EGT2
ENGINEERING TRIPOS PART IIA

Friday 1 May 201514.00 to 15.30

Module 3D7

## FINITE ELEMENT METHODS

Answer not more than three questions.
All questions carry the same number of marks.
The approximate percentage of marks allocated to each part of a question is indicated in the right margin.

Write your candidate number not your name on the cover sheet.

## STATIONERY REQUIREMENTS

Single-sided script paper

## SPECIAL REQUIREMENTS TO BE SUPPLIED FOR THIS EXAM

CUED approved calculator allowed
Attachment: 3D7 Data Sheet (3 pages).
Engineering Data Book

## 10 minutes reading time is allowed for this paper.

You may not start to read the questions printed on the subsequent pages of this question paper until instructed to do so.

## Version FC/3

1 (a) Figure 1(a) shows the finite element model for a plate with a hole under axial tension. In attempting to solve this problem with a commercial finite element package the program stops with the error message 'the system matrix is singular'. Suggest the cause of this error message and how the finite element model can be fixed.
(b) For computing the shape functions of a three-noded triangular element we consider the function $T(x, y)=\alpha_{0}+\alpha_{1} x+\alpha_{2} y$ with the parameters $\alpha_{i}$ to be determined. Explain why you should not alternatively use the function $T(x, y)=\alpha_{0}+\alpha_{1} x^{2}+\alpha_{2} y^{2}$.
(c) Consider the four-noded element shown in Fig. 1(b). The location of node 3 is $(a, b)$.
(i) Show that the isoparametric map is given by

$$
\begin{aligned}
& x=\frac{1}{4}(1+\xi)[2(1-\eta)+a(1+\eta)] \\
& y=\frac{1}{4}(1+\eta)[2(1-\xi)+b(1+\xi)]
\end{aligned}
$$

(ii) Determine the conditions on the location of node 3 by considering the Jacobian at the point with the coordinates $\xi=1$ and $\eta=1$ in the parent element.
(d) Determine the shape functions $N_{1}$ and $N_{5}$ of the five-node element shown in Fig. 1(c).

Version FC/3


Fig. 1

## Version FC/3

2 On the domain $\Omega$ with boundary $\Gamma$, the following modified two-dimensional heat equation is given

$$
k\left(\frac{\partial^{2} T}{\partial x^{2}}+\frac{\partial^{2} T}{\partial y^{2}}\right)+c \frac{\partial T}{\partial x}+s=0
$$

where $T$ is the temperature, $k$ and $c$ are constants and $s$ is a prescribed domain heat source. The temperature boundary condition over the entire boundary is $T=0$.
(a) Derive the weak form of this modified heat equation.
(b) The following integral is to be discretised and evaluated

$$
c \int_{\Omega} w \frac{\partial T}{\partial x} d \Omega
$$

where $c$ is a constant and $w$ is the test function.
(i) For the triangular three-node element shown in Fig. 2(a), compute the $(1,2)$ and $(2,1)$ components of the resulting matrix.
(ii) For the four-node quadrilateral element shown in Fig. 2(b) what is the minimum number of Gauss quadrature points needed in the $\xi$ and $\eta$ directions to perform exact integration. Justify your answer.


Fig. 2

## Version FC/3

3 A particular problem on the domain $(0, L)$ involves finding the function $u$ such that

$$
\int_{0}^{L} \alpha \frac{d^{2} v}{d x^{2}} \frac{d^{2} u}{d x^{2}} d x+\int_{0}^{L} \beta v u d x+\left.\gamma v(u-g)\right|_{x=L}=0
$$

for all admissible test functions $v$, where $\alpha, \beta, \gamma$ and $g$ are prescribed constants.
(a) Identify the strong form of the differential equation that is satisfied by a solution of this problem and identify any boundary conditions that are implied.
(b) The appropriate shape functions for this problem are Hermitian shape functions. Explain briefly why Hermitian shape functions are required for this problem.
(c) Sketch the Hermitian shape functions for a two-node element.
(d) Formulate a system of equations for computing the Hermitian shape functions for an element that runs from $x_{1}$ to $x_{2}\left(x_{1}<x_{2}\right)$.
(e) If this problem is solved using two-node Hermitian elements and Gauss quadrature, how many quadrature points would you recommend using? Provide a brief justification for your answer.

## Version FC/3

4 A particular time dependent problem involves the semi-discrete finite element system

$$
M \ddot{a}+\alpha K \dot{a}+K a=f
$$

where $\boldsymbol{M}$ is the mass matrix, $\boldsymbol{K}$ is the stiffness matrix, $\alpha \geq 0$ is a constant and $\boldsymbol{f}$ is prescribed.
(a) Using the Verlet time stepping scheme

$$
\begin{aligned}
y_{n+1} & =y_{n}+\Delta t \dot{y}_{n}+\frac{\Delta t^{2}}{2} \ddot{y}_{n} \\
\dot{y}_{n+1} & =\dot{y}_{n}+\frac{\Delta t}{2}\left(\ddot{y}_{n+1}+\ddot{y}_{n}\right)
\end{aligned}
$$

formulate a fully discrete version of the finite element problem.
(b) Would you recommend this time stepping scheme for $\alpha>0$ ? Briefly justify your answer.
(c) Why is a lumped mass matrix appropriate for this problem, and how can a lumped mass matrix be constructed?
(d) Consider a scenario in which this problem is solved using an implicit time integration scheme and the linear solver has complexity $O\left(n^{2}\right)$, where $n$ is the number of degrees of freedom. Estimate the change in the solution time per time step for 2D and 3D problems if the mesh is refined such that the number of elements in each direction is doubled.

