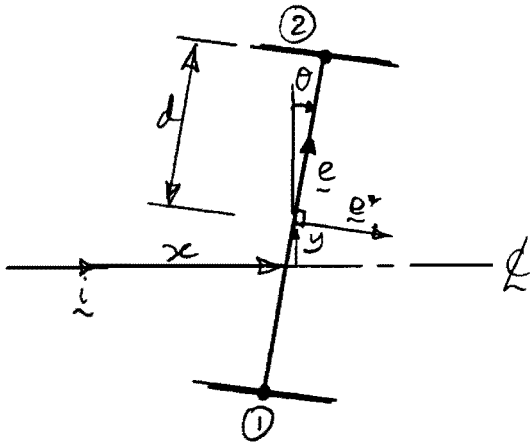


Q1



Position vector of wheel ②:

$$\underline{r}_2 = x \underline{i} + y \underline{j} + d \underline{e}$$

$$\dot{\underline{r}}_2 = \dot{x} \underline{i} + \dot{y} \underline{j} + d \dot{\theta} \underline{e}^* \quad \text{--- ①}$$

Now $\dot{x} = u$

$$\underline{i} = \sin \theta \underline{e} + \cos \theta \underline{e}^* \approx \theta \underline{e} + \underline{e}^*$$

$$\underline{j} = \cos \theta \underline{e} - \sin \theta \underline{e}^* \approx \underline{e} - \theta \underline{e}^*$$

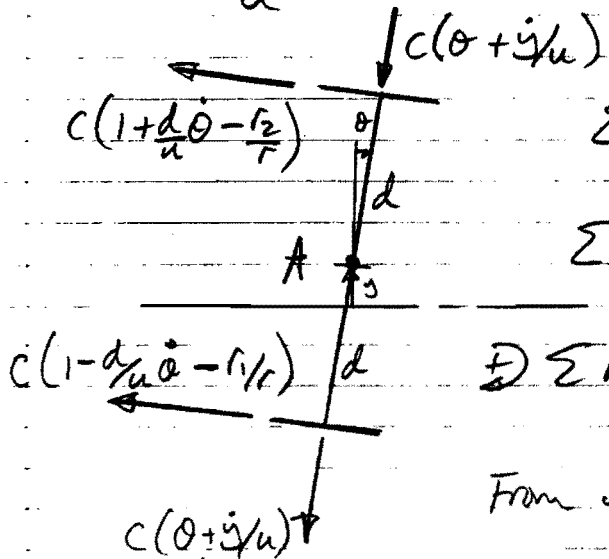
so ① becomes: $\dot{\underline{r}}_2 = (u + d \dot{\theta} - y \theta) \underline{e}^* + (y + u \theta) \underline{e}$ --- ②

neglect (small term)

If the wheelset rotates with ang. vel $\omega = u/r$, then the longitudinal creep velocity is $v_x = u + d \dot{\theta} - \frac{r_2 u}{r}$ --- ③

Assuming the lateral and long'l creep forces are

$$X = -\frac{C v_x}{u} \quad \& \quad Y = -\frac{C v_y}{u} \quad \text{gives} \quad \text{--- ④}$$



$$\Sigma \text{ lateral forces} = 2c(\theta + y/u) \quad \text{--- ⑤}$$

$$\Sigma \text{ longitudinal forces} = 0$$

$$\Sigma \text{ Moments at A} = dC \left[\left(\frac{r_2 - r_1}{r} \right) - \frac{2d \dot{\theta}}{u} \right]$$

