

Part IIB 2006 Module 4C9 Solutions.1 (a) (i) to find $\underline{a} \cdot (\underline{b} \times \underline{c})$ in subscript notation

$$\underline{b} \times \underline{c} = \epsilon_{ijk} b_j c_k$$

$$\underline{u} \cdot \underline{v} = u_i v_i$$

$$\therefore \underline{a} \cdot (\underline{b} \times \underline{c}) = a_i \epsilon_{ijk} b_j c_k$$

$$= \underline{\epsilon_{ijk} a_i b_j c_k}$$

$$\epsilon_{ijk} a_i b_j c_k$$

$$= \epsilon_{123} a_1 b_2 c_3 + \epsilon_{231} a_2 b_3 c_1 + \epsilon_{312} a_3 b_1 c_2$$

$$+ \epsilon_{132} a_1 b_3 c_2 + \epsilon_{213} a_2 b_1 c_3 + \epsilon_{321} a_3 b_2 c_1$$

$$= a_1 b_2 c_3 + a_2 b_3 c_1 + a_3 b_1 c_2$$

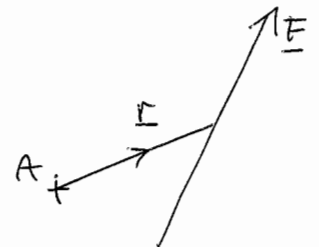
$$- a_1 b_3 c_2 - a_2 b_1 c_3 - a_3 b_2 c_1$$

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So if  $\underline{c} = \underline{F} = (3, 2, 4)$

$\underline{b} = \underline{r} = (2, 1, 3)$

$\underline{a} = \underline{e} = \frac{1}{\sqrt{53}} (1, 4, 6)$



$$\underline{a} \cdot (\underline{b} \times \underline{c}) = \underline{e} \cdot (\underline{r} \times \underline{F}) = \left( 1 \cdot 1 \cdot 4 + 4 \cdot 3 \cdot 3 + 6 \cdot 2 \cdot 2 \right. \\ \left. - 1 \cdot 3 \cdot 2 - 4 \cdot 2 \cdot 4 - 6 \cdot 1 \cdot 3 \right) \frac{1}{\sqrt{53}}$$

$$= \frac{1}{\sqrt{53}} (4 + 36 + 24 - 6 - 32 - 18)$$

$$= \frac{8}{\sqrt{53}} \text{ Nm}$$

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$$(ii) \quad v_k = e_{kji} u_{ij}$$

$$\begin{aligned} e_{k\ell m} v_k &= e_{k\ell m} e_{kji} u_{ij} \\ &= (\delta_{\ell j} \delta_{mi} - \delta_{\ell i} \delta_{mj}) u_{ij} \\ &= \delta_{\ell j} \delta_{mi} u_{ij} - \delta_{\ell i} \delta_{mj} u_{ij} \end{aligned}$$

$$\text{i.e. } e_{k\ell m} v_k = u_{m\ell} - u_{\ell m}$$

But if u_{ij} is skew-symmetric

$$u_{ij} = -u_{ji}$$

$$\therefore e_{k\ell m} v_k = 2u_{m\ell}$$

$$\text{i.e. } u_{m\ell} = \frac{1}{2} e_{k\ell m} v_k$$

$$\text{or } \underline{u_{ij} = \frac{1}{2} e_{kji} v_k}$$

1. (b)

$$(i) \quad \epsilon_y = \epsilon_z = -\mu \epsilon_x \quad (\text{total strains})$$

$$\epsilon_y^e = \epsilon_z^e = -\nu \epsilon_x^e \quad (\text{elastic strains})$$

$$\epsilon_x^p + \epsilon_y^p + \epsilon_z^p = 0 \quad (\text{plastic incompressibility})$$

$$\Rightarrow \epsilon_x + \epsilon_y + \epsilon_z = \epsilon_x - 2\mu \epsilon_x = (1-2\mu)\epsilon_x$$

$$\text{But } \epsilon_x + \epsilon_y + \epsilon_z = \epsilon_x^e + \epsilon_y^e + \epsilon_z^e = (1-2\nu)\epsilon_x^e$$

$$\Rightarrow \epsilon_x = \frac{1-2\nu}{1-2\mu} \epsilon_x^e$$

$$\text{Now, } \sigma_x = E_{sec} \epsilon_x = E \epsilon_x^e \quad \Rightarrow \quad \epsilon_x = \frac{E}{E_{sec}} \epsilon_x^e$$

