

1.

- (a.i) From the discharge record, the base flow is 10 m³/s.
So, the runoff due to 1 hour excess rainfall is:

Duration (h)	0-1	1-2	2-3	3-4	4-5	5-6	...
Discharge (m ³ s ⁻¹)	6	18	12	4	0	0	...

(a.ii)

First, calculate the excess rainfall in each hour:

$$\text{Infiltration: } \int_{t_1}^{t_2} f \cdot dt = f_c(t_2 - t_1) - \frac{1}{K_f}(f_0 - f_c)(e^{-K_f t_2} - e^{-K_f t_1})$$

$$\text{Infiltration in the 1st hour: } 2(1-0) - \frac{1}{0.5}(10-2)(e^{-0.5 \times 1} - e^{-0.5 \times 0}) = 8.3 \text{ mm}$$

$$\text{Infiltration in the 2nd hour: } 2(2-1) - \frac{1}{0.5}(10-2)(e^{-0.5 \times 2} - e^{-0.5 \times 1}) = 5.8 \text{ mm}$$

$$\text{Excess rainfall in the 1st hour: } 20 - 8.3 = 11.7 \text{ mm}$$

$$\text{Excess rainfall in the 2nd hour: } 10 - 5.8 = 4.2 \text{ mm}$$

According to the unit hydrograph theory:

Duration (h)	0-1	1-2	2-3	3-4	4-5	5-6	6-7
Direct runoff in 1 st hour	6×1.17	18×1.17	12×1.17	4×1.17	0	0	0
Direct runoff in 2 nd hour		6×0.42	18×0.42	12×0.42	4×0.42	0	0
Base flow	10	10	10	10	10	10	10
Total (m ³ s ⁻¹)	17.02	33.58	31.6	19.72	11.68	10	10

The peak flow rate is 33.58 m³ s⁻¹.

(b.i)

Construct the S curve:

Time (hour)	0	2	6	10	14	18	22
Runoff (%)	0	3	21	56	83	95	100

The time of the concentration is 22 hour.

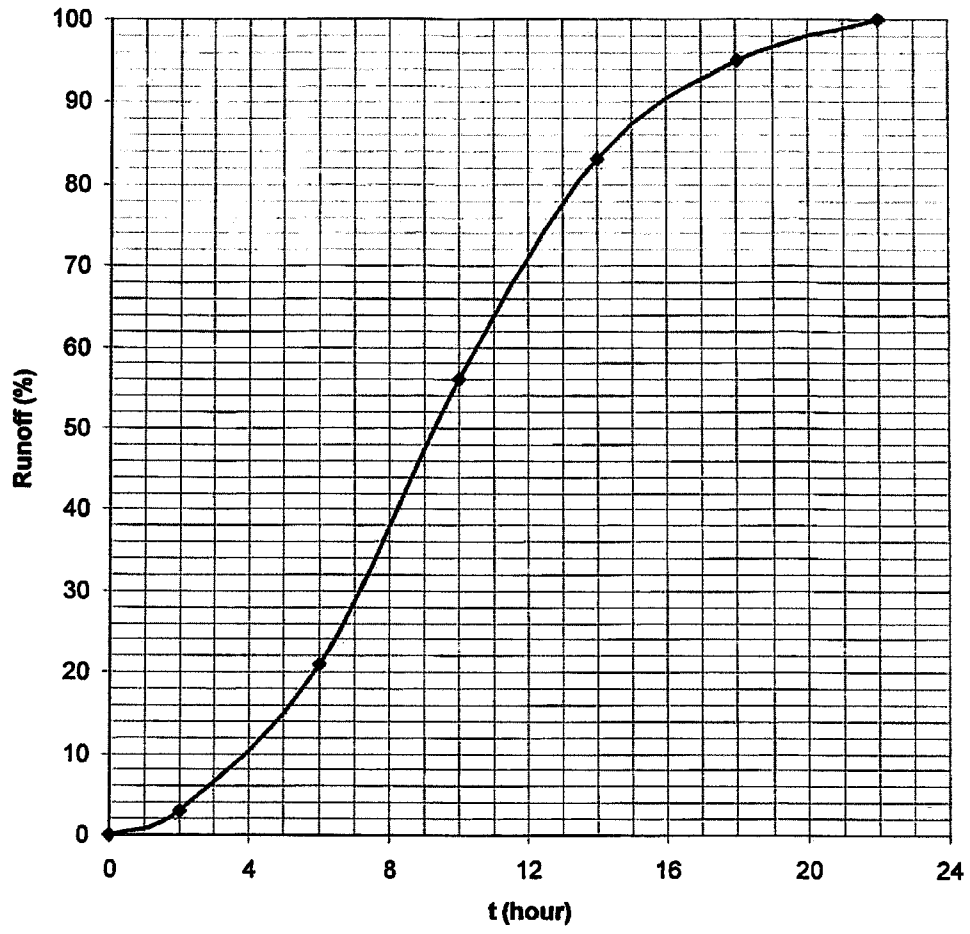
(b.ii)