From wheel geometry $r_2 - r_1 = 2\epsilon y$

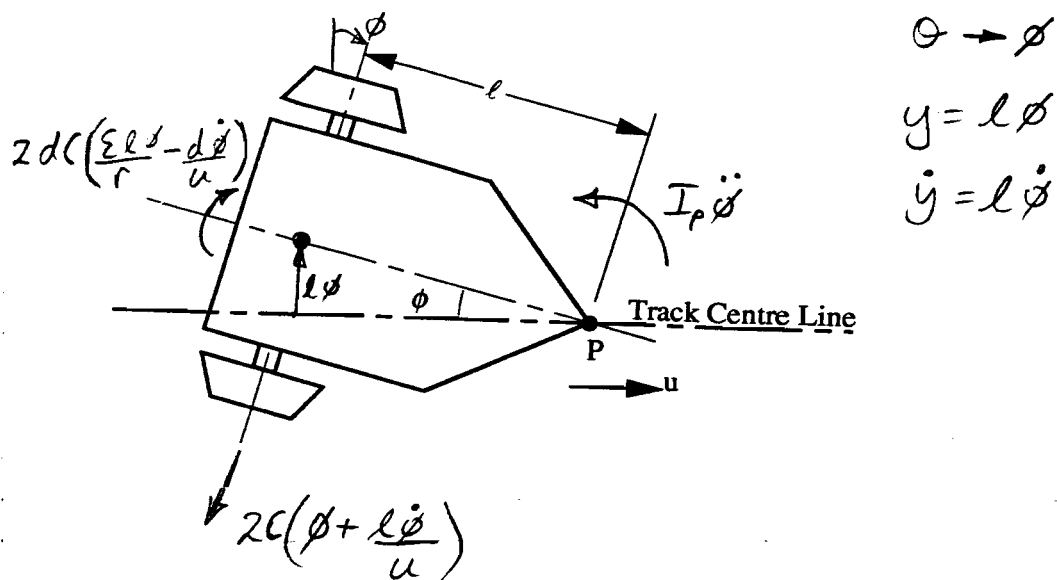
Assumptions

- $C_{11} = C_{22} = C$
- linear creep
- No spin creep
- Small y, θ (10%)

So $\Sigma M_A = 2dC \left[\frac{\epsilon y}{r} - \frac{d \dot{\theta}}{u} \right]$ --- ⑥

(40%)

Q1 (Cont)



$$\sum M_P: l \cdot 2c\left(\phi + \frac{l\dot{\phi}}{u}\right) + I_P \ddot{\phi} - 2dc\left(\frac{\epsilon l \dot{\phi}}{r} - \frac{d\dot{\phi}}{u}\right) = 0$$

$$\Rightarrow I_P \ddot{\phi} + \frac{2c}{u} (\epsilon l^2 + d^2) \dot{\phi} + 2cl\left(1 - \frac{d\epsilon}{r}\right) \phi = 0$$

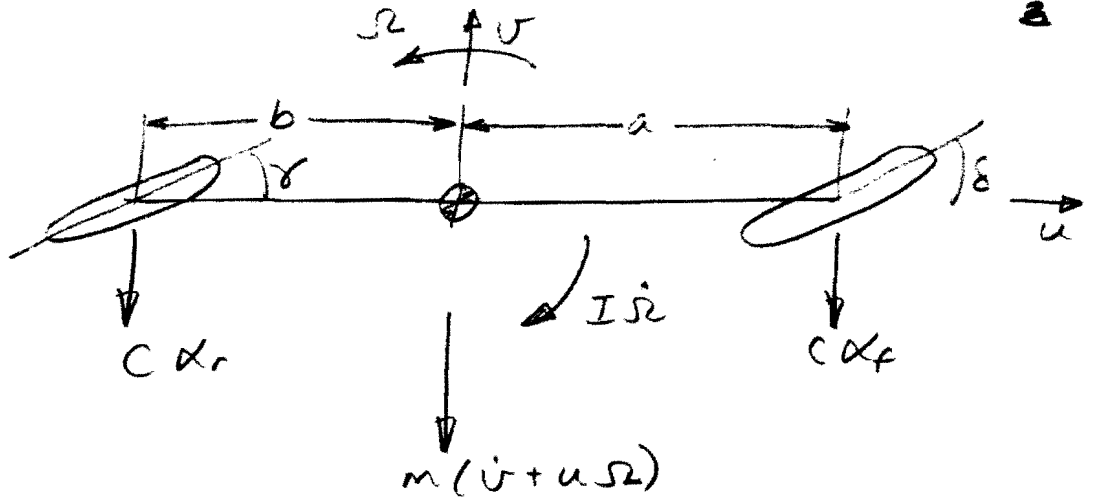
Motion is stable if all coefficients are positive

$$\text{i.e. } u > 0 \text{ and } 1 - \frac{d\epsilon}{r} > 0 \Rightarrow \epsilon < \frac{r}{d} \quad (30\%)$$

This implies an upper limit on the allowable concavity and no lower limit - the concavity can even be negative. This is quite different to the free wheelset which requires $\epsilon > 0$.

