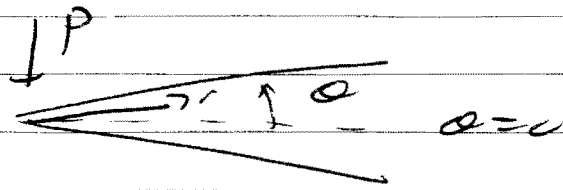


3C7 2013

Q1.

a)



$$\phi = -Cr\theta \cos\theta$$

$$\frac{\partial\phi}{\partial r} = -C\theta \cos\theta$$

$$\nabla_{\theta\theta} = \frac{\partial^2\phi}{\partial r^2} = \frac{\partial^2\phi}{\partial r^2} = 0$$

$$\frac{\partial\phi}{\partial\theta} = -Cr \cos\theta + Cr\theta \sin\theta$$

$$\frac{\partial^2\phi}{\partial\theta^2} = 2Cr \sin\theta + Cr\theta \cos\theta$$

$$\nabla_{rr} = -\frac{\partial}{\partial r} \left( \frac{1}{r} \frac{\partial\phi}{\partial\theta} \right) = 0$$

$$\nabla_{rr} = -\frac{C\theta \cos\theta}{r} + \frac{C\theta \cos\theta}{r} + \frac{2C \sin\theta}{r}$$

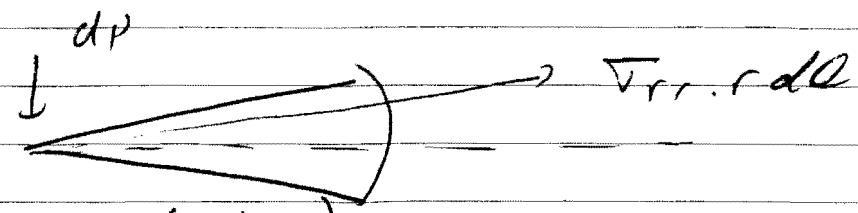
$$= \frac{2C \sin\theta}{r}$$

BC on free edges:  $\nabla_{\theta\theta} = 0$  ✓

$\nabla_{rr} = 0$  ✓

both satisfied.

Q1 (b) Symmetry : anti-symmetry about  $\theta = 0$



Equilibrium : (vertical)

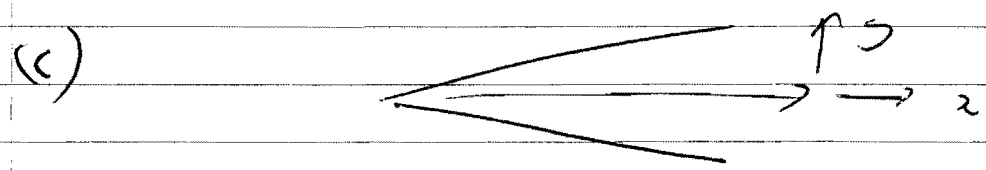
$$dP = \nabla_{rr} \cdot r d\theta \sin \theta$$

$$= 2C \sin^2 \theta d\theta$$

Integrate,  $P = 2C \int_{-\alpha}^{\alpha} \sin^2 \theta d\theta$

$$\rightarrow C = P / (2\alpha - \sin 2\alpha)$$

Horizontal equilibrium satisfied since  $\nabla_{rr}$  anti-symmetric in  $\theta$ .

