

1. (a) Line tension T is the force acting along the dislocation line, which equals the energy per unit length of the dislocation:

$$T \sim Gb^2/2$$

To minimise the energy (and hence the tension T), the dislocation tends to be straight in the absence of stresses. (15%)

(b) After bowing, a segment of length dx of the dislocation has the new length $\sqrt{1+u'^2} dx$, with $u' \equiv du/dx$. The total length of dislocation is $\int_{-l/2}^{l/2} \sqrt{1+u'^2} dx$, and the total energy due to line tension is

$$\int_{-l/2}^{l/2} T \sqrt{1+u'^2} dx$$

On the other hand, during bowing, work is done against resistance τb , namely

$$\int_{-l/2}^{l/2} \tau b u(x) dx$$

(25%)

\Rightarrow Total energy of the bowed segment is

$$E = \int_{-l/2}^{l/2} T \sqrt{1+u'^2} dx - \int_{-l/2}^{l/2} \tau b u(x) dx$$

(c) The configuration of the bowed-out segment that minimises the energy is obtained by solving:

$$\frac{d}{dx} \left[\frac{T u'}{\sqrt{1+u'^2}} \right] = -\tau b$$

\Rightarrow

$$\frac{T u'}{\sqrt{1+u'^2}} = -\tau b x + C_1$$

1. (c) continued

$$\Rightarrow \frac{u'^2}{1+u'^2} = \left(\frac{\tau b x}{T} + c \right)^2$$

From this, it can be shown that

$$\frac{du}{dx} = \pm \frac{\frac{\tau b x}{T} + c}{\sqrt{1 - \left(\frac{\tau b x}{T} + c \right)^2}} \quad c=0 \quad \frac{du}{dx} = 0 \text{ at } x=0$$

Choose "-" because $\frac{du}{dx} < 0$ when $x > 0$

Integration

$$\begin{aligned} \Rightarrow u(x) &= - \int \frac{\tau b x / T}{\sqrt{1 - (\tau b x / T)^2}} dx + d \\ &= \frac{T}{\tau b} \sqrt{1 - \left(\frac{\tau b x}{T} \right)^2} + d = \sqrt{\left(\frac{T}{\tau b} \right)^2 - x^2} + d \end{aligned}$$

But $u(\pm \frac{l}{2}) = 0 \Rightarrow d = - \sqrt{\left(\frac{T}{\tau b} \right)^2 - \frac{l^2}{4}} \quad (45\%)$

$$\Rightarrow u(x) = \sqrt{\left(\frac{T}{\tau b} \right)^2 - x^2} - \sqrt{\left(\frac{T}{\tau b} \right)^2 - \frac{l^2}{4}}$$

(d) When $|u'| \ll 1$, the differential equation simplifies to

$$T \frac{d^2 u}{dx^2} = -\tau b \quad \text{or} \quad \frac{d^2 u}{dx^2} = -\frac{\tau b}{T}$$

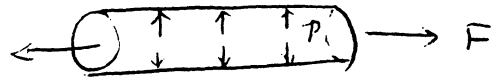
integration

$$\Rightarrow u(x) = -\frac{\tau b}{2T} x^2 + c_1 x + d_1 \quad (15\%)$$

But $u(\pm \frac{l}{2}) = 0 \Rightarrow c_1 = 0, d_1 = \frac{\tau b}{2T} \frac{l^2}{4}$

$$\Rightarrow u(x) = \frac{\tau b}{2T} \left(\frac{l^2}{4} - x^2 \right)$$

2. (a)



$$\sigma_H = \frac{PR}{t}, \quad \sigma_L = \frac{F}{2\pi R t}, \quad \sigma_T = 0$$

If $\sigma_H > \sigma_L$, the ordering of principal stresses is $\sigma_1 = \sigma_H, \sigma_2 = \sigma_L, \sigma_3 = 0$

If $\sigma_H < \sigma_L$, $\sigma_1 = \sigma_L, \sigma_2 = \sigma_H, \sigma_3 = 0$

