{zz} = mR^2 = \frac{mRL}{2\pi}$$

$$I_{xy} = 0$$



note: $\int \theta \cos \theta d\theta = \theta \sin \theta - \int \sin \theta d\theta = \theta \sin \theta + \cos \theta$
 $\int \theta \sin \theta d\theta = -\theta \cos \theta + \int \cos \theta d\theta = -\theta \cos \theta + \sin \theta$
 so $\int_{-\pi}^{\pi} \theta \cos \theta d\theta = 0$ and $\int_{-\pi}^{\pi} \theta \sin \theta d\theta = 2\pi$

$$S_o \quad [I] = \frac{m}{12} \begin{bmatrix} 6R^2 + L^2 & 0 & 0 \\ 0 & 6R^2 + L^2 & -\frac{6RL}{\pi} \\ 0 & -\frac{6RL}{\pi} & 12R^2 \end{bmatrix} \quad [50\%]$$

(b) $R=0$ $I_{xx} = I_{yy} = \frac{mL^2}{12}$, $I_{zz} = 0$ as in data book for straight rod
 $L=0$ $I_{xx} = I_{yy} = \frac{1}{2} mR^2$, $I_{zz} = mR^2$ ✓

$$L=2R \quad [I] = \frac{mR^2}{6} \begin{bmatrix} 5 & 0 & 0 \\ 0 & 5 & -\frac{6}{\pi} \\ 0 & -\frac{6}{\pi} & 6 \end{bmatrix} \quad [10\%]$$

(c) By parallel axis theorem $I = I_G + M \begin{bmatrix} y^2+z^2 & -xy & -xz \\ -yx & x^2+z^2 & -yz \\ -zx & -zy & x^2+y^2 \end{bmatrix}$

Five coils with z being, say, $-2L, -L, 0, L$ and $2L$
 $x = y = 0$

$$\therefore I = \frac{5MR^2}{6} \begin{bmatrix} 5 & 0 & 0 \\ 0 & 5 & -6/\pi \\ 0 & -6/\pi & 6 \end{bmatrix} + M \begin{bmatrix} (k^2+1+0+1+k)L^2 & 0 & 0 \\ 0 & (k^2+1+0+1+k)L^2 & 0 \\ 0 & 0 & 0 \end{bmatrix}$$

$$= \frac{5MR^2}{6} \begin{bmatrix} 5 & 0 & 0 \\ 0 & 5 & -6/\pi \\ 0 & -6/\pi & 6 \end{bmatrix} + 10ML^2 \begin{bmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 0 \end{bmatrix}$$

and with $L = 2R$ (gives $\frac{265}{6} MR^2$ on leading diagonal, etc). [20%]

(d) For the couple, $Q = \dot{h} = \Omega \mathbf{k} \times \mathbf{h}$
 with $\mathbf{h} = \begin{bmatrix} I_{xx} & -I_{xz} & -I_{xz} \\ I_{yy} & -I_{yz} & -I_{yz} \\ I_{zz} & & \end{bmatrix} \begin{bmatrix} 0 \\ 0 \\ \Omega \end{bmatrix} \begin{matrix} \hat{i} \\ \hat{j} \\ \hat{k} \end{matrix} = (-I_{xz} \hat{i} - I_{yz} \hat{j} + I_{zz} \hat{k}) \Omega$

so $Q = (-I_{xz} \hat{j} + I_{yz} \hat{i}) \Omega^2$

but $I_{xz} = 0$

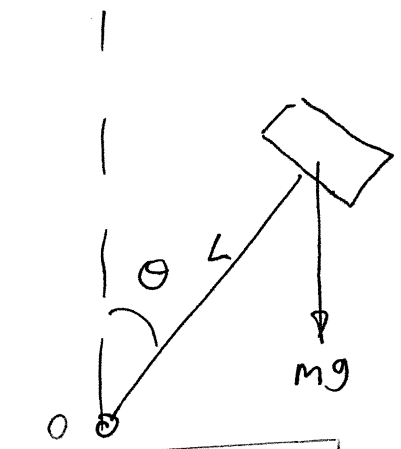
$I_{yz} = \frac{6}{\pi} \frac{5MR^2}{6}$

so $Q = \frac{5MR^2 \Omega^2}{\pi} \hat{i}$

[20%]

2/

Use gyro equation 2



Take the gyro equations relative to the fixed point O.