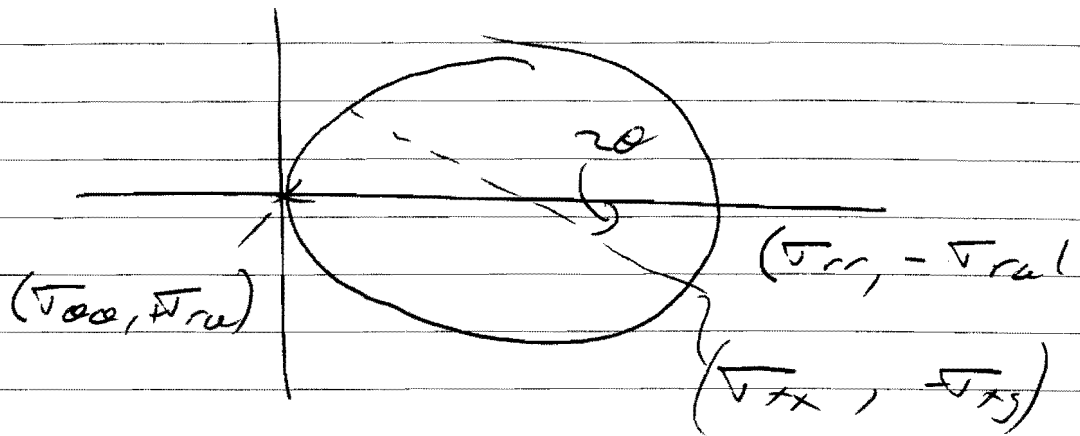


$$r = (x^2 + y^2)^{1/2}, \quad \sin \theta = y / (x^2 + y^2)^{1/2}$$

$$\theta = \tan^{-1} (y/x) \quad | \quad \cos \theta = x / (x^2 + y^2)^{1/2}$$

$$\nabla_{rr} = \frac{2C y}{(x^2 + y^2)}$$

## Mohr's circle



$$\tau_{xy} = \frac{\sigma_{xx} \sin 2\theta}{2}$$

$$\sin 2\theta = 2 \sin \theta \cos \theta$$

$$\rightarrow \tau_{xy} = \frac{2 \cdot C \cdot x \cdot y^2}{(x^2 + y^2)^2}$$

- Max shear stress at top & bottom.

Q2. (i) (ii)

Use Lami

$$\sigma_{rr} = A - \frac{B}{r^2} - \frac{E\alpha}{r^2} \int_0^r r T dr$$

$$\sigma_{\theta\theta} = A + \frac{B}{r^2} + \frac{E\alpha}{r^2} \int_0^r r T dr - E\alpha T$$

Finite stresses @  $r=0 \rightarrow B=0$ 

$$\begin{aligned} \text{Nud } \frac{E\alpha}{r^2} \int_0^r r T dr &= \frac{2E\alpha T_0}{r^2} \int_0^r \left( R - \frac{R^3}{D^2} \right) dR \\ &= E\alpha T_0 \left[ 1 - \frac{1}{2} \left( \frac{r}{D} \right)^2 \right] \end{aligned}$$

$$\sigma_{rr} = 0 \text{ @ } r = D/2$$

$$\rightarrow \cancel{\sigma_{rr}} = A = \frac{7}{8} E\alpha T_0$$

$$\sigma_{rr} = -\frac{E\alpha T_0}{8} + \frac{E\alpha T_0}{2} \left( \frac{r}{D} \right)^2$$

$$\begin{aligned} \sigma_{\theta\theta} &= \frac{15}{8} E\alpha T_0 - \frac{E\alpha T_0}{2} \left( \frac{r}{D} \right)^2 - 2E\alpha T_0 \\ &\quad + 2E\alpha T_0 \left( \frac{r}{D} \right)^2 \end{aligned}$$

$$= -\frac{E\alpha T_0}{8} + \frac{3E\alpha T_0}{2} \left( \frac{r}{D} \right)^2$$

Q2 (a)(ii)

At  $r = 0$   $\sigma_{rr} = \frac{-E\alpha T_0}{8}$

$\sigma_{\theta\theta} = \frac{-E\alpha T_0}{8}$

At  $r = D/2$ ,  $\sigma_{rr} = 0$ ,  $\sigma_{\theta\theta} = \frac{2E\alpha T_0}{8}$

Max case @  $r = D/2$

$\frac{2E\alpha T_0}{8} = 189 \text{ MPa}$

Safety factor =  $\frac{350}{189} = 1.85$

B) Find edge expansion in unrestrained case:

