

ENGINEERING TRIPOS

PART IIB

Friday 13 May 2011

9.00 to 10.30

Module 4C9

CONTINUUM MECHANICS

*Answer not more than **two** questions.*

All questions carry the same number of marks.

*The **approximate** percentage of marks allocated to each part of a question is indicated in the right margin.*

Attachment:

4C9 datasheet (6 pages).

STATIONERY REQUIREMENTS

Single-sided script paper

SPECIAL REQUIREMENTS

Engineering Data Book

CUED approved calculator allowed

**You may not start to read the questions
printed on the subsequent pages of this
question paper until instructed that you may
do so by the Invigilator**

1 The Kronecker delta and the permutation tensor are denoted by the symbols δ_{ij} and ϵ_{ijk} , respectively.

(a) Determine the component f_2 for the following vector expressions

(i) $f_i = C_{ij}t_j$, [10%]

(ii) $f_i = \epsilon_{ijk}T_{jk}$. [10%]

(b) Using the $\epsilon - \delta$ identity, write $\epsilon_{pqs}\epsilon_{sqr}$ in terms of the Kronecker delta. [15%]

(c) Given the coordinate transformation law $A_{ij} = a_{pi}a_{qj}A'_{pq}$, where a_{mn} denotes the cosine of the angle between the m^{th} primed and the n^{th} unprimed coordinate axes, show that

(i) A_{ii} ,

(ii) $A_{ij}A_{ij}$ and

(iii) $\epsilon_{ijk}\epsilon_{kjp}A_{ip}$

are invariant under coordinate transformations. [45%]

(d) If B_{ij} is skew-symmetric (i.e. $B_{ij} = -B_{ji}$) and A_{ij} is symmetric (i.e. $A_{ij} = A_{ji}$) show that $A_{ij}B_{ij} = 0$. [20%]

- 2 (a) For a function $\phi(x, y)$ in Cartesian co-ordinates show that

$$\frac{\partial \phi}{\partial y} = \frac{\partial \phi}{\partial r} \sin \theta + \frac{\partial \phi}{\partial \theta} \frac{\cos \theta}{r}$$

where the polar coordinates (r, θ) are related to the Cartesian coordinates (x, y) via the usual relations $x = r \cos \theta$ and $y = r \sin \theta$. [20%]

- (b) The Airy stress function

$$\phi = A \left(\theta + \frac{\sin 2\theta}{2} \right),$$

where A is a constant, has been proposed to determine the stress field in an elastic half-space subjected to a surface couple M as shown in Fig. 1(a). Show that the stresses σ_{rr} , $\sigma_{\theta\theta}$ and $\sigma_{r\theta}$ calculated from ϕ satisfy the boundary conditions and hence determine A in terms of M . [30%]

- (c) We wish to determine the stress field in the case of two equal and opposite couples of magnitude M acting a small distance a apart as shown in Fig. 1(b). Using superposition, or otherwise, determine an appropriate Airy stress function in the limit of a vanishingly small a . (Hint: Use the co-ordinate transformation relation derived in part (a)). [50%]

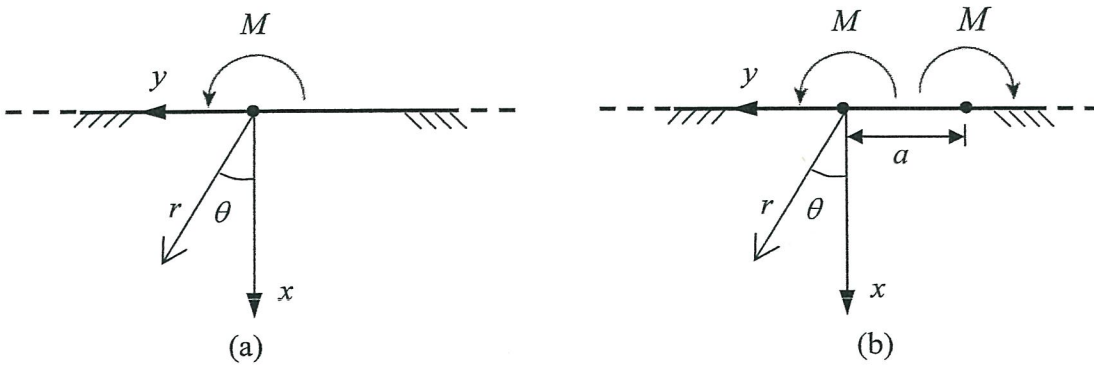


Fig. 1

(TURN OVER)

