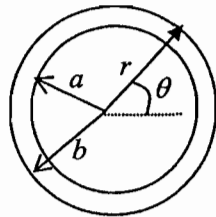
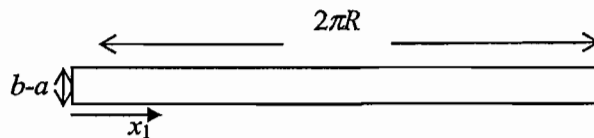


Solutions

PROF A P DOWLING

- Qu1 a) When the annulus height $(b-a)$ is small compared with the radius a , the duct curvature is not important and the annulus can be unwound [10%] as a rectangular duct of height $h = b-a$ and length $2\pi R$, where R is the average radius $(a+b)/2$.

annular cross-section
of duct

rectangular approximation

- b) There are rigid wall boundary conditions on the surfaces $x_2=0$ and h which correspond to the inner and outer radii of the annulus;

$$\frac{\partial p'}{\partial x_2} = 0 \quad \text{on } x_2=0 \text{ and } h.$$

The requirement that in the annular duct $p'(r, \theta, x_3, t) = p'(r, \theta + 2\pi, x_3, t)$ leads to $p'(x_1, x_2, x_3, t) = p'(x_1 + 2\pi R, x_2, x_3, t)$ in the rectangular duct [10%]

- c) Writing the solution in the form

$$p'(x_1, x_2, x_3, t) = e^{i\omega t} f(x_1) g(x_2) h(x_3)$$

and substituting into the wave equation leads to:

$$\frac{\ddot{g}}{g} + \frac{\ddot{h}}{h} + \frac{\omega^2}{c^2} = -\frac{\ddot{f}}{f} = -k^2 \text{ say}$$

$$f = A \cos(kx_1) + B \sin(kx_1)$$

The periodic boundary condition $f(x_1 + 2\pi R) = f(x_1)$ requires that

$$k = \frac{n}{R} \quad \text{for integer } n.$$

With $\frac{\ddot{g}}{g} = k^2$ giving solutions $g(x_2) = C \cos(k_2 x_2) + D \sin(k_2 x_2)$

The rigid wall boundary condition $\frac{dg}{dx_2} = 0$ on $x_2=0$ leads to $D=0$, $k_2 = m\pi/(b-a)$ for m integer.

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Qu1 cont.)

The equation for $h(x_3)$ is therefore

$$\frac{\ddot{h}}{h} = -\frac{\omega^2}{c_0^2} - \frac{\ddot{f}}{f} - \frac{\ddot{g}}{h} = -\frac{\omega^2}{c_0^2} + \left(\frac{n}{R}\right)^2 + \left(\frac{m\pi}{b-a}\right)^2 = -k_{mn}^2 \text{ say}$$

The x_3 -dependence is

$$h(x_3) = E e^{-ik_{mn}x_3} + F e^{ik_{mn}x_3}$$

where E and F are constants and the axial wavenumber k_{mn} is

$$k_{mn} = \sqrt{\frac{\omega^2}{c_0^2} - \left(\frac{n}{R}\right)^2 - \left(\frac{m\pi}{b-a}\right)^2} \quad [60\%]$$

and n is the circumferential mode number.

d) Modes are 'cut-off' if k_{mn} is purely imaginary since they then decay exponentially along the duct, i.e. if

$$\frac{\omega^2}{c_0^2} < \left(\frac{n}{R}\right)^2 + \left(\frac{m\pi}{b-a}\right)^2 \quad [10\%]$$

For $\omega < \underline{nc_0/R}$, the $m=0$ and all higher m modes are cut-off.

e) A rotor with angular velocity Ω produces disturbances which are functions of $\Omega t - \theta$. Hence $\omega/n = \Omega$.

The condition for 'cut-off' disturbances requires

$$\frac{\omega}{n} = \Omega < c_0/R$$

i.e. $\Omega R < c_0$

That is the fan tip speed must be subsonic if [10%]
rotor alone noise is to be cut-off.