Need to find the runoff proportions due to the 3 hour uniform rainfall.

Read from S curve:

Time (hour)	0	1.5	4.5	7.5	10.5	13.5	16.5	19.5	22.5	...
S-curve value	0	2	13	33	60	80	92	97	100	...
Shift S-curve by 3 hour		0	2	13	33	60	80	92	97	...
3-hour hydrograph	0	2	11	20	27	20	12	5	3	0

So, the peak flow rate occurs 10.5 hour (between 9 hours and 12 hours) after the rainfall starts.



2 (a.i)

Manning formula:
$$U = \frac{1}{n} \cdot R_h^{2/3} \cdot S_b^{1/2} = \frac{1}{n} \cdot \left(\frac{B \cdot h_0}{B + 2h_0} \right)^{2/3} \cdot S_b^{1/2}$$

$$Q = U \cdot B \cdot h_0 = \frac{1}{n} \cdot \frac{(B \cdot h_0)^{5/3}}{(B + 2h_0)^{2/3}} \cdot S_b^{1/2}$$

$$10 = \frac{1}{0.013} \cdot \frac{(6 \cdot h_0)^{5/3}}{(6 + 2h_0)^{2/3}} \cdot 0.001^{1/2}$$

$h_0 = 0.88$ m is the solution.

(a.ii)

Critical flow:
$$U = \sqrt{gh_c}$$

$$Q = U \cdot B \cdot h_c = \sqrt{gh_c} \cdot B \cdot h_c = g^{1/2} \cdot h_c^{3/2} \cdot B$$

$$10 = 9.81^{1/2} \cdot h_c^{3/2} \cdot 6$$

$h_c = 0.66$ m

(a.iii)

Gradually varied flow:
$$\frac{dh}{dx} = \frac{S_b - S_f}{1 - Fr^2} = \frac{S_b - \frac{n^2 \cdot U^2}{R_h^{4/3}}}{1 - Fr^2}$$

When $h = 0.85$ m:
$$U = \frac{10}{6 \times 0.85} \approx 1.96 \text{ m/s}$$

$$R_h = \frac{6 \times 0.85}{6 + 2 \times 0.85} \approx 0.662 \text{ m}$$

$$S_f = \frac{n^2 \cdot U^2}{R_h^{4/3}} = \frac{0.013^2 \cdot 1.96^2}{0.662^{4/3}} \approx 0.001125$$

$$Fr = \frac{U}{\sqrt{gh}} = \frac{1.96}{\sqrt{9.81 \times 0.85}} \approx 0.6788$$

$$\frac{dh}{dx} = \frac{S_b - S_f}{1 - Fr^2} = \frac{0.001 - 0.001125}{1 - 0.6788^2} \approx -2.32 \times 10^{-4}$$