3 (a) Explain the physical basis for the plastic deformation of metals. Why is volume conserved during plastic deformation? [15%]

(b) Volumetric plastic straining can occur in a porous metallic solid. Explain this observation. [15%]

(c) A thin-walled cylindrical tube of length l , diameter d and wall thickness t is made from an isotropic, elastic-plastic solid. The solid obeys the von Mises yield criterion and has an initial yield strength σ_Y . Strain hardening of this solid is linear such that in post-yield uniaxial tension, the tensile stress σ is related to the axial plastic strain ε^P by a constant plastic modulus $h = d\sigma / d\varepsilon^P$. Elastic straining is characterised by a Young's modulus E and Poisson ratio ν . The tube is subjected to an end tensile load P and torque Q which increase with time in fixed proportion $Q = \alpha Pd$, where α is a dimensionless constant.

(i) Determine the stress state in the tube wall in terms of Q and P . [10%]

(ii) Obtain the yield load P_Y for $\alpha = 0$ and $\alpha = 1$. [20%]

(iii) For $\alpha = 1$, obtain an expression for the hardening rate dP / du at $P \geq P_Y$, where u is the axial extension. You may neglect the possibility of necking. [40%]

END OF PAPER

ENGINEERING TRIPOS Part IIB

Module 4C9 Data Sheet

SUBSCRIPT NOTATION

Repeated suffix implies summation

$$\underline{a} = a_1 \underline{e}_1 + a_2 \underline{e}_2 + a_3 \underline{e}_3$$

$$a_i \underline{e}_i$$

$$\underline{a} \cdot \underline{b}$$

$$a_i b_i \equiv a_i b_j \delta_{ij}$$

$$\underline{c} = \underline{a} \times \underline{b}$$

$$c_i = e_{ijk} a_j b_k$$

$$\underline{d} = \underline{a} \times (\underline{b} \times \underline{c})$$

$$d_k = -e_{ijk} e_{irs} a_j b_r c_s = a_j b_k c_j - a_i b_i c_k$$

Kronecker delta δ_{ij}

$$\delta_{ij} = 1 \text{ for } i = j \text{ and } \delta_{ij} = 0 \text{ for } i \neq j$$

$$e_{ijk}$$

$e_{ijk} = 1$ when indices cyclic; $= -1$ when indices anticyclic
and $= 0$ when any indices repeat

$e - \delta$ identity

$$e_{ijk} e_{ilm} \equiv \delta_{jl} \delta_{km} - \delta_{jm} \delta_{kl}$$

trace a

$$\text{tr } a = a_{ii} = a_{11} + a_{22} + a_{33}$$

$$\frac{\partial \sigma_{ij}}{\partial x_i} = \frac{\partial \sigma_{1j}}{\partial x_1} + \frac{\partial \sigma_{2j}}{\partial x_2} + \frac{\partial \sigma_{3j}}{\partial x_3}$$

$$\sigma_{ij,i}$$

$$\text{grad } \phi = \nabla \phi$$

$$\frac{\partial \phi}{\partial x_i} = \phi_{,i}$$

$\text{div } \underline{V}$

$$V_{i,i}$$

$$\text{curl } \underline{V} \equiv \underline{\nabla} \times \underline{V}$$

$$e_{ijk} V_{k,j}$$

Rotation of Orthogonal Axes

If $01'2'3'$ is related to 0123 by rotation matrix a_{ij}

vector v_i becomes

$$v'_\alpha = a_{\alpha i} v_i$$

tensor σ_{ij} becomes

$$\sigma'_{\alpha\beta} = a_{\alpha i} a_{\beta j} \sigma_{ij}$$

Evaluation of principal stresses

deviatoric stress $s_{ij} = \sigma_{ij} - \frac{1}{3}\sigma_{kk}\delta_{ij}$

$$\sigma^3 - I_1\sigma^2 + I_2\sigma - I_3 = 0$$

$$I_1 = \sigma_{ii} = \text{tr}\sigma$$

$$I_2 = \frac{1}{2}(\sigma_{ii}\sigma_{jj} - \sigma_{ij}\sigma_{ij})$$

$$I_3 = \frac{1}{6}(e_{ijk}e_{pqr}\sigma_{ip}\sigma_{jq}\sigma_{kr})$$

$$s^3 - J_2s - J_3 = 0$$

$$J_1 = s_{ii} = \text{trs} ; J_2 = \frac{1}{2}s_{ij}s_{ij} ; J_3 = \frac{1}{3}s_{ij}s_{jk}s_{ki}$$