$$\Rightarrow \frac{1-2\nu}{1-2\mu} = \frac{E}{E_{sec}}$$

$$\text{Re-arranging } \Rightarrow \quad \mu = \frac{1}{2} - \frac{E_{sec}}{2E} (1-2\nu) \quad \#$$

$$(ii) \quad \epsilon_{ij}^p = \frac{3}{2} \left(\frac{1}{E_{sec}} - \frac{1}{E} \right) s_{ij} \quad (J_2 \text{ deformation theory})$$

$$\begin{aligned} \Rightarrow \epsilon_{ij} &= \epsilon_{ij}^e + \epsilon_{ij}^p = \frac{1}{E} [(1+\nu)\sigma_{ij} - \nu\sigma_{kk}\delta_{ij}] + \frac{3}{2} \left(\frac{1}{E_{sec}} - \frac{1}{E} \right) (\sigma_{ij} - \frac{1}{3}\sigma_{kk}\delta_{ij}) \\ &= \frac{1}{E} \left(\frac{2\nu-1}{2}\sigma_{ij} - \frac{2\nu-1}{2}\sigma_{kk}\delta_{ij} \right) + \frac{3}{2E_{sec}} \left(\sigma_{ij} - \frac{1}{3}\sigma_{kk}\delta_{ij} \right) \end{aligned}$$

$$\xrightarrow{\text{(i) result}} = \frac{2\mu-1}{2E_{sec}} (\sigma_{ij} - \sigma_{kk}\delta_{ij}) + \frac{3}{2E_{sec}} \left(\sigma_{ij} - \frac{1}{3}\sigma_{kk}\delta_{ij} \right)$$

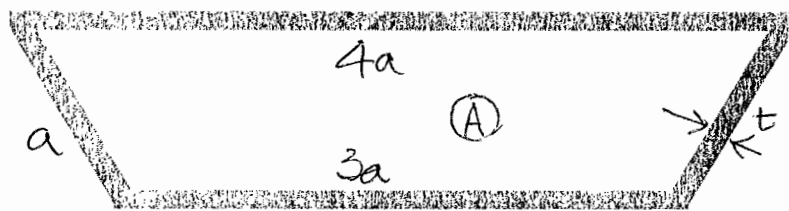
$$= \frac{1}{E_{sec}(\sigma_e)} [(1+\mu)\sigma_{ij} - \mu\sigma_{kk}\delta_{ij}]$$

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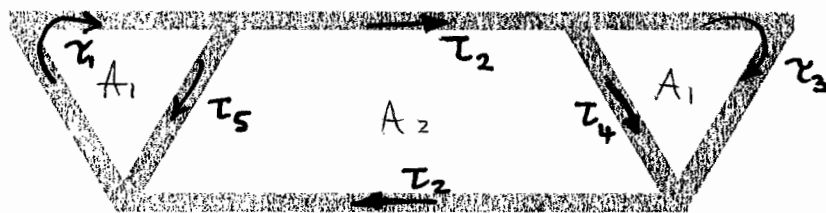
For Box section enclosed area $A = \frac{7}{2}a \cdot \frac{\sqrt{3}}{2}a = \frac{7\sqrt{3}a^2}{4}$

$$\frac{\oint ds}{dt} = \frac{9a}{t} \quad \therefore \frac{I}{G\alpha} = \frac{4A^2}{\oint ds/t} = \frac{4 \times 49.3a^4 t}{16 \times 9a}$$

$$\therefore \frac{T}{G\alpha} = \frac{49}{12}a^3t = 4.08a^3t$$



(a)



(b)

$$\begin{cases} 2GA_1\alpha = \tau_1 2a + \tau_5 a \\ 2GA_2\alpha = \tau_2 \cdot 2a + \tau_4 a + \tau_3 \cdot 3a - \tau_5 a \\ 2GA_3\alpha = \tau_3 2a - \tau_4 \cdot a \end{cases}$$

assume $d_1 > d_2 > d_3$
effective
neutral
displacement

$$\begin{cases} \tau_1 = \frac{d_1}{t} & \tau_2 = \frac{d_2}{t} & \tau_3 = \frac{d_1 - d_2}{t} & \tau_4 = \frac{d_2 - d_3}{t} \\ \tau_3 = \frac{d_3}{t} & \text{shear flows} & & \end{cases}$$

$$\frac{2GA_1\alpha}{a} = 2\tau_1 + \tau_5 = 2\tau_3 - \tau_4$$

But $\tau_1 - \tau_2 = \tau_5$ $\tau_2 - \tau_3 = \tau_4$ anti/lt.

