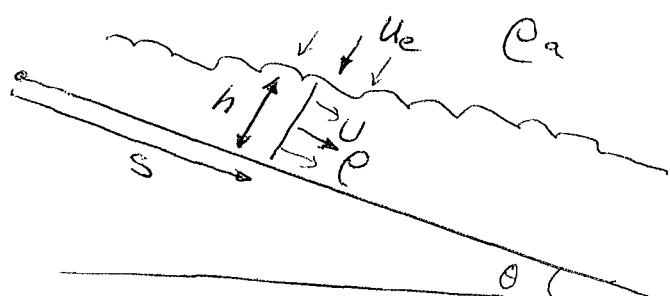


(1)



a) Derive equations;

Volume conservation

$$\frac{d(Uh)}{ds} = U_e = \alpha U \quad (1)$$

Mass:

$$\frac{d(\rho Uh)}{ds} = \rho_a U_e = \rho_a \alpha U \quad (2)$$

Momentum

$$\frac{d(\rho U^2 h)}{ds} = (\rho - \rho_a)gh \sin \theta \quad (3)$$

Take  $(2) - \rho_a \times (1)$ 

$$\frac{d((\rho - \rho_a)Uh)}{ds} = 0 \quad (4)$$

① cont'd

Assume power law behaviour

$$U = a_1 s^{b_1}$$

$$h = a_2 s^{b_2}$$

$$(e - e_a) = a_3 s^{b_3}$$

look at eqn ④

$$\frac{d}{ds} \left( a_1 a_2 a_3 s^{b_1 + b_2 + b_3} \right) = 0$$

$$(b_1 + b_2 + b_3) a_1 a_2 a_3 s^{b_1 + b_2 + b_3 - 1} = 0$$

$$b_1 + b_2 + b_3 = 0$$

look at eqn ①

$$\frac{d}{ds} \left( a_1 a_2 s^{b_1 + b_2} \right) = \alpha a_1 s^{b_1}$$

$$(b_1 + b_2) a_1 a_2 s^{b_1 + b_2 - 1} = \alpha a_1 s^{b_1}$$

$$b_1 + b_2 - 1 = b_1 \Rightarrow b_2 = \cancel{-1}$$

$$(b_1 + 1) \cancel{\times} a_2 = \alpha \quad a_2 = \frac{\alpha}{b_1 + 1}$$

③

- a)  $\lambda \frac{\partial T}{\partial x_3}$  is the heat transferred by conduction  
 and  $\rho C_p \overline{u_3 \theta}$  is the heat transferred  
 by the convective fluctuations  
 (turbulent heat flux)

Very near the surface the fluctuations  
 $u_3$  must go to zero so all  
 heat is transferred by conduction.

Further away from the wall the  
 turbulent heat flux dominates  
 since it is more effective at  
 transporting heat than conduction

- (b) Since there is no mean velocity there  
 is no production due to shear and  
 overall the balance is between the  
 dissipation,  $\epsilon$ , and the production due  
 to buoyancy.

$$\epsilon = \frac{g \overline{\theta u_3}}{T}$$

Q cont'd

$$\left. \begin{aligned} h &= \alpha s \\ (\rho - \rho_a) &= a_3 s^{-1} \\ U &= a_1 \end{aligned} \right\}$$

$$a_1^2 a_2 = \frac{g}{\rho_a} a_2 a_3 \sin \theta$$

$$a_1^2 = \frac{g}{\rho_a} a_3 \sin \theta$$

$$\text{OR } a_3 = \frac{\rho_a a_1^2}{g \sin \theta}$$

$$\text{So } h = \alpha s$$

$$(\rho - \rho_a) = \frac{\rho_a a_1^2}{g \sin \theta} s^{-1}$$

$$U = a_1$$

$$(\rho - \rho_a) U h = \text{constant} = B, \text{ say,}$$

$$a_3 \cancel{s}^1 a_1 \cancel{a_1} \cancel{\alpha} \cancel{s} = B$$

$$B = \alpha a_1 a_3$$

(2)

- a)  $\lambda \frac{\partial T}{\partial x_3}$  is the heat transferred by conduction  
 and  $\rho C_p \bar{u}_3 \theta$  is the heat transferred  
 by the deviatoric fluctuations  
 (turbulent heat flux)

Very near the surface the fluctuations  $u_3'$  must go to zero so all heat is transferred by conduction.

Further away from the wall the turbulent heat flux dominates since it is more effective at transporting heat than conduction.