equilibrium

$$\sigma_{ij,i} + b_j = 0$$

small strains

$$\varepsilon_{ij} = \frac{1}{2}\left(\frac{\partial u_i}{\partial x_j} + \frac{\partial u_j}{\partial x_i}\right) \equiv \frac{1}{2}(u_{i,j} + u_{j,i})$$

compatibility

$$\varepsilon_{ij,kl} + \varepsilon_{kl,ij} - \varepsilon_{ij,ki} - \varepsilon_{kl,ji} - e_{pik}e_{qjl}\varepsilon_{ij,kl} = 0$$

equivalent to $e_{pik}e_{qjl}\varepsilon_{ij,kl} \equiv e_{pik}e_{qjl}\frac{\partial^2\varepsilon_{ij}}{\partial x_k\partial x_l} = 0$

Linear elasticity

$$\sigma_{ij} = C_{ijkl}\varepsilon_{kl}$$

Hooke's law

$$E\varepsilon_{ij} = (1+\nu)\sigma_{ij} - \nu\sigma_{kk}\delta_{ij}$$

Lamé's equations

$$\sigma_{ij} = \lambda\varepsilon_{kk}\delta_{ij} + 2\mu\varepsilon_{ij}$$

von Mises equivalent stress

$$\sigma_e \equiv \bar{\sigma} = \sqrt{\frac{3}{2}s_{ij}s_{ij}} = \sqrt{3J_2}$$

equivalent strain increment

$$d\bar{\varepsilon} = \sqrt{\frac{2}{3}d\varepsilon_{ij}d\varepsilon_{ij}}$$

Elastic torsion of prismatic bars

Warping function $\Psi(x_1, x_2)$ satisfies $\nabla^2\Psi = \Psi_{,ii} = 0$

If Prandtl stress function $\phi(x_1, x_2)$ satisfies $\nabla^2\phi = \phi_{,ii} = -2G\alpha$ where α is the twist per unit length then

$$\sigma_{31} = \phi_{,2} = \frac{\partial\phi}{\partial x_2} , \sigma_{32} = -\phi_{,1} = -\frac{\partial\phi}{\partial x_1} \text{ and } T = 2\iint_A \phi(x_1, x_2)dx_1dx_2$$

Equivalence of elastic constants

	E	ν	$G=\mu$	λ
E, ν	-	-	$\frac{E}{2(1+\nu)}$	$\frac{\nu E}{(1+\nu)(1-2\nu)}$
E, G	-	$\frac{E-2G}{2G}$	-	$\frac{(2G-E)G}{E-3G}$
E, λ	-	$\frac{E-\lambda+R}{4\lambda}$	$\frac{E-3\lambda+R}{4}$	-
ν, G	$2G(1+\nu)$	-	-	$\frac{2G\nu}{1-2\nu}$
ν, λ	$\frac{\lambda(1+\nu)(1-2\nu)}{\nu}$	-	$\frac{\lambda(1-2\nu)}{2\nu}$	-
G, λ	$\frac{G(3\lambda+2G)}{\lambda+G}$	$\frac{\lambda}{2(\lambda+G)}$	-	-

$$R = \sqrt{E^2 + 2E\lambda + 9\lambda^2}$$

Two-dimensional Airy Stress function

Biharmonic equation $\nabla^4 \phi \equiv \phi_{,\alpha\alpha\beta\beta} = 0$

Stresses $\sigma_{\alpha\beta} = e_{\alpha\gamma} e_{\beta\delta} \phi_{,\gamma\delta}$

where $e_{\alpha\beta} \equiv e_{3\alpha\beta} = \begin{cases} 1 & \text{if } \alpha=1, \beta=2 \\ 0 & \text{if } \alpha=\beta \\ -1 & \text{if } \alpha=2, \beta=1 \end{cases}$