Above the upper limit, $\epsilon = r/d$, the yawing moment of the long's forces: $2dc\left(\frac{\epsilon l \dot{\phi}}{r}\right)$ exceeds $l \cdot 2c\phi$ and therefore destabilises the system. However r/d is a very large concavity - much larger than practical values. (20%)

Q2.



Slip angles: $\alpha_f = \frac{V + a\Omega}{u} - \delta$
 $\alpha_r = \frac{V - b\Omega}{u} - \gamma$ } ①

(a) Force and moment eqns:

$$\left. \begin{aligned} m(\dot{v} + u\dot{\Omega}) + C\alpha_f + C\alpha_r &= 0 \\ I\dot{\Omega} + aC\alpha_f - bC\alpha_r &= 0 \end{aligned} \right\} \textcircled{2}$$

① into ② gives

$$\left. \begin{aligned} m(\dot{v} + u\dot{\Omega}) + 2C\frac{v}{u} + (a-b)C\frac{\Omega}{u} &= C(\delta + \gamma) \\ I\dot{\Omega} + c(a-b)\frac{v}{u} + c(a^2 + b^2)\frac{\Omega}{u} &= C(a\delta - b\gamma) \end{aligned} \right\} \textcircled{3}$$

Steady state turn: $\dot{v} = \dot{\Omega} = 0$, $\gamma = K\delta$
 & $u = R\Omega$

③ becomes

$$\begin{bmatrix} mu^2 + (a-b)c & 2c \\ c(a^2 + b^2) & c(a-b) \end{bmatrix} \begin{Bmatrix} \Omega/u \\ v/u \end{Bmatrix} = c\delta \begin{Bmatrix} 1+K \\ a-bK \end{Bmatrix}$$

Q2 (cont)

$$\frac{\delta_0}{u} \begin{Bmatrix} \frac{\delta}{u} \\ v/u \end{Bmatrix} = \begin{bmatrix} c(a-b) & -2c \\ -c(a^2+b^2) & mu^2 + (a-b)c \end{bmatrix} c\delta \begin{Bmatrix} 1+k \\ a-bk \end{Bmatrix}$$

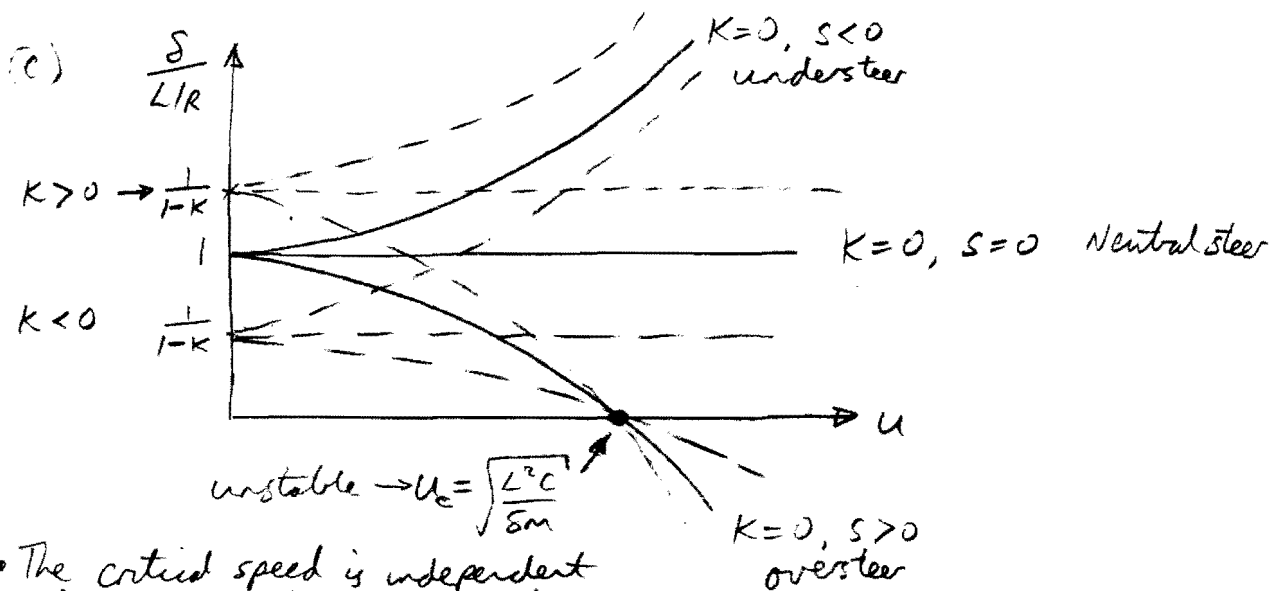
$$\frac{1}{[mu^2 + (a-b)c][c(a-b) - 2c^2(a^2+b^2)]}$$

