

1) a) Standard proof: $\psi = 2(1 - \Lambda - \phi \tan \alpha_1)$ GIVE IT!

b) i) $\Lambda = 0.5, \alpha_1 = 0$

$$\psi = 2(1 - 0.5) = 1$$

$$\Delta h_0 \text{ stage} = \psi U_m^2 = 1 \times 180^2 = 32.4 \text{ kJ kg}^{-1}$$

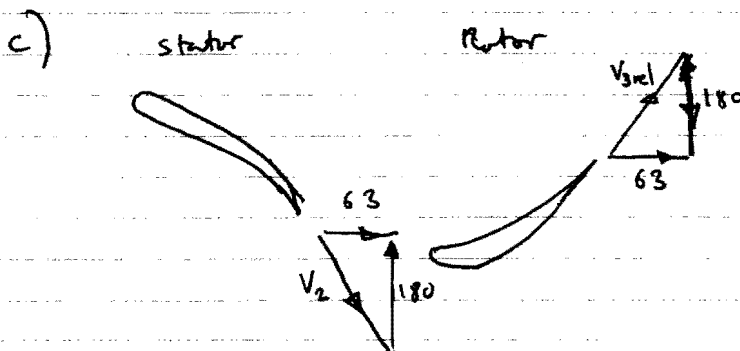
$$\therefore \text{Number of stages required } \frac{32.4}{3.50} = 10.8 \text{ so } \underline{\underline{11 \text{ stages}}}$$

ii) $\Lambda = 0.3, \alpha_1 = -22^\circ, \phi = 0.3$

$$\psi = 2(1 - 0.3 - 0.3 \tan -22^\circ) = 1.64$$

$$\Delta h_0 \text{ stage} = \psi U_m^2 = 1.64 \times 180^2 = 53.2 \text{ kJ kg}^{-1}$$

$$\therefore \text{Number of stages required } \frac{53.2}{3.50} = 6.6 \text{ so } \underline{\underline{7 \text{ stages}}}$$



So % reaction design

$$\text{Flow coefficient } \phi = \frac{V_{x2}}{U_m} \text{ so } V_{x2} = 180 \times 0.35 = 63 \text{ ms}^{-1}$$

$$\text{So } V_2 = V_{3rel} = \sqrt{63^2 + 180^2} = 190.7 \text{ ms}^{-1}$$

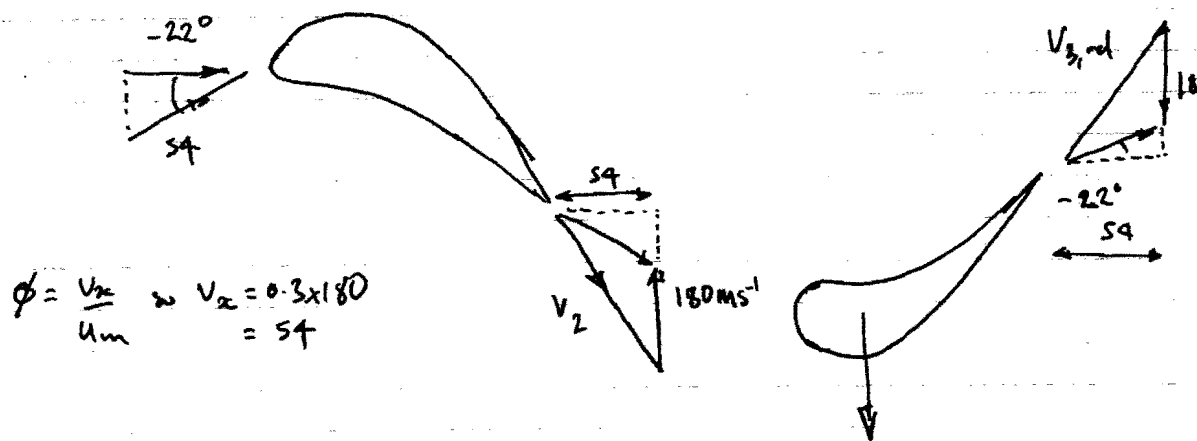
$$\therefore \eta_{tt} = \frac{1 - 0.05(2 \times 190.7^2)}{1 \times 180^2} = \underline{\underline{88.8\%}}$$