$\epsilon_{\theta\theta} = \frac{1}{E} (\sigma_{\theta\theta} - \nu \sigma_{rr}) + \alpha \Delta T$

At  $r = \frac{D}{2}$ ,  $\sigma_{rr} = 0$ ,  $\Delta T = \frac{3T_0}{2} \Rightarrow \Delta T = \frac{T_0}{2}$

$\epsilon_{\theta\theta} = \frac{\alpha T_0}{4} + \frac{\alpha T_0}{2} = \frac{3\alpha T_0}{4}$

Need to provide change in uniform stress to give equal but opposite strain:

$\Delta \sigma = \frac{E}{1-\nu^2} \left( -\frac{\alpha T_0}{4} - \frac{\alpha T_0 \nu}{2} \right)$   
 $= \frac{-E}{1-\nu} \frac{D\alpha T_0}{4}$

$$\therefore \Delta \sigma_{rr} = \frac{F}{4(1-\nu)} - \frac{E\alpha T_0}{(1-\nu)}$$

$$\Delta \sigma_{\theta\theta} = \frac{F}{4(1-\nu)} - \frac{E\alpha T_0}{(1-\nu)}$$

Does not change  $\sigma_{\theta\theta} - \sigma_{rr}$

Consider  $r=0$  (worst case compressive stress)

$$\begin{aligned} \sigma_{rr} - \sigma_{\theta\theta} &= -\frac{E\alpha T_0}{8(1-\nu)} \left[ 1-\nu + 8\nu \right] \\ &= \frac{-E\alpha T_0 (\nu)}{8(1-\nu)} = \sigma_{rr} - \sigma_{z_z} \\ &\quad \uparrow \\ &\quad 0 \end{aligned}$$

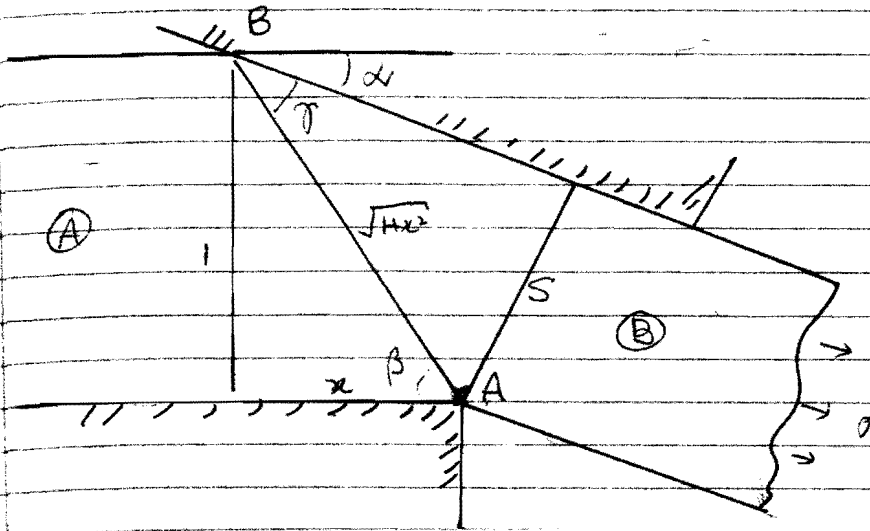
$$\rightarrow \quad \cancel{1984.8 \text{ MPa}} \quad 1310 \text{ MPa}$$

$$\text{Safety factor} \quad \frac{350}{\cancel{1984.5}} = \cancel{0.17}$$

$$\frac{350}{1310} = 0.26$$

NOT SAFE!