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Solutions

Qu2. a) Conservation of mass in the plenum:

$$V \frac{dp_2}{dt} = m_1 - m_2$$

For linear isentropic disturbances,

$$p_2' = c_0^2 p_2'$$

Hence $\frac{V}{c_0^2} \frac{dp_2'}{dt} = m_1 - m_2$.

The duct flow is given to be such that $p_2' = \frac{1}{\kappa} \frac{dm_2}{dt}$

and hence we obtain $\frac{V}{c_0^2 \kappa} \frac{d^2 m_2}{dt^2} = m_1 - m_2$.

For disturbances of radian frequency ω ,

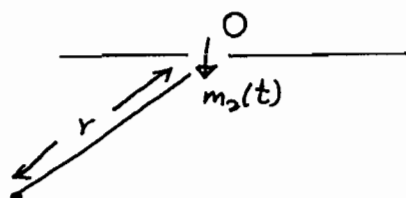
$$-\frac{V \omega^2 m_2'}{c_0^2 \kappa} = m_1' - m_2'$$

i.e. $m_2' \left(1 - \frac{V \omega^2}{c_0^2 \kappa} \right) = m_1'$

or
$$\underline{\underline{m_2'(t) = \frac{m_1'(t)}{1 - V \omega^2 / (\kappa c_0^2)}}}$$

[30%]

b)



Mass ejection at a rate $m_2(t)$ is a monopole point source and in free space would generate a sound field

$$p'(r, t) = \frac{\dot{m}_2(t - r/c)}{4\pi r} \quad \text{at a distance } r \text{ from the source.}$$

Here, the source is on large rigid surface and so there is an equal image source, resulting in a doubling of sound pressure:

$$\underline{\underline{p'(r, t) = \frac{2 \dot{m}_2(t - r/c)}{4\pi r}}}$$

[20%]

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Qu 2 cont.)

c) To determine the far-field sound we just need to combine results from a) and b). For fluctuations of frequency ω

$$p_{\text{rms}}(r) = \frac{\omega}{2\pi r} m_{2\text{rms}} = \frac{\omega m_{1\text{rms}}}{2\pi r (1 - V\omega^2 / (\kappa c_0^2))}$$

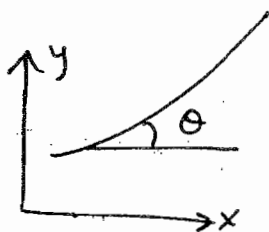
Substituting the given values gives

$$\begin{aligned} p_{\text{rms}}(200) &= \frac{2\pi 500}{2\pi 200} 2 \times 10^{-4} \frac{1}{1 - \frac{4.8 \times 10^{-4}}{0.04} \left(\frac{2\pi 500}{343}\right)^2} \\ &= 5 \times 10^{-4} \frac{1}{1 - 1.006683} = 0.0748 \text{ N/m}^2 \\ &= \underline{71.5 \text{ dB}} \quad [40^\circ] \end{aligned}$$

d) The problem with the volume of $4.8 \times 10^{-4} \text{ m}^3$ is that the plenum and short exit duct with $\kappa = 0.04 \text{ m}$ are resonant at 500 Hz. We want to move the volume away from this resonance while still keeping it small enough for the compact approximation of a uniform pressure fluctuation throughout the plenum (otherwise there might be additional resonances from standing waves in the plenum). For example, for $V = 1.5 \times \text{original volume}$,

$$\begin{aligned} p_{\text{rms}} &= 5 \times 10^{-4} \frac{1}{1 - 1.5 \times 1.006683} = 9.8 \times 10^{-4} \text{ N/m}^2 \\ &= \underline{33.8 \text{ dB}} \end{aligned}$$

3(a) (i)



$$\text{Snell: } \frac{\sin \theta}{c(x)} = \text{constant}$$

Suppose launched from $(0,0)$ at angle θ .