$$\eta_{ts} = \eta_{tt} - \text{exit K.E.}$$

$$= 81.8 - \frac{\frac{1}{2} 68^2}{180^2}$$

$$= \underline{\underline{82.65\%}}$$

low reaction design



$$\phi = \frac{V_{\alpha_2}}{u_m} \approx V_{\alpha_2} = 0.3 \times 180 = 54$$

$$\Delta h_0 = u \Delta V_{\alpha} \Rightarrow \Delta V_{\alpha} = 1.64 \times 180 = 295.2 \text{ ms}^{-1}$$

$$V_{\alpha_1} = V_{\alpha} \tan \alpha_1 = -21.82 \text{ ms}^{-1}$$

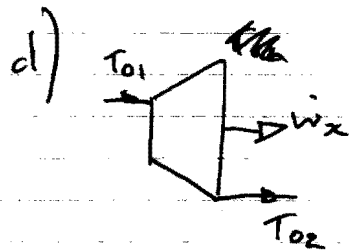
$$V_{\alpha_2} = \Delta V_{\alpha} + V_{\alpha_1} = 273.4 \text{ ms}^{-1} \quad V_2 = 278.7 \text{ ms}^{-1}$$

$$V_{3,rel} = \sqrt{\{ 54^2 + (180 + 21.82)^2 \}} = 208.9 \text{ ms}^{-1}$$

$$\eta_{tt} = 1 - 0.05 \left(\frac{278.7^2 + 208.9^2}{1.64 \times 180^2} \right) = \underline{\underline{88.6\%}}$$

$$\text{Stage exit K.E.} = \frac{1}{2} (54^2 + 21.82^2)$$

$$\eta_{ts} = \eta_{tt} - \frac{\frac{1}{2} (54^2 + 21.82^2)}{1.64 \times 180^2} = \underline{\underline{85.4\%}}$$



whole stage.

$$T_{02} = T_{01} - \frac{sh_0}{c_r}$$

$$= (540 + 273.15) - \frac{350}{1.8}$$

$$= 618.7 \text{ K}$$

$$\frac{P_{03}}{P_{01}} = \left(\frac{T_{02}}{T_{01}} \right)^{\frac{\gamma}{\gamma-1} \frac{1}{\eta_p}}$$

$$\eta_p = 88.8\% \quad \frac{\gamma}{\gamma-1} \frac{1}{\eta_p} = 4.881$$

$$\frac{P_{02}}{P_{01}} = 0.2639$$

$$\frac{T_{02is}}{T_{01}} = (0.2639)^{\frac{\gamma-1}{\gamma}} = 0.735$$

$$T_{02is} = 612.5 \text{ K}$$

$$\eta_{is} = \frac{T_{01} - T_{02}}{T_{01} - T_{02is}} = 90.2\%$$

Examiner's note:

A very popular and straightforward question, well-answered by most candidates. Almost all candidates were able to complete the proof and calculate the number of stages. Candidates began to struggle a little more with the velocity diagrams. When calculating the stage efficiencies, the most common mistake was to use the overall work output and an Euler work equation approach which often led to errors. The candidates who spotted that they had already calculated the stage-loading coefficient s for each design in the previous part of the question found it much easier.

$$2) \quad M_1 = 0.68 \quad \alpha_1 = 42^\circ \quad \alpha_2 = 18^\circ \quad P_{01} = 1 \times 10^5 \quad T_{01} = 300K$$

$$\frac{P_1}{P_{01}} = \left[1 + \frac{\gamma-1}{2} M_1^2 \right]^{-\frac{\gamma}{\gamma-1}} = 0.7338$$

$$\frac{P_{01} - P_{02}}{P_{01} - P_1} = 0.035 \quad \frac{P_{02}}{P_{01}} = -0.035 \left[1 - \frac{P_1}{P_{01}} \right] + 1 = \underline{\underline{0.9907}}$$

$$\gamma = 1.4, \quad c_p = 1005$$

$$\frac{\dot{m}}{A_1} \frac{\sqrt{c_p T_{01}}}{P_{01}} = 1.1549 \quad \Rightarrow \quad \frac{\dot{m}}{A_1} = \frac{1.1549 \times 10^5}{\sqrt{1005 \times 300}} = 210.24 \text{ kg s}^{-1} \text{ m}^{-2}$$

$$A_1 = A_{x_1} \cos 42 \quad \Rightarrow \quad \frac{\dot{m}}{A_{x_1}} = \frac{210.24}{\cos 42} = 156.24 \text{ kg s}^{-1} \text{ m}^{-2}$$