## END OF PAPER

## 3D7 DATA SHEET

## Element relationships

Elasticity

Displacement
Strain
Stress (2D/3D)
Element stiffness matrix

$$
\boldsymbol{k}_{e}=\int_{V_{e}} \boldsymbol{B}^{T} \boldsymbol{D} \boldsymbol{B} d V
$$

Element force vector

$$
\begin{aligned}
& \boldsymbol{u}=\boldsymbol{N} \boldsymbol{a}_{e} \\
& \boldsymbol{\varepsilon}=\boldsymbol{B} \boldsymbol{a}_{e} \\
& \boldsymbol{\sigma}=\boldsymbol{D} \boldsymbol{\varepsilon}
\end{aligned}
$$

(body force only)

Heat conduction
Temperature
Temperature gradient

$$
\begin{aligned}
& T=\boldsymbol{N} \boldsymbol{a}_{e} \\
& \nabla T=\boldsymbol{B} \boldsymbol{a}_{e} \\
& \boldsymbol{q}=-\boldsymbol{D} \nabla T \\
& \boldsymbol{k}_{e}=\int_{V_{e}} \boldsymbol{B}^{T} \boldsymbol{D} \boldsymbol{B} d V
\end{aligned}
$$

Beam bending
Displacement

$$
\begin{aligned}
& v=\boldsymbol{N} \boldsymbol{a}_{e} \\
& \kappa=\boldsymbol{B} \boldsymbol{a}_{e} \\
& \boldsymbol{k}_{e}=\int_{V_{e}} \boldsymbol{B}^{T} E I \boldsymbol{B} d V
\end{aligned}
$$

Curvature
Element stiffness matrix

## Elasticity matrices

2D plane strain

$$
\boldsymbol{D}=\frac{E}{(1+v)(1-2 v)}\left[\begin{array}{ccc}
1-v & v & 0 \\
v & 1-v & 0 \\
0 & 0 & \frac{1-2 v}{2}
\end{array}\right]
$$

2D plane stress

$$
\boldsymbol{D}=\frac{E}{\left(1-v^{2}\right)}\left[\begin{array}{ccc}
1 & v & 0 \\
v & 1 & 0 \\
0 & 0 & \frac{1-v}{2}
\end{array}\right]
$$

Heat conductivity matrix (2D, isotropic)

$$
\boldsymbol{D}=\left[\begin{array}{ll}
k & 0 \\
0 & k
\end{array}\right]
$$

## Shape functions


$N_{1}=\left(\left(x_{2} y_{3}-x_{3} y_{2}\right)+\left(y_{2}-y_{3}\right) x+\left(x_{3}-x_{2}\right) y\right) / 2 A$
$N_{2}=\left(\left(x_{3} y_{1}-x_{1} y_{3}\right)+\left(y_{3}-y_{1}\right) x+\left(x_{1}-x_{3}\right) y\right) / 2 A$
$N_{3}=\left(\left(x_{1} y_{2}-x_{2} y_{1}\right)+\left(y_{1}-y_{2}\right) x+\left(x_{2}-x_{1}\right) y\right) / 2 A$
$A=$ area of triangle


$$
\begin{aligned}
& N_{1}=1-\xi-\eta \\
& N_{2}=\xi \\
& N_{3}=\eta
\end{aligned}
$$



$$
\begin{aligned}
& N_{1}=2(1-\xi-\eta)^{2}-(1-\xi-\eta) \\
& N_{2}=2 \xi^{2}-\xi \\
& N_{3}=2 \eta^{2}-\eta \\
& N_{4}=4 \xi(1-\xi-\eta) \\
& N_{5}=4 \eta \xi \\
& N_{6}=4 \eta(1-\xi-\eta)
\end{aligned}
$$



$$
\begin{aligned}
& N_{1}=(1-\xi)(1-\eta) / 4 \\
& N_{2}=(1+\xi)(1-\eta) / 4 \\
& N_{3}=(1+\xi)(1+\eta) / 4 \\
& N_{4}=(1-\xi)(1+\eta) / 4
\end{aligned}
$$

$$
\xrightarrow{\longrightarrow} \quad \begin{aligned}
& x_{2} \\
& \text { Hermitian element }
\end{aligned} \begin{aligned}
& N_{1}=\frac{-\left(x-x_{2}\right)^{2}\left(-l+2\left(x_{1}-x\right)\right)}{l^{3}} \\
& N_{2}=\frac{\left(x-x_{1}\right)\left(x-x_{2}\right)^{2}}{l^{2}} \\
& M_{2}=\frac{\left(x-x_{1}\right)^{2}\left(l+2\left(x_{2}-x\right)\right)}{l^{3}} \\
& l^{2}
\end{aligned}
$$

## Gauss integration in one dimension on the domain $(-1,1)$

Using $n$ Gauss integration points, a polynomial of degree $2 n-1$ is integrated exactly.

| number of points $n$ | location $\xi_{i}$ | weight $w_{i}$ |
| :--- | ---: | ---: |
| 1 | 0 | 2 |
| 2 | $-\frac{1}{\sqrt{3}}$ | 1 |
|  | $\frac{1}{\sqrt{3}}$ | 1 |
| 3 | $-\sqrt{\frac{3}{5}}$ | $\frac{5}{9}$ |
|  | 0 | $\frac{8}{9}$ |
|  | $\sqrt{\frac{3}{5}}$ | $\frac{5}{9}$ |

## Numerical Answers

1. Condition on location of node 3: $a+b>2$
2. $\quad K_{12}=1 / 3, K_{21}=-1 / 3$
3. Change in solution time: $16(2 D), 64(3 D)$
