

Solution for 1

$$\textcircled{a} \frac{1}{U} = R_i + \sum r_b + R_e$$

$$0.123 \quad \cancel{1.13} \cdot \frac{1}{1.13} \times 0.14 \quad \frac{1}{0.77} \times 0.1$$

$$\frac{1}{0.045} \times 0.1 + 0.08 \quad \text{total } 2.678$$

$$\therefore \underline{U = 0.373 \text{ W/m}^2\text{K}} \quad [20\%]$$

$$\textcircled{b} \text{ (i) Convection heat gain: } 10 \times \frac{10}{24} \times 100 = 417 \text{ W}$$

$$\text{Solar gain: } 8 \times 207 \times 0.62 = \underline{1,027 \text{ W}}$$

$$1,447 \text{ W}$$

$$Q_c = \sum UA = \overset{8 \times 2.9 + 312}{\cancel{5 \times 64}} \times 0.373 = \overset{140}{\cancel{119}} \text{ W/K}$$

$$Q_r = 5 \times 8 \times 64 \times \frac{1200}{3600} = \underline{853 \text{ W/K}}$$

$$\underline{993 \text{ W/K}}$$

$$\therefore \text{Temp. increment} = 1.48 \text{ }^\circ\text{C}$$

$$\therefore \text{Mean environ. temp} = 18.7 + 1.48 = \underline{20.2 \text{ }^\circ\text{C}} \quad [30\%]$$

$$\text{(ii) Saving in solar gain: } 8 \times 561 \times 0.44 = 1,975 \text{ W}$$

$$\text{Saving in convection gain: } 1000 - 417 = 583$$

$$Q_g = 8 \times 2.9 + 853 = 876 \times 5.8 = \underline{5,082}$$

$$\underline{7640 \text{ W}}$$

specific heat loss factor now

$$8 \times 3.1 + 312 \times 4.6 + 853 = 2,313 \text{ W/K}$$

$$\therefore \text{swing in enviro. temp } \frac{7640}{2313} = \underline{3.30} \text{ } ^\circ\text{C}$$

$$\therefore \text{peak enviro. temp : } 20.2 + 3.3 = \underline{23.5} \text{ } ^\circ\text{C}$$

[4.5%]

Gelbing quite hot - discuss relative to Fanger etc.

[1.5%]

Solution to 2

(a) (i) use equation in Data Sheet

$$\Delta p = 0.043 h (\bar{T}_i - \bar{T}_o) \quad \begin{array}{l} h = 4 \\ \bar{T}_i - \bar{T}_o = 3 \end{array}$$
$$= 0.576 \text{ Pa}$$

$$A = \frac{4 \times 4}{\sqrt{32}} = 2.828 \text{ m}^2$$

$$Q = 0.827 \times 2.828 \times (0.576)^{1/2}$$
$$= \underline{1.68 \text{ m}^3/\text{s}} \quad (= 6048 \text{ m}^3/\text{hour})$$

\therefore hourly air-change rate volume 450 m^3

$$\text{is } \frac{6048}{450} = \underline{13.5} \text{ high!}$$

$$\text{air movement through opening: } \frac{1.68}{4} = \underline{0.42 \text{ m/s}}$$

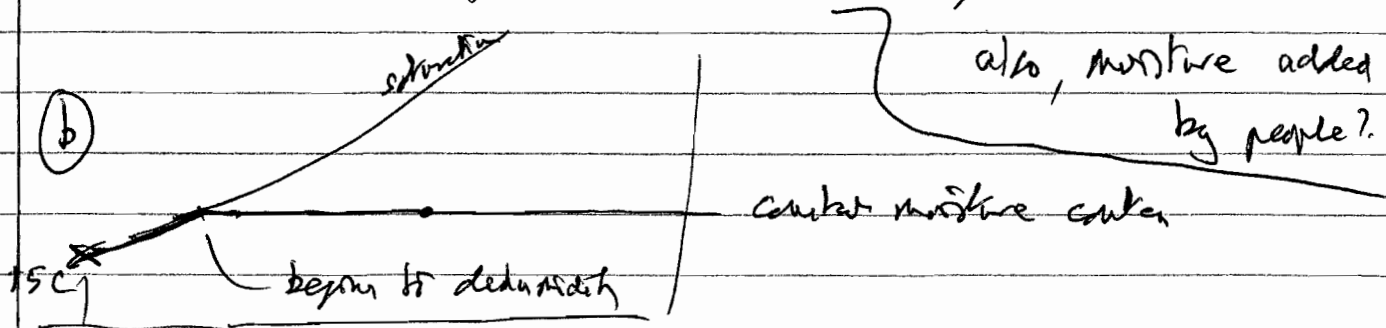
not high

$$\text{Ventilation heat loss: } 1.68 \times 1200 \times 3 = 6.05 \text{ kW}$$

$\frac{\text{m}^3}{\text{s}} \quad \frac{\text{J}}{\text{m}^3 \text{K}} \quad \text{K} \quad *$

