

EGT3  
ENGINEERING TRIPOS PART IIB

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Monday 20 April 2015 2 to 3.30

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**Module 4A13**

**COMBUSTION AND IC ENGINES**

*Answer not more than **three** questions.*

*All questions carry the same number of marks.*

*The **approximate** percentage of marks allocated to each part of a question is indicated in the right margin.*

*Write your candidate number **not** your name on the cover sheet.*

**STATIONERY REQUIREMENTS**

Single-sided script paper

**SPECIAL REQUIREMENTS TO BE SUPPLIED FOR THIS EXAM**

CUED approved calculator allowed

Engineering Data Book

Attachment: None

**10 minutes reading time is allowed for this paper.**

**You may not start to read the questions printed on the subsequent pages of this question paper until instructed to do so.**

1 (a) Describe what is meant by the terms *laminar burning velocity*, *flammability limits* and *quenching distance* for a fuel-air mixture. What are the physical and/or chemical origins of the flammability limits? [30%]

(b) A premixed flame with laminar burning velocity  $S_L$  is stabilised on the rim of a vertical circular tube with radius  $R$ . Write down an expression for the angle  $\theta(r)$  between the flame and the axis of the tube when the exit velocity  $V(r)$

(i) is uniform; [5%]

(ii) varies as a parabola given by

$$V(r) = V_0 \left( 1 - \frac{r^2}{R^2} \right)$$

where  $V_0$  is the centreline velocity. Sketch the flame shape by assuming that  $V_0 > S_L$  and ignoring the region close to the tube wall where  $V < S_L$ . [15%]

(c) The products of methane-air combustion at an equivalence ratio of 2.0 are in chemical equilibrium at 10 bar and 1600 K. Calculate their volumetric composition, assuming that the only species present are  $\text{CO}_2$ ,  $\text{CO}$ ,  $\text{H}_2\text{O}$ ,  $\text{H}_2$  and  $\text{N}_2$ . [50%]

2 (a) Describe briefly flame *blow-off*, using a carefully drawn graph for the physics of a well-stirred reactor. [20%]

(b) An approximate theory suggests that flame blow-off occurs in a combustor when

$$\tau_{\text{chem}} > C \tau_{\text{res}}$$

where  $C$  is a constant. The chemical time scale of a flammable mixture at an initial temperature  $T_0$  is defined as  $\tau_{\text{chem}} = \lambda / (\rho_0 c_p S_L^2)$  and the residence time is  $\tau_{\text{res}} = L/U$  with  $U$  as the bulk-mean velocity of the mixture entering the combustor of length  $L$ . The isobaric specific heat capacity of the mixture is  $c_p$  and is taken to be constant. The thermal conductivity  $\lambda$  increases with temperature as  $\lambda/\lambda_{\text{ref}} = (T_0/T_{\text{ref}})^{1/2}$  and  $\rho_0$  is the density of the mixture entering the combustor. The laminar burning velocity  $S_L$  increases with reactant temperature as  $S_L/S_{L,\text{ref}} = (T_0/T_{\text{ref}})^2$ .

Find the percentage increase in mass flow rate at blow-off when  $T_0$  is increased from 300 to 600 K and the combustor length is doubled, with all other parameters kept constant. [45%]

(c) Briefly describe the various mechanisms of nitric oxide generation from combustion. Discuss strategies used to mitigate nitric oxide emission from common combustion environments. [35%]

3 For a naturally aspirated gasoline engine

- (a) Draw a typical maximum torque characteristic on a *bme<sub>p</sub>* versus *engine speed* plot and justify what you have drawn. [30%]
- (b) What limits the maximum torque? [10%]
- (c) Add to this plot a typical top-gear road load characteristic with justifications. [10%]
- (d) Describe how and why the *sfc* varies within the operating envelope. [20%]
- (e) In the context of this plot, describe five approaches to reduce *sfc* which are being implemented. [10%]
- (f) For all of the above, describe briefly how your commentary would vary in the case of a naturally aspirated *diesel* engine. [20%]

- 4 (a) Prove that the expression

$$W_{12} = \frac{p_1 V_1}{\gamma - 1} \left[ 1 - \left( \frac{p_2}{p_1} \right)^{\frac{\gamma - 1}{\gamma}} \right]$$

is the work done during a reversible adiabatic (isentropic) compression of a perfect gas between states 1 and 2. The ratio of specific heat capacities is  $\gamma$  and the symbols  $p$  and  $V$  denote pressure and volume respectively. [10%]

- (b) Figure 1 below shows an idealised, throttled, four-stroke engine cycle. Assuming the working fluid is a perfect gas, find expressions for the work done during the compression and expansion strokes in terms of  $p_e$ ,  $p_i$ ,  $p_3$ ,  $V_m$  and  $V_c$ . [10%]

- (c) If the compression ratio is 10, the manifold inlet temperature is 288 K,  $\gamma = 1.4$ , the temperature rise on combustion is 1300 K, and the exhaust and inlet manifold pressures are 1.05 bar and 0.5 bar respectively, determine  $p_3$ , and hence the gross *imep*. [40%]

- (d) Determine the pumping work, and hence the *pmep* and the net *imep*. [30%]

- (e) Sketch the  $p$ - $V$  diagram of Fig. 1 in your script, and add to this a sketch of a cycle operating between the same minimum and maximum volumes, which would produce the same net work, based on late inlet valve closing. You may assume that the unthrottled inlet pressure is equal to  $p_e$ . No calculations are required, but justify your sketch with comments. [10%]

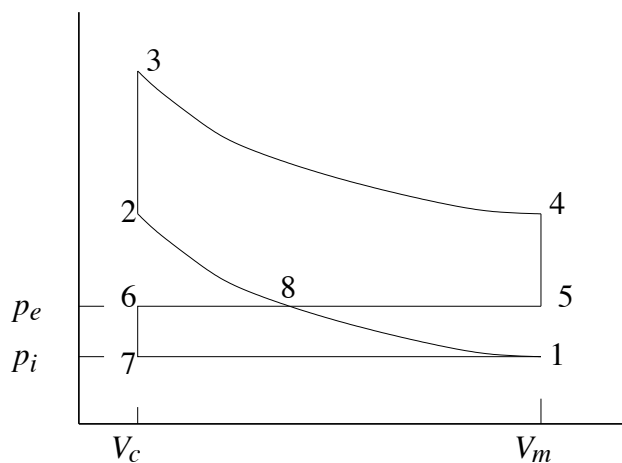


Fig. 1

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## Numerical Answers - 2015

1. (c)  $x_{CO_2} = 0.0277$ ,  $x_{CO} = 0.1202$ ,  $x_{H_2O} = 0.1202$   
 $x_{H_2} = 0.1755$ ,  $x_{N_2} = 0.5564$
2. (b) 466% increase in mass flow rate
4. (c)  $p_3 = 35.13$  bar    gross imep = 3.78 bar  
(d) pumping work =  $-0.358V_m$ ,  $pmep = 0.398$  bar, net imep = 3.382 bar