

ENGINEERING TRIPOS PART IIA

Thursday 6 May 2010 2.30 to 4

Module 3D7

FINITE ELEMENT METHODS

*Answer not more than **three** questions.*

All questions carry the same number of marks.

*The **approximate** percentage of marks allocated to each part of a question is indicated in the right margin.*

Attachment: 3D7 Data Sheet (3 pages).

STATIONERY REQUIREMENTS

Single-sided script paper

SPECIAL REQUIREMENTS

Engineering Data Book

CUED approved calculator allowed

**You may not start to read the questions
printed on the subsequent pages of this
question paper until instructed that you
may do so by the Invigilator**

1 Consider on the interval running from $x = 0$ to $x = L$ the problem governed by the differential equation

$$\alpha \frac{du}{dx} - \frac{d}{dx} \left(\beta \frac{du}{dx} \right) = f$$

where α and β are positive constants, and with the boundary conditions

$$\begin{aligned} u &= 0 & \text{at } x &= 0 \\ \beta \frac{du}{dx} &= h & \text{at } x &= L \end{aligned}$$

(a) Show that a weak version of this problem involves finding u such that

$$\int_0^L \alpha v \frac{du}{dx} dx + \int_0^L \beta \frac{dv}{dx} \frac{du}{dx} dx = \int_0^L v f dx + v h|_{x=L} \quad (1)$$

holds for almost all test functions v , and state the restrictions on v . [25%]

(b) For a linear element of length l , compute the element stiffness matrix \mathbf{k}_e for the formulation in Equation (1). [50%]

(c) Formulate an alternative version of the weak form to that in Equation (1) which would still be suitable for finite element analysis. [25%]

2 Figure 1(a) shows a triangular domain made of two isotropic materials with heat conductivities $k_1 = 3$ and $k_2 = 6$. The heat flux on the edge AC is prescribed to be $\bar{q} = 10$ and the edge CB is insulated. A constant temperature $\bar{T} = 10$ is prescribed on edge AB . The domain is discretized with two three-noded triangular elements as shown in Fig. 1(b).

- (a) Compute the global source vector f . [25%]
- (b) Compute the global conductance matrix \mathbf{K} . [40%]
- (c) Compute the temperature at the four finite element nodes and determine the corresponding heat flux across the domain edge CB . Briefly comment on the computed energy flux across the domain edge CB and suggest how its accuracy could be improved. [35%]

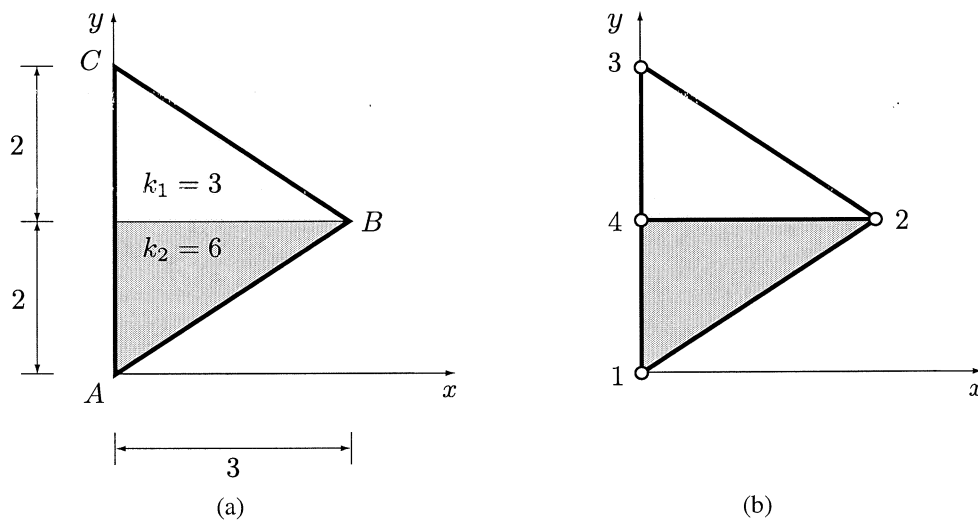


Fig. 1

(TURN OVER

3 (a) Figure 2(a) shows a four-noded elastic element with the following displacement vector

$$\mathbf{a}_e = [u_{x1} \ u_{y1} \ u_{x2} \ u_{y2} \ u_{x3} \ u_{y3} \ u_{x4} \ u_{y4}]^T = [1 \ 0 \ -1 \ 0 \ 1 \ 0 \ -1 \ 0]^T$$