$$y' = \tan \theta, \quad \frac{y'}{\sqrt{1+y'^2}} = \frac{\sin \theta_0}{\beta}$$

$$\frac{\alpha x + \beta}{\beta}$$

$$(y')^2 \left\{ 1 - \frac{\sin^2 \theta_0}{\beta^2} (\alpha x + \beta)^2 \right\} = \frac{\sin^2 \theta_0}{\beta^2} (\alpha x + \beta)^2$$

$$\rightarrow y' = \left(\frac{\alpha x + \beta}{\beta} \right) \sin \theta_0$$

$$\left[1 - \frac{(\alpha x + \beta)^2 \sin^2 \theta_0}{\beta^2} \right]^{1/2}$$

$$\int dy = \int \frac{\alpha x + \beta \sin \theta_0}{\beta \left[1 - \frac{(\alpha x + \beta)^2 \sin^2 \theta_0}{\beta^2} \right]^{1/2}} dx$$

$$y = -\frac{\beta}{\alpha \sin \theta_0} \left[1 - \frac{(\alpha x + \beta)^2 \sin^2 \theta_0}{\beta^2} \right]^{1/2} + k$$

$$\text{when } x=0, y=0 \therefore k = \frac{\beta}{\alpha \sin \theta_0} \left[1 - \frac{\beta^2 \sin^2 \theta_0}{\beta^2} \right] = \frac{\beta}{\alpha \tan \theta_0}$$

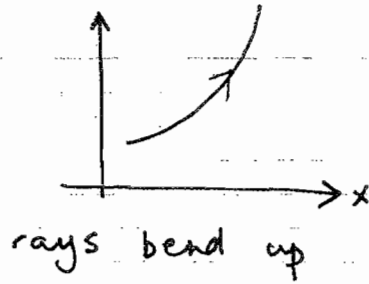
$$\therefore y - \frac{\beta \cot \theta_0}{\alpha} = -\frac{\beta}{\alpha \sin \theta_0} \left[1 - \frac{(\alpha x + \beta)^2 \sin^2 \theta_0}{\beta^2} \right]^{1/2}$$

$$\rightarrow \left(x + \frac{\beta}{\alpha} \right)^2 + \left(y - \frac{\beta \cot \theta_0}{\alpha} \right)^2 = \frac{\beta^2}{\alpha^2 \sin^2 \theta_0}$$

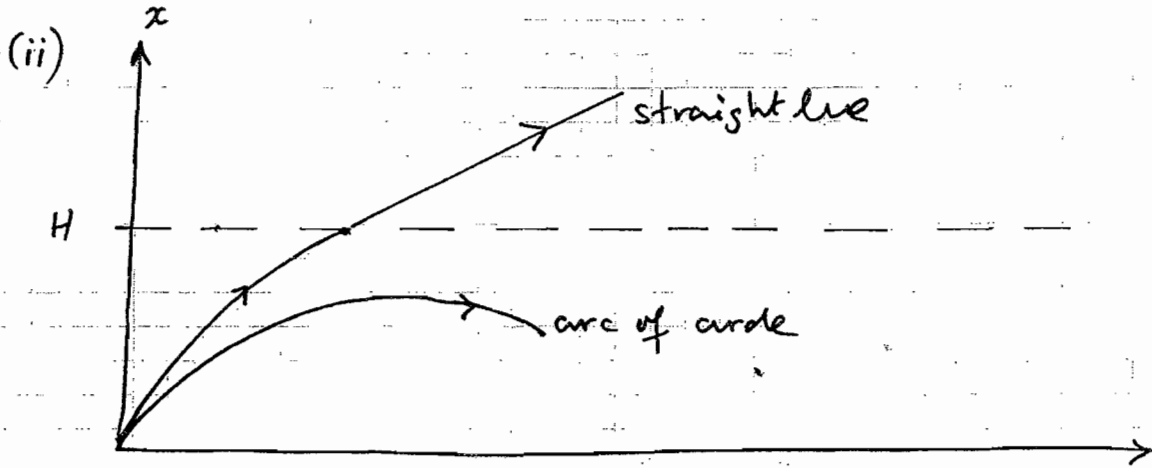
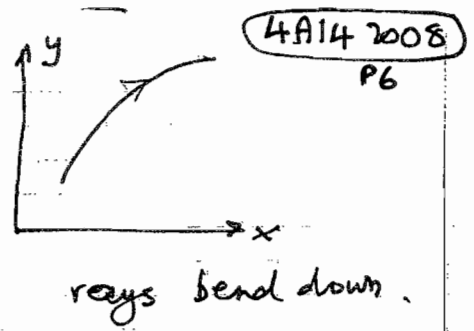
\therefore circle centre $-\frac{\beta}{\alpha}, \frac{\beta \cot \theta_0}{\alpha}$