Continuity:

$$\frac{\dot{m} \sqrt{c_p T_{02}}}{A_2 P_{02}} = \frac{\dot{m} \sqrt{c_p T_{01}}}{A_1 P_{01}} \cdot \frac{P_{01}}{P_{02}} \cdot \frac{A_1}{A_2} \sqrt{\frac{T_{02}}{T_{01}}}$$

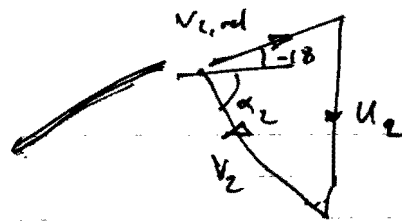
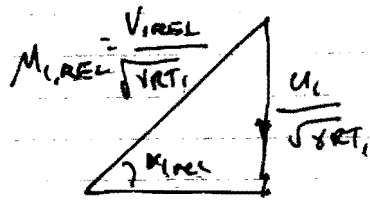
$$\Rightarrow \sqrt{\frac{T_{02}}{T_{01}}} = 1, \quad \frac{A_1}{A_2} = \frac{A_{x_1} \cos 42}{A_{x_1} \cos 18} = 0.7814$$

$$\Rightarrow \frac{\dot{m} \sqrt{c_p T_{02}}}{A_2 P_{02}} = 0.8937 \quad \Rightarrow \quad M_2 = 0.453$$

$$\frac{P_2}{P_1} = \frac{P_2}{P_{02}} \cdot \frac{P_{02}}{P_{01}} \cdot \frac{P_{01}}{P_1} \quad \frac{P_2}{P_{02}} = 0.867$$

$$\Rightarrow \frac{P_2}{P_1} = 1.1705 //$$

c)



$$\frac{V_1}{\sqrt{\gamma RT_1}} = M_1 = M_{1,rel} \cos \alpha_{1,rel}$$

$$V_2 = M_1 \sqrt{\gamma RT_1} \quad \gamma R = (\gamma - 1) C_p \quad C_p T_1 = C_p T_{01} - \frac{1}{2} V_1^2$$

$$V_2^2 = M_{1,rel}^2 \cos^2 \alpha_2 \left((\gamma - 1) C_p T_{01} - \frac{(\gamma - 1)}{2} V_1^2 \right)$$

$$V_1 = 171.2 \text{ m s}^{-1}$$

$$u_1 = V_1 \tan 42 = 154.13 \text{ m s}^{-1}$$

$$V_{1,rel} = 230.3 \text{ m s}^{-1}$$

$$\text{SFEE} \quad T_{01,rel} = T_{01} - \frac{u_1 V_{01}}{C_p} + \frac{1}{2} \frac{u_1^2}{C_p} = 300 + \frac{154.13^2}{2 \times 1005} = \underline{312 \text{ K}}$$

In relative frame $\frac{p_2}{p_1} = 1.1705 \quad M_{2,rel} = 0.455$

$$\text{Constant radius. } T_{02,rel} = T_{01,rel} = 312.0 \quad T_2 = \frac{T_{02,rel}}{1 + \frac{\gamma - 1}{2} M_{2,rel}^2} = 305.7$$

$$V_{2,rel} = \sqrt{\left\{ \text{radius} \cdot (T_{02,rel} - T_2) \right\}} = 112.5 \text{ m s}^{-1}$$

$$\alpha_2 = \tan^{-1} \left[\frac{V_{2,rel} \cos \alpha_{2,rel} + u_1}{V_{2,rel} \cos \alpha_{2,rel}} \right] = \underline{\underline{48^\circ}}$$

d) $V_3 = V_1 = 171.2 \text{ m/s } \alpha_1 = 0$

$$V_2 = \underline{V_2 \cos \alpha_2} = 144.0 \text{ m/s}^{-1}$$

$$\Lambda = \frac{\Delta h_{rotor}}{\Delta h_{stage}} \quad \Delta h_{rotor} = \frac{1}{2} (230.3^2 - 112.5^2)$$
$$= 2.02 \times 10^4$$

$$\Delta h_{stage} = \frac{1}{2} V_2 V_{02}$$
$$= 154.1 \times 144 \cos 48^\circ$$
$$= 1.366 \times 10^4$$