For aluminum $\dot{\epsilon}_i = \lambda \sigma_i' = \lambda (\sigma_i - \sigma_m)$, $\dot{\epsilon}_2 = \lambda (\sigma_2 - \sigma_m)$, $\dot{\epsilon}_3 = \lambda (\sigma_3 - \sigma_m)$

mean stress $= (\sigma_1 + \sigma_2 + \sigma_3) / 3$

$$\Rightarrow (\dot{\epsilon}_1 - \dot{\epsilon}_2) = \lambda (\sigma_1 - \sigma_2) \text{ and so on}$$

$$\Rightarrow \dot{\epsilon}_e = \sqrt{\frac{2}{9} [(\dot{\epsilon}_1 - \dot{\epsilon}_2)^2 + (\dot{\epsilon}_2 - \dot{\epsilon}_3)^2 + (\dot{\epsilon}_3 - \dot{\epsilon}_1)^2]}$$

$$= \sqrt{\frac{2}{9} \lambda^2 [(\sigma_1 - \sigma_2)^2 + (\sigma_2 - \sigma_3)^2 + (\sigma_3 - \sigma_1)^2]} = \frac{2}{3} \lambda \sigma_e$$

But $\dot{\epsilon}_e = \dot{\epsilon}_0 \left(\frac{\sigma_e}{\sigma_0}\right)^6 \Rightarrow \lambda = \frac{3}{2} \frac{\dot{\epsilon}_0}{\sigma_0} \left(\frac{\sigma_e}{\sigma_0}\right)^5$ (40%)

$$\Rightarrow \dot{\epsilon}_1 = \frac{3}{2} \frac{\dot{\epsilon}_0}{\sigma_0} \left(\frac{\sigma_e}{\sigma_0}\right)^5 (\sigma_1 - \sigma_m), \quad \dot{\epsilon}_2 = \dots, \quad \dot{\epsilon}_3 = \dots$$

(b) For copper

Assume $\sigma_H > \sigma_L$

$$\sigma_1 = \sigma_H = \frac{PR}{t}, \quad t_f^{Cu} = A \sigma_H^{-b} \quad (\text{swap } \sigma_H \text{ with } \sigma_L \text{ if } \sigma_H < \sigma_L)$$

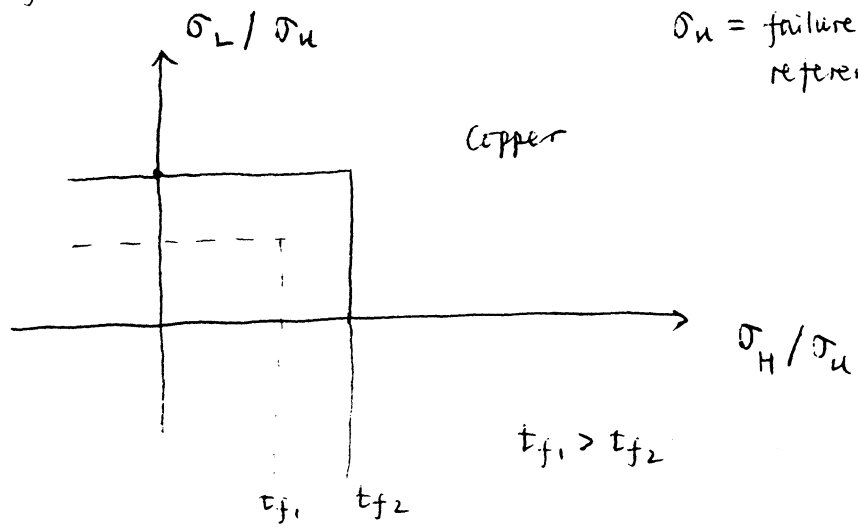
For aluminum alloy

$$\sigma_e = \sqrt{\frac{1}{2} [(\sigma_1 - \sigma_2)^2 + (\sigma_2 - \sigma_3)^2 + (\sigma_3 - \sigma_1)^2]} = \sqrt{\frac{1}{2} [\sigma_H^2 + \sigma_L^2 + (\sigma_H - \sigma_L)^2]}$$

$$= \frac{\sigma_H}{\sqrt{2}} [1 + \alpha^2 + (1 - \alpha)^2]^{1/2}, \quad \alpha = \frac{\sigma_L}{\sigma_H}$$