total internal heat production is 10 kW — so actually [50%]

the internal temperature would be higher than 25°C d.f.



water - flow needed.

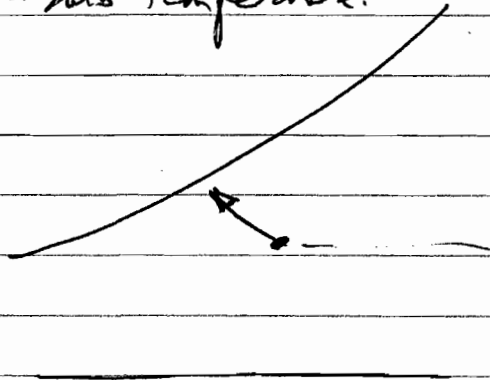
Look at enthalpy of air in and out.

— difference is heat extracted to water

— hence flow-rate required to keep temp. at 1°C
[25%]

(c) water is sprayed in at 19°C , which is exactly the WB temp. of the external air. So the air would travel along the line of constant wet-bulb temperature.

Wet-bulb temperature.



Mix is effective at

reducing db temp of air

— and also it

adding moisture.

So the cooling coil would be preferable, as it alters the db temp. of air much more, and removes moisture — which is likely to improve comfort for the students. Or, use some combined system, e.g. with bypass of some incoming air. [25%]

* This calculation uses average internal temperature — in fact the air existing will be hotter than average, so perhaps OK.

Solution for 3

$$\textcircled{a} \text{ (i) } V = 14 \times 9 \times 4.5 = 567 \text{ m}^3$$

$$A = 2 \times 9 \times 4.5 \times 0.02 = 1.62$$

$$+ 14 \times 4.5 \times 0.10 = 6.3$$

$$14 \times 4.5 \times 0.05 = 3.15$$

$$14 \times 9 \times 0.05 = 6.3$$

$$14 \times 9 \times 0.15$$

$$\frac{18.9}{36.27} \text{ m}^2$$

$$\therefore T = 0.16 \frac{V}{A} = 2.5 \text{ sec.} \quad \left[\begin{array}{l} \text{very high for speech} \\ 25\% \end{array} \right]$$

$$\text{audience } 80 \times 1 \times 0.3 = \frac{24 \text{ m}^3}{24} \Rightarrow T = 1.5 \text{ s}$$

still too high.

$$\textcircled{b} \text{ for } T = 1.1 \text{ s, need } 1.1 = 0.16 \times \frac{567}{A}$$

$$\therefore A = 82.5 \text{ m}^2$$

$$\therefore \text{extra (in empty hall) } \underline{46 \text{ m}^2}$$

$$\left[15\% \right]$$

area of back wall: 40 m^2 only

so even if coeff. nearly 1, need more than

whole wall.

Also better to have reflection from front part of ceiling
— remove acoustic plaster

— to provide say double height of ceiling

3.5m x 9m. plus 4m near rear of side wall.

Extra absorption as floor not possible.

? control according to audience

[40%]

(c) This refers to time for lead-in course,
about the reverberation time of large spaces — e.g.

golfers clubhouse, development of chant in one

note to improve intelligibility, stocker hall, with latter

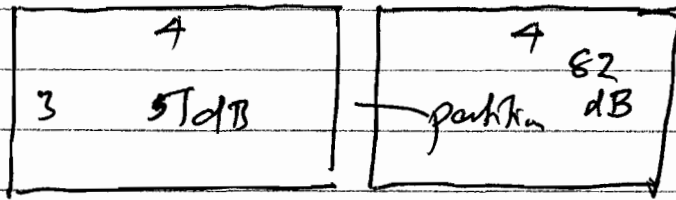
musical characteristics (RT about 2s) and plenty of

reflections etc.