(i) Compute the strain components ϵ_{xx} and ϵ_{xy} along the top edge between nodes 3 and 4. [50%]

(ii) If the given displacements are due to an element force vector

$$\mathbf{f}_e = [1 \ 0 \ -1 \ 0 \ 1 \ 0 \ -1 \ 0]^T$$

comment if the strains found in (i) are physically meaningful. [20%]

(b) Consider the eight-noded hexahedral three-dimensional element shown in Fig. 2(b).

(i) Give an analytical expression for the shape function of node 1. [20%]

(ii) Suggest an integration rule for the numerical integration of the stiffness matrix using Gauss integration. Give reasons for your answer. [10%]

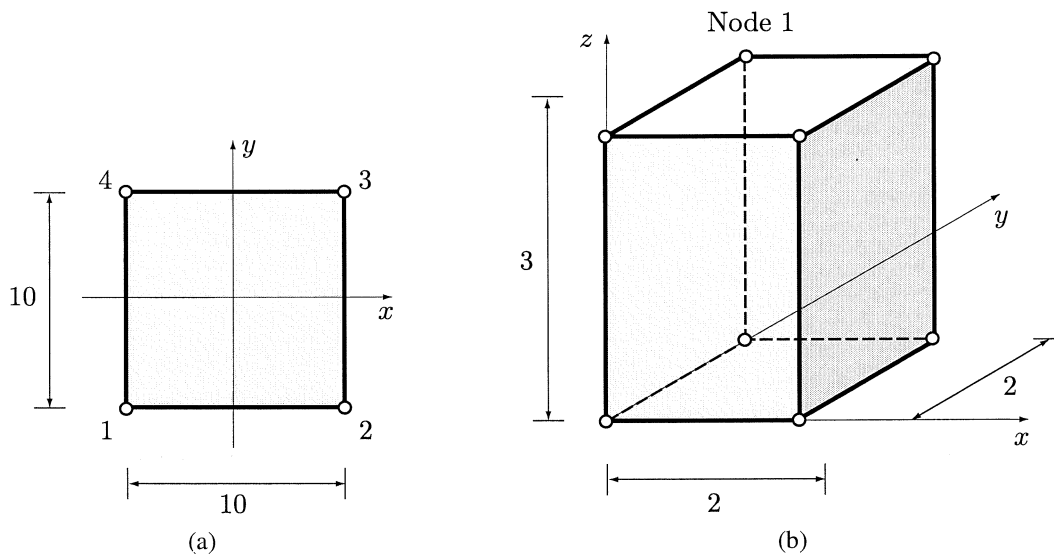


Fig. 2

4 (a) A particular semi-discrete finite element elastodynamics problem is in semi-discrete form written as

$$\mathbf{M}\ddot{\mathbf{a}} + \mathbf{K}\mathbf{a} = \mathbf{b} \quad (2)$$

Using the time stepping method

$$\frac{y_{n+1} - y_n}{\Delta t} = (1 - \theta)\dot{y}_n + \theta\dot{y}_{n+1}$$

where θ is a constant and $0 \leq \theta \leq 1$:

(i) re-formulate equation (2) into a fully discrete problem such that it can be solved; [60%]

(ii) under what conditions can solving a system of equations be avoided for this problem? [10%]