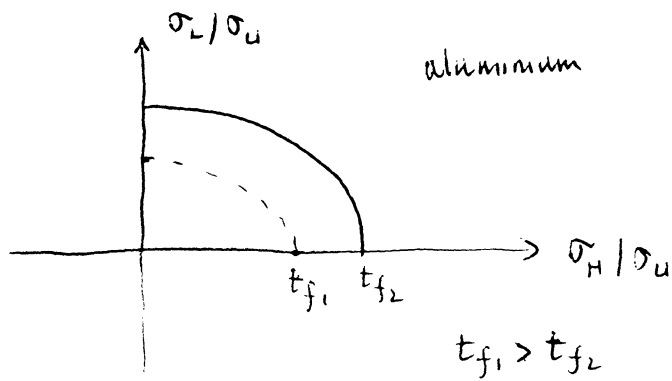
$$t_f^{Al} = A \sigma_e^{-b}$$

σ_u = failure stress at a reference temp



(25%)

2. (b) continued



(c) If $\sigma_H > \sigma_L \Rightarrow \alpha < 1$

$$\Rightarrow I = \frac{t_f^{Cu}}{t_f^{Al}} = \left(\frac{\sigma_e}{\sigma_H} \right)^6 = \left\{ \frac{1}{2} [1 + \alpha^2 + (1-\alpha)^2] \right\}^3$$

If $I = 1$, $1 + \alpha^2 + (1-\alpha)^2 = 2 \Rightarrow \alpha(\alpha-1) = 0$, $\alpha = 0$ or 1

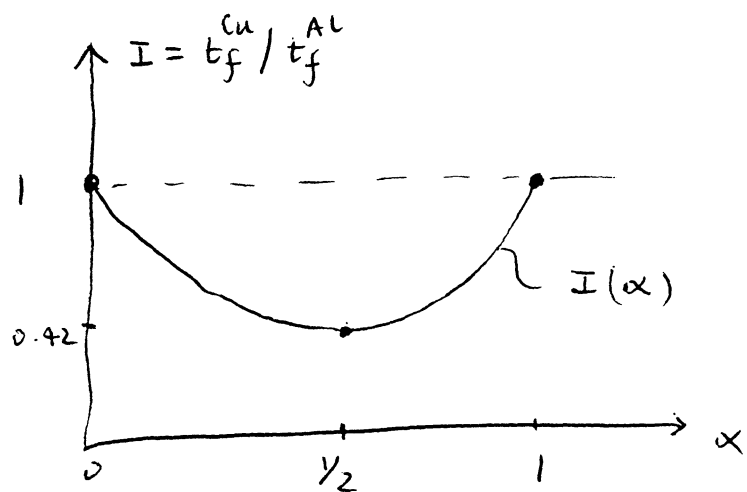
At minimum of I , $\frac{dI}{d\alpha} = 0$

$$\Rightarrow 2\alpha - 2(1-\alpha) = 0 \Rightarrow \alpha = \frac{1}{2}$$

$$\Rightarrow \frac{t_f^{Cu}}{t_f^{Al}} = \begin{cases} 1, & \alpha = 0 \quad (\sigma_L = 0) \\ 1, & \alpha = 1 \quad (\sigma_L = \sigma_H) \\ \text{minimum}, & \alpha = \frac{1}{2} \quad (\sigma_L = 2\sigma_H) \end{cases} \quad (35\%)$$

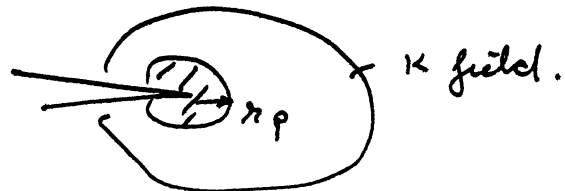
$$I_{\text{minimum}} = 0.42$$

Note that at $I = I_{\text{minimum}}$, $\sigma_L = 2\sigma_H$, which corresponds to the case of internally pressurized cylindrical vessel with closed ends.