Plane stress and plane strain

$$G\varepsilon_{11} = \frac{1}{8} \{ \sigma_{11}(1+\kappa) + \sigma_{22}(\kappa-3) \}$$

$$G\varepsilon_{22} = \frac{1}{8} \{ \sigma_{22}(1+\kappa) + \sigma_{11}(\kappa-3) \}$$

$$G\varepsilon_{12} = \frac{\sigma_{12}}{2}$$

where $\begin{cases} \kappa = (3-\nu)/(1+\nu) & \text{in plane stress and} \\ \kappa = 3-4\nu & \text{in plane strain} \end{cases}$

Plasticity

von Mises yield criterion

$$f = \sigma_e - Y = 0$$

J_2 flow rule

$$\dot{\epsilon}_{ij}^{PL} = \frac{3}{2} \frac{s_{ij}}{\sigma_e} \frac{\dot{\sigma}_e}{h}$$

Slip Line Fields

Henky equations

$$p + 2k\phi = \text{constant along } \alpha \text{ line}$$

$$p - 2k\phi = \text{constant along } \beta \text{ line}$$

Geiringer equations

$$\frac{dv_\alpha}{ds} = v_\beta \frac{d\phi}{ds} \quad \text{along } \alpha \text{ line}$$

$$\frac{dv_\beta}{ds} = -v_\alpha \frac{d\phi}{ds} \quad \text{along } \beta \text{ line}$$

Table I – The Michell solutions — stress components

$\phi(r, \theta)$	σ_{rr}	$\sigma_{\theta\theta}$	$\sigma_{r\theta}$
r^2	2	2	0
$r^2 \ln r$	$2 \ln r + 1$	$2 \ln r + 3$	0
$\ln r$	$1/r^2$	$-1/r^2$	0
θ	0	0	$1/r^2$
$r^3 \cos \theta$	$2r \cos \theta$	$6r \cos \theta$	$2r \sin \theta$
$r\theta \sin \theta$	$2 \cos \theta / r$	0	0
$r \ln r \cos \theta$	$\cos \theta / r$	$\cos \theta / r$	$\sin \theta / r$
$\cos \theta / r$	$-2 \cos \theta / r^3$	$2 \cos \theta / r^3$	$-2 \sin \theta / r^3$
$r^3 \sin \theta$	$2r \sin \theta$	$6r \sin \theta$	$-2r \cos \theta$
$r\theta \cos \theta$	$-2 \sin \theta / r$	0	0
$r \ln r \sin \theta$	$\sin \theta / r$	$\sin \theta / r$	$-\cos \theta / r$
$\sin \theta / r$	$-2 \sin \theta / r^3$	$2 \sin \theta / r^3$	$2 \cos \theta / r^3$
$r^{n+2} \cos n\theta$	$-(n+1)(n-2)r^n \cos n\theta$	$(n+1)(n+2)r^n \cos n\theta$	$n(n+1)r^n \sin n\theta$
$r^{-n+2} \cos n\theta$	$-(n+2)(n-1)r^{-n} \cos n\theta$	$(n-1)(n-2)r^{-n} \cos n\theta$	$-n(n-1)r^{-n} \sin n\theta$
$r^n \cos n\theta$	$-n(n-1)r^{n-2} \cos n\theta$	$n(n-1)r^{n-2} \cos n\theta$	$n(n-1)r^{n-2} \sin n\theta$
$r^{-n} \cos n\theta$	$-n(n+1)r^{-n-2} \cos n\theta$	$n(n+1)r^{-n-2} \cos n\theta$	$-n(n+1)r^{-n-2} \sin n\theta$
$r^{n+2} \sin n\theta$	$-(n+1)(n-2)r^n \sin n\theta$	$(n+1)(n+2)r^n \sin n\theta$	$-n(n+1)r^n \cos n\theta$
$r^{-n+2} \sin n\theta$	$-(n+2)(n-1)r^{-n} \sin n\theta$	$(n-1)(n-2)r^{-n} \sin n\theta$	$n(n-1)r^{-n} \cos n\theta$
$r^n \sin n\theta$	$-n(n-1)r^{n-2} \sin n\theta$	$n(n-1)r^{n-2} \sin n\theta$	$-n(n-1)r^{n-2} \cos n\theta$
$r^{-n} \sin n\theta$	$-n(n+1)r^{-n-2} \sin n\theta$	$n(n+1)r^{-n-2} \sin n\theta$	$n(n+1)r^{-n-2} \cos n\theta$