$$\therefore 2\tau_1 + \tau_1 - \tau_2 = 2\tau_3 - \tau_2 + \tau_3$$

$$\therefore \underline{\tau_1 = \tau_3} \quad \text{Symmetry}$$

$$A_1 = \frac{\sqrt{3}a^2}{4}; \quad A_2 = \frac{5\sqrt{3}a^2}{4}$$

$$\therefore \begin{cases} 24 \frac{\sqrt{3}}{4} a^2 \alpha = 2\tau_1 + \tau_1 - \tau_2 \\ 2 \cdot 5 \frac{\sqrt{3}}{4} a^2 \alpha = 2\tau_2 + \tau_2 - \tau_1 + 3\tau_2 - \tau_1 + \tau_2 \end{cases}$$

$$\begin{cases} 3\tau_1 - \tau_2 = \frac{\sqrt{3}}{2} Ga\alpha \\ -2\tau_1 + 7\tau_2 = \frac{5\sqrt{3}}{2} Ga\alpha \end{cases}$$

$$\therefore \tau_2 = \frac{17\sqrt{3}}{19.2} Ga\alpha$$

$$\begin{cases} 6\tau_1 - 2\tau_2 = \sqrt{3} Ga\alpha \\ -6\tau_1 + 21\tau_2 = \frac{15\sqrt{3}}{2} Ga\alpha \end{cases}$$

and

$$\tau_1 = \frac{12\sqrt{3}}{19.2} Ga\alpha$$

$$\text{But } T = 2A_1d_1 + 2A_2d_2 + 2A_1d_3$$

$$= 2A_1\tau_1t + 2A_2\tau_2t + 2A_1\tau_1t \quad (\tau_2 \equiv \tau_1)$$

$$= 4 \cdot \frac{\sqrt{3}}{4} a^2 t \frac{12\sqrt{3}}{19.2} Ga\alpha + 2 \frac{5\sqrt{3}}{4} a^2 t \frac{17\sqrt{3}}{19.2} Ga\alpha$$

$$= \left(\frac{12 \cdot 3}{19.2} + \frac{15 \cdot 17 \cdot 3}{2 \cdot 19.2} \right) Ga^3 t \alpha$$

$$\therefore \frac{T}{Ga} = \frac{72 + 15 \cdot 17}{19.4} a^3 t = \frac{327}{76} a^3 t \Rightarrow \underline{4.303 a^3 t}$$

$$\text{So increase in stiffness } \frac{4.303}{4.08} = \underline{5.5\%}$$

(b) Fully plastic torque can be evaluated by sand-will analogy.

Consider shaft with section ABCD in which the inscribed circle just touches all sides, as in (i) & (ii)

Then pyramid formed by "sand" has apex of height h where $h = k \times R$

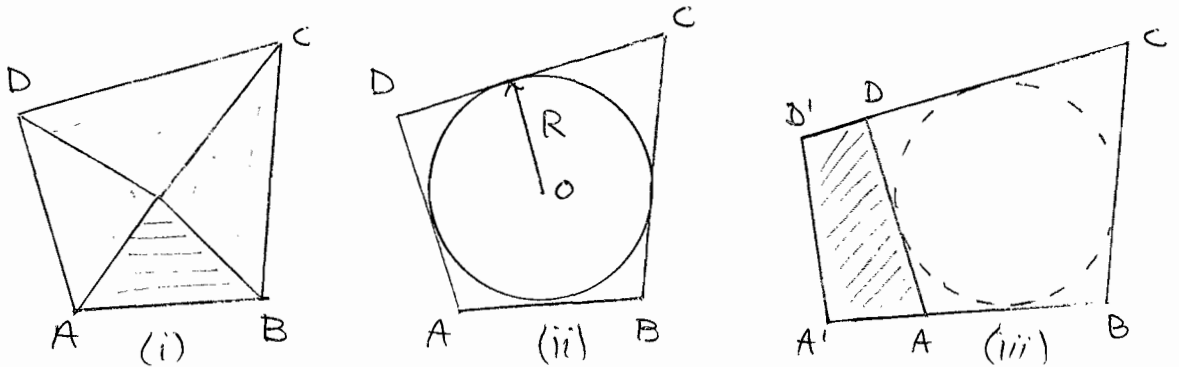
Torque \Rightarrow 2 volume of pyramid

But since volume of pyramid is $\frac{1}{3} \times \text{base} \times \text{height}$