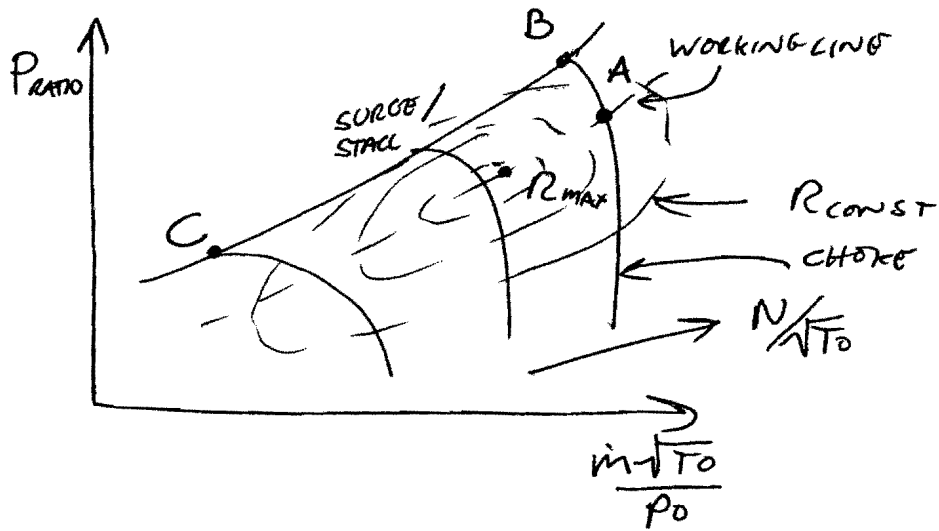
$$\Lambda = 0.68$$

Examiner's note:

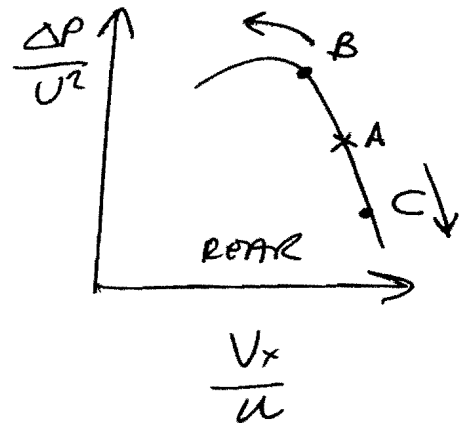
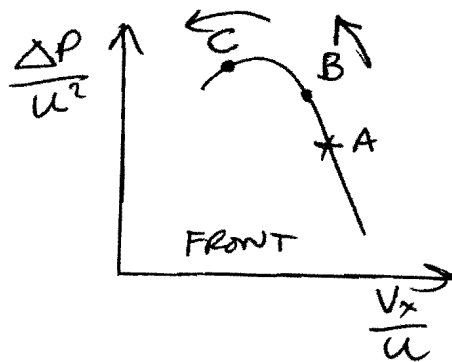
Another very popular question. Almost all candidates tackled the first part very well, calculating the correct exit Mach number from the tables in the Databook. However many were not able to manipulate the pressure ratios. Part c) was well answered. For this section the answers were given and the candidates had to show their methodology. However there were almost no successful attempts to the next part. In this case they had to find the absolute exit flow angle. This time the answer was not given - but the calculation itself was no complicated.

3 a

7



b



- A - DESIGN POINT
- B - HIGH SPEED CROSSING STALL LINE / SURGE LINE
INCREASED ΔP IN FRONT STAGES REDUCES V_x INTO NEXT STAGE SINCE P IS INCREASED RELATIVE TO DESIGN, ΔP OF NEXT STAGE RISES MORE. HENCE REAR STAGES $\frac{V_x}{U}$ MUCH LESS AND STALL
- C - LOWER SPEED CROSSING STALL LINE / SURGE LINE
LOWER U GIVES SMALLER RISE IN P THAN AT DESIGN \therefore 2ND STAGE LESS HEAVILY LOADED THAN FIRST. REAR STAGES CHOKE WHILE FRONT STALL.