Table II – The Michell solutions — displacement components

For plane strain $\kappa = 3 - 4\nu$; for plane stress $\kappa = (3 - \nu)/(1 + \nu)$

$\phi(r, \theta)$	$2Gu_r$	$2Gu_\theta$
r^2	$(\kappa - 1)r$	0
$r^2 \ln r$	$(\kappa - 1)r \ln r - r$	$(\kappa + 1)r\theta$
$\ln r$	$-1/r$	0
θ	0	$-1/r$
$r^3 \cos \theta$	$(\kappa - 2)r^2 \cos \theta$	$(\kappa + 2)r^2 \sin \theta$
$r\theta \sin \theta$	$0.5[(\kappa - 1)\theta \sin \theta - \cos \theta$ $+ (\kappa + 1) \ln r \cos \theta]$	$0.5[(\kappa - 1)\theta \cos \theta - \sin \theta$ $- (\kappa + 1) \ln r \sin \theta]$
$r \ln r \cos \theta$	$0.5[(\kappa + 1)\theta \sin \theta - \cos \theta$ $+ (\kappa - 1) \ln r \cos \theta]$	$0.5[(\kappa + 1)\theta \cos \theta - \sin \theta$ $- (\kappa - 1) \ln r \sin \theta]$
$\cos \theta / r$	$\cos \theta / r^2$	$\sin \theta / r^2$
$r^3 \sin \theta$	$(\kappa - 2)r^2 \sin \theta$	$-(\kappa - 2)r^2 \cos \theta$
$r\theta \cos \theta$	$0.5[(\kappa - 1)\theta \cos \theta + \sin \theta$ $- (\kappa + 1) \ln r \sin \theta]$	$0.5[-(\kappa - 1)\theta \sin \theta - \cos \theta$ $- (\kappa + 1) \ln r \cos \theta]$
$r \ln r \sin \theta$	$0.5[-(\kappa + 1)\theta \cos \theta - \sin \theta$ $+ (\kappa - 1) \ln r \sin \theta]$	$0.5[(\kappa + 1)\theta \sin \theta + \cos \theta$ $+ (\kappa - 1) \ln r \cos \theta]$
$\sin \theta / r$	$\sin \theta / r^2$	$-\cos \theta / r^2$
$r^{n+2} \cos n\theta$	$(\kappa - n - 1)r^{n+1} \cos n\theta$	$(\kappa + n + 1)r^{n+1} \sin n\theta$
$r^{-n+2} \cos n\theta$	$(\kappa + n - 1)r^{-n+1} \cos n\theta$	$-(\kappa - n + 1)r^{-n+1} \sin n\theta$
$r^n \cos n\theta$	$-nr^{n-1} \cos n\theta$	$nr^{n-1} \sin n\theta$
$r^{-n} \cos n\theta$	$nr^{-n-1} \cos n\theta$	$nr^{-n-1} \sin n\theta$
$r^{n+2} \sin n\theta$	$(\kappa - n - 1)r^{n+1} \sin n\theta$	$-(\kappa + n + 1)r^{n+1} \cos n\theta$
$r^{-n+2} \sin n\theta$	$(\kappa + n - 1)r^{-n+1} \sin n\theta$	$(\kappa - n + 1)r^{-n+1} \cos n\theta$
$r^n \sin n\theta$	$-nr^{n-1} \sin n\theta$	$-nr^{n-1} \cos n\theta$
$r^{-n} \sin n\theta$	$nr^{-n-1} \sin n\theta$	$-nr^{-n-1} \cos n\theta$

JAW//NAF

Numerical answers to 4C9(2011)

1. (a)
(i) $f_2 = C_{21}t_1 + C_{22}t_2 + C_{23}t_3$
(ii) $T_{31} - T_{13}$

(b) $-2\delta_{pr}$

2. (b) $A = \frac{M}{\pi}$

(c) $\phi_1 = -\frac{2Ma}{\pi r} \cos^3 \theta$

3.

(c)

$$\sigma_a = \frac{P}{\pi dt}$$

(i)

$$\tau = \frac{2Q}{\pi d^2 t}$$

$$P_Y = \sigma_Y \pi dt$$

(ii)

$$P_Y = \frac{\sigma_Y \pi dt}{\sqrt{13}}$$

(iii) $\frac{dP}{du} = \frac{\pi dth}{l}$