$$\text{Torque } T_p = 2 \times \frac{1}{3} A_0 \times k \times R$$

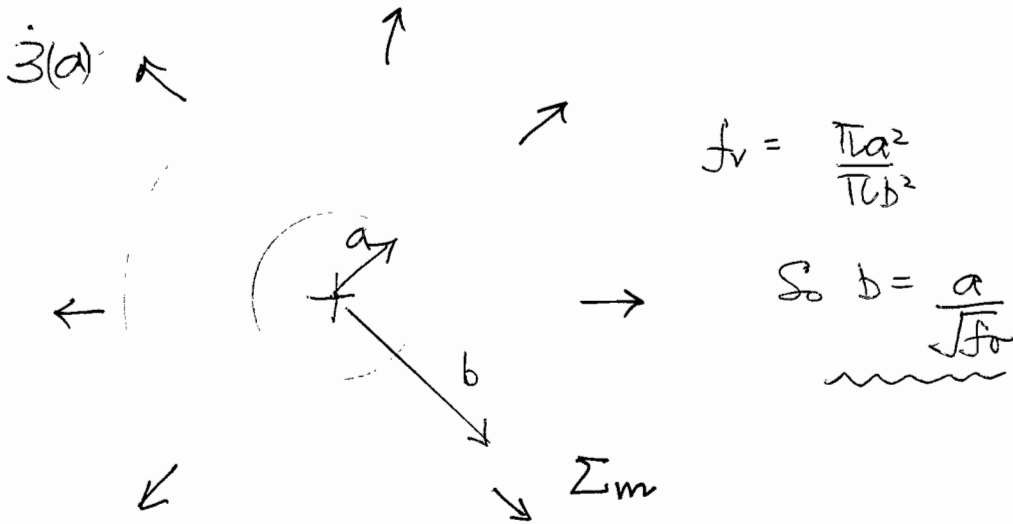
$$T_p = \frac{2}{3} A_0 R k$$

So expression is exact.



But adding material cannot reduce plastic resistance so that fully plastic torque T_p' for $A'ABCDD'$ must be greater than that for ABCD

ie. $T_p = \frac{2}{3} A_0 R k$ is a lower bound on T_p'



$$u^* = Ar + \frac{B}{r}$$

$$u^* \Rightarrow 0 \text{ at } r=a \quad \therefore 0 = Aa + \frac{B}{a}$$

$$B = -Aa^2$$

$$\therefore u^* = Ar \left(1 - \frac{a^2}{r^2}\right) = Ar - \frac{Aa^2}{r}$$

Strain rates $\dot{\epsilon}_r^* = \frac{du^*}{dr} = A + \frac{Aa^2}{r^2} = A \left(1 + \frac{a^2}{r^2}\right)$

$u^* \Rightarrow 0$ $\dot{\epsilon}_\theta^* = \frac{u^*}{r} = A \left(1 - \frac{a^2}{r^2}\right)$

$$\dot{\epsilon}_e^* = \sqrt{\frac{2}{3} \dot{\epsilon}_j^* \dot{\epsilon}_j^*} = \sqrt{\frac{2}{3} (\dot{\epsilon}_r^{*2} + \dot{\epsilon}_\theta^{*2})}$$

$$= A \sqrt{\frac{2}{3}} \left\{ 1 + \frac{2a^2}{r^2} + \frac{a^4}{r^4} + 1 - \frac{2a^2}{r^2} + \frac{a^4}{r^4} \right\}^{1/2}$$

$$\dot{\epsilon}_e^* = \frac{2A}{\sqrt{3}} \left\{ 1 + \frac{a^4}{r^4} \right\}^{1/2}$$

Now UB theorem $\int_S T_i^L u_i^* dS \leq \int_V \sigma_y^M \dot{\epsilon}_e^* dV$

So that

$$\sum_m^L u_b^* 2\pi b \leq \int_a^b \sigma_y^m \cdot \frac{2A}{\sqrt{3}} \left\{ 1 + \frac{a^4}{r^4} \right\}^{1/2} \cdot 2\pi r dr$$

$$u_b^* = Ab(1 - a^2/b^2)$$

$$\sum_m^L \cancel{Ab(1 - a^2/b^2)} \cancel{2\pi b} \leq \frac{2A}{\sqrt{3}} \cdot \cancel{2\pi} \sigma_y^m \int_a^b r \left\{ 1 + \frac{a^4}{r^4} \right\}^{1/2} dr$$

