

ENGINEERING TRIPOS  
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PART IIB  
PART IIA

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Friday 30 April 2004

2.30 to 4

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Module 4C14

NATURAL AND MICRO-ARCHITECTURED MATERIALS

*Answer not more than **three** questions.*

*All questions carry the same number of marks.*

*The **approximate** percentage of marks allocated to each part of a question is indicated in the right margin.*

*Attachments:*

*Special datasheet(s) (2 pages).*

**You may not start to read the questions  
printed on the subsequent pages of this  
question paper until instructed that you may  
do so by the Invigilator**

(TURN OVER)

1 Explain the following

(a) The cell membrane of red blood cells has a very low elastic modulus, but a large lock-up strain and a high ultimate strength. [25%]

(b) Wood is strongly anisotropic, with a compressive strength along the grain an order of magnitude greater than that across the grain. [25%]

(c) Proteins are transported within cells at much faster rates than diffusion can provide. [25%]

(d) Plants and animal cells have different strategies for harvesting energy. [25%]

2 (a) Describe the physical basis for the Young's modulus of biological tissues by explaining how the following concepts dictate the modulus:

(i) persistence length [25%]

(ii) nodal connectivity [25%]

(b) Qualitatively describe the myosin crossbridge cycle with reference to conversions of ATP to ADP. [25%]

(c) Describe the tension versus length curve of a single muscle fibre. Suppose that the tension decreased nonlinearly with increasing length for striation spacings greater than  $2.5 \mu\text{m}$ . Would this invalidate the theory that the crossbridges working independently generate the tension? [25%]

3 In the Huxley crossbridge model for a muscle,  $n(x)$  is the fraction of attached crossbridges, where  $x$  is the position of an actin binding site from the equilibrium position of a myosin head. Assume that the attachment and detachment of the crossbridges is governed by a first order kinetic scheme with attachment and detachment constants  $f(x)$  and  $g(x)$ , respectively.

(a) Determine the steady-state  $n(x)$  in terms of  $f(x)$  and  $g(x)$  for a muscle in isometric tension. [10%]

(b) Given that:

$$\begin{array}{lll}
 f(x) = 0 ; & g(x) = g_1 ; & x < 0 ; \\
 f(x) = f_0 ; & g(x) = g_0 ; & 0 \leq x \leq h ; \\
 f(x) = 0 ; & g(x) = g_0 & x > h ;
 \end{array}$$

determine  $n(x)$  for shortening at a constant velocity  $V = -dx/dt$ . Here  $g_0$ ,  $f_0$  and  $h$  are constants. [50%]

(c) Writing any appropriate equations, explain how one might use the Huxley crossbridge dynamics model to calculate the response of a muscle in Hill's quick-release experiments (step change in tension). [40%]

(TURN OVER)

4 (a) Describe the functions of arteries and veins in blood flow. Comment on why there is no reverse flow in the veins. [15%]

(b) Describe the role of capillaries in blood flow. [15%]

(c) Consider a thin-walled cylindrical tube of length  $L$ , radius  $r$  and wall thickness  $t$  subjected to an internal pressure  $P$ . The radius of the tube is  $r_0$  at zero pressure, and the tube is made of an elastic material having Young's modulus  $E$ . If the cross-sectional area of the tube varies linearly with pressure, show that the compliance per unit length of the tube can be approximated as  $c = 2\pi r^3 / (Et)$ . [25%]

(d) Comment on why pulmonary and systemic capillaries can be modelled as resistance vessels whereas large arteries cannot, and explain why veins are more compliant than arteries (for a fixed vessel radius). [15%]

(e) There is blood flow across the cylindrical vessel described in (c), with inlet pressure  $P_0$  and outlet pressure  $P_1$ . It can be shown that the flux of blood is given by:

$$Q = \frac{\pi r^4}{24\mu L} \frac{(1 + \gamma P_0)^3 - (1 + \gamma P_1)^3}{\gamma}$$

where  $\gamma = c / (\pi r^2)$  and  $\mu$  is the viscosity of blood.

(i) Explain why the pressure drop in the veins can be much less than in the arteries.

(ii) Show that, in the limit  $c \rightarrow 0$ , the blood vessel reduces to a resistance vessel, with its resistance per unit length given by  $\rho = 8\mu / (\pi r^4)$ .