When $h = 0.75$ m:
$$U = \frac{10}{6 \times 0.75} \approx 2.22 \text{ m/s}$$

$$R_h = \frac{6 \times 0.75}{6 + 2 \times 0.75} \approx 0.60 \text{ m}$$

$$S_f = \frac{n^2 \cdot U^2}{R_h^{4/3}} = \frac{0.013^2 \cdot 2.22^2}{0.60^{4/3}} \approx 0.001616$$

$$Fr = \frac{U}{\sqrt{gh}} = \frac{2.22}{\sqrt{9.81 \times 0.75}} \approx 0.8184$$

$$\frac{dh}{dx} = \frac{S_b - S_f}{1 - Fr^2} = \frac{0.001 - 0.001616}{1 - 0.8184^2} \approx -0.001865$$

So
$$\frac{0.75 - 0.85}{\Delta x} = \frac{0.001 - \frac{0.001125 + 0.001616}{2}}{1 - \left(\frac{0.6788 + 0.8184}{2} \right)^2} = -8.43 \times 10^{-4}$$

$$\Delta x = 118.6 \text{ m}$$

Or
$$\frac{0.75 - 0.85}{\Delta x} = \frac{-2.32 \times 10^{-4} - 0.001865}{2} \approx -1.05 \times 10^{-3}$$

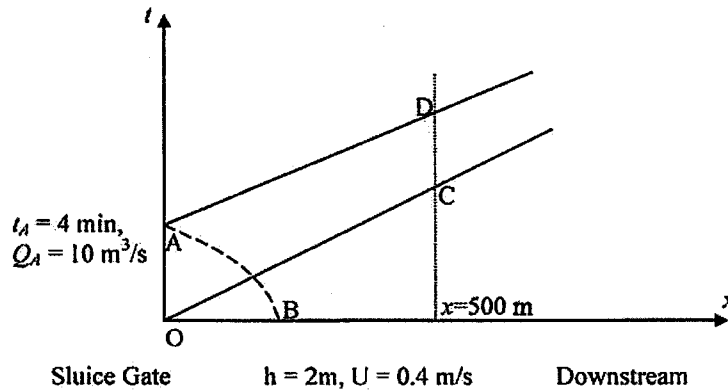
$$\Delta x = 95.2 \text{ m}$$

Or using
$$\frac{d}{dx} \left(h + \frac{U^2}{2g} \right) = S_b - S_f$$

$$\frac{\left(0.75 + \frac{2.22^2}{2 \times 9.81} \right) - \left(0.85 + \frac{1.96^2}{2 \times 9.81} \right)}{\Delta x} = 0.001 - \frac{0.001125 + 0.001616}{2}$$

$$\Delta x = 120.4 \text{ m}$$

(b.i)



Along the negative line AB: $(U - 2\sqrt{gh}) = \text{const}$

$$U_A - 2\sqrt{9.81h_A} = 0.4 - 2\sqrt{9.81 \times 2} = -8.46$$

Also,
$$Q_A = 5U_A h_A = 10$$

It can be easily shown that $h_A = 2.23 \text{ m}$ and $U_A = 0.897 \text{ m/s}$ are the solutions.

(b.ii) According to the straight positive line OC: $\frac{dx}{dt} = U_0 + \sqrt{gh_0}$

$$\frac{500}{t_C - 0} = 0.4 + \sqrt{9.81 \times 2} \Rightarrow t_C = 103.5 \text{ s}$$

(b.iii) According to the straight positive line AD: $\frac{dx}{dt} = U_A + \sqrt{gh_A}$

$$\frac{500}{t_D - 4 \times 60} = 0.897 + \sqrt{9.81 \times 2.23} \Rightarrow t_D = 329.7 \text{ s}$$

3 (a.i)

$$U = \frac{Q}{Bh} = \frac{200}{50 \times 2} = 2 \text{ m/s}$$

For uniform flow: $\tau_b = \rho g R_h S_b = \frac{g}{C^2} \rho \cdot U^2$

$$C = \sqrt{\frac{1}{R_h S_b}} \cdot U = \sqrt{\frac{1}{2 \times 0.002}} \times 2 = 31.6 \text{ m}^{1/2} \text{ s}^{-1}$$