Q3
(a)

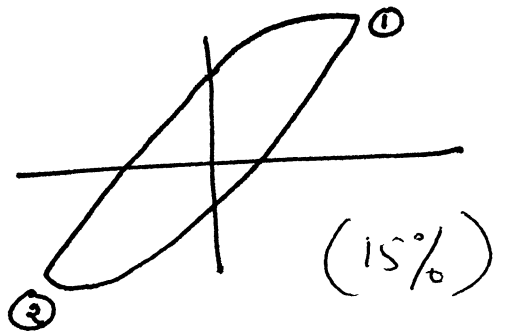
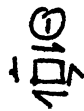
Small scale yielding: In small scale yielding, the plastic zone size $r_p \sim \frac{K^2}{\pi \sigma_Y^2}$. This is much less than the leading dimensions like plate size l , crack length a etc.



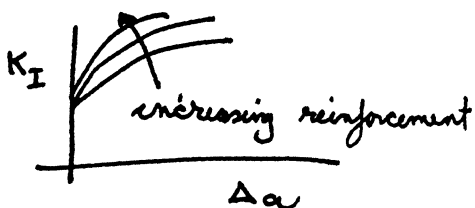
In SSY, the non-linear zone is completely embedded within an outer elastic K-field which determines the state in the non-linear zone & thus K is an adequate parameter to correlate fracture.

In large scale yielding, the plastic zone is of the order of the leading dimensions. No ~~K~~ K-field exists but J might be used to correlate fracture. (20%)

(b) Hysteresis in the stress versus strain curve is the origin of R-curves in metals



The R-curve effect increases with fibre re-inforcement i.e.



due to more crack bridging by the fibres in the wake of the crack.

c
(1)

$$K_I = \frac{2P}{\sqrt{2\pi L}} \quad \& \quad \sigma_{yy} = \frac{K_I}{\sqrt{2\pi x}} \quad @ \quad \theta = 0^\circ$$

By the Irwin criterion (for plane stress) yielding occurs

when

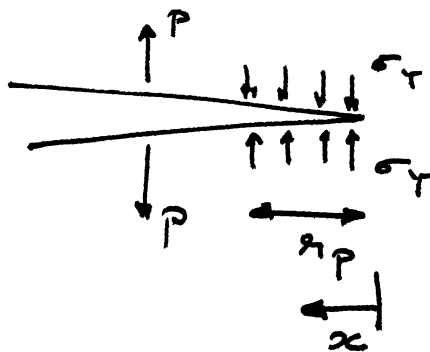
$$\sigma_{yy} = \sigma_Y$$

$$\text{i.e. } r_p = \frac{1}{2\pi} \left(\frac{K_I}{\sigma_Y} \right)^2$$

$$r_p = \frac{L}{\pi^2} \left(\frac{P}{\sigma_Y L} \right)^2$$

(15%)

(ii)



$$K_I^{(P)} \text{ at fictitious crack tip} = \frac{2P}{\sqrt{2\pi(L+r_p)}}$$

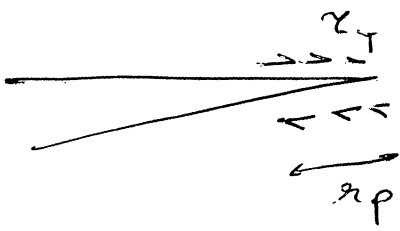
(25%)

$$K_I^{(\sigma_Y)} = - \int_0^{r_p} \frac{2\sigma_Y}{\sqrt{2\pi x}} dx = - \frac{4\sigma_Y r_p^{\frac{1}{2}}}{\sqrt{2\pi}}$$

$$K_I = K_I^{(P)} + K_I^{(\sigma_Y)} = 0$$

$$\text{i.e. } r_p = \frac{L}{2} \left[\left\{ 1 + \left(\frac{P}{L\sigma_Y} \right)^2 \right\}^{\frac{1}{2}} - 1 \right]$$

(d)