[Hint:  $x^3 - y^3 = (x - y)(x^2 + xy + y^2)$ .] [30%]

**END OF PAPER**

**Paper G4: Mechanics of Solids**  
**ELASTICITY and PLASTICITY FORMULAE**

**1. Axi-symmetric deformation : discs, tubes and spheres**

	<u>Discs and tubes</u>	<u>Spheres</u>
Equilibrium	$\sigma_{\theta\theta} = \frac{d(r\sigma_{rr})}{dr} + \rho\omega^2 r^2$	$\sigma_{\theta\theta} = \frac{1}{2r} \frac{d(r^2\sigma_{rr})}{dr}$
Lam's equations (in elasticity)	$\sigma_{rr} = A - \frac{B}{r^2} - \frac{3+\nu}{8} \rho\omega^2 r^2 - \frac{E\alpha}{r^2} \int_r^c rTdr$	$\sigma_{rr} = A - \frac{B}{r^3}$
	$\sigma_{\theta\theta} = A + \frac{B}{r^2} - \frac{1+3\nu}{8} \rho\omega^2 r^2 + \frac{E\alpha}{r^2} \int_r^c rTdr - E\alpha T$	$\sigma_{\theta\theta} = A + \frac{B}{2r^3}$

**2. Plane stress and plane strain**

	<u>Cartesian coordinates</u>	<u>Polar coordinates</u>
Strains	$\epsilon_{xx} = \frac{\partial u}{\partial x}$ $\epsilon_{yy} = \frac{\partial v}{\partial y}$ $\gamma_{xy} = \frac{\partial u}{\partial y} + \frac{\partial v}{\partial x}$	$\epsilon_{rr} = \frac{\partial u}{\partial r}$ $\epsilon_{\theta\theta} = \frac{u}{r} + \frac{1}{r} \frac{\partial v}{\partial \theta}$ $\gamma_{r\theta} = \frac{\partial v}{\partial r} + \frac{1}{r} \frac{\partial u}{\partial \theta} - \frac{v}{r}$
Compatibility	$\frac{\partial^2 \gamma_{xy}}{\partial x \partial y} = \frac{\partial^2 \epsilon_{xx}}{\partial y^2} + \frac{\partial^2 \epsilon_{yy}}{\partial x^2}$	$\frac{\partial}{\partial r} \left\{ r \frac{\partial \gamma_{r\theta}}{\partial \theta} \right\} = \frac{\partial}{\partial r} \left\{ r^2 \frac{\partial \epsilon_{\theta\theta}}{\partial r} \right\} - r \frac{\partial \epsilon_{rr}}{\partial r} + \frac{\partial^2 \epsilon_{rr}}{\partial \theta^2}$
or (in elasticity)	$\left\{ \frac{\partial^2}{\partial x^2} + \frac{\partial^2}{\partial y^2} \right\} (\sigma_{xx} + \sigma_{yy}) = 0$	$\left\{ \frac{\partial^2}{\partial r^2} + \frac{1}{r} \frac{\partial}{\partial r} + \frac{1}{r^2} \frac{\partial^2}{\partial \theta^2} \right\} (\sigma_{rr} + \sigma_{\theta\theta}) = 0$
Equilibrium	$\frac{\partial \sigma_{xx}}{\partial x} + \frac{\partial \sigma_{xy}}{\partial y} = 0$ $\frac{\partial \sigma_{yy}}{\partial y} + \frac{\partial \sigma_{xy}}{\partial x} = 0$	$\frac{\partial}{\partial r} (r\sigma_{rr}) + \frac{\partial \sigma_{r\theta}}{\partial \theta} - \sigma_{\theta\theta} = 0$ $\frac{\partial \sigma_{\theta\theta}}{\partial \theta} + \frac{\partial}{\partial r} (r\sigma_{r\theta}) + \sigma_{r\theta} = 0$
$\nabla^4 \phi = 0$ (in elasticity)	$\left\{ \frac{\partial^2}{\partial x^2} + \frac{\partial^2}{\partial y^2} \right\} \left\{ \frac{\partial^2 \phi}{\partial x^2} + \frac{\partial^2 \phi}{\partial y^2} \right\} = 0$	$\left\{ \frac{\partial^2}{\partial r^2} + \frac{1}{r} \frac{\partial}{\partial r} + \frac{1}{r^2} \frac{\partial^2}{\partial \theta^2} \right\}$ $\times \left\{ \frac{\partial^2 \phi}{\partial r^2} + \frac{1}{r} \frac{\partial \phi}{\partial r} + \frac{1}{r^2} \frac{\partial^2 \phi}{\partial \theta^2} \right\} = 0$
Airy Stress Function	$\sigma_{xx} = \frac{\partial^2 \phi}{\partial y^2}$ $\sigma_{yy} = \frac{\partial^2 \phi}{\partial x^2}$ $\sigma_{xy} = -\frac{\partial^2 \phi}{\partial x \partial y}$	$\sigma_{rr} = \frac{1}{r} \frac{\partial \phi}{\partial r} + \frac{1}{r^2} \frac{\partial^2 \phi}{\partial \theta^2}$ $\sigma_{\theta\theta} = \frac{\partial^2 \phi}{\partial r^2}$ $\sigma_{r\theta} = -\frac{\partial}{\partial r} \left\{ \frac{1}{r} \frac{\partial \phi}{\partial \theta} \right\}$