- (b) Since there is no mean velocity there is no production due to shear and overall the balance is between the dissipation,  $\epsilon$ , and the production due to buoyancy.

$$\epsilon = \frac{g \bar{\theta} u_3'}{T}$$

(2)<sup>(b)</sup> cont'd

Now we need  $\bar{u_3 \theta}$  and we can use the enthalpy equation  
 $\rightarrow$  drop the first term as small in the main region of the flow

$$\bar{u_3 \theta} = \frac{\dot{Q}}{\rho C_p}$$

Hence  $\varepsilon = \frac{g Q}{\rho C_p T}$

Now the reason we need  $\varepsilon$  is that the smallest eddies are Kolmogorov eddies with a size of

$$\eta = \left( \frac{v^3}{\varepsilon} \right)^{\frac{1}{4}} = \left( \frac{v^3 \rho C_p T}{g Q} \right)^{\frac{1}{4}}$$

(c) Now the large eddy turnover time

$$t_{te} = \frac{L}{u'} \quad \text{where } u' \text{ is the turbulent}$$

fluctuation. To find  $u'$  use

$$\varepsilon = \frac{u'^3}{L} \Rightarrow u' = \left( \frac{g Q L}{\rho C_p T} \right)^{\frac{1}{3}}$$

②(c) cont'd

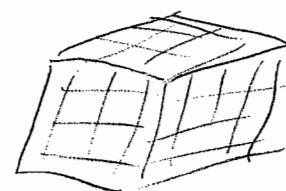
$$t_{LE} = \frac{L}{\left(\frac{gQL}{\rho C_p T}\right)^{\frac{1}{3}}} = \left(\frac{\rho C_p T L^2}{gQ}\right)^{\frac{1}{3}}$$

(d) Grid points. Smallest spacing =  $\eta$  (Kolmog.)

Box size =  $L$

No pt on one side is  $\approx \frac{L}{\eta}$

Total No pts  $\left(\frac{L}{\eta}\right)^3$



$$= \left(\frac{gQL^4}{\nu^3 \rho C_p T}\right)^{\frac{3}{4}}$$

(e) Shortest time is the Kolmogorov time

$$\tau = \sqrt{\frac{\nu}{\epsilon}}$$

Longest time is  $t_{LE}$  - large eddy turnover

$$N_{\text{timesteps}} = \frac{10t_{LE}}{\tau} = \frac{10 \left(\frac{\rho C_p T L^2}{gQ}\right)^{\frac{1}{3}}}{\sqrt{\nu \rho C_p T / gQ}}$$

2e) contd

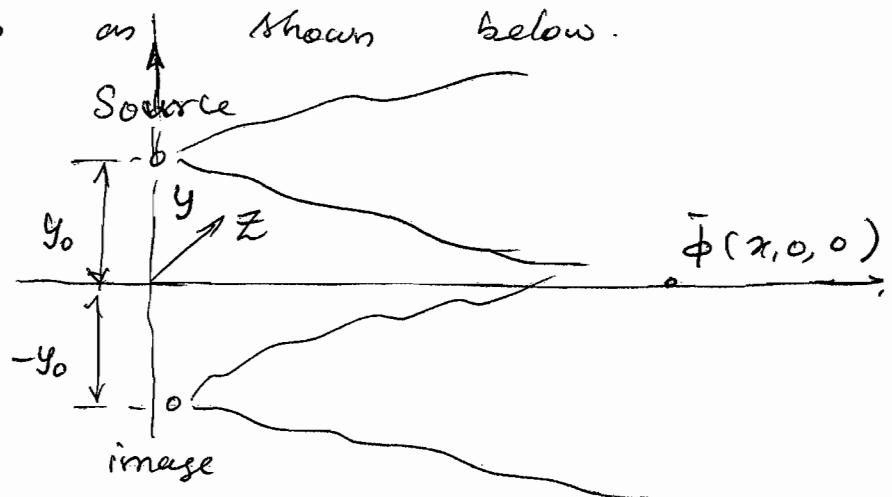
$$N_{\text{timesteps}} = \frac{10}{V^{\frac{1}{2}}} \frac{(gQ)^{\frac{1}{6}} (\rho C_p T)^{-\frac{1}{6}} L^{\frac{2}{3}}}{}$$

$$= \left( \frac{gQ}{\rho C_p T} \right)^{\frac{1}{6}} \frac{L^{\frac{2}{3}}}{V^{\frac{1}{2}}}$$

(can check dimensions to see if there  
are any mistakes)

- ③ a) 1) Point Source - Power plants, Cement Plants Stacks
- 2) Line Source - Automobiles on the motorway
- 3) Area Source - Forest fires, evaporation from oil spills.
- A) Volume Source - Emission from a large paint shop,

b) The ground effects are included via image sources as shown below.