3 (c)

THROTTLE IS CHOKED AS $P_{RATIO} > CHOKING\ VALUE$

$$\therefore \frac{\dot{m} \sqrt{\gamma P_0 T_0}}{A P_0} = \text{CONSTANT}$$

$$\text{AT } P_{RATIO} = 12 \quad \dot{m} = 80 \text{ kg/s}^{-1}$$

$$\text{FOR } \dot{m} = 50 \text{ kg/s}$$

$$\frac{\dot{m}_{(50)} \sqrt{\gamma P_{03(50)} T_{03(50)}}}{A P_{03(50)}} = \frac{\dot{m}_{(80)} \sqrt{\gamma P_{03(80)} T_{03(80)}}}{A P_{03(80)}}$$

$$\frac{P_{03(50)}}{P_{03(80)}} = \frac{50}{80} \sqrt{\frac{T_{03(50)}}{T_{03(80)}}} \quad (2)$$

$$\frac{T_{03(50)}}{T_{03(80)}} = \frac{P_{03(50)}^{\frac{\gamma-1}{\gamma}}}{P_{03(80)}^{\frac{\gamma-1}{\gamma}}} = \left[\frac{P_{03(50)}}{P_{03(80)}} \right]^{\frac{0.4}{1.4 \times 0.88}} \quad (1)$$

SUB (1) INTO (2)

$$\frac{P_{03(50)}}{P_{03(80)}} = \frac{50}{80} \left[\frac{P_{03(50)}}{P_{03(80)}} \right]^{0.3267 \times 0.5}$$

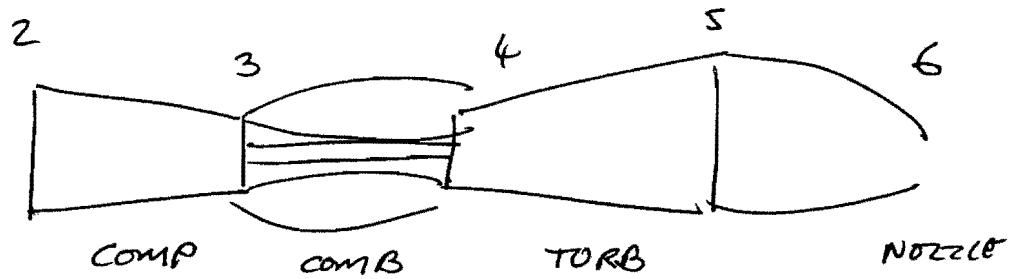
$$\left[\frac{P_{03(50)}}{P_{03(80)}} \right]^{[1 - 0.3267 \times 0.5]} = \frac{50}{80} \quad \frac{P_{03(50)}}{P_{03(80)}} = 0.576$$

$$\therefore P_{03(50)} = 0.5706 \times 12 = \underline{\underline{6.8472}}$$

PRESSURE RATIO OF COMPRESSOR AT $\dot{m} = 50$

IS 6.8472

(3) d



$$\left(\frac{\dot{m} \sqrt{c_p T_0}}{A P_0} \right)_4 = \left(\frac{\dot{m} \sqrt{c_p T_0}}{A P_0} \right)_6 = \text{CONST}$$

NB ASSUME NOZZLE OF TURBINE FLOW EXIT NOZZLE CHOKED

NB ASSUME NO PRESSURE LOSS OR HEAT TRANSFER IN EXIT NOZZLE [VALID]

$$\frac{T_{04}}{T_{05}} = \left(\frac{P_{04}}{P_{05}} \right)^{\frac{\gamma-1}{\gamma}}$$

NB ASSUME POLYTROPIC TURBINE FLOW [VALID AT HIGHER SPEEDS]

$$\Rightarrow \frac{T_{04}}{T_{05}} = \text{CONST} \quad \frac{P_{04}}{P_{05}} = \text{CONST}$$

$$T_{04} - T_{05} = T_{04} \left(1 - \frac{T_{05}}{T_{04}} \right) = \text{CONST} \times T_{04}$$

$$c_{pT} (T_{04} - T_{05}) = c_p (T_{03} - T_{02})$$

NB ASSUME c_p, c_{pT} CONSTANT THROUGH OPERATION

$$\therefore T_{03} - T_{02} = \text{CONST} \times T_{04}$$

$$\left(\frac{\dot{m} \sqrt{c_p T_0}}{A P_0} \right)_4 = \text{CONST}$$

NB ASSUME $\frac{P_{04}}{P_{03}} = \text{CONST}$

$$\therefore \frac{\dot{m} \sqrt{c_p (T_{03} - T_{02})}}{P_{03}} = \text{CONST}$$
