

①

$$\frac{Dk}{Dt} = \frac{\partial k}{\partial t} + U \frac{\partial k}{\partial x} + \dots$$

Total or material derivative; change due to temporal and spatial variations; change following a fluid element

$\nu_t \left[ \frac{\partial U_i}{\partial x_j} \right]^2$  Production of turbulent kinetic energy from product of shear stress and velocity gradient

$\frac{\partial}{\partial x_j} \left( \frac{\nu_t}{\sigma_k} \frac{\partial k}{\partial x_j} \right)$  Turbulent diffusion of kinetic energy  
 $\Sigma$  Dissipation of turbulent kinetic energy to internal energy

$$-\overline{u'v'} = u_x^2 \quad \frac{\partial u}{\partial y} = u_x / xy \quad \Sigma = u_x^3 / xy$$

$$\nu_t = C_\mu k^2 / \Sigma = \cancel{u_x} y = C_\mu k^2 / u_x^3$$

$$\therefore C_\mu = u_x^4 / k^2 = (-0.3)^2 = 0.09$$

5c10

$k \sim x^{-1}$  turbulent kinetic energy  
 $L \sim (x+C)^{1/2}$

$$E(k_w) = A \Sigma^{2/3} k_w^{-5/3}$$

$$\Sigma \sim k^{3/2} / L \sim \frac{x^{3/2}}{x^{1/2}} \sim x^{-2}$$

( wavenumber )

$$E(k_w) = A x^{-4/3} k_w^{-5/3}$$

$$u'^2 \sim x^{-1}$$

decays faster;

Relatively more energy in the larger eddies  
 5c10

② Kinematic momentum flux / unit width

$$M'_0 = U^2 b = \text{constant}$$

$$\frac{d}{dx} U b = 2u_e = 2\alpha U$$

$$b \sim x^1 ; l \sim b$$

$$U \sim x^{-1/2}$$

$$\varepsilon \sim U^3 / b \sim \frac{U^3}{b} \sim \frac{x^{-3/2}}{x} \sim x^{-5/2}$$

$$\eta \sim \left( \frac{U^3}{\varepsilon} \right)^{1/4} \sim \left( x^{5/2} \right)^{1/4} \sim x^{5/8}$$

10<sup>10</sup>, 10<sup>11</sup>  
20<sup>10</sup>

20<sup>10</sup>

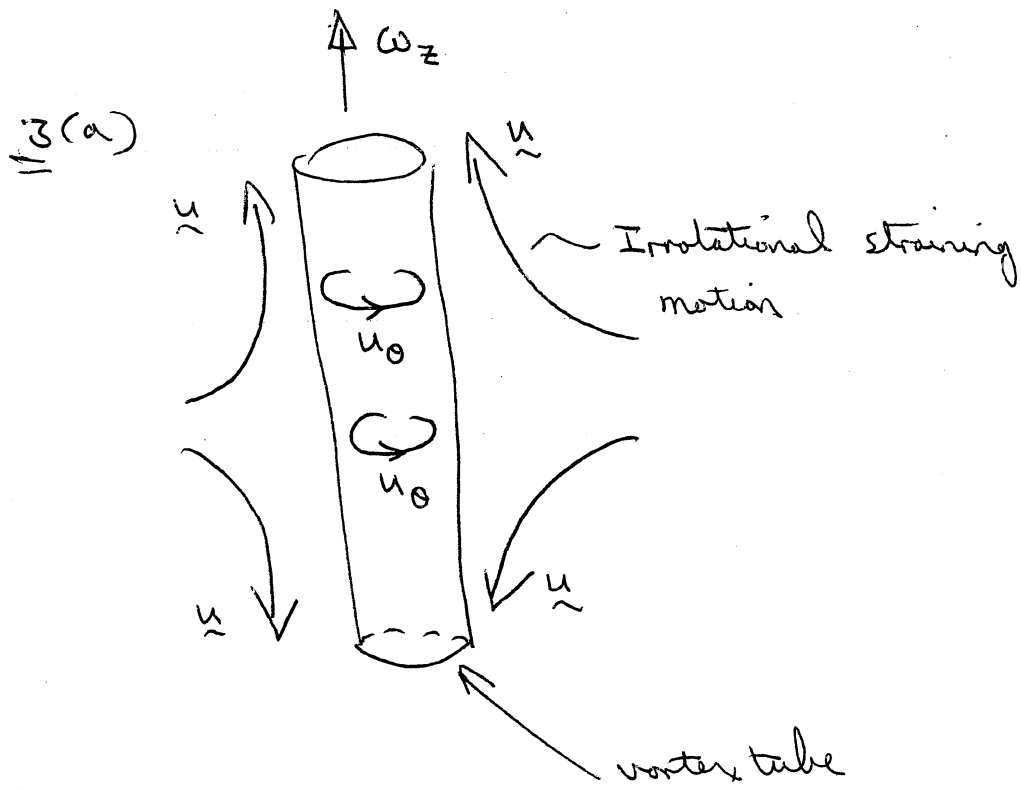
20<sup>10</sup>

Get properties independent of position?  
 Obtained by integrating ~~across~~ variables over  
 cross section i.e. by definition of similarity  
 arguments.

$$u'/U = \text{constant.}$$

Required on dimensional grounds can be  
 local similarity.