Grain-related value: $C' = 7.8 \ln\left(\frac{12h}{k_s'}\right) = 7.8 \ln\left(\frac{12 \times 2}{0.01}\right) = 60.7 \text{ m}^{1/2} \text{ s}^{-1}$

$$\tau_b = \rho g R_h S_b = \frac{g}{C^2} \rho \cdot U^2$$

$$\theta = \frac{\tau_b}{g(\rho_s - \rho)d} = \frac{\rho g R_h S_b}{g(\rho_s - \rho)d} = \frac{h S_b}{(s-1)d} = \frac{2 \times 0.002}{(2.65-1) \times 0.01} = 0.24$$

$$\frac{q_b}{\sqrt{g(s-1) \cdot d^3}} = 8 \left[\left(\frac{C}{C'} \right)^{1.5} \theta - 0.047 \right]^{1.5}$$

$$\frac{q_b}{\sqrt{9.81 \times (2.65-1) \times 0.01^3}} = 8 \left[\left(\frac{31.6}{60.7} \right)^{1.5} \times 0.24 - 0.047 \right]^{1.5} = 0.0717$$

$$q_b = 2.9 \times 10^{-4} \text{ m}^3/(\text{s} \cdot \text{m}) = 2.9 \times 10^{-4} \times 2650 \times 50 = 38 \text{ kg/s}$$

(a.ii)

$$u_* = \sqrt{\frac{\tau_b}{\rho}} = \sqrt{g R_h S_b} = \sqrt{9.81 \times 2 \times 0.002} = 0.198 \text{ m/s}$$

$$D_y = 0.15 h u_* = 0.15 \times 2 \times 0.198 = 0.0594 \text{ m}^2/\text{s}$$

Consider the image of one channel bank:

$$\bar{c}(x, y) = 2 \times \frac{\dot{M}/h}{U \sqrt{4\pi \frac{x}{U} D_y}} \exp\left(-\frac{y^2}{4D_y x/U}\right)$$

According to $\bar{c}(x=1000, y=0) = 3 \text{ ppm}$

$$3 = 2 \times \frac{\dot{M}/2}{2 \times \sqrt{4 \times 3.14 \times \frac{1000}{2} \times 0.0594}}$$

$$\dot{M} = 115.6 = c_{\text{source}} \cdot Q_{\text{source}} = 100 \cdot Q_{\text{source}}$$

$$Q_{\text{source}} \approx 1.2 \text{ m}^3/\text{s}$$

(b.i)

$$d_* = d \cdot \left(\frac{g(s-1)}{\nu^2} \right)^{1/3} = 0.12 \times 10^{-3} \times \left(\frac{9.81 \times (2.65-1)}{10^{-12}} \right)^{1/3} = 3.036$$

$$w_s = \frac{v}{d} \left[\sqrt{10.36^2 + 1.049 \cdot d_*^3} - 10.36 \right]$$

$$w_s = \frac{10^{-6}}{0.12 \times 10^{-3}} \left[\sqrt{10.36^2 + 1.049 \times 3.036^3} - 10.36 \right] = 0.0111 \text{ m/s}$$

(b.ii)

$$\bar{c}(z=2) = 2.2 \text{ kg m}^{-3}, \quad \bar{c}(a=1) = 4.2 \text{ kg m}^{-3}$$

$$\frac{\bar{c}(z)}{\bar{c}(a)} = \left(\frac{h-z}{z} \cdot \frac{a}{h-a} \right)^{\frac{w_s}{k u_*}}$$

$$\frac{2.2}{4.2} = \left(\frac{3-2}{2} \cdot \frac{1}{3-1} \right)^{\frac{0.0111}{0.4 \times u_*}}$$

$$\ln(0.524) = \ln(0.25) \times \frac{0.0111}{0.4 \times u_*}$$

$$u_* = 0.0595 \text{ m/s}$$

$$\tau_b = \rho g h S_b = \rho \cdot u_*^2$$

$$9.81 \times 3 \times S_b = 0.0595^2$$

$$S_b = 1.2 \times 10^{-4}$$

4. (a)

$$\frac{1}{\sqrt{\lambda}} = -2 \log_{10} \left(\frac{k_s}{3.7D} + \frac{2.51}{\text{Re} \sqrt{\lambda}} \right), \text{ where } \text{Re} = \frac{UD}{\nu}$$