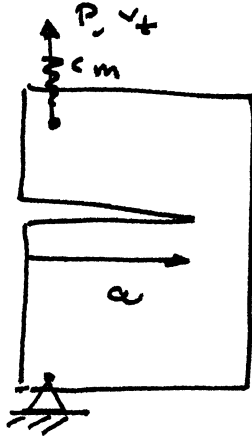


$$K_{II} = \int_0^{x_p} P \frac{2x_T}{\sqrt{2\pi x}} dx$$
$$= x_T \sqrt{\frac{2}{\pi}} 2 x_p^{1/2}$$

$$x_p = \frac{II}{8} \left(\frac{K_{II}}{x_T} \right)^2$$

(25%)

4(a)



$v = c(a) P$
 ↑
 compliance of cracked body

$$v_t = v + c_m P$$

$$E = \frac{1}{2} P v + \frac{1}{2} P (v_t - v)$$

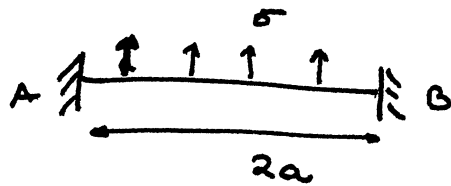
$$= \frac{1}{2} \frac{v^2}{c} + \frac{1}{2} \frac{(v_t - v)^2}{c_m}$$

$$G = -\frac{\partial E}{B \partial a} = \frac{1}{2B} \left[\frac{v^2}{c^2} \frac{dc}{da} - \frac{v}{c} \frac{dv}{da} + \frac{v_t - v}{c_m} \frac{dv}{da} \right]$$

$= 0 \because P = \frac{v}{c} = \frac{v_t - v}{c_m}$

ie $G = \frac{1}{2B} P^2 \frac{dc}{da}$ (30%)

(b) (i)



bending moment $M_A = -\frac{\sigma B a^2}{3}$

ie $M = -\frac{\sigma B}{6} (2a^2 - 6ax + 3x^2)$
 $0 \leq x \leq a$

$$U = 4 \int_0^a \frac{M^2 dx}{2EI} = \frac{8}{15} \frac{\sigma^2 a^5 B}{E h^3} \quad \left(\because I = \frac{B h^3}{12} \right)$$

$G = \frac{\partial U}{\partial A}$ where $A = 2aB$ (+ sign \because constant load)

$$G = \frac{4 \sigma^2 a^4}{3 E h^3} \quad (40\%)$$

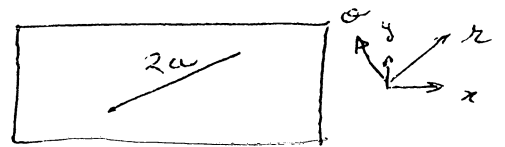
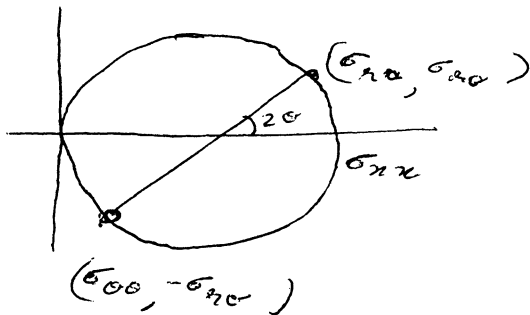
(ii)

$$K_I = \sqrt{E\epsilon} \quad (\text{plane stress})$$

$$= \frac{2\sigma a^2}{\sqrt{3h^3}}$$

(10%)

(c) $\sigma_{xx} = \frac{F}{2\pi R t}$, $\sigma_{yy} = 0$, $\sigma_{xy} = 0$



$$\sigma_{yy} = \frac{F}{4\pi R t} (1 - \cos 2\theta)$$

$$\sigma_{xy} = \frac{F}{4\pi R t} \sin 2\theta$$

(20%)

$$K_I = \frac{F}{4\pi R t} (1 - \cos 2\theta) \sqrt{\pi a}$$

$$K_{II} = \frac{F}{4\pi R t} \sin 2\theta$$