(b) For a system of equations of the form $\mathbf{M}\dot{\mathbf{a}} + \mathbf{K}\mathbf{a} = \mathbf{b}$, the critical time step for a conditionally-stable time-stepping scheme is typically proportional to the square of a measure of the element size. For meshes with uniform element size, if the number of elements is doubled, estimate the change in the critical time step for a one-dimensional problem, a two-dimensional problem and a three-dimensional problem. [20%]

(c) Explain why the modal analysis approach is not applicable to non-linear time-dependent problems? [10%]

END OF PAPER

3D7 DATA SHEET

Element relationships

Elasticity

$$\text{Displacement} \quad \mathbf{u} = \mathbf{N}\mathbf{a}_e$$

$$\text{Strain} \quad \boldsymbol{\varepsilon} = \mathbf{B}\mathbf{a}_e$$

$$\text{Stress (2D/3D)} \quad \boldsymbol{\sigma} = \mathbf{D}\boldsymbol{\varepsilon}$$

$$\text{Element stiffness matrix} \quad \mathbf{k}_e = \int_{V_e} \mathbf{B}^T \mathbf{D} \mathbf{B} dV$$

$$\text{Element force vector} \quad \mathbf{f}_e = \int_{V_e} \mathbf{N}^T \mathbf{f} dV$$

(body force only)

Heat conduction

$$\text{Temperature} \quad T = \mathbf{N}\mathbf{a}_e$$

$$\text{Temperature gradient} \quad \nabla T = \mathbf{B}\mathbf{a}_e$$

$$\text{Heat flux} \quad \mathbf{q} = -\mathbf{D}\nabla T$$

$$\text{Element conductance matrix} \quad \mathbf{k}_e = \int_{V_e} \mathbf{B}^T \mathbf{D} \mathbf{B} dV$$

Beam bending

$$\text{Displacement} \quad v = \mathbf{N}\mathbf{a}_e$$

$$\text{Curvature} \quad \kappa = \mathbf{B}\mathbf{a}_e$$

$$\text{Element stiffness matrix} \quad \mathbf{k}_e = \int_{V_e} \mathbf{B}^T E I \mathbf{B} dV$$

Elasticity matrices

2D plane strain

$$\mathbf{D} = \frac{E}{(1+\nu)(1-2\nu)} \begin{bmatrix} 1-\nu & \nu & 0 \\ \nu & 1-\nu & 0 \\ 0 & 0 & \frac{1-2\nu}{2} \end{bmatrix}$$

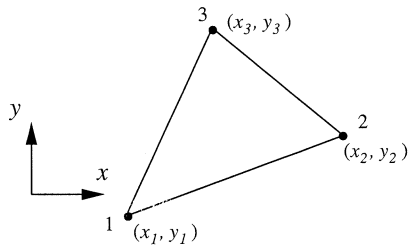
2D plane stress

$$\mathbf{D} = \frac{E}{(1-\nu^2)} \begin{bmatrix} 1 & \nu & 0 \\ \nu & 1 & 0 \\ 0 & 0 & \frac{1-\nu}{2} \end{bmatrix}$$

Heat conductivity matrix (2D, isotropic)