radius $\left| \frac{\beta}{\alpha \sin \theta_0} \right|$

Qu 3d(i) cont.
 $\alpha > 0$



$\alpha < 0$



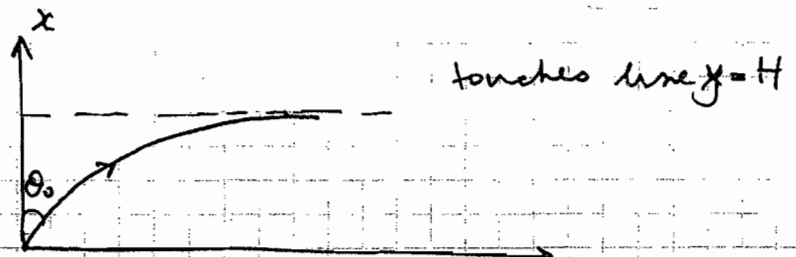
Here must have α positive to make sense

critical case:

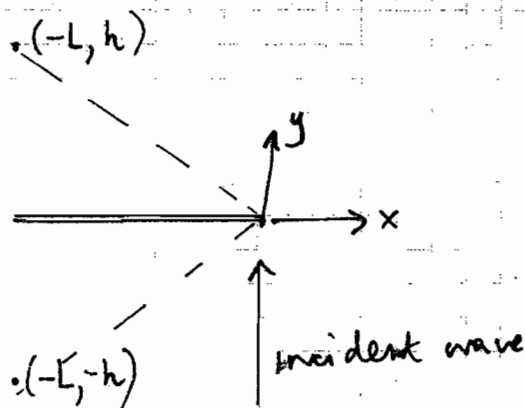
$$\frac{\sin \theta}{c_0} = \text{constant}$$

$$\therefore \frac{\sin \theta_0}{\alpha H} = \frac{\sin(\pi/2)}{2\alpha H}$$

$$\therefore \sin \theta_0 = 1/2, \quad \theta_0 = \underline{\underline{\sin^{-1}(1/2) = 30^\circ}}$$



3(b)



Qu 3b cont.)
In notation of data sheet $\theta_0 = -\pi/2$

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(i) $\theta = \pi - \tan^{-1}(h/L)$

$$Pd = \underset{=1}{\overset{P_{inc}}{\uparrow}} \left(\frac{z}{\pi k_0 r} \right)^{1/2} \frac{\sin(-\frac{\pi}{4}) \sin(\frac{\pi}{2} - \frac{1}{2} \tan^{-1}(\frac{h}{L}))}{\cos(\pi - \tan^{-1}(\frac{h}{L})) + \cos(-\pi/2)} e^{-i\frac{\pi}{4} - ik_0 r}$$

