EGT3
ENGINEERING TRIPOS PART IIB

Friday 27 April $2018 \quad 9.30$ to 11.10

## Module 4A10

## FLOW INSTABILITY

Answer not more than three questions.
All questions carry the same number of marks.
The approximate percentage of marks allocated to each part of a question is indicated in the right margin.

Write your candidate number not your name on the cover sheet.

## STATIONERY REQUIREMENTS

Single-sided script paper

## SPECIAL REQUIREMENTS TO BE SUPPLIED FOR THIS EXAM

CUED approved calculator allowed
Engineering Data Book
Attachment: 4A10 Flow Instability data sheet (2 pages).

## 10 minutes reading time is allowed for this paper at the start of the exam. <br> You may not start to read the questions printed on the subsequent pages of this question paper until instructed to do so.

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1 The centrifugal instability is analysed theoretically by letting a liquid of viscosity $v$ occupy the gap between two infinitely long concentric circular cylinders. The inner cylinder has radius $r_{1}$ and angular velocity $\Omega_{1}$, and the outer cylinder has radius $r_{2}>r_{1}$ and angular velocity $\Omega_{2}$. The width $d$ of the gap is narrow so that $d=r_{2}-r_{1} \ll r_{1}$. The cylinders rotate in the same sense ( $\Omega_{1}>0, \Omega_{2}>0$ ) and at similar velocities ( $\Omega_{1} \approx \Omega_{2}$ ). The vertical axis of each cylinder is aligned with the coordinate $z$. The radial coordinate is denoted $r$. The dimensionless coordinate $x$, defined as $x=\left(r-r_{1}\right) / d$, spans the gap such that the liquid is confined by cylinder walls at $x=0$ and $x=1$.

To investigate the stability of the steady, purely rotary, base flow it is subjected to infinitesimal axisymmetric disturbances. For the velocity perturbation in the radial direction $u_{r}^{\prime}$, normal mode solutions are sought of the form

$$
u_{r}^{\prime}=\hat{u}_{r}(r) \cos (n z) e^{s t}
$$

where $n$ denotes the vertical wavenumber and $s$ is the growth rate of the disturbance. You may assume that the radial velocity $\hat{u}_{r}(r)$ is governed by the following differential equation

$$
\begin{equation*}
\left[\left(\frac{d^{2}}{d x^{2}}-(n d)^{2}\right)-\frac{s d^{2}}{v}\right]^{2}\left(\frac{d^{2}}{d x^{2}}-(n d)^{2}\right) \hat{u}_{r}=-(n d)^{2} T \hat{u}_{r} \tag{1}
\end{equation*}
$$

where the Taylor number is

$$
T=\frac{2\left(\Omega_{1} r_{1}^{2}-\Omega_{2} r_{2}^{2}\right) d^{3}}{v^{2} r_{1}}\left(\frac{\Omega_{1}+\Omega_{2}}{2}\right)
$$

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(a) Using $N$ to denote the radial wavenumber of the marginal state, solve Equation (1) subject to the boundary conditions

$$
\hat{u}_{r}=\frac{d^{2} \hat{u}_{r}}{d x^{2}}=\frac{d^{4} \hat{u}_{r}}{d x^{4}} \quad \text { at } \quad x=0 \text { and } x=1
$$

to show that the condition for marginal stability may be expressed as

$$
T=\frac{1}{(n d)^{2}}\left[N^{2} \pi^{2}+(n d)^{2}\right]^{3}
$$

(b) Derive the minimum value of the Taylor number that gives rise to marginal stability under the boundary conditions given in part (a) and comment on the implications for the vertical structure of the flow.
(c) Indicate on a clearly annotated sketch of Taylor number against dimensionless vertical wavenumber the regions of parameter space that correspond to stable and unstable flow.