In hydraulically smooth flow, the friction factor does not depend on k_s , so

$$\frac{1}{\sqrt{\lambda}} = -2 \log_{10} \left(\frac{2.51}{\text{Re} \sqrt{\lambda}} \right)$$

$$\frac{\sqrt{8g}}{\sqrt{\lambda}} = -2\sqrt{8g} \log_{10} \left(\frac{2.51\nu\sqrt{8g}}{UD\sqrt{8g}\sqrt{\lambda}} \right)$$

Note $C = \sqrt{\frac{8g}{\lambda}}$ and $R_h = \frac{D}{4}$, so

$$C = -2\sqrt{8g} \log_{10} \left(\frac{2.51\nu C}{4UR_h\sqrt{8g}} \right) = -2\sqrt{8g} \log_{10} \left(\frac{\nu \cdot C}{14.1 \cdot UR_h} \right)$$

$$C = 17.72 \cdot \log_{10} \left(\frac{14.1 \cdot UR_h}{\nu \cdot C} \right) = 17.72 \cdot \frac{\ln \left(\frac{14.1 \cdot UR_h}{\nu \cdot C} \right)}{\ln(10)} = 7.7 \cdot \ln \left(\frac{14.1 \cdot UR_h}{\nu \cdot C} \right)$$

(b.i)

$$H_f = \lambda \frac{L U^2}{D 2g} = 0.025 \times \frac{70}{0.3} \times \frac{\left(\frac{Q}{3.14 \times 0.3^2 / 4} \right)^2}{2 \times 9.81} = 59.6 Q^2$$

$$H_l = \sum \zeta \frac{U^2}{2g} = 2.5 \times \frac{\left(\frac{Q}{3.14 \times 0.3^2 / 4} \right)^2}{2 \times 9.81} = 25.5 Q^2$$

$$\text{System curve: } H = 15 + H_f + H_l = 15 + 85.1 Q^2$$

Equating the system and pump curves:

$$15 + 85.1 Q^2 = 22.9 + 10.7 \cdot Q - 111.0 \cdot Q^2$$

So, $Q = 0.23 \text{ m}^3/\text{s}$, $H = 19.5 \text{ m}$

(b.ii)

For two pumps in parallel, the pumping head remains the same at double discharge.

The characteristic curve can be written as:

$$H = 22.9 + 10.7 \cdot \frac{Q}{2} - 111.0 \cdot \left(\frac{Q}{2} \right)^2$$

Proceeding as before, the duty point is: $Q = 0.29 \text{ m}^3/\text{s}$, $H = 22.2 \text{ m}$ or 22.1

For two pumps in series, the pumping head doubles at the same discharge.

The characteristic curve can be written as:

$$H = (22.9 + 10.7 \cdot Q - 111.0 \cdot Q^2) \times 2$$

Proceeding as before, the duty point is: $Q = 0.35 \text{ m}^3/\text{s}$, $H = 25.4 \text{ m}$ or 26.1 m

(b.iii)

The dimensionless form of the pump characteristic should be the same for all speeds. As the impellor size is fixed, $\frac{Q}{N}$ and $\frac{H}{N^2}$ have the unique relationship.

Express the pump characteristic in terms of $\frac{Q}{N}$ and $\frac{H}{N^2}$, where $N = 2900$ rpm

$$H = 22.9 + 10.7 \cdot Q - 111.0 \cdot Q^2$$

$$2900^2 \times \left(\frac{H}{N^2}\right) = 22.9 + 10.7 \times (2900) \times \left(\frac{Q}{N}\right) - 111.0 \times 2900^2 \times \left(\frac{Q}{N}\right)^2$$

This equation applies to all N. Letting $N = 3450$ rpm to get the new characteristic.

$$\left(\frac{2900}{3450}\right)^2 \times H = 22.9 + 10.7 \times \left(\frac{2900}{3450}\right) \times (Q) - 111.0 \times \left(\frac{2900}{3450}\right)^2 \times Q^2$$

$$H = 32.41 + 12.73 \cdot Q - 111.0 \cdot Q^2$$