$$r = \sqrt{h^2 + L^2}$$

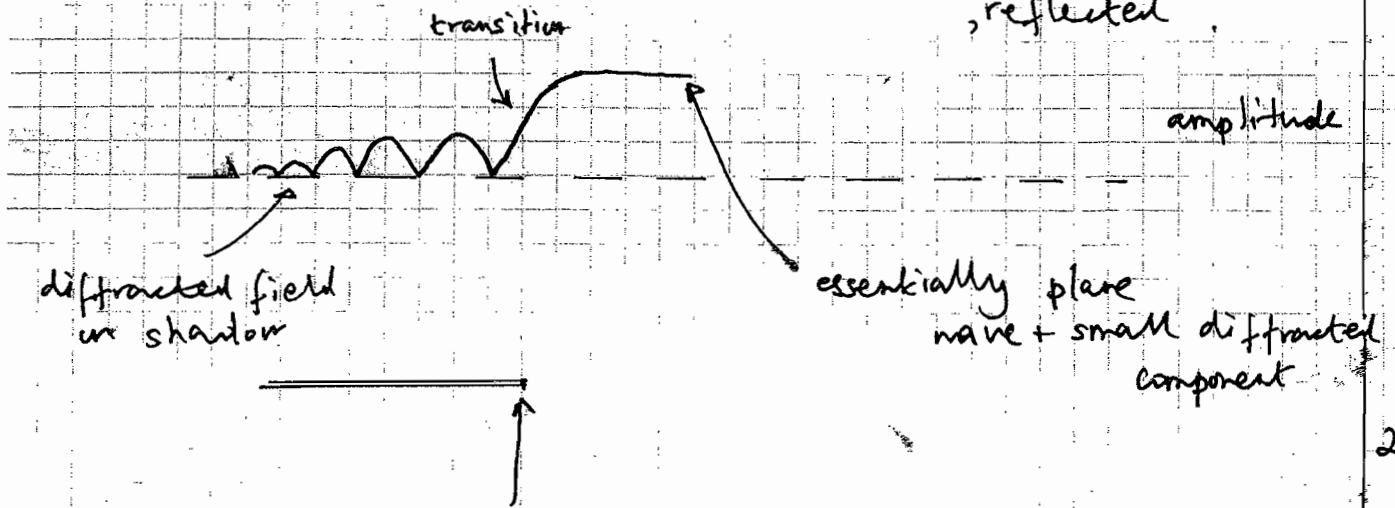
$$\rightarrow \left(\frac{z}{\pi k_0 r} \right)^{1/2} \frac{+1}{\sqrt{2}} \frac{\cos(\frac{1}{2} \tan^{-1}(\frac{h}{L}))}{\cos(\tan^{-1}(\frac{h}{L}))} e^{-i\pi/4 - ik_0 r}$$

(ii) $\theta = -\pi + \tan^{-1}(h/L)$

$$\therefore Pd = - \left(\frac{z}{\pi k_0 r} \right)^{1/2} \frac{1}{\sqrt{2}} \frac{\cos(\frac{1}{2} \tan^{-1}(\frac{h}{L}))}{\cos(\tan^{-1}(h/L))} e^{-i\frac{\pi}{4} + ik_0 r}$$

In case (i) geometrical optics field = 0 (shadow)

In (ii) " " " = $\exp(i\omega t + ik_0 y)$, reflected



4 (a) Starting from $p'(x, t) = \frac{\partial^2}{\partial x_i \partial x_j} \int \frac{T_{ij}(\underline{y}, t - \frac{|\underline{x} - \underline{y}|}{c_0}) d^3 y}{|\underline{x} - \underline{y}|}$
 (data sheet)

Replace $\frac{1}{|\underline{x} - \underline{y}|}$ by $\frac{1}{|\underline{x}|}$, as assume observer in far field

further, assume retarded-time variation negligible across source - compact

$$\therefore p'(x, t) = \frac{\partial^2}{\partial x_i \partial x_j} \int \frac{T_{ij}(\underline{y}, t - \frac{|\underline{x}|}{c_0}) d^3 y}{|\underline{x}|}$$

Take $\frac{1}{|\underline{x}|}$ outside derivatives, as only want $\frac{1}{|\underline{x}|}$ term

in far field

$$\begin{aligned} p' &\sim \frac{1}{|\underline{x}|} \frac{\partial^2}{\partial x_i \partial x_j} \int \frac{T_{ij}(\underline{y}, t - \frac{|\underline{x}|}{c_0}) d^3 y}{|\underline{x}|} \\ &= \frac{1}{|\underline{x}|} \frac{\partial^2}{\partial x_i \partial x_j} S_{ij}(t - |\underline{x}|/c_0), \quad S_{ij} \equiv \int T_{ij} \\ &= \frac{1}{|\underline{x}|} \frac{x_i x_j}{|\underline{x}|^2} \frac{1}{c_0^2} \ddot{S}_{ij} \end{aligned}$$