Q3



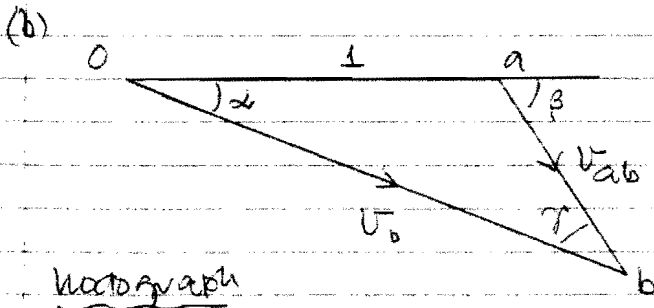
(a) Consider angles at B  $(\frac{\pi}{2} - \beta) + \gamma + \alpha = \frac{\pi}{2}$

$$\alpha = \beta - \gamma \quad \gamma = \beta - \alpha$$

$$\sin \alpha = \sin(\beta - \gamma) = \sin \beta \cos \gamma - \cos \beta \sin \gamma$$

But  $\sin \beta = \frac{1}{\sqrt{1+x^2}}$ ;  $\cos \beta = \frac{x}{\sqrt{1+x^2}}$  and  $\sin \gamma = \frac{s}{\sqrt{1+x^2}}$ ;  $\cos \gamma = \frac{\sqrt{1+x^2-s^2}}{\sqrt{1+x^2}}$

$$\sin \alpha = \frac{\sqrt{1+x^2-s^2} - xs}{1+x^2}$$



Velocity triangle

Upper Bound given by

$$\text{or } s \cdot U_b = r \cdot \text{lab} \cdot U_{ab}$$

from velocity triangle

$$\frac{U_b}{\sin(\pi - \beta)} = \frac{U_{ab}}{\sin \alpha}$$

$$\text{i.e. } U_b = \frac{\sin \beta}{\sin \alpha} \cdot U_{ab}$$

$$\text{T.S. } \frac{\sin \beta}{\sin \alpha} \cdot U_{ab} = r \cdot \frac{1}{\sin \beta} \cdot U_{ab}$$

$$\frac{r s}{r} = \frac{\sin \alpha}{\sin^2 \beta} = \frac{\sqrt{1+x^2-s^2} - xs}{1+x^2} \cdot \frac{1+x^2}{1}$$

$$\text{i.e. } \underline{\underline{\sigma^2 S / R}} = \frac{\sqrt{1+x^2-s^2}}{s} - x$$

~~Given~~

(c) Now minimize  $\sigma$  for given  $S$

$$\frac{d(\sigma^2 S / R)}{dx} = \frac{1}{2} (1+x^2-s^2)^{-1/2} \cdot 2x - S = 0$$

$$\text{When } x(1+x^2-s^2)^{-1/2} = S$$

$$\text{i.e. } x = S(1+x^2-s^2)^{1/2}$$

$$\text{or } x^2 = S^2(1+x^2-s^2)$$

$$x^2(1-s^2) = S^2(1-s^2)$$

$$\text{i.e. } \underline{\underline{x = S}}$$

$$\text{then } \sigma^2 S = \frac{\sqrt{1+S^2-S^2}}{1+S^2} - S^2 = \frac{1-S^2}{1+S^2}$$

(d) we have  $\underline{\underline{\frac{\sigma}{R} = \frac{\sqrt{1+x^2-s^2}}{s} - x}}$

$$\text{So if } x=S \quad \frac{\sigma}{R} = \frac{1-S^2}{S}$$

$$\text{If } \sigma = 2R \quad \text{then } \frac{1-S^2}{S} = 2$$

$$\text{in unit } \text{i.e. } 1-S^2 = 2S$$

$$\text{or } S^2 + 2S - 1 = 0$$

$$\therefore S = \frac{-2 \pm \sqrt{8}}{2} \quad \text{i.e. } S = \sqrt{2} - 1 = \underline{\underline{0.414}}$$

$$\text{i.e. reduction of area } \Rightarrow 1-S = \underline{\underline{58.6\%}}$$

$$\text{When } \alpha = \frac{1 - (\sqrt{2}-1)^2}{1 + (\sqrt{2}-1)^2} = \frac{1 - 2 + 2\sqrt{2} - 1}{1 + 2 - 2\sqrt{2} + 1} = \frac{-2 + 2\sqrt{2}}{4 - 2\sqrt{2}} \quad \therefore \alpha = \underline{\underline{45^\circ}}$$

$$\text{So } \beta = \frac{1}{(4-2\sqrt{2})^2} = 67.5^\circ$$



Q4

(a)  $(\sigma_1 - \sigma_2)^2 + (\sigma_2 - \sigma_3)^2 + (\sigma_3 - \sigma_1)^2 + 2\sigma_3(\rho - 1)(\sigma_1 + \sigma_2 + \sigma_3) = 2\rho\sigma_3^2$