$$Q_2 = A\dot{\Omega}_2 + (A\Omega_3 - c\omega_3)\Omega_1$$

with $Q_2 = mgL \sin \theta$

and $\dot{\Omega}_2 = 0$ in steady state

and $\Omega_3 = \dot{\phi} \cos \theta$

$$\Omega_1 = -\dot{\phi} \sin \theta$$

$$\therefore mgL \sin \theta = -(A\dot{\phi} \cos \theta - c\omega_3) \dot{\phi} \sin \theta$$

$$\therefore A \cos \theta \dot{\phi}^2 - c\omega_3 \dot{\phi} + mgL = 0 \tag{1}$$

a/ Fast spin $\omega_3 \gg \dot{\phi}$ \therefore ignore $\dot{\phi}^2$ term

$$\therefore \dot{\phi} \approx \frac{mgL}{c\omega_3} \quad \text{precession for fast spin at all } \theta \text{ [30\%]}$$

b/ For non-fast spin there is only a solution to (1)

if the discriminant Δ is positive

$$\text{ie } \Delta = "b^2 - 4ac" = (c\omega_3)^2 - 4A \cos \theta mgL > 0$$

$$\therefore \omega_3^2 > \frac{4A \cos \theta mgL}{c^2}$$

and for $\theta < \frac{\pi}{2}$, $\cos \theta$ is positive hence there is always a speed below which precession is impossible. [30%]

c/ for $\theta > \frac{\pi}{2}$ $\cos \theta$ is negative so Δ is always positive meaning that precession always occurs. [10%]

d/ The full solution to (1) is

$$\dot{\phi} = \frac{c\omega_3 \pm \sqrt{(c\omega_3)^2 - 4A \cos \theta mgL}}{2A \cos \theta}$$

and for θ close to π then $\cos\theta \approx -1$

(4)

$$\text{so } \dot{\phi} \approx \frac{C\omega_3 \pm \sqrt{(C\omega_3)^2 + 4AMgL}}{2A}$$

and for $C\omega_3 \ll 4AMgL$

$$\begin{aligned} \text{then } \sqrt{(C\omega_3)^2 + 4AMgL} &= \sqrt{4AMgL} \sqrt{1 + \frac{(C\omega_3)^2}{4AMgL}} \\ &\approx \sqrt{4AMgL} \left(1 + \frac{1}{2} \frac{(C\omega_3)^2}{4AMgL}\right) \end{aligned}$$

$$\text{so } \dot{\phi} \approx \pm \sqrt{\frac{MgL}{A}} + \frac{C\omega_3}{2A}$$

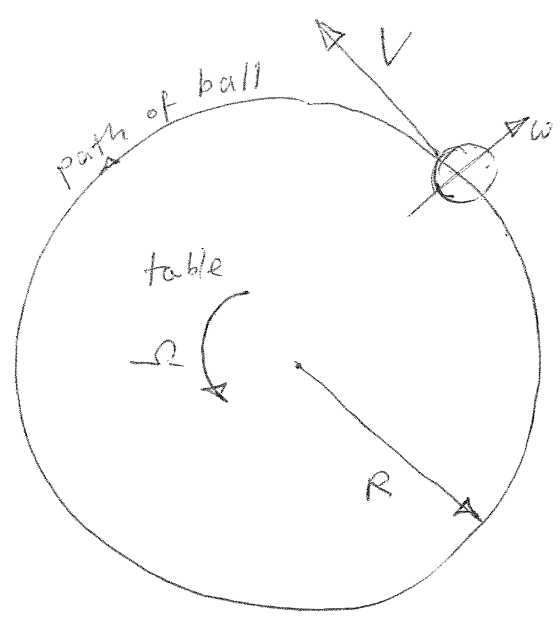
Note that $A \approx ML^2$ if the rotor is treated

as a point mass $\therefore \dot{\phi} \approx \pm \sqrt{\frac{g}{L}} + \frac{C\omega_3}{2A}$

↑

this is the frequency of a conical pendulum. The \pm allows for the conical motion being "clockwise" or "anticlockwise". the small spin ω_3 means that precession one way is slightly faster than the other way. [30%]

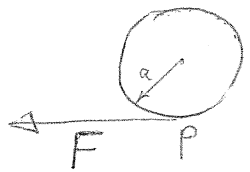
3.



a/ Ball no slip at P
 ω = ball spin
 V = ball speed
 $R\Omega$ = table speed

$\therefore a\omega = R\Omega - V$
 $\therefore \omega = \frac{R\Omega - V}{a}$ [20%]

b/ Ball moves on a circular path so use
 $F = \frac{mV^2}{R}$ for the friction force at P



so couple $Q = Fa = \frac{mV^2 a}{R}$ [30%]

c/ Use " $Q = J\Omega_1\omega$ " for gyroscopic effect
 with $J = \frac{2}{5}ma^2$ and $\Omega_1 = \frac{V}{R}$ = precession rate