Take $T_{ij} = O(\rho u_0^2)$ $u_0 =$ velocity scale

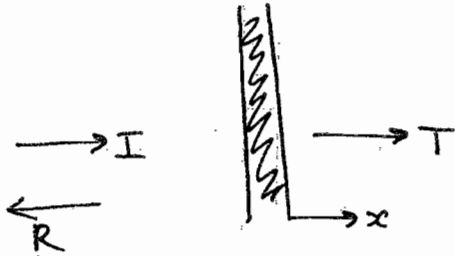
$S_{ij} = \int T_{ij} = O(\rho_0 u_0^2 l^3)$ $l =$ length scale
 time derivative $= O(u_0/l)$, i.e. inverse time scale

$$\begin{aligned} \therefore p' &= O\left(\frac{1}{|\underline{x}|} \cdot \frac{1}{c_0^2} \frac{u_0^2}{l^2} \rho_0 u_0^2 l^3\right) \\ &= O\left(\frac{l}{|\underline{x}|} \rho_0 c_0^2 \left[\frac{u_0}{c_0}\right]^4\right) \end{aligned}$$

Acoustic intensity $\propto p'^2 \propto \left[\frac{u_0}{c_0}\right]^8$

Qu
4(b)

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$$x < 0 \quad p' = I e^{i\omega t - i\omega x/c_0} + R e^{i\omega t + i\omega x/c_0}$$

$$x > 0 \quad p' = T e^{i\omega t - i\omega x/c_0}$$

Pressure difference across the wall drives wall motion.

Eqⁿ of motion is

$$m \frac{d}{dt} (V e^{i\omega t}) = p'(x=-0, t) - p'(x=+0, t)$$

$$\Rightarrow mi\omega V = -T + I + R. \quad (1)$$

Acoustic velocity at $x=+0$ is $\frac{T}{\rho_0 c_0}$ ← plane wave impedance

" " " $x=-0$ is $\frac{I-R}{\rho_0 c_0}$

$$\text{Continuity} \Rightarrow \frac{T}{\rho_0 c_0} = \frac{I-R}{\rho_0 c_0} = V \quad (2)$$

put (2) into (1)

$$mi\omega V = \rho_0 c_0 [-V] + I + I - \rho_0 c_0 V$$

$$\Rightarrow V = \frac{2I}{mi\omega + 2\rho_0 c_0}$$

$$(2) \Rightarrow T = \frac{2\rho_0 c_0 I}{2\rho_0 c_0 + mi\omega}$$

$$\therefore \frac{|T|}{|I|} = \frac{2\rho_0 c_0}{\sqrt{m^2 \omega^2 + 4\rho_0^2 c_0^2}}$$

Qn 4b) cont.
 $\frac{|T|}{|I|}$

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$$= \left[\left(\frac{mw}{\rho_0 c_0} \right)^2 + 4 \right]^{1/2}$$

one wavelength is $\frac{2\pi c_0}{\omega}$, mass of air is $\frac{2\pi c_0 \rho_0}{\omega}$

per unit wall area

$$\therefore \frac{\text{mass of wall per unit area}}{\text{mass of sound in one } \lambda \text{ " " "}} = \left(\frac{mw}{\rho_0 c_0} \right) \frac{1}{2\pi}$$

Hence $\frac{mw}{\rho_0 c_0}$ controls level of transmission

$\frac{mw}{\rho_0 c_0} \gg 1 \Rightarrow$ heavy wall, long waves, $|T| \ll |I|$

$\frac{mw}{\rho_0 c_0} \ll 1 \Rightarrow$ light wall, short waves, $|T| \approx |I|$

2

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