$$\frac{\delta}{u\delta} = \frac{v_R}{\delta} = \frac{c^2(a-b)(1+k) - 2c^2(a-bk)}{mu^2(a-b) + c^2(a-b)^2 - 2c^2(a^2+b^2)}$$

$$= \frac{cL(k-1)}{mu^2(a-b) - cL^2}$$

where $L = a+b$
and $s = a-b$

$$\delta_0 \frac{\delta}{L/R} = \frac{1 - \frac{sM}{L^2C} u^2}{1-k}$$

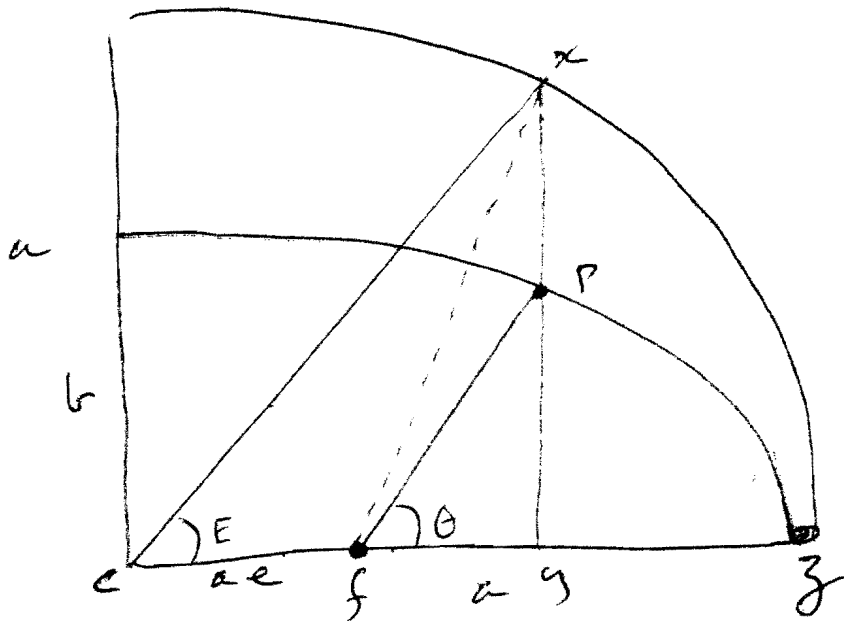


- The critical speed is independent of K so stability is not altered by this 4WS control strategy
- Understeer - more steer angle needed for higher speed turn $\frac{d\delta}{du} > 0$ ($s < 0$)
- Oversteer - $\frac{d\delta}{du} < 0$ ($s > 0$)
- If $K < 0$, then $d\delta/du$ is decreased for a given s making the vehicle more neutral steering, which is desirable
- If $K > 0$, $d\delta/du$ increases - so best design has $K < 0$



$$3a) \text{ Mean anomaly} = \frac{2\pi \times \text{area swept since pericapsis}}{\text{total area}}$$

mean Motion = rate of change of mean anomaly



$$M = \frac{2\pi \times \text{area } fPz}{\pi ab} \rightarrow fPz = \frac{abM}{2}$$

$$\therefore f \times z = fPz \times \frac{a}{b} = \frac{Ma^2}{2}$$

$$f \times z + f \times c = \frac{a^2 E}{2}$$

$$\frac{Ma^2}{2} + \frac{ae a \sin E}{2} = \frac{a^2 E}{2}$$

$$\rightarrow M = E - e \sin E$$

36) i) Signal time < reference time, so weekly roll-over has occurred
 $t = 3001 + 604800 - 560,670 = 47131 \text{ s.}$

ii) Mean motion = $\sqrt{\frac{\mu}{a^3}} = \sqrt{\frac{398603}{26562^3}}$
 $n = 145.84 \times 10^{-6} \text{ rad/s}$