$\therefore \frac{mV^2 a}{R} = \frac{2}{5}ma^2 \frac{V}{R} \frac{R\Omega - V}{a}$

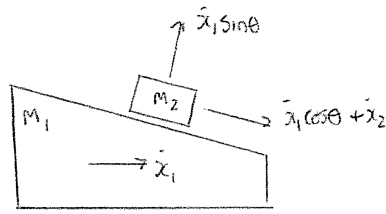
$\therefore 5V = 2(R\Omega - V)$

$\therefore V = \frac{2}{7}R\Omega$ [50%]

Note $Q = J\Omega\omega$ was covered in Part JB mechanics. To derive this result from the gyro equations, \underline{k} must be aligned along ω . The second gyro equation then gives:

$A\dot{\Omega}_2 + (A\Omega_3 - C\omega_3)\Omega_1 = Q_2$ Plan view $\hat{i} \otimes \hat{k} \rightarrow \hat{j} \uparrow V$
 $\downarrow \quad \downarrow \quad \downarrow$
 $0 \quad 0 \quad -\frac{mV^2 a}{R}$

4 a) Velocities:



Kinetic energy: $T = \frac{1}{2} M_1 \dot{x}_1^2 + \frac{1}{2} M_2 \{ (\dot{x}_1 \sin \theta)^2 + (\dot{x}_1 \cos \theta + \dot{x}_2)^2 \}$

$\Rightarrow T = \frac{1}{2} M_1 \dot{x}_1^2 + \frac{1}{2} M_2 (\dot{x}_1^2 + \dot{x}_2^2 + 2 \dot{x}_1 \dot{x}_2 \cos \theta)$

Potential energy: $U = \frac{1}{2} k x_2^2 + m_2 g (-x_2 \sin \theta)$

Generalised Force: $\delta W = P \delta (x_1 \cos \theta + x_2) = (P \cos \theta) \delta x_1 + P \delta x_2 \Rightarrow Q_1 = P \cos \theta, Q_2 = P$

Lagrange for x_1 : $\frac{d}{dt} \left[\frac{\partial T}{\partial \dot{x}_1} \right] - \frac{\partial T}{\partial x_1} + \frac{\partial V}{\partial x_1} = Q_1$

$\Rightarrow M_1 \ddot{x}_1 + M_1 \ddot{x}_2 + m_2 \ddot{x}_2 \cos \theta = P \cos \theta$

Lagrange for x_2 : $\frac{d}{dt} \left[\frac{\partial T}{\partial \dot{x}_2} \right] - \frac{\partial T}{\partial x_2} + \frac{\partial V}{\partial x_2} = Q_2$

$\Rightarrow M_2 \ddot{x}_2 + m_2 \ddot{x}_1 \cos \theta + k x_2 - m_2 g \sin \theta = P$

In matrix form:
$$\begin{pmatrix} M_1 + m_2 & m_2 \cos \theta \\ M_2 \cos \theta & M_2 \end{pmatrix} \begin{pmatrix} \ddot{x}_1 \\ \ddot{x}_2 \end{pmatrix} + \begin{pmatrix} 0 & 0 \\ 0 & k \end{pmatrix} \begin{pmatrix} x_1 \\ x_2 \end{pmatrix} = \begin{pmatrix} P \cos \theta \\ P + m_2 g \sin \theta \end{pmatrix}$$

[40%]

b) For free vibration $|- \omega^2 M + k| = 0 \Rightarrow \begin{vmatrix} -(M_1 + M_2) \omega^2 & -M_2 \cos \theta \omega^2 \\ -M_2 \cos \theta \omega^2 & -M_2 \omega^2 + k \end{vmatrix} = 0$

$\Rightarrow \omega^2 \{ -(M_1 + M_2) (k - M_2 \omega^2) - M_2^2 \omega^2 \cos^2 \theta \} = 0$

$\omega^2 \{ -(M_1 + M_2) k + M_2 \omega^2 (M_1 + M_2 - M_2 \cos^2 \theta) \} = 0$