The ground level concentration is

$$\bar{\phi}(x, 0, 0) = \bar{\phi}(x, 0, 0) \text{ due to Source}$$

+

$$\bar{\phi}(x, 0, 0) \text{ due to Image Source}$$

+ - is used if the ground is non-absorbing

- - is used if the ground is absorbing.

(Q3 contd ii)  $\bar{v}_y = Bx, A_1 = (A/B)$

$$\Rightarrow \bar{\phi}(x, 0) = \frac{A_1}{x} \exp\left(-\frac{y_0^2}{2B^2x^2}\right)$$

$$x \bar{\phi} = A_1 \exp\left(-\frac{y_0^2}{2B^2x^2}\right)$$

$$\Rightarrow \bar{\phi}'(x, 0) = \frac{A_1}{x^2} \exp\left(-\frac{y_0^2}{2B^2x^2}\right) \left\{ \frac{y_0^2}{B^2x^2} - 1 \right\}$$

for  $\bar{\phi}' = 0, x = \pm (y_0/B)$

Take the +ve root  $x^* = y_0/B$

$$\bar{\phi}_{\max}(x^*, 0) = \frac{A}{y_0} \exp\left(-\frac{1}{2}\right) = 0.607 \frac{A}{y_0}$$

$$\boxed{\bar{\phi}_{\max} = 0.607 \frac{A}{y_0}}$$

d)  $(Q/L) = 1 \text{ kg/m-sec.}$

$$U = 10 \text{ km/hr} \approx 2.78 \text{ m/sec.}, y_0 = 10 \text{ m}$$

$$\Rightarrow A = \frac{2Q}{UL\sqrt{2\pi}} = \frac{2}{2.78 \sqrt{2\pi}} = 0.287 \text{ kg/m}^2$$

$$\bar{\phi}_{\max} = 0.607 \frac{A}{y_0} = 0.0174 \text{ kg/m}^3 //$$

e) The maximum ground level concentration becomes one-half, if the height,  $y_0$ , is doubled.

(4)

a)

$$Da_1 = \frac{T_{\text{Turb}}}{T_{\text{ch}}} ; \quad Da_2 = \frac{T_k}{T_{\text{ch}}} = Da_1 R_{\text{et}}^{-1/2}$$

$T_{\text{Turb}}$  - Turbulence time scale

$T_k$  - Kolmogorov time scale

$R_{\text{et}}$  - Turbulence Reynolds number.

Thus Damköhler number signifies the relative rate of fluid dynamic & chemically reactive processes.

Limits: i)  $Da=0$  chemically non-reactive.  
( $\omega = 0$ ;  $T_{\text{ch}} \rightarrow \infty$ )

ii)  $Da \gg 1$ , - "Fast chemistry limit"  
chemical rates  $\gg$  fluid dynamic processes' rate  
- mixing controlled reaction.

iii)  $Da \ll 1$  - "Slow chemistry rate"

Chemical Reaction rate  $\ll$  rate of fluid dynamic process

iv)  $Da \approx 1$  - "Intermediate chemistry"

For this  $Da_1 \gg 1$ ,  $Da_2 \ll 1$

Chemical reactions are slow compared to small-scale (Kolmogorov scale) turbulence process but the reactions are faster than large scale turbulence process.

(4) cont'd

b) i) for mean concentration:

$$\frac{\partial \bar{\phi}}{\partial t} + \gamma_j \frac{\partial \bar{\phi}}{\partial x_j} = \cancel{\frac{\partial}{\partial x_j} (K \frac{\partial \bar{\phi}}{\partial x_j})} + \bar{\omega}$$