So mean anomaly
 $= 0.1703 + 47131 \times 145.84 \times 10^{-6}$
 $= 7.0439$

$> 2\pi$, so subtract 2π to give

$M = 0.7607 \text{ radians}$

iii) Let $E = 0.7754 \text{ radians}$

$M = E - e \sin E$
 $= 0.7754 - 0.021 \times 0.700$
 $= \underline{0.7607}$

$\therefore E = 0.7754$

Now $\cos \theta = \frac{\cos E - e}{1 - e \cos E} = \frac{0.6935}{0.9850}$

$= 0.704$
 $\rightarrow \underline{\theta = 45.25^\circ}$ or 0.7898 radians

$$3b) iv) r = \frac{a(1-e^2)}{1+e \cos \theta} = \frac{26550}{1.01478}$$

$$= \underline{26,163.5 \text{ km}}$$

$$\begin{aligned} \text{Argument of latitude} &= 0.27574 + 0.7898 \\ &= 1.0472 \text{ radians} \\ &\quad (60.0^\circ) \end{aligned}$$

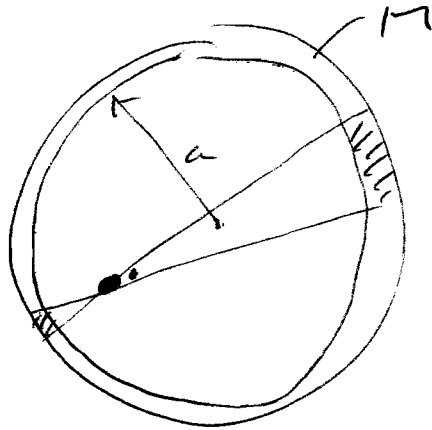
$$\begin{aligned} \text{So distance from equatorial plane} \\ \text{is } 26163.5 \text{ in } 60 \text{ in SS} \\ &= \underline{18,561 \text{ km}} \end{aligned}$$

$$\begin{aligned} v) \text{ Right ascension has changed by} \\ 7.83 \times 10^{-9} \times 47131 = 0.0004 \text{ radians} \\ \text{So correct R.A. is } \underline{0.2879} \end{aligned}$$

$$\begin{aligned} \text{Distance of satellite from Earth's} \\ \text{polar axis is } \sqrt{26163^2 - 18561^2} \\ = 18439 \text{ km} \end{aligned}$$

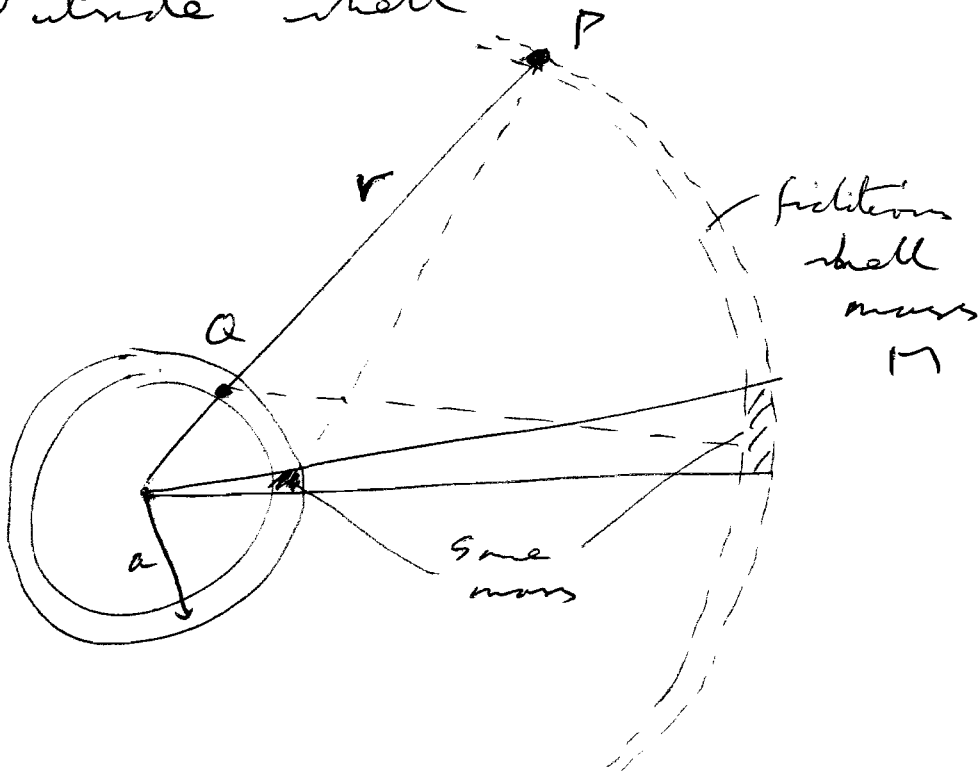
$$\therefore \text{ error is } 18439 \times 0.0004 = \underline{7.4 \text{ km}}$$