### 3. Torsion of prismatic bars

Prandtl stress function:  $\sigma_{zx} (= \tau_x) = \frac{dF}{dy}$  ,  $\sigma_{zy} (= \tau_y) = -\frac{dF}{dx}$

Equilibrium:  $T = 2 \int_A F dA$

Governing equation for elastic torsion:  $\nabla^2 F = -2G\beta$  where  $\beta$  is the angle of twist per unit length.

### 4. Total potential energy of a body

$$\Pi = U - W$$

where  $U = \frac{1}{2} \int_V \underline{\varepsilon}^T [D] \underline{\varepsilon} dV$  ,  $W = \underline{p}^T \underline{u}$  and  $[D]$  is the elastic stiffness matrix.

### 5. Principal stresses and stress invariants

Values of the principal stresses,  $\sigma_p$ , can be obtained from the equation

$$\begin{vmatrix} \sigma_{xx} - \sigma_p & \sigma_{xy} & \sigma_{xz} \\ \sigma_{xy} & \sigma_{yy} - \sigma_p & \sigma_{yz} \\ \sigma_{xz} & \sigma_{yz} & \sigma_{zz} - \sigma_p \end{vmatrix} = 0$$

This is equivalent to a cubic equation whose roots are the values of the 3 principal stresses, i.e. the possible values of  $\sigma_p$ .

Expanding:  $\sigma_p^3 - I_1 \sigma_p^2 + I_2 \sigma_p - I_3 = 0$  where  $I_1 = \sigma_{xx} + \sigma_{yy} + \sigma_{zz}$ ,

$$I_2 = \begin{vmatrix} \sigma_{yy} & \sigma_{yz} \\ \sigma_{yz} & \sigma_{zz} \end{vmatrix} + \begin{vmatrix} \sigma_{xx} & \sigma_{xz} \\ \sigma_{xz} & \sigma_{zz} \end{vmatrix} + \begin{vmatrix} \sigma_{xx} & \sigma_{xy} \\ \sigma_{xy} & \sigma_{yy} \end{vmatrix} \quad \text{and} \quad I_3 = \begin{vmatrix} \sigma_{xx} & \sigma_{xy} & \sigma_{xz} \\ \sigma_{xy} & \sigma_{yy} & \sigma_{yz} \\ \sigma_{xz} & \sigma_{yz} & \sigma_{zz} \end{vmatrix}$$

### 6. Equivalent stress and strain

Equivalent stress  $\bar{\sigma} = \sqrt{\frac{1}{2} \{ (\sigma_1 - \sigma_2)^2 + (\sigma_2 - \sigma_3)^2 + (\sigma_3 - \sigma_1)^2 \}}^{1/2}$

Equivalent strain increment  $d\bar{\varepsilon} = \sqrt{\frac{2}{3} \{ d\varepsilon_1^2 + d\varepsilon_2^2 + d\varepsilon_3^2 \}}^{1/2}$

### 7. Yield criteria and flow rules

#### Tresca

Material yields when maximum value of  $|\sigma_1 - \sigma_2|$ ,  $|\sigma_2 - \sigma_3|$  or  $|\sigma_3 - \sigma_1| = Y = 2k$ , and then,

if  $\sigma_3$  is the intermediate stress,  $d\varepsilon_1 : d\varepsilon_2 : d\varepsilon_3 = \lambda(1 : -1 : 0)$  where  $\lambda \neq 0$ .

#### von Mises

Material yields when,  $(\sigma_1 - \sigma_2)^2 + (\sigma_2 - \sigma_3)^2 + (\sigma_3 - \sigma_1)^2 = 2Y^2 = 6k^2$ , and then

$$\frac{d\varepsilon_1}{\sigma_1} = \frac{d\varepsilon_2}{\sigma_2} = \frac{d\varepsilon_3}{\sigma_3} = \frac{d\varepsilon_1 - d\varepsilon_2}{\sigma_1 - \sigma_2} = \frac{d\varepsilon_2 - d\varepsilon_3}{\sigma_2 - \sigma_3} = \frac{d\varepsilon_3 - d\varepsilon_1}{\sigma_3 - \sigma_1} = \lambda = \frac{3}{2} \frac{d\bar{\varepsilon}}{\bar{\sigma}}$$