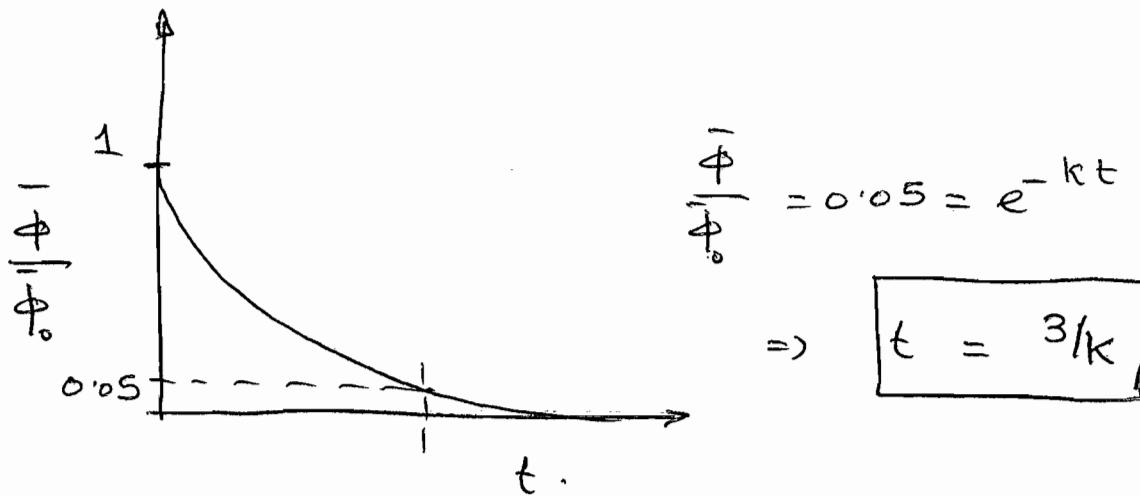
$\downarrow$

$\bar{\phi}_0$  (still day)       $\bar{\phi}_0^0$  (homogeneous)

$$\Rightarrow \frac{\partial \bar{\phi}}{\partial t} = \bar{\omega} = -k \overline{[NO_2]} = -k \bar{\phi}$$

$$\Rightarrow \boxed{\bar{\phi}(t) = \bar{\phi}_0 e^{-kt}}$$

$\text{kg/m}^3$ .



ii) Variance equation: (from Data card)

$$\frac{\partial g}{\partial t} + \gamma_j \frac{\partial g}{\partial x_j} = \cancel{\frac{\partial}{\partial x_j} (K \frac{\partial \bar{\phi}}{\partial x_j})} + 2k \cancel{\left( \frac{\partial \bar{\phi}}{\partial x_j} \right)^2}$$

$\downarrow$

$$-2 \frac{g}{T_{\text{turb}}} + 2 \overline{\phi' \omega'}$$

$$\Rightarrow \frac{\partial g}{\partial t} = -2 \frac{g}{T_{\text{turb}}} + 2 \overline{\phi' \omega'}$$

~ ④ cont'd

$$\ddot{\omega} = -K\phi = -K(\bar{\phi} + \phi')$$

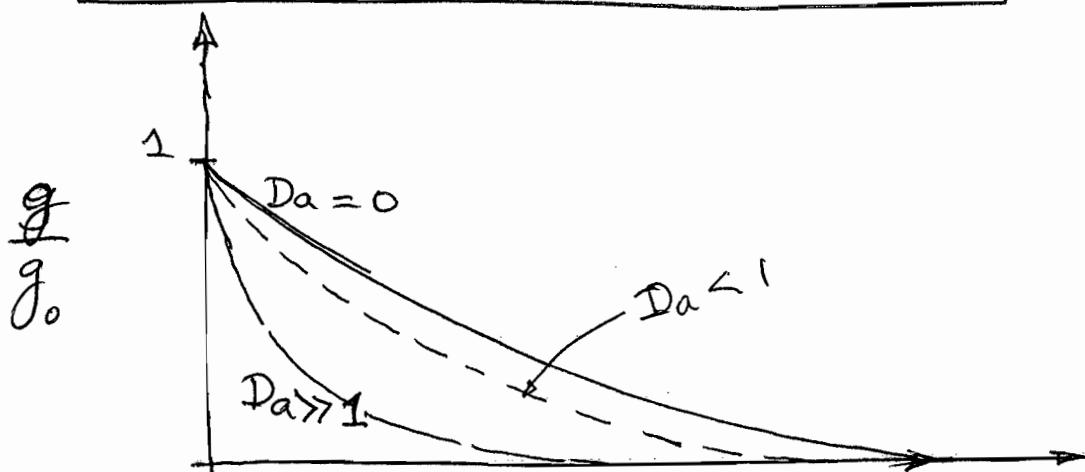
$$\Rightarrow \phi' \ddot{\omega}' = -K \phi'^2 \Rightarrow \overline{\dot{\omega}' \phi'} = -Kg$$

$$\therefore \frac{dg}{dt} = -\frac{2g}{T_{\text{Turb}}} - 2kg.$$

$$= -2 \frac{g}{T_{\text{Turb}}} (1 + KT_{\text{Turb}})$$

$$= -\frac{2g}{T_{\text{Turb}}} (1 + Da)$$

$$\Rightarrow g(t) = g_0 \exp \left[ -\frac{2t}{T_{\text{Turb}}} (1 + Da) \right]$$



for  $K=0$ ;  $\Rightarrow Da=0$ .

$t/T_{\text{Turb}}$

$K \gg 1 \Rightarrow Da \gg 1$  "Fast chemistry limit"

$Ka < 1 \Rightarrow Da < 1$  "Slow chemistry"