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2 Consider the two-dimensional, inviscid and incompressible flow of two homogeneous layers. The upper layer $(z>0)$ has a density $\rho_{1}$ and flows with a uniform horizontal velocity $U_{1}$ in the $x$-direction. The lower layer $(z<0)$ has a density $\rho_{2}>\rho_{1}$ and flows with a uniform horizontal velocity $U_{2}$. In the base state, a horizontal interface at $z=0$ separates the layers and pressure $p$ is

$$
p= \begin{cases}p_{0}-\rho_{1} g z & z>0 \\ p_{0}-\rho_{2} g z & z<0\end{cases}
$$

where $p_{0}=p(z=0)$ and $g$ is the acceleration due to gravity.
(a) By considering small amplitude disturbances to the interface in time $t$ and space $x$ of the form

$$
\eta(x, t)=\hat{\eta} e^{i k x+s t}
$$

for horizontal wavenumber $k$ and growth rate $s$, investigate the temporal stability of the base flow to show that the dispersion relationship is

$$
\rho_{1}\left[k g-\left(s+U_{1} i k\right)^{2}\right]=\rho_{2}\left[k g+\left(s+U_{2} i k\right)^{2}\right]
$$

(b) Discuss the implications for the stability of this two-layer system to small amplitude disturbances as the layer velocities and layer densities are varied.

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3 A horizontal strut on a structure in a tidal river has square cross-section with side length $d=0.24 \mathrm{~m}$ and mass per unit length $m_{a}=70 \mathrm{~kg} \mathrm{~m}^{-1}$. The natural frequency is $\omega_{a}=315 \mathrm{rad} \mathrm{s}^{-1}$ when not submerged, and the damping factor $\zeta=0.01$ when submerged. The damping factor is defined such that the equation of motion for the displacement $y$ of an unforced mass-spring-damper system is $m \ddot{y}+2 m \zeta \omega \dot{y}+k y=0$, where $m$ is the notional mass, $\omega$ is the frequency and $k$ is the stiffness.

The following information may be useful in this question: the density of seawater is $1025 \mathrm{~kg} \mathrm{~m}^{-3}$; the added mass per unit length for a square cross-section is $1.51 \rho \pi d^{2}$ where $\rho$ denotes density; and for small angles of attack, $\alpha$ (in radians), the vertical force coefficient $c_{y}$ for this shape is $2 \alpha$.
(a) The tide comes in and water at speed $U$ completely submerges the strut. Calculate the speed at which the strut will start oscillating. Explain physically what causes this oscillation.
(b) To prevent these oscillations, an engineer proposes to attach a mass to the strut using a spring and a damper. Explain physically why this might reduce the oscillation amplitude. If the spring stiffness is to be $2 \times 10^{4} \mathrm{Nm}^{-1}$, what mass should be used? What should be taken into account if this mass is also submerged?
(c) Propose better ways to damp or prevent the oscillations of the strut when it is submerged.

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4 Figure 1 is a snapshot of a vertical round helium jet discharging into still air. Figure 2 is the power spectral density of the signal from a hot wire anemometer placed one jet radius downstream of the exit plane along the centreline.
(a) By interpreting Figs 1 and 2, describe the motion of the helium jet.
(b) Using the concepts of absolute and convective instability, explain how this motion arises.
(c) A local stability analysis is performed on the flow at the streamwise location of the hot wire anemometer. It assumes that perturbations have the form $\mathrm{e}^{i(k x-\omega t)}$, where $k$ is the complex wavenumber and $\omega$ is the complex angular frequency. The coordinate $x$ is measured along the centreline of the jet with origin at the exit plane and $t$ is time. Give the physical meaning of the real and imaginary components of $k$ and $\omega$. Sketch contours of $\omega_{i}$ in the complex $k$-plane, where $\omega_{i}$ is the complex part of $\omega$; indicate the contour or contours that have $\omega_{i}=0$; and highlight any other relevant features.
(d) Explain why a spatial stability analysis of the flow at this streamwise location has no physical meaning.

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Fig. 1


Fig. 2

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