$$\frac{\sum_m^L}{\sigma_y^m} \cdot \frac{\sqrt{3}}{2} b^2 (1 - a^2/b^2) \leq \underbrace{\int_a^b r \left\{ 1 + \frac{a^4}{r^4} \right\}^{1/2} dr}_I$$

$$I = \frac{a^2}{2} \int_{\theta_1}^{\theta_2} \sec^2 \theta \csc \theta d\theta \quad \begin{cases} \tan \theta_1 = 1; \theta_1 = \pi/4 \\ \tan \theta_2 = b^2/a^2 = 1/\sqrt{3} \end{cases}$$

$$\therefore \frac{2I}{a^2} = \left[\csc \theta + \tan \theta \right]_{\theta_1}^{\theta_2} + \int_{\theta_1}^{\theta_2} \tan \theta \frac{\cos \theta}{\sin^2 \theta} d\theta$$

$$\text{i.e. } \frac{2I}{a^2} = \left[\sec \theta \right]_{\theta_1}^{\theta_2} = \int_{\theta_1}^{\theta_2} \frac{\sin \theta}{\cos \theta} \cdot \frac{\cos \theta}{\sin^2 \theta} d\theta = \int_{\theta_1}^{\theta_2} \csc \theta d\theta$$

i.e.

$$\frac{2I}{a^2} = \sec \theta_2 + \sec \theta_1 = \left[\ln \left(\tan \frac{\theta}{2} \right) \right]_{\theta_1}^{\theta_2}$$

$$\frac{\sum_m^L}{\sigma_y^m} \frac{\sqrt{3}}{2} b^2 (1 - a^2/b^2) \cdot \frac{2}{a^2} = \sec \theta_2 - \sec \theta_1 + \ln \left[\frac{\tan(\theta_2/2)}{\tan(\theta_1/2)} \right]$$

$$\text{But } \frac{a^2}{b^2} = f_v$$

$$\therefore \sqrt{3} \frac{\Sigma_M^L}{\sigma_4^M} \cdot \frac{1-f_v}{f_v} = \sqrt{1 + (1/f_v)^2} - \sqrt{2} + \ln \left[\frac{\tan(\theta_2/2)}{\tan(\pi/8)} \right]$$

If $f_v \Rightarrow 0$ then

$$\sqrt{3} \frac{\Sigma_M^L}{\sigma_4^M} \cdot \frac{1}{f_v} \approx \frac{1}{f_v}$$

$$\text{i.e. } \Sigma_M^L = \frac{\sigma_4^M}{\sqrt{3}}$$

If $f_v \Rightarrow 1$

$$\text{So } \Sigma_M^L \Rightarrow \frac{f_v}{1-f_v} (\) \Rightarrow \infty \quad \text{no yield} \checkmark$$

$$(b) \quad u^* = \left(Cr + \frac{D}{r} \right) \sin 2\theta; \quad v^* = \left(Cr + \frac{D}{r} \right) \cos 2\theta$$

again at $r=a$ $u^* = v^* \Rightarrow 0$

$$\therefore D = -Ca^2$$

$$\text{So } u^* = Cr \left(1 - \frac{a^2}{r^2} \right) \sin 2\theta; \quad v^* = Cr \left(1 - \frac{a^2}{r^2} \right) \cos 2\theta$$

$$E_r^* = \frac{\partial u^*}{\partial r} = C \left(1 + \frac{a^2}{r^2} \right) \sin 2\theta$$

$$E_\theta^* = \frac{u^*}{r} + \frac{1}{r} \frac{\partial v^*}{\partial \theta} = C \left(1 - \frac{a^2}{r^2} \right) \sin 2\theta - 2C \left(1 - \frac{a^2}{r^2} \right) \cos 2\theta$$

$$= -C \left(1 - \frac{a^2}{r^2} \right) \sin 2\theta$$

$$\gamma_{r\theta}^* = \frac{\partial v^*}{\partial r} + \frac{1}{r} \frac{\partial u^*}{\partial \theta} - \frac{v^*}{r}$$

$$\Rightarrow C \left(1 + \frac{a^2}{r^2} \right) \cos 2\theta + 2C \left(1 - \frac{a^2}{r^2} \right) \cos 2\theta - C \left(1 - \frac{a^2}{r^2} \right) \cos 2\theta$$

$$= C \left[1 + \frac{a^2}{r^2} + 2 - \frac{2a^2}{r^2} - 1 + \frac{a^2}{r^2} \right] \cos 2\theta$$

$$\gamma_{r\theta}^* = 2C \cos 2\theta$$