[25%]

Soln to Q 4

(a)



equation $\Delta(\text{SPL}) = R - 10 \log(S/A)$ (Dk Steel)

$$S = 4 \times 3 = 12 \text{ m}^2$$

$$A = 0.2 (4 \times 12 + 2 \times 16) = 16 \text{ m}^2$$

$$\Delta(\text{SPL}) = 82 - 57 = 31 \text{ dB}$$

$$\therefore \underline{R = 29.75 \text{ dB}} \quad [30 \text{ dB}]$$

(b) Assumption: that all sound transmitted between source and receiving room is going through the partition - i.e.

ignore any flanking paths, through ceiling, floor, ducts etc. Also, white noise being used, incoherent - no coincidence with any natural frequency in the partition.

derivation: $R = 10 \log\left(\frac{1}{\tau}\right)$ where τ is transmission coeff. (transmitted / incident intensity)

equation follows from considering sound coming through

partition being absorbed in receiving room. To allow for door, need to find its transmission coeff and that of rest of wall - hence obtain an "average" α , weighted by areas of the different materials, hence modified R. [35%]

(c) This refers to the lectures by Lewers on the different requirements of different spaces.

Meeting room - everyone there to hear clearly, but reasonable sound insulation provided by walls, so probably no need for background noise.

Open-plan office - do not wish to hear distant conversations - maybe have reasonable background noise generated to mask distant sounds.

For un-walled space within office, probably have partitions, and perhaps a reflector above the meeting table, to keep workers sound in, and reinforced

- while hopefully damping down unwanted sound from the rest of the office.

[40%]

[40%]

Solution to 5

(a) 'Daylight factor' is the ratio of illuminance (in lux) ^{simultaneously} on the horizontal working plane in a room, to the illuminance on the horizontal plane due to an unobstructed overcast sky outside. It gives a guide to the use of daylight/natural light in a room — deep rooms having less daylight as the back away from windows. Can be used to help lighting design, topping up daylight with artificial light etc — or to decide, can be used to consider the effect of deep reveals of windows, light shelves, anti-solar-gain blinds etc. [20%]

(b) Daylight factor at a point is hard to calculate (sky component, externally-reflected component, internally-reflected etc) — we need to study whole room to get proper idea — especially if windows on more than one side. More precision, but more effort. Simple formulae for average daylight factor rather crude, take no account of depth/shape of room, reflectance from external obstruction etc. [20%]

$$c) DF_{are} = \frac{TWdM}{(1-r^2)} A_i$$

In this case $T = 0.72$, $W = 5.4 \text{ m}^2$ $d = 53^\circ$

$$M = 0.8 \quad r = 0.5 \quad \Rightarrow DF_{are} = \underline{\underline{1.76\%}}$$

$$A_i = 2 \times 1.75 \times 4.8 + 3(1.75 \times 2 + 4.8 + 1.8)$$

$$= 125 \text{ m}^2$$

[20%]

d) 300 lux internal, @ 1.76% requires 17,000 lux outside.

$$89 + 86 + 82 + 71 + 78 + 57 \times 2 + 27 + 22 + 6 + 4 + 11$$

$$= \underline{\underline{480}}$$

[20%]

e) Altering DF_{are} without altering total window area:

(i) raise and lengthen windows: max length 4.8 m
 \therefore depth 1.125. $\Rightarrow 56^\circ$ (only 6% improvement)

(ii) raise maintenance factor — i.e. clean windows

(iii) increase internal reflectance: e.g. to 60% gives 17% better

(iv) have relatively shallower rooms for some areas — may not be possible.

et.

[20%]

Solution to 6

This question is about the conflict between the need for illumination, to help people view and appreciate art — usually measured in lux on a surface — and the fact that light energy can cause deterioration of the exhibits (e.g. loss of colour in water-colour paintings). So architects have struggled with how to resolve this problem. The final lecture by Prof. Steeman is devoted entirely to this matter.

(a) Discuss rooms why light can cause deterioration — and whether it is the maximum intensity, or integral over time which matters. Need also to consider the requirements for display — low background light, so contrast with exhibits even if actual light level on the art is low to limit deterioration. Daylight good for appreciation. [30%]

(b) Two strategies — (i) entirely artificial

light, hardly any daylight, special lights on exhibits — with examples. : end (ii) using some daylight, because it the better color rendering and psychological benefits, with special ways of lighting exhibits all carefully engineered. (40%)

(e) More of a technical question — may not appeal to engineers. But they can bring examples from history to bear — all covered in final lecture by Steeman (30%)