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Burger's vortex is thought to be a good model of the smallest scales in turbulence, called "worms".

$$(b) \quad (\underline{\omega} \cdot \nabla) \underline{u}_\omega = \omega_z \frac{\partial}{\partial z} (\underline{u}_\omega) = 0$$

( $\underline{u}_\omega$  not a function of  $z$ )

$$(\underline{u}_\omega \cdot \nabla) \underline{\omega} = u_\theta \frac{1}{r} \frac{\partial}{\partial \theta} (\underline{\omega}) = 0$$

( $\underline{\omega}$  not a function of  $\theta$ )

$$\underline{3} \text{ (c) } (\underline{u} \cdot \nabla) \underline{\omega} = u_x \frac{\partial \omega_z}{\partial x} \hat{e}_z = (-\alpha x) \left( \frac{-2x}{\delta^2} \right) \omega_z \hat{e}_z$$

$$(\underline{\omega} \cdot \nabla) \underline{u} = \omega_z \frac{\partial u}{\partial z} = \omega_z \alpha \hat{e}_z$$

$$v \nabla^2 \underline{u} = v \frac{\partial^2 \omega_z}{\partial x^2} \hat{e}_z = v \left[ \left( \frac{-2x}{\delta^2} \right)^2 \omega_z - \frac{2}{\delta^2} \omega_z \right] \hat{e}_z$$

$$\text{But } (\underline{\omega} \cdot \nabla) \underline{u} + v \nabla^2 \underline{u} = (\underline{u} \cdot \nabla) \underline{\omega}$$

$$\Rightarrow \alpha \omega_z + v \left[ \left( \frac{-2x}{\delta^2} \right)^2 - \frac{2}{\delta^2} \right] \omega_z = \frac{2\alpha x^2}{\delta^2} \omega_z$$

$$\Rightarrow \alpha + v \left[ \frac{4x^2}{\delta^4} - \frac{2}{\delta^2} \right] = \frac{2\alpha x^2}{\delta^2}$$

$$\Rightarrow \alpha - \frac{2v}{\delta^2} = \frac{2\alpha x^2}{\delta^2} \left[ \alpha - \frac{2v}{\delta^2} \right]$$

This is satisfied if

$$\alpha = \frac{2v}{\delta^2}$$

$$\Rightarrow \underline{\underline{\delta = \sqrt{2v/\alpha}}}$$

(d) If  $u_s = 0$  then

$$\frac{\partial \omega}{\partial t} = v \nabla^2 \omega$$

The vortex sheet will then thicken by diffusion,  
at a rate  $\delta \sim (v t)^{1/2}$ .

4 (a) (i) The vortex lines are 'glued' into the fluid.

(ii) The flux of vorticity down a vortex tube is the same at all cross-sections and independent of time.

4. (b)

$$\begin{aligned}
 \frac{D}{Dt}(\underline{u} \cdot \underline{\omega}) &= \underline{u} \cdot \frac{D\underline{\omega}}{Dt} + \underline{\omega} \cdot \frac{D\underline{u}}{Dt} \\
 &= \underline{u} \cdot [(\underline{\omega} \cdot \nabla)\underline{u}] + \underline{\omega} \cdot [-\nabla(p/\rho)] \\
 &= (\underline{\omega} \cdot \nabla)(u^2/2) - \nabla \cdot (\underline{\omega} p/\rho) \\
 &= \nabla \cdot (\underline{\omega} u^2/2) - \nabla \cdot (\underline{\omega} p/\rho) \\
 &= \nabla \cdot [(u^2/2 - p/\rho)\underline{\omega}]
 \end{aligned}$$

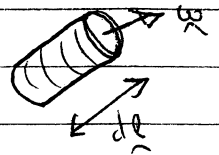
$$\frac{D}{Dt}(\underline{u} \cdot \underline{\omega}) \delta V = \delta V \frac{D}{Dt}(\underline{u} \cdot \underline{\omega}) \quad \text{since } \frac{D}{Dt} \delta V = 0$$

$$\begin{aligned}
 \Rightarrow \frac{D}{Dt} \int (\underline{u} \cdot \underline{\omega}) dV &= \int \frac{D}{Dt}(\underline{u} \cdot \underline{\omega}) dV \\
 &= \int \nabla \cdot [(u^2/2 - p/\rho)\underline{\omega}] dV \\
 &= \oint \left( \frac{u^2}{2} - \frac{p}{\rho} \right) \underline{\omega} \cdot d\underline{s} \\
 &= 0
 \end{aligned}$$

$$(c) \quad \underline{\omega} dV = \underline{\omega} A dl = |\underline{\omega}| A dl = \Phi dl$$

Thus, for one tube

$$H = \int_{C_1} \underline{\omega} dV = \oint_{C_1} \underline{u} \cdot (\Phi dl)$$



$$\text{For both tubes, } H = \oint_{C_1} \underline{u} \cdot (\Phi_1 dl) + \oint_{C_2} \underline{u} \cdot (\Phi_2 dl)$$

Since  $\Phi_1$  and  $\Phi_2$  are constant around the tubes, Stokes' theorem gives us,

$$H = \Phi_1 \oint_{C_1} \underline{u} \cdot dl + \Phi_2 \oint_{C_2} \underline{u} \cdot dl = \Phi_1 \Gamma_1 + \Phi_2 \Gamma_2$$

4 (d) If  $\omega$  in one tube is reversed the linkage of the tubes becomes left-handed, so the application of Stokes theorem introduces a minus sign and  $H = -2 \Phi_1 \Phi_2$

4 (e)  $H$  is not conserved if the fluid is viscous or subject to a buoyancy force because the vortex lines are no longer frozen into the fluid so that the linkage of the two tubes is not preserved.