If  $\rho = 3$  and conditions are plane stress i.e.  $\sigma_3 = 0$

$(\sigma_1 - \sigma_2)^2 + \sigma_2^2 + \sigma_1^2 + 4(\sigma_1 + \sigma_2)\sigma_3 = 6\sigma_3^2$

If  $\sigma_1 = Y, \sigma_2 = 0$  then

$Y^2 + Y^2 + 4Y\sigma_3 = 6\sigma_3^2$

i.e.  $Y^2 + 2Y\sigma_3 = 3\sigma_3^2$

So satisfied by  $Y = \sigma_3$

hence Stassi condition becomes

$(\sigma_1 - \sigma_2)^2 + \sigma_2^2 + \sigma_1^2 + 4Y(\sigma_1 + \sigma_2) = 6Y^2$

Von Mises for plane stress would be

$(\sigma_1 - \sigma_2)^2 + \sigma_2^2 + \sigma_1^2 = 2Y^2$

(b) In simple compression (say)  $\sigma_2 = -\sigma, \sigma_1 = 0$

Stassi  $2\sigma^2 - 4Y\sigma = 6Y^2$

letting  $\sigma = \sigma/Y$   $\sigma^2 - 2\sigma - 3 = 0$

So that  $|\sigma| = \frac{2 \pm \sqrt{4+12}}{2} = 1 \pm 2$  i.e.  $\sigma = -3$

Von Mises  $2\sigma^2 = 2Y^2$   $\frac{\sigma}{Y} = -1$

Equal biaxial tension  $\sigma_1 = \sigma_2 = \sigma$  say

Von Mises  $2\sigma^2 = 2Y^2$   $\sigma/Y = 1$

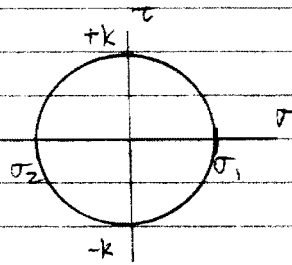
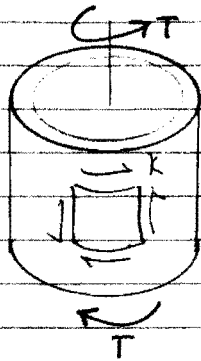
Stassi  $\sigma^2 + \sigma^2 + 8\sigma = 6$

i.e.  $\sigma^2 + 4\sigma - 3 = 0$   $|\sigma| = \frac{-4 \pm \sqrt{16+12}}{2}$

$\frac{\sigma}{Y} = 0.645$

$\frac{\sqrt{25-4}}{2}$   
 ~~$\frac{\sqrt{25-4}}{2}$~~   
 ~~$\frac{\sqrt{25-4}}{2}$~~

(c) thin walled tube in torsion



Mohr's circle

$$\sigma_1 = k\tau, \sigma_2 = -k\tau$$

Von Mises

$$4k^2 + k^2 + k^2 = 2Y^2$$

$$\therefore k = \frac{Y}{\sqrt{3}}$$

$$\text{i.e. } \frac{\sigma_1}{Y} = \frac{1}{\sqrt{3}}; \frac{\sigma_2}{Y} = -\frac{1}{\sqrt{3}}$$

Stassi

$$4k^2 + k^2 + k^2 = 6Y^2$$

$$\therefore k = Y$$

$$\therefore \frac{\sigma_1}{Y} = 1; \frac{\sigma_2}{Y} = -1$$

(d) Comparison of Stassi & von Mises yield loci in principal stress axes shown below