$$\mathbf{D} = \begin{bmatrix} k & 0 \\ 0 & k \end{bmatrix}$$

Shape functions

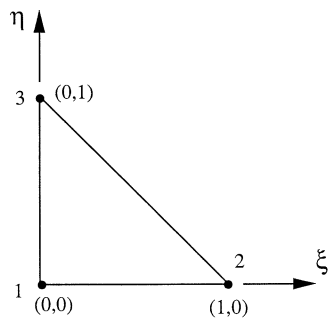


$$N_1 = ((x_2 y_3 - x_3 y_2) + (y_2 - y_3)x + (x_3 - x_2)y) / 2A$$

$$N_2 = ((x_3 y_1 - x_1 y_3) + (y_3 - y_1)x + (x_1 - x_3)y) / 2A$$

$$N_3 = ((x_1 y_2 - x_2 y_1) + (y_1 - y_2)x + (x_2 - x_1)y) / 2A$$

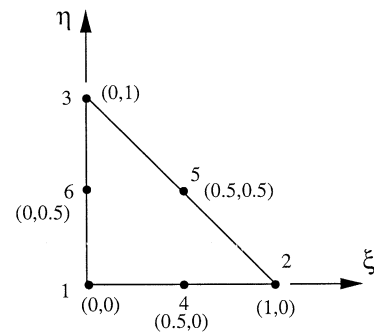
A = area of triangle



$$N_1 = 1 - \xi - \eta$$

$$N_2 = \xi$$

$$N_3 = \eta$$



$$N_1 = 2(1 - \xi - \eta)^2 - (1 - \xi - \eta)$$

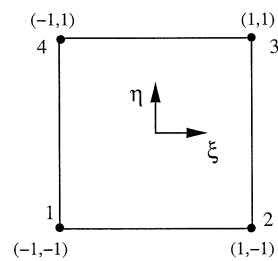
$$N_2 = 2\xi^2 - \xi$$

$$N_3 = 2\eta^2 - \eta$$

$$N_4 = 4\xi(1 - \xi - \eta)$$

$$N_5 = 4\eta\xi$$

$$N_6 = 4\eta(1 - \xi - \eta)$$

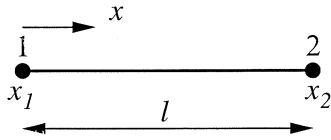


$$N_1 = (1 - \xi)(1 - \eta) / 4$$

$$N_2 = (1 + \xi)(1 - \eta) / 4$$

$$N_3 = (1 + \xi)(1 + \eta) / 4$$

$$N_4 = (1 - \xi)(1 + \eta) / 4$$



Hermitian element

$$N_1 = \frac{-(x-x_2)^2(-l+2(x_1-x))}{l^3}$$

$$M_1 = \frac{(x-x_1)(x-x_2)^2}{l^2}$$

$$N_2 = \frac{(x-x_1)^2(l+2(x_2-x))}{l^3}$$

$$M_2 = \frac{(x-x_1)^2(x-x_2)}{l^2}$$

Gauss integration in one dimension on the domain $(-1, 1)$ Using n Gauss integration points, a polynomial of degree $2n - 1$ is integrated exactly.

number of points n	location ξ_i	weight w_i
1	0	2
2	$-\frac{1}{\sqrt{3}}$	1
	$\frac{1}{\sqrt{3}}$	1
3	$-\sqrt{\frac{3}{5}}$	$\frac{5}{9}$
	0	$\frac{8}{9}$
	$\sqrt{\frac{3}{5}}$	$\frac{5}{9}$

The list of numerical answers for 3D7

1b)

$$\mathbf{k}_e = \begin{bmatrix} -\frac{\alpha}{2} + \frac{\beta}{l} & \frac{\alpha}{2} - \frac{\beta}{l} \\ -\frac{\alpha}{2} - \frac{\beta}{l} & \frac{\alpha}{2} + \frac{\beta}{l} \end{bmatrix}$$

2a)

$$\mathbf{f}^T = [-10 \quad 0 \quad -10 \quad -20]$$

2b)

$$\mathbf{K} = \begin{bmatrix} \text{---} & \text{---} & \text{---} & \text{---} \\ \text{---} & \text{---} & \text{---} & \text{---} \\ \text{---} & \text{---} & \frac{9}{4} & -\frac{9}{4} \\ \text{---} & \text{---} & -\frac{9}{4} & \left(\frac{13}{4} + \frac{13}{4} \cdot 2\right) \end{bmatrix}$$

2c)

$$\begin{bmatrix} T_3 \\ T_4 \end{bmatrix} = \begin{bmatrix} -8.44 \\ -4 \end{bmatrix}$$

$$q_n = 1.7285$$

3a)

$$\epsilon_{xx} = \frac{1}{5}$$

$$\epsilon_{xx} = \frac{1}{50}$$

- 4b) 1D: Reduction factor 4
2D: Reduction factor 2
3D: Reduction factor 1.59