$\Rightarrow \underline{\omega^2 = 0 \text{ or } \omega^2 = \frac{k(M_1 + M_2)}{M_2 (M_1 + M_2 \sin^2 \theta)}}$

for mode shapes $\begin{pmatrix} -M_2 \cos \theta \omega^2 & -M_2 \omega^2 + k \end{pmatrix} \begin{pmatrix} x_1 \\ x_2 \end{pmatrix} = 0 \xrightarrow{\omega = 0} \begin{pmatrix} x_1 \\ x_2 \end{pmatrix} = \begin{pmatrix} 1 \\ 0 \end{pmatrix}$ rigid body motion

$\begin{pmatrix} -(M_1 + M_2) \omega^2 & -M_2 \cos \theta \omega^2 \end{pmatrix} \begin{pmatrix} x_1 \\ x_2 \end{pmatrix} = 0 \xrightarrow{\omega = \omega_2} \begin{pmatrix} x_1 \\ x_2 \end{pmatrix} = \begin{pmatrix} \frac{-M_2 \cos \theta}{M_1 + M_2} \\ 1 \end{pmatrix}$

[50%]

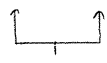
c) Generalised momentum = $\frac{\delta T}{\delta \dot{q}_i}$

For x_1 : $\frac{\delta T}{\delta \dot{x}_1} = (M_1 + M_2) \dot{x}_1 + M_2 \dot{x}_2 \cos \theta \longrightarrow$ zero (constant) for both modes 1 and 2
since no other forces in horizontal direction

For x_2 : $\frac{\delta T}{\delta \dot{x}_2} = M_2 \dot{x}_2 + M_2 \dot{x}_1 \cos \theta \longrightarrow$ not zero for mode 2, spring provides a force. [20%]

5 a) Lagrange's equation:

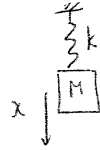
$$\frac{d}{dt} \left[\frac{\partial T}{\partial \dot{q}_i} \right] - \frac{\partial T}{\partial q_i} + \frac{\partial V}{\partial q_i} = Q_i$$



 gravity can be included in
 one of these terms, but not both.

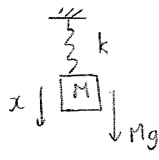
(i) Mass-spring system:

Gravity in potential



$$\left. \begin{aligned} T &= \frac{1}{2} M \dot{x}^2 \\ U &= -Mgx \\ Q &= 0 \end{aligned} \right\} \rightarrow \text{Lagrange } \underline{M\ddot{x} + kx = Mg}$$

Gravity in generalised force



$$\left. \begin{aligned} T &= \frac{1}{2} M \dot{x}^2 \\ U &= 0 \\ \delta W &= Mg \delta x \\ \Rightarrow Q &= Mg \end{aligned} \right\} \rightarrow \text{Lagrange } \underline{M\ddot{x} + kx = Mg} \quad [15\%]$$

(ii) pendulum system

Gravity in potential



$$\left. \begin{aligned} T &= \frac{1}{2} M l^2 \dot{\theta}^2 \\ U &= Mgl(1 - \cos\theta) \\ Q &= 0 \end{aligned} \right\} \rightarrow \text{Lagrange } \underline{Ml^2 \ddot{\theta} + Mgl \sin\theta = 0}$$

Gravity in generalised force



$$\left. \begin{aligned} T &= \frac{1}{2} M l^2 \dot{\theta}^2 \\ U &= 0 \\ \delta W &= -Mg \delta(1 - \cos\theta) l \\ &= -Mg \sin\theta \delta\theta l \\ \Rightarrow Q &= -Mgl \sin\theta \end{aligned} \right\} \rightarrow \text{Lagrange } \underline{Ml^2 \ddot{\theta} + Mgl \sin\theta = 0} \quad [20\%]$$

b) $T = \frac{1}{2} M \dot{x}^2$, $U = \frac{1}{2} kx^2$, $R = \frac{1}{2} \lambda \dot{x}^2$

$$\frac{d}{dt} \left[\frac{\partial T}{\partial \dot{x}} \right] - \frac{\partial T}{\partial x} + \frac{\partial U}{\partial x} + \frac{\partial R}{\partial x} = 0 \Rightarrow \underline{M\ddot{x} + \lambda\dot{x} + kx = 0} \quad [15\%]$$

$$\left. \begin{aligned} \text{c) (i) } T &= \frac{1}{2} \underline{\dot{q}}^T M \underline{\dot{q}} = \frac{1}{2} \sum_i \sum_j M_{ij} \dot{q}_i \dot{q}_j \\ U &= \frac{1}{2} \underline{q}^T k \underline{q} = \frac{1}{2} \sum_i \sum_j k_{ij} q_i q_j \end{aligned} \right\} \frac{d}{dt} \left[\frac{\partial T}{\partial \dot{q}_i} \right] - \frac{\partial T}{\partial q_i} + \frac{\partial U}{\partial q_i} = Q_i$$

$$\Rightarrow \sum_j M_{ij} \ddot{q}_j + \sum_j k_{ij} q_j = Q_i \text{ for all } i$$

$$\Rightarrow \underline{M \ddot{\underline{q}} + k \underline{q} = \underline{Q}} \quad [20\%]$$