4a) Inside shell



Effects cancel, so potential is constant
 \therefore calculate at centre = $\frac{M}{a}$

Outside shell



Potential at P from real piece of shell is same as potential at Q from fictitious piece

\therefore Potential from whole shell = $\frac{M}{r}$

i.e. if M was at origin.

46) i) If $r > a$, potential from core is

$$U(r) = \frac{4}{3} \pi a^3 \rho_0 \frac{G}{r} = 4\pi \rho_0 G a^2 \left(\frac{a}{3r} \right)$$

For area between a & r , potential is

~~$$U(r) = \int_a^r 4\pi x^2 \rho_0 \left(\frac{a}{x} \right)^4 \frac{G}{x} dx$$~~

$$U(r) = \int_{x=a}^r 4\pi x^2 \rho_0 \left(\frac{a}{x} \right)^4 \frac{G}{x} dx$$

$$= \frac{4\pi \rho_0 a^4 G}{r} \left[-\frac{1}{x} \right]_{x=a}^{x=r}$$

$$= 4\pi \rho_0 G a^2 \left(\frac{a}{r} - \frac{a^2}{r^2} \right)$$

For area outside r , potential is

$$U(r) = \int_{x=r}^{\infty} 4\pi x^2 \rho_0 \left(\frac{a}{x} \right)^4 \frac{G}{x} dx$$

$$= 4\pi \rho_0 G a^4 \left[-\frac{1}{2x^2} \right]_{x=r}^{x=\infty}$$

$$= 4\pi \rho_0 G a^2 \left(\frac{a^2}{2r^2} \right)$$

Adding gives $U(r) = 4\pi \rho_0 G a^2 \left(\frac{4a}{3r} - \frac{a^2}{2r^2} \right)$

46) ii) Central force = $f(u) = \nabla u = \frac{\partial u}{\partial r} \underline{e}_r$

\therefore force \rightarrow centre = $-4\pi\rho_0\gamma a^2 \left(\frac{a^2}{r^3} - \frac{4a}{3r^2} \right)$
 $= 4\pi\rho_0\gamma a^2 \left(a^2 u^3 - \frac{4au^2}{3} \right)$

From formula in D.B.,

$$\frac{\partial^2 u}{\partial \theta^2} + u = - \frac{4\pi\rho_0\gamma a^2}{h^2} \left(a^2 u - \frac{4a}{3} \right)$$

$$\frac{\partial^2 u}{\partial \theta^2} + u \left(1 + \frac{4\pi\rho_0\gamma a^4}{h^2} \right) = \frac{16\pi\rho_0\gamma a^3}{3h^2}$$

For which the general solution is

$$u = \frac{1}{r} = A \cos \left(\sqrt{1 + \frac{4\pi\rho_0\gamma a^4}{h^2}} \theta + \theta_0 \right) + \frac{16\pi\rho_0\gamma a^3}{3h^2 + 12\pi\rho_0\gamma a^4}$$