(9)

(ii) By direct analogy with the above, if $F = \frac{1}{2} \dot{\underline{q}}^T \underline{C} \dot{\underline{q}} = \frac{1}{2} \sum_i \sum_j C_{ij} \dot{q}_i \dot{q}_j$

$$\Rightarrow \frac{\partial F}{\partial \dot{q}_i} = \sum_j C_{ij} \dot{q}_j \quad (\text{if } C \text{ is symmetric})$$

$\Rightarrow \underline{C} \dot{\underline{q}}$ in equation of Motion

[20%]

(iii) The term $\frac{\partial F}{\partial \dot{q}_i}$ is linear in $\dot{\underline{q}}$ $\Rightarrow F$ is second order in $\dot{\underline{q}}$

↓

Taylor series expression $F \equiv \sum_i \sum_j \underbrace{\frac{\partial^2 F}{\partial \dot{q}_i \partial \dot{q}_j}}_{\text{symmetric}} \dot{q}_i \dot{q}_j$

\Rightarrow only symmetric damping matrices can be produced in this way.

[10%]

3C5 Dynamics: Answers to Tripos Paper 2010

1. (a) $\mathbf{I} = \frac{m}{12} \begin{pmatrix} 6R^2 + L^2 & 0 & 0 \\ 0 & 6R^2 + L^2 & -6RL/\pi \\ 0 & -6RL/\pi & 12R^2 \end{pmatrix}.$
- (b) $\mathbf{I} = \frac{5mR^2}{6} \begin{pmatrix} 53 & 0 & 0 \\ 0 & 53 & -6/\pi \\ 0 & -6/\pi & 6 \end{pmatrix}.$
- (c) $\mathbf{Q} = (5mR^2\Omega^2/\pi)\mathbf{i}.$
2. (a) $\dot{\phi} = mgL/(C\omega_3).$
- (b) Stable for $\omega_3^2 > (4AmgL/C^2)\cos\theta.$
- (c) Always stable because $\cos\theta < 0.$
- (d) For $A \approx mL^2$ the two solutions are $\dot{\phi} = \frac{C\omega_3}{2A} \pm \sqrt{\frac{g}{L}}.$
3. (b) $Q = maV^2/R$ in the direction of the velocity $V.$
- (c) $\alpha = 2/7.$
4. (a) $\begin{pmatrix} m_1 + m_2 & m_2 \cos\theta \\ m_2 \cos\theta & m_2 \end{pmatrix} \begin{pmatrix} \ddot{x}_1 \\ \ddot{x}_2 \end{pmatrix} + \begin{pmatrix} 0 & 0 \\ 0 & k \end{pmatrix} \begin{pmatrix} x_1 \\ x_2 \end{pmatrix} = \begin{pmatrix} P \cos\theta \\ P + m_2 g \sin\theta \end{pmatrix}.$
- (b) $\omega_1 = 0, \mathbf{u}_1 = \begin{pmatrix} 1 \\ 0 \end{pmatrix}; \omega_2^2 = \frac{k(m_1 + m_2)}{m_2(m_1 + m_2 \sin^2\theta)}, \mathbf{u}_2 = \begin{pmatrix} -m_2 \cos\theta \\ m_1 + m_2 \\ 1 \end{pmatrix}.$
- (c) $p_1 = (m_1 + m_2)\dot{x}_1 + m_2\dot{x}_2 \cos\theta$ (conserved)
 $p_2 = m_2\dot{x}_2 + m_2\dot{x}_1 \cos\theta$ (not conserved)
5. (b) $M\ddot{x} + \lambda\dot{x} + Kx = Mg.$
- (c) (i) $T = (1/2)\dot{\mathbf{q}}^T \mathbf{M}\dot{\mathbf{q}}, \quad V = (1/2)\mathbf{q}^T \mathbf{K}\mathbf{q}.$
- (ii) $F = (1/2)\dot{\mathbf{q}}^T \mathbf{C}\dot{\mathbf{q}},